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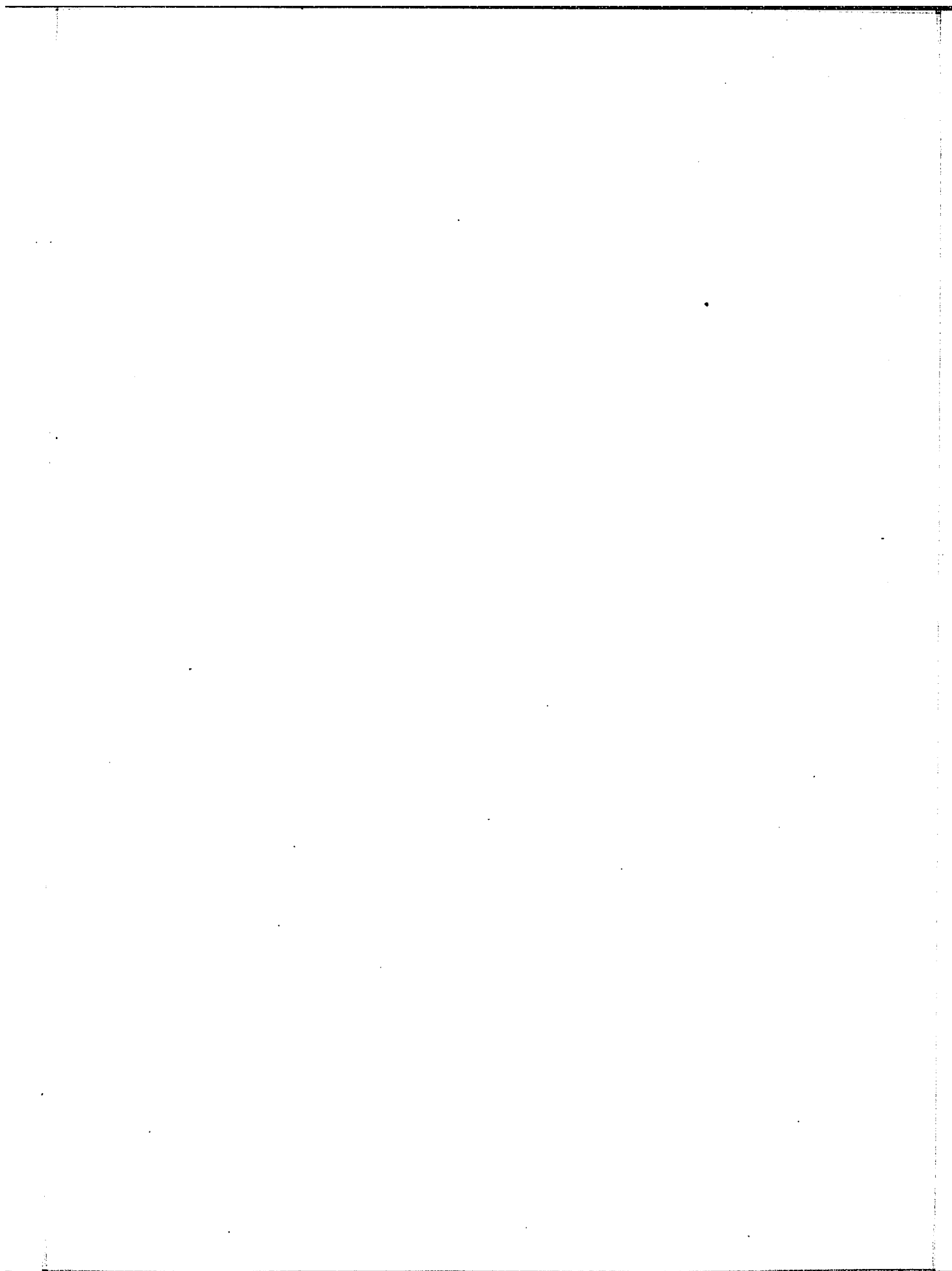
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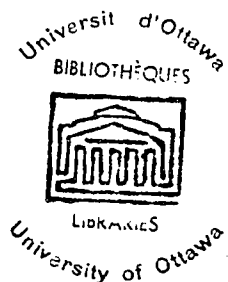


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STRESS DISTRIBUTION IN RECTANGULAR PLATES WITH
INSERTS UNDER AXIAL LOADING

by

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Submitted to the Faculty of Pure and Applied Science
of the University of Ottawa

In partial fulfillment of the requirements

for the degree of

MASTER OF APPLIED SCIENCE

in Mechanical Engineering

September, 1970

UMI Number: EC52396

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ACKNOWLEDGEMENT

The author expresses his profound gratitude to Dr. A. Feingold and Dr. S. Mirza, under whose supervision this research project has been carried out, for their valuable suggestions, critical discussions and constant encouragement.

He is also grateful to the National Research Council of Canada for their financial assistance in the execution of this project. Thanks are also due to Dr. Feingold for granting additional financial support in the form of Teaching Assistantship. The courtesies extended by the personnel of the Computing Centre of the University of Ottawa are highly appreciated.

The help and encouragement for the completion of the thesis, from colleagues, Mr. A.V. Singh, and Mr. C.P. Khulbe in particular, is gratefully acknowledged.

ABSTRACT

The stress distribution in finite plates containing rectangular inserts and subjected to axial loads has been investigated in this thesis. For obtaining the solutions, the basic differential equations have been reduced in terms of finite difference form.

The influence of modulus of elasticity, shape and orientation of insert on the stress concentration produced has also been examined here.

Two-dimensional models containing inserts of rectangular shape and various orientations have been analysed further, by means of the photoelastic techniques. Experimental data have been obtained along the sections where rapid stress variations are expected. Shear difference procedure has been employed to calculate the principal stress and shear stress distributions along these sections.

NOMENCLATURE

X, Y	Rectangular co-ordinates
P_x, P_y	Normal stress in x and y direction
τ_{xy}	Shear stress
P_1, P_2	Principal stresses
u, v	Displacements in x and y directions
ϵ_x, ϵ_y	Strains in x and y directions
γ_{xy}	Shear strain
E	Young's modulus of elasticity
μ	Poisson's ratio
ϕ, Φ	Airy's stress function
N	Fringe order
f_σ	Stress optic coefficient
a, b	Dimensions of the rectangular insert
ρ	Radius of curvature
h	Step size
H	Thickness of the material

TABLE OF CONTENTS

SECTION	Page
1. INTRODUCTION	1
2. FORMULATION OF THE PROBLEM	4
2.1 FUNDAMENTAL RELATIONS	5
2.2 BOUNDARY VALUES OF THE STRESS FUNCTION	8
2.3 INTERFACE CONDITIONS	14
2.4 NUMERICAL PROCEDURE-FINITE DIFFERENCE TREATMENT	18
2.5 EQUATIONS OF BOUNDARY	32
3. COMPUTATION	33
4. EXPERIMENTAL INVESTIGATION	36
4.1 THEORY	36
4.2 SEPARATION TECHNIQUES	38
4.3 MATERIALS FOR MODELS	41
4.4 CALIBRATION OF MATERIAL	47
4.5 EXPERIMENTAL PROCEDURE	47
4.6 COMPUTATION	51
5. DISCUSSION OF RESULTS AND CONCLUSIONS	55
6. APPENDICES	70
7. REFERENCES	88

CHAPTER 1

1. INTRODUCTION

All engineering structures and machine parts have to be designed so that they are strong and sufficiently rigid. At the same time, because of economic considerations, the quantity of the material used should be kept to a minimum. When designing structures and machines, various factors have to be taken into account. One of these is stress concentration or stress distribution in general. Stress distribution plays an important role in design problems. Its knowledge helps a designer to produce balanced designs.

The problem of stress distribution around holes, grooves, notches, shoulder fillets etc. has been treated extensively in the literature [1] but little work has been done on the stress distribution in a finite plate with inserts of different material.

In this thesis an attempt has been made to study the stress distribution in a finite plate with a rectangular insert. Both the theoretical and experimental investigations have been carried out. The object was to develop and evaluate methods, both mathematical and experimental which could then be employed in more complicated practical situations.

The well known classical solutions for the strip with central hole under tension and bending were obtained by Howland and Stevenson [2,3]. This work was later extended to the cases of eccentric circular holes and elliptical holes in infinite strips by Isida [4,5]. Other papers were published on the associated problem of stress concentration with two or more holes in infinite plates [6, 7] and the corresponding problems in strips of finite width [8,9].

The case of rectangular holes, with round corners, in infinite plates have also been studied [10]. The rectangular cut outs with different ratios of their sides and orientations have been analysed by Savin [11].

Isida [12] has studied the effects of rounding the corners of the cutout in an infinite strip which is an extension of Savin's problem.

Researchers have made attempts to determine the effect of reinforcements around the holes of various shapes [13, 14, 15] .

The work of Goodier [16] and Donell [17] also includes the problems of the stress distribution in infinite plates having circular and elliptical inclusions. The results for the case of a circular inclusion were confirmed experimentally by photoelastic method by Thibodeau and Wood [18] .

Edmonds and Beevers [19] have studied the stress distribution existing in and around hard inclusions in solids subjected to a uniaxially applied stress. Two-dimensional models containing inclusions of various shapes and orientations have been analysed by means of photoelastic technique. The influence of modulus of elasticity, shape and orientation of inclusions on the stress concentration has been further examined by a mathematical treatment. However, this study is confined mainly to the circular, elliptical and pentagonal inclusions and for particular values of modulus of elasticity.

The theoretical solution essentially consists of the evaluation of a certain stress function ϕ , which must satisfy the biharmonic equation $\nabla^4 \phi = 0$ at every point in the plate. This function is defined such that

$$P_x = \frac{\partial^2 \phi}{\partial y^2}$$

$$P_y = \frac{\partial^2 \phi}{\partial x^2}$$

$$\tau_{xy} = - \frac{\partial^2 \phi}{\partial x \partial y}$$

This is a mixed boundary value problem. Along the external boundary the conditions are prescribed in terms of stresses, while at the interface between the two materials the deformations as well as the stresses

must be compared.

The computer program written for the solution of the problem has the flexibility in terms of any values of E , μ , different orientations and insert sizes. For experimental analysis, photoelasticity method, which gives complete exploration of stresses, is adopted. The experimental investigation has been carried out on a sophisticated portable polariscope (comparator type). The method of shear differences is employed for the separation of stresses. The moduli of elasticity for the two materials used in the experimental investigation are 340×10^3 psi and 1×10^3 psi. Inserts of different sizes and orientations are placed in a finite rectangular plate which is subjected to a uniform tension along its length.

CHAPTER 2

2. FORMULATION OF THE PROBLEM

The theoretical study of this problem mainly involves the calculation of stress function ϕ which should satisfy the biharmonic equation , $\nabla^4 \phi = 0$, at each point in the plate. Due to the complex nature of the problem, a numerical method using finite difference techniques has been employed. These techniques replace the governing equations and boundary conditions by finite number of unknown values of the dependent variables at a number of discrete points within or just outside the domain of integration.

The finite difference equations form a set of linear algebraic equations. The large number of simultaneous equations thus formed are solved on the digital computer.

Two orientations of insert with respect to the plate have been studied.

Case I : When the insert is parallel to the sides of the plate.

Case II: When the insert is inclined at an angle of 45° to the sides of the plate.

In each of the above two cases three different sizes of inserts have been studied.

In order to simplify the mathematical formulation of the problem, the insert has been kept parallel to x, y co-ordinates and the outside plate has been rotated. when required. The solution in each case has been obtained in three steps:

step 1, the generation of a system of non-homogeneous simultaneous equations at nodal points ;

step 2, the solutions of these equations, i.e. the determination of the value of the stress function ϕ at each nodal point;

step 3, calculation of the stresses from the stress function.

2.1. FUNDAMENTAL RELATIONS

In this problem of plane stress two possible formulations can be used. The problem can be set up in terms of a stress function or alternatively the governing equations could be written in terms of displacements. However, in this case use is made of Airy stress function. This is primarily because of the reason that the boundary conditions are prescribed in terms of stresses. Thus, the solution of the problem reduces to the integration of the differential equations of equilibrium together with the compatibility equations.

Neglecting the body and temperature forces, the differential equations of equilibrium for a two dimensional problem, in terms of cartesian co-ordinates are [20]

$$\begin{aligned}\frac{\partial P_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} &= 0 \\ \frac{\partial P_y}{\partial y} + \frac{\partial \tau_{xy}}{\partial x} &= 0\end{aligned}\tag{2.1.1}$$

These two equations of equilibrium (2.1.1) are not sufficient for the determination of three unknown stress components P_x , P_y and τ_{xy} . The necessary third equation is derived by using the elastic properties of the body. In the case of two dimensional problem, the linear strain displacement relations are

$$\begin{aligned}\epsilon_x &= \frac{\partial u}{\partial x} \\ \epsilon_y &= \frac{\partial v}{\partial y} \\ \gamma_{xy} &= \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}\end{aligned}\tag{2.1.2}$$

where u and v are displacements in the x and y directions. Differentiating the first of these eqns. (2.1.2) twice with respect to y , the second twice with respect to x , and the third once with respect

to x and once with respect to y , we obtain the compability equation:

$$\frac{\partial^2 \epsilon_x}{\partial y^2} + \frac{\partial^2 \epsilon_y}{\partial x^2} = \frac{\partial^2 \gamma_{xy}}{\partial x \partial y} \quad (2.1.3)$$

From Hooke's law, the stress-strain relations in a two dimensional problem are written as

$$\begin{aligned} \epsilon_x &= A (P_x - B P_y) \\ \epsilon_y &= A (P_y - B P_x) \\ \gamma_{xy} &= 2 A (1+B) \tau_{xy} \end{aligned} \quad (2.1.4)$$

where $A = \frac{1}{E}$ and $B = \mu$ for plane stress.

Substituting relations (2.1.4) into eqn. (2.1.3) and simplifying, we obtain the compatibility equation in terms of stress components:

$$\left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) (P_x + P_y) = 0 \quad (2.1.5)$$

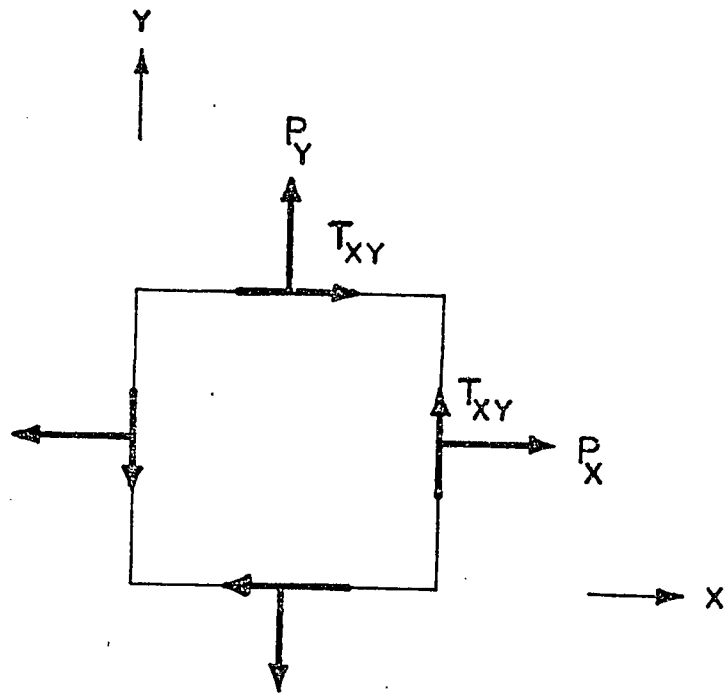
In order to solve eqns. (2.1.1) and (2.1.5), we introduce the stress function ϕ , which is defined such that

$$\begin{aligned} P_x &= \frac{\partial^2 \phi}{\partial y^2} \\ P_y &= \frac{\partial^2 \phi}{\partial x^2} \\ \tau_{xy} &= - \frac{\partial^2 \phi}{\partial x \partial y} \end{aligned} \quad (2.1.6)$$

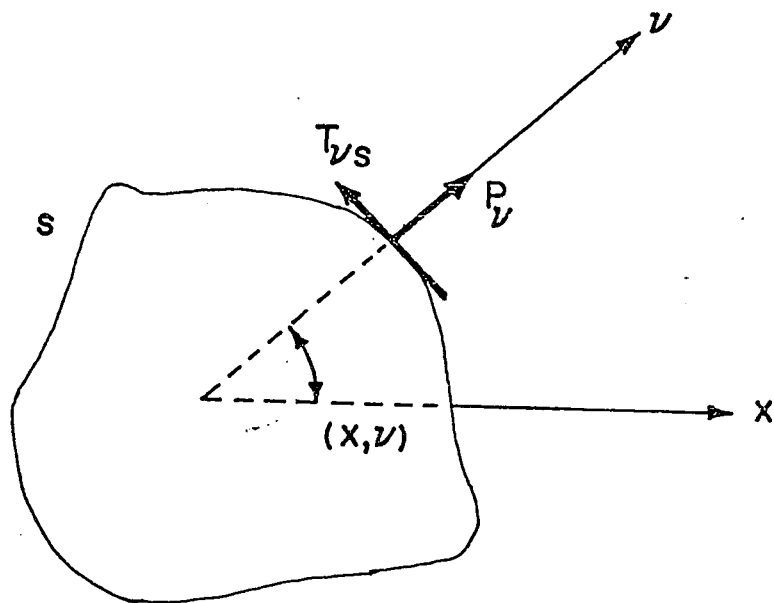
which identically satisfies eqn. (2.1.1).

Now substituting eqn. (2.1.6) into eqn. (2.1.5), we find that the stress function ϕ must satisfy the following equation :

$$\frac{\partial^4 \phi}{\partial x^4} + 2 \frac{\partial^4 \phi}{\partial x^2 \partial y^2} + \frac{\partial^4 \phi}{\partial y^4} = 0 \quad (2.1.7)$$



(a)



(b)

FIG.- I

which can also be written as

$$\nabla^4 \phi = 0 \quad (2.1.7)$$

where ∇^4 is the Laplacian operator

$$\nabla^4 \equiv \frac{\partial^4}{\partial x^4} + 2 \frac{\partial^4}{\partial x^2 \partial y^2} + \frac{\partial^4}{\partial y^4}$$

The problem is now reduced to that of finding a function ϕ which satisfies eqn. (2.1.7) at every point within the boundary and at the boundary.

2.2. BOUNDARY VALUES OF THE STRESS FUNCTION

For the complete solution of the problem, it is necessary that the boundary values of the stress function and its gradients be specified. As the stress components are represented by the second derivatives of ϕ , the addition of any linear function of co-ordinates of the form

$$C_1 x + C_2 y + C_3 = 0$$

where C_1, C_2, C_3 are constants, will not affect the stress system.

Therefore it is possible, always, to assume arbitrarily convenient values of ϕ , $\frac{\partial \phi}{\partial x}$ and $\frac{\partial \phi}{\partial y}$ at some boundary point. If the external forces are known these values can now be determined at all points of the boundary by a simple integration [21].

Let P_v and τ_{vs} represent the values of normal direct stress and the tangential shear stress respectively at a point on the boundary. The v denotes the direction of the outward drawn normal at the edge and s the direction of the tangent in a counter clockwise circuit in fig. 1.

We know that at the boundary, the components of stresses P_{vx} and P_{vy} in x and y direction respectively can be given by

$$P_{vx} = P_v \cos(x, v) - \tau_{vs} \sin(x, v) \quad \text{and}$$

$$P_{vy} = P_v \sin(x, v) + \tau_{vs} \cos(x, v)$$

But also

$$P_{vx} = P_x \cos(x, v) + \tau_{xy} \sin(x, v) \quad (2.2.1)$$

and

$$P_{vy} = P_y \sin(x, v) + \tau_{xy} \cos(x, v) \quad (2.2.2)$$

Now substituting in the above equations for P_x and P_y in terms of ϕ and for $\cos(x, v)$ and $\sin(x, v)$, we get

$$P_{vx} = \frac{\partial^2 \phi}{\partial y^2} \cdot \frac{\partial y}{\partial s} + \frac{\partial^2 \phi}{\partial x \partial y} \cdot \frac{\partial x}{\partial s}$$

which can also be written as

$$P_{vx} = \frac{\partial}{\partial s} \left(\frac{\partial \phi}{\partial y} \right)$$

and similarly

$$P_{vy} = - \frac{\partial^2 \phi}{\partial x^2} \cdot \frac{\partial x}{\partial s} - \frac{\partial^2 \phi}{\partial x \partial y} \cdot \frac{\partial y}{\partial s}$$

or

$$= - \frac{\partial}{\partial s} \left(\frac{\partial \phi}{\partial x} \right)$$

Hence by line integration round the boundary we can obtain boundary values for $\frac{\partial \phi}{\partial x}$ and $\frac{\partial \phi}{\partial y}$ as

$$\begin{aligned} \frac{\partial \phi}{\partial x} &= - \int P_{vy} ds \\ \frac{\partial \phi}{\partial y} &= \int P_{vx} ds \end{aligned} \quad (2.2.3)$$

To find ϕ we use the equation

$$\frac{\partial \phi}{\partial s} = \frac{\partial \phi}{\partial x} \cdot \frac{dx}{ds} + \frac{\partial \phi}{\partial y} \cdot \frac{dy}{ds}$$

which, after integration by parts, gives

$$\phi = x \frac{\partial \phi}{\partial x} + y \frac{\partial \phi}{\partial y} - \int \left(x \frac{\partial \phi}{\partial s \partial x} + y \frac{\partial^2 \phi}{\partial s \partial y} \right) ds \quad (2.2.4)$$

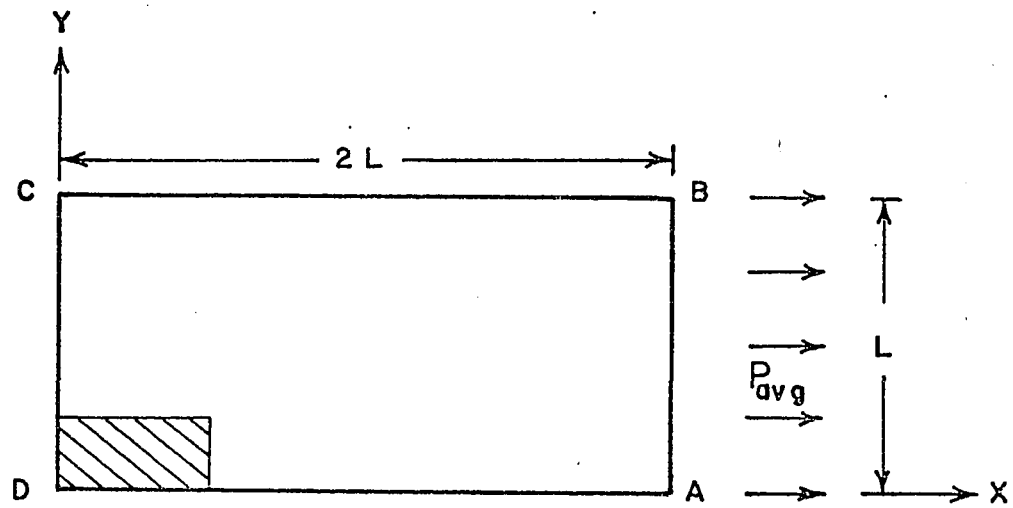


FIG.- 2

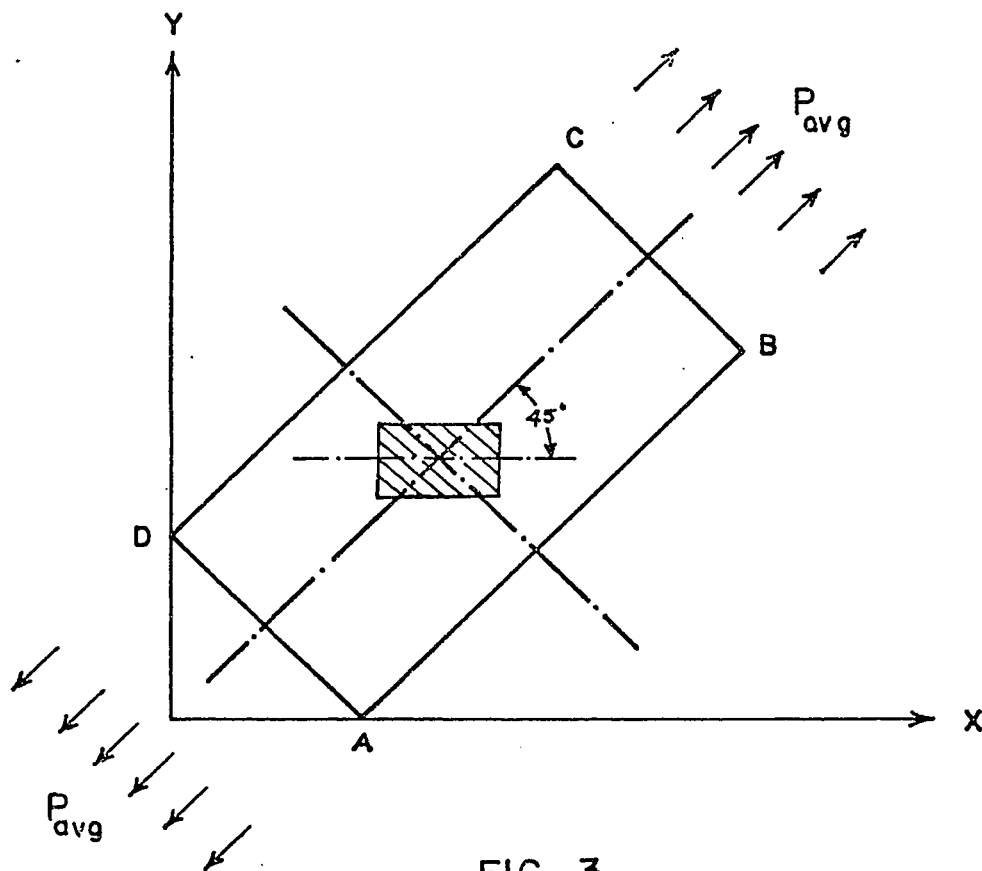


FIG.- 3

Substituting in this equation the values of the derivatives given by eqn. (2.2.3), we calculate the boundary values of ϕ .

The boundary values of the stress function for the two different orientations of insert have been given below

Case I - 0° Orientation (Refer fig. 2)

In this case the plate has two-fold symmetry. Therefore it is sufficient to work with one quarter of the plate. The half width, AB, of the plate is L and the half length DA is 2L. The intensity of the applied tension along AB is 100 (a non-dimensional value).

Then the stress boundary conditions on ABC are

$$P_{vy} = 0 \quad \text{everywhere}$$

$$P_{vx} = 100 \quad \text{on AB}$$

$$= 0 \quad \text{on BC}$$

Now substituting these values in eqn. (2.2.3), we get

$$\frac{\partial \phi}{\partial x} = 0 \quad \text{everywhere} \quad (\because \frac{\partial \phi}{\partial x} = 0 \quad \text{at C due to symmetry})$$

$$\begin{aligned} \text{and } \frac{\partial \phi}{\partial y} &= 100 y + C_1 \quad \text{along AB} \\ &= 100 y \quad (C_1 = 0 \text{ because } \frac{\partial \phi}{\partial y} = 0 \text{ at A due to symmetry}). \end{aligned}$$

Similarly,

$$\frac{\partial \phi}{\partial y} = 100 \quad \text{on BC}$$

Now substituting the values of $\frac{\partial \phi}{\partial x}$ and $\frac{\partial \phi}{\partial y}$ in eqn. (2.2.4)

we obtain

$$\begin{aligned} \phi &= 50 y^2 \quad \text{on AB} \\ &= 50 \quad \text{on BC} \end{aligned}$$

where constant of integration is chosen to be such that ϕ vanishes at A.

Case II - 45° orientation (Refer fig. 3)

The orientation of the plate A B C D with respect to the x-y axes requires that the plate A B C D be taken as the working diagram.

Let the length of the plate be $2L$ and its width be L . Again the intensity of applied tension along A D and B C is 100 (non-dimensional value).

Using eqns. (2.2.3) and (2.2.4), we obtain the values of $\frac{\partial \phi}{\partial x}$, $\frac{\partial \phi}{\partial y}$ and ϕ along the boundary A B C D as follows.

Along the boundary A B :

$$\frac{\partial \phi}{\partial x} = 150 \sqrt{2}$$

$$\frac{\partial \phi}{\partial y} = 100 \sqrt{2}$$

$$\phi_{AB} = 150 \sqrt{2} x + 100 \sqrt{2} y - 325$$

Along the boundary B C :

$$\frac{\partial \phi}{\partial x} = 100 x$$

$$\frac{\partial \phi}{\partial y} = 100 y$$

$$\phi_{BC} = 50x^2 + 50y^2$$

Along the boundary C D :

$$\frac{\partial \phi}{\partial x} = 100 \sqrt{2}$$

$$\frac{\partial \phi}{\partial y} = 150 \sqrt{2}$$

$$\phi_{CD} = 100 \sqrt{2} x + 150 \sqrt{2} y - 325$$

Along the boundary A D :

$$\frac{\partial \phi}{\partial x} = 100 x + 100 \sqrt{2}$$

$$\frac{\partial \phi}{\partial y} = 100 y + 100 \sqrt{2}$$

$$\phi_{AD} = 50x^2 + 50y^2 + 100 \sqrt{2} x + 100 \sqrt{2} y - 300$$

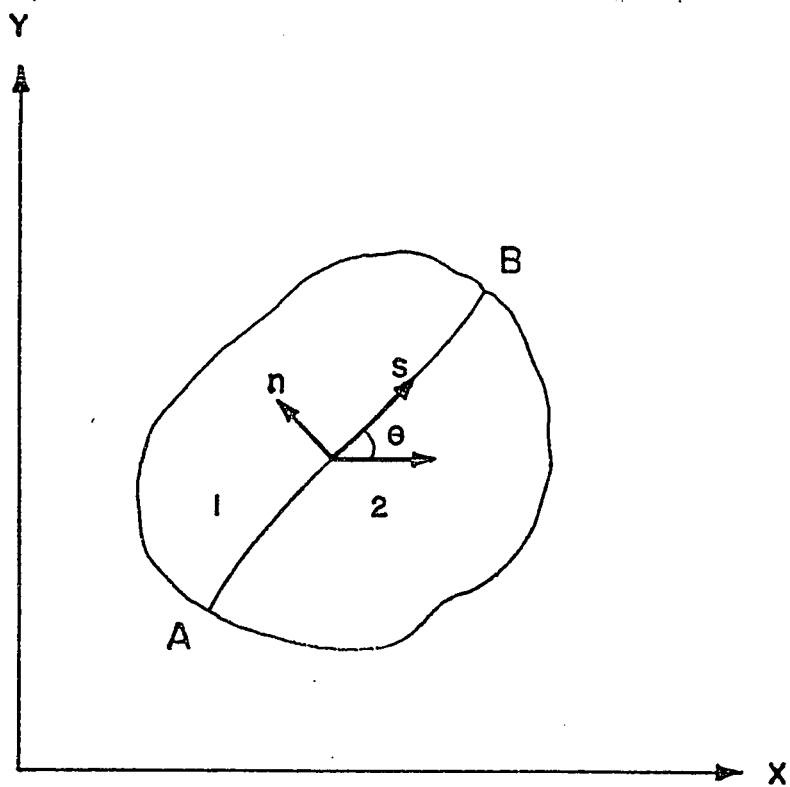


FIG.- 4

For detailed derivation of this case see appendix (I.1).

2.3 INTERFACE CONDITIONS

Some special conditions have to be imposed on the stress function for the interface separating the two materials having different elastic properties [22]. These conditions are

1. Interface displacement conditions
2. Interface stress conditions

2.3.1. Interface Displacement Conditions

Let the line AB in fig.4 denote an interface between the two regions of different elastic properties. It is assumed that on this interface no slippage is possible and therefore the stresses in each region must be such that a "perfect fit" of displacement is assured. If u'_1 and v'_1 represent respectively the tangential and normal components of displacement for region 1 along any surface s and u'_2 , v'_2 represent the similar displacement for region 2, then

$$\frac{\partial u'_1}{\partial s} = \frac{\partial u'_2}{\partial s}$$

and

$$\frac{\partial^2 v'_1}{\partial s^2} = \frac{\partial^2 v'_2}{\partial s^2}$$

which ensures that the tangential 'stretching' and curvatures of the two regions are identical.

With the stresses given in terms of the appropriate stress functions, the interface conditions expressed in terms of two stress functions ϕ_1 and ϕ_2 are [22]

$$A_1 \left[\frac{\partial^2 \phi_1}{\partial n^2} - B_1 \left(\frac{\partial^2 \phi_1}{\partial s^2} - \frac{1}{\rho} \frac{\partial \phi_1}{\partial n} \right) \right] = A_2 \left[\frac{\partial^2 \phi_2}{\partial n^2} - B_2 \left(\frac{\partial^2 \phi_2}{\partial s^2} - \frac{1}{\rho} \frac{\partial \phi_2}{\partial n} \right) \right]$$

(2.3.1)

and

$$-A_1 \left[\frac{\partial}{\partial n} (\nabla^2 \phi_1 + (1+B_2) \frac{\partial}{\partial s} \left(\frac{\partial^2 \phi_1}{\partial n \partial s} \right)) \right] = -A_2 \left[\frac{\partial}{\partial n} (\nabla^2 \phi_2 + (1+B_2) \frac{\partial}{\partial s} \left(\frac{\partial^2 \phi_2}{\partial n \partial s} \right)) \right]$$

in which $\frac{1}{\rho} = \frac{\partial \theta}{\partial s}$ represents the curvature of the boundary

$$A = \frac{1}{E}$$

and $B = \mu$

with appropriate suffix.

2.3.2. Interface Stress Conditions

In addition to the displacement conditions it is necessary that the normal and tangential stress in both regions be equal on the interface which is a condition from the considerations of static equilibrium.

The value of normal stress in terms of stress function is given

as

$$\begin{aligned} P_n &= \frac{\partial^2 \phi}{\partial x^2} \cos^2 \theta + \frac{\partial^2 \phi}{\partial y^2} \sin^2 \theta + \frac{\partial^2 \phi}{\partial x \partial y} \sin 2\theta \\ &\equiv \frac{\partial^2 \phi}{\partial s^2} - \frac{1}{\rho} \frac{\partial \phi}{\partial n} \end{aligned}$$

and the tangential stress is given as

$$\begin{aligned} \tau_{ns} &= \left(\frac{\partial^2 \phi}{\partial x^2} - \frac{\partial^2 \phi}{\partial y^2} \right) \frac{1}{2} \sin 2\theta - \frac{\partial^2 \phi}{\partial x \partial y} \cos 2\theta \\ &\equiv - \frac{\partial^2 \phi}{\partial n \partial s} \end{aligned}$$

Thus the stress equality interface conditions become

$$\frac{\partial^2 \phi_1}{\partial s^2} - \frac{1}{\rho} \frac{\partial \phi_1}{\partial n} = \frac{\partial^2 \phi_2}{\partial s^2} - \frac{1}{\rho} \frac{\partial \phi_2}{\partial n} \quad (2.3.3)$$

and

$$\frac{\partial^2 \phi_1}{\partial n \partial s} = \frac{\partial^2 \phi_2}{\partial n \partial s} \quad (2.3.4)$$

The eqns.(2.3.3) and (2.3.4) can be further simplified using the fact that an addition of any linear function of x and y to either stress function does not alter the stresses. Therefore, arbitrarily, the equality of the two stress functions and their slopes is imposed at some point. Hence the eqns. (2.3.3) and (2.3.4) reduce to

$$\phi_1 = \phi_2 \quad (2.3.5)$$

and

$$\frac{\partial \phi_1}{\partial n} = \frac{\partial \phi_2}{\partial n} \quad (2.3.6)$$

In addition, in each region the stress function has to satisfy the general governing equation

$$\nabla^4 \phi_1 = \nabla^4 \phi_2 = 0$$

Now, in the case of straight interface between two regions parallel to x-axis in which the stress functions are defined as $\phi_1 = \phi$ and $\phi_2 = \bar{\phi}$ and which have the elastic constants A1, B1 and A2, B2 respectively

$$\frac{1}{\rho} = 0, \quad \partial n = \partial y, \quad \partial s = \partial x.$$

Therefore, substituting these values in the eqns.(2.3.1), (2.3.2), (2.3.5) and (2.3.6), the following conditions are obtained:

$$\phi = \bar{\phi} \quad (2.3.7)$$

$$\frac{\partial \phi}{\partial y} = \frac{\partial \bar{\phi}}{\partial y} \quad (2.3.8)$$

$$A_1 \left[\frac{\partial^2 \phi}{\partial y^2} - B_1 \frac{\partial^2 \phi}{\partial x^2} \right] = A_2 \left[\frac{\partial^2 \bar{\phi}}{\partial y^2} - B_2 \frac{\partial^2 \bar{\phi}}{\partial x^2} \right] \quad (2.3.9)$$

$$-A_1 \left[\frac{\partial^3 \phi}{\partial y^3} + (2+B_1) \frac{\partial^2 \phi}{\partial x^2 \partial y} \right] = -A_2 \left[\frac{\partial^3 \bar{\phi}}{\partial y^3} + (2+B_2) \frac{\partial^2 \bar{\phi}}{\partial x^2 \partial y} \right] \quad (2.3.10)$$

Similarly in the case of the straight interface parallel to y-axis we have

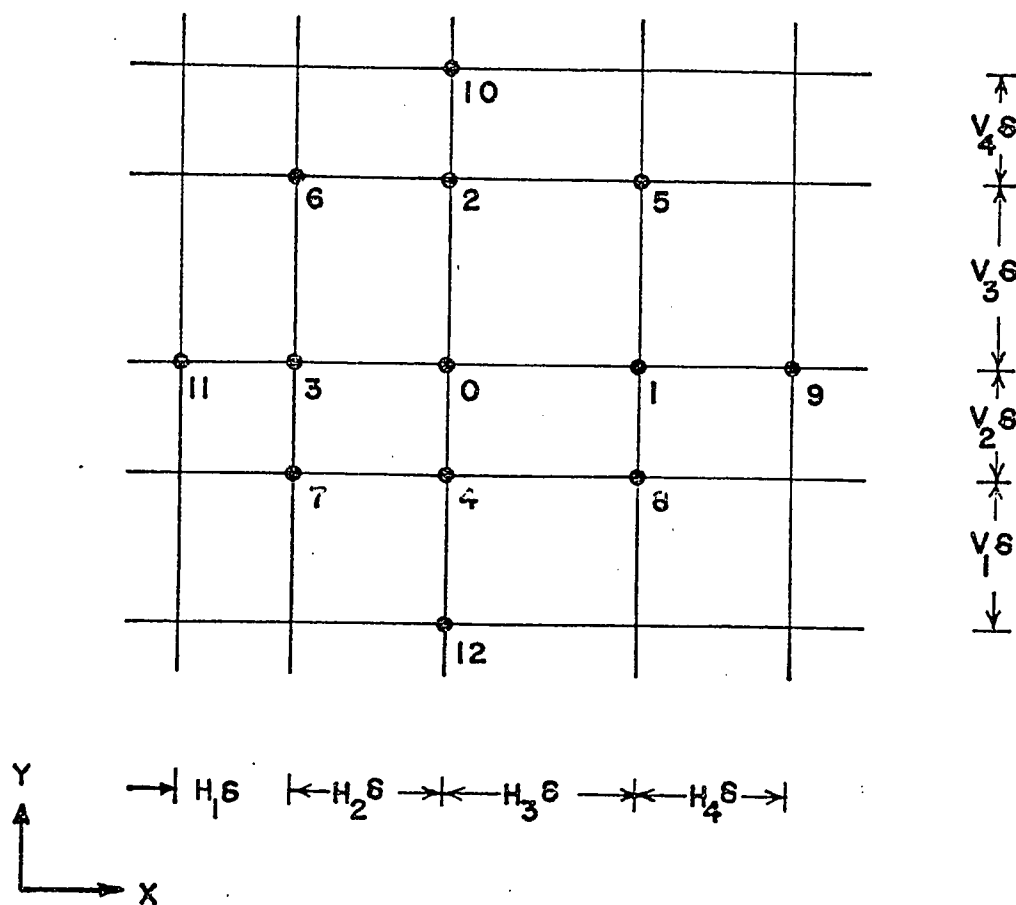


FIG.- 5

$$\frac{1}{\rho} = 0$$

$$\partial n = \partial x$$

$$\partial s = \partial y$$

Substitution of these values in eqns. (2.3.5), (2.3.6), (2.3.1) and (2.3.2) yields

$$\phi = \bar{\phi} \quad (2.3.11)$$

$$\frac{\partial \phi}{\partial x} = \frac{\partial \bar{\phi}}{\partial x} \quad (2.3.12)$$

$$A_1 \left[\frac{\partial^2 \phi}{\partial x^2} - B_1 \frac{\partial^2 \phi}{\partial y^2} \right] = A_2 \left[\frac{\partial^2 \bar{\phi}}{\partial x^2} - B_2 \frac{\partial^2 \bar{\phi}}{\partial y^2} \right] \quad (2.3.13)$$

and

$$-A_1 \left[\frac{\partial^3 \phi}{\partial x^3} + (2+B_1) \frac{\partial^3 \phi}{\partial y^2 \partial x} \right] = -A_2 \left[\frac{\partial^3 \bar{\phi}}{\partial x^3} + (2+B_2) \frac{\partial^3 \bar{\phi}}{\partial y^2 \partial x} \right] \quad (2.3.14)$$

2.4. NUMERICAL PROCEDURE - FINITE DIFFERENCE TREATMENT

The numerical values of the function ϕ are obtained at the nodes of the network covering the region of integration. The size of the network is kept as small as possible in order to achieve better accuracy. Various forms of the biharmonic operators in finite difference have been used for these nodal points depending upon their position relative to other points of the network. These forms in their string pattern are given below. For detailed derivation and notations see appendix II.

2.4.1 General Form

This form of biharmonic operator has been used for the nodal points where all the points associated with the operator fall within one region. When using for points near the outside boundary, the fictitious values for the points outside are calculated as described in the Art 2.4.2. Referring to fig. 5 the string pattern for this case can be given as shown

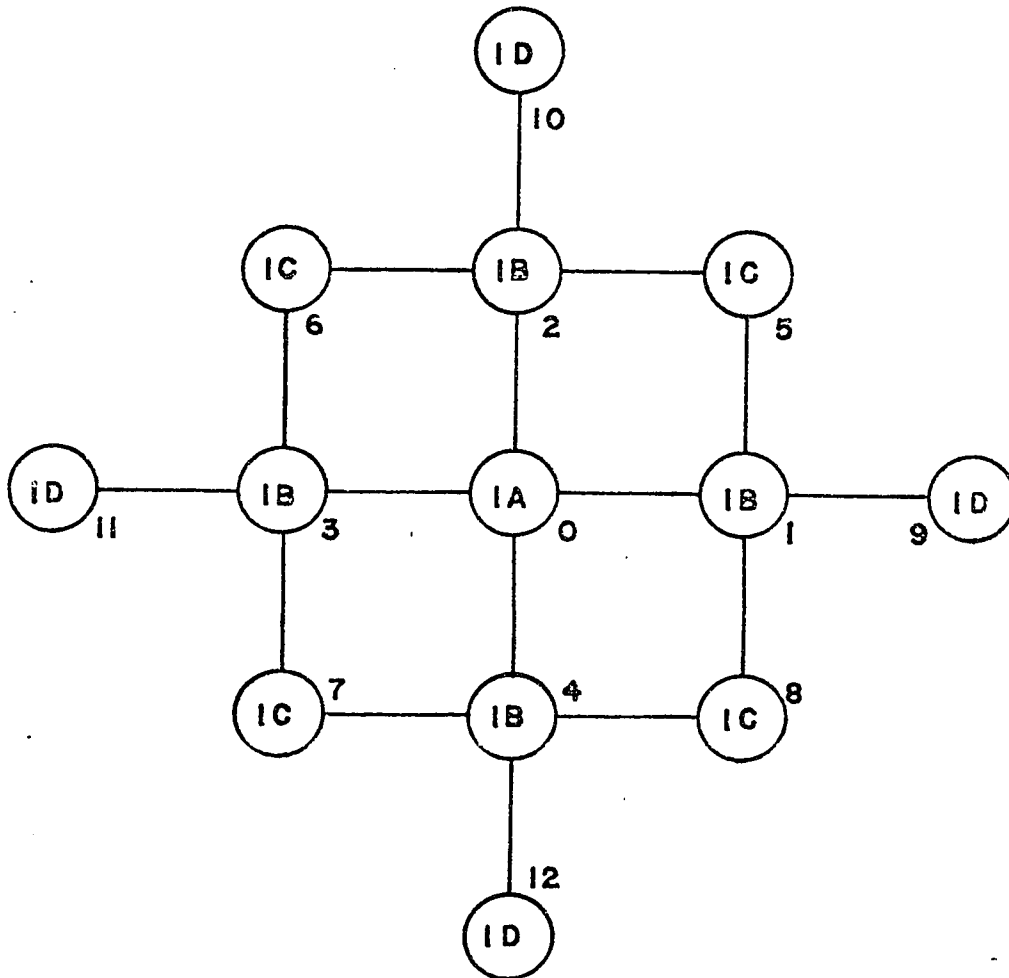


FIG.- 6

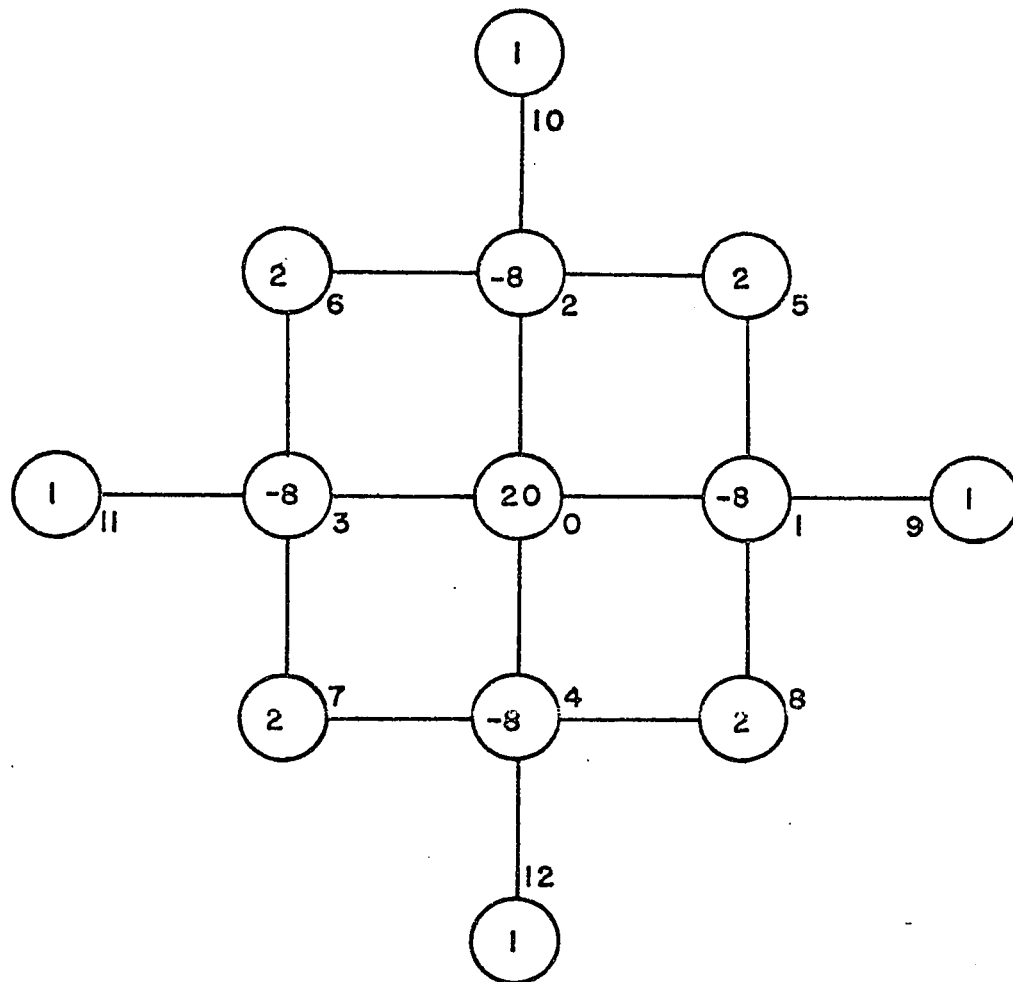


FIG.- 7

in fig. 6.

When the mesh is square, the above pattern reduces to the form shown in fig. 7.

2.4.2. Rectilinear Boundaries

a. When mesh line coincides with boundary: The two boundary conditions are appropriately associated with a biharmonic equation at all points of the boundary.

While writing the biharmonic equation at the nodal points which are only one mesh length within the boundary, a fictitious value is involved in the equation. This fictitious value can be removed by using one of the boundary conditions in finite difference form [21].

b. When the mesh and the boundary do not coincide: When the boundary intersects the mesh at other than nodal points irregular stars exist and therefore the following method has to be used for finding the values at the fictitious nodes and at the nodes which are less than one mesh length away from the boundary. Both cases, when the points are along the increasing value of ϕ and when they are along the decreasing value of ϕ have been dealt with here [21].

Case I

The values of ϕ can be attached to node 0 by the following relations (fig.8)

$$\phi_9 = \frac{4\xi K}{(1+\xi)^2} + \frac{2(1-\xi)hk}{1+\xi} + \frac{(1-\xi)^2}{(1+\xi)^2} \phi_0, \quad (2.4.2I)$$

$$\phi_1 = \frac{(1+2\xi)K}{(1+\xi)^2} - \frac{\xi hk}{1+\xi} + \frac{\xi^2 \phi_0}{(1+\xi)^2}$$

where $\frac{\partial \phi}{\partial x}$ or $\frac{\partial \phi}{\partial y}$ at B are denoted by k and ϕ at B by K.

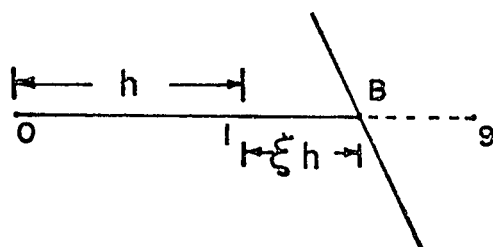


FIG- 8

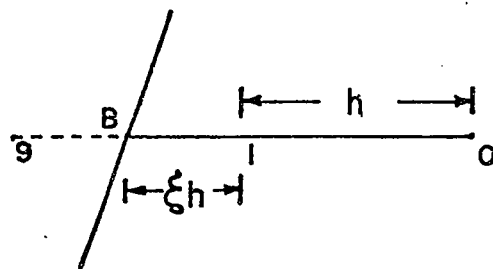


FIG.- 9

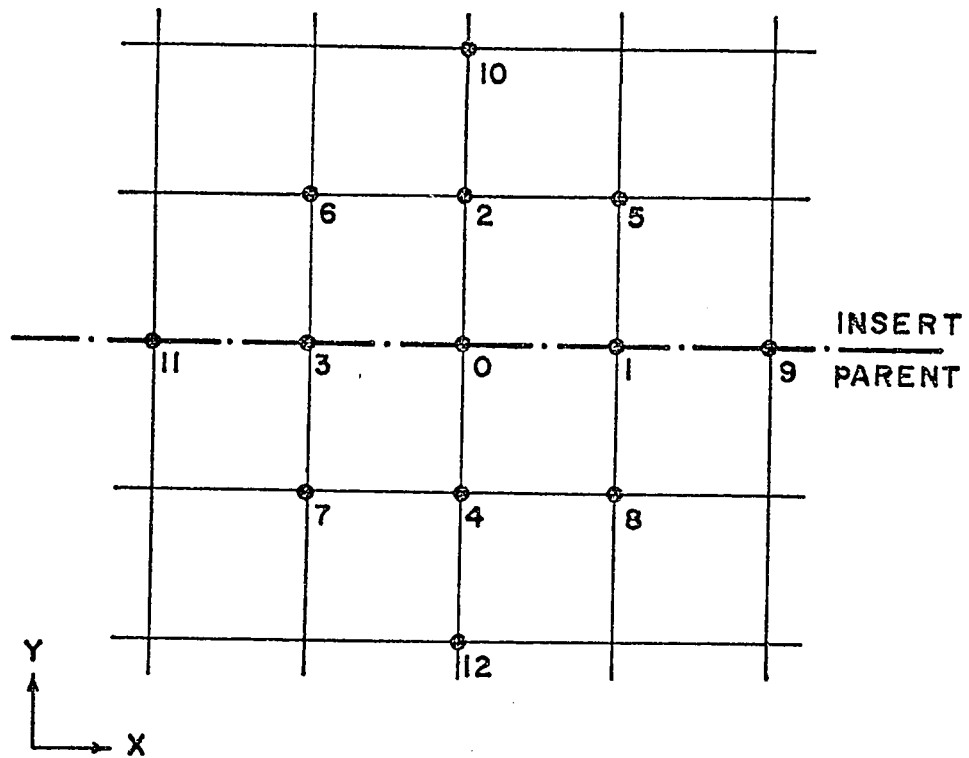


FIG.- 10

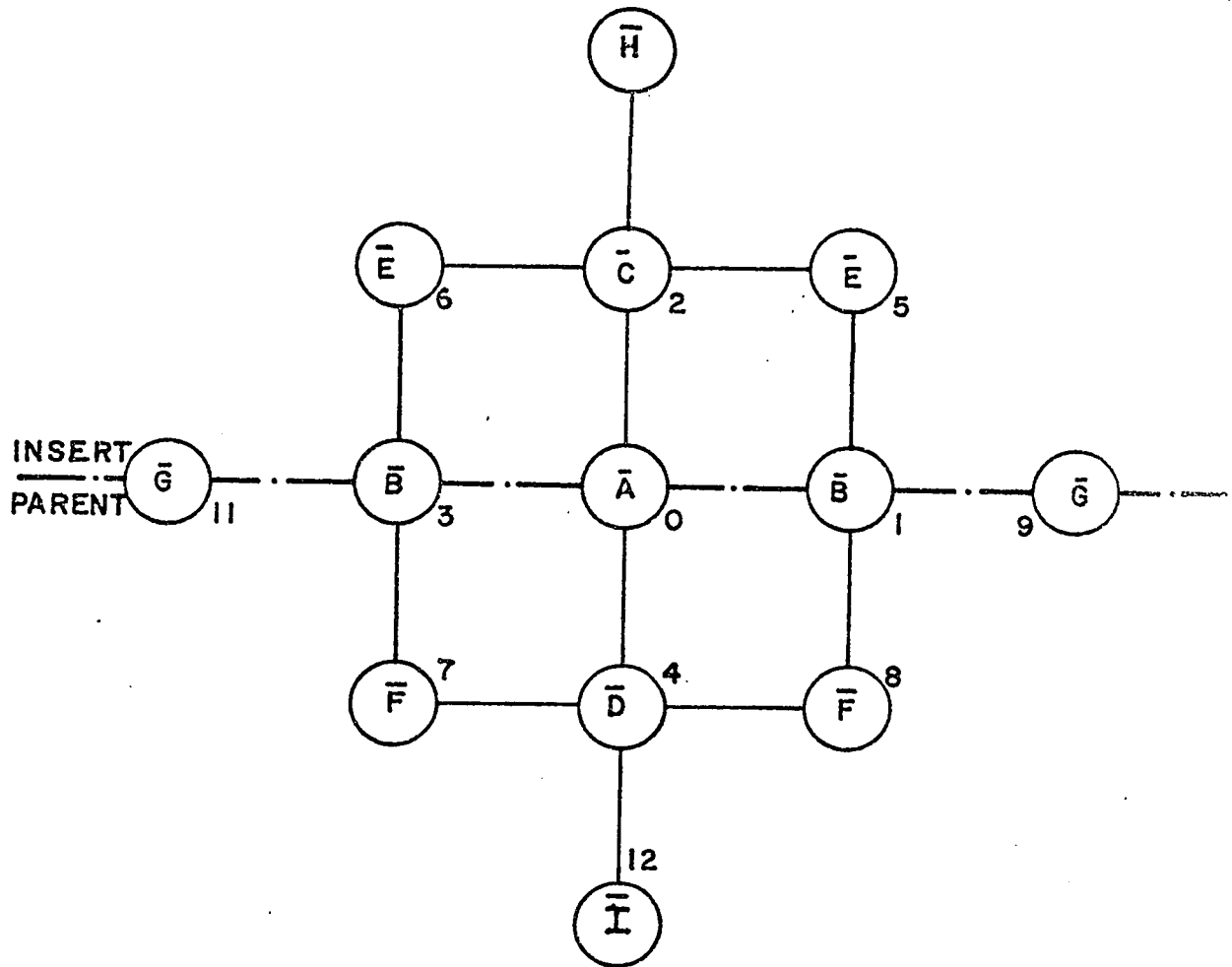


FIG.-II

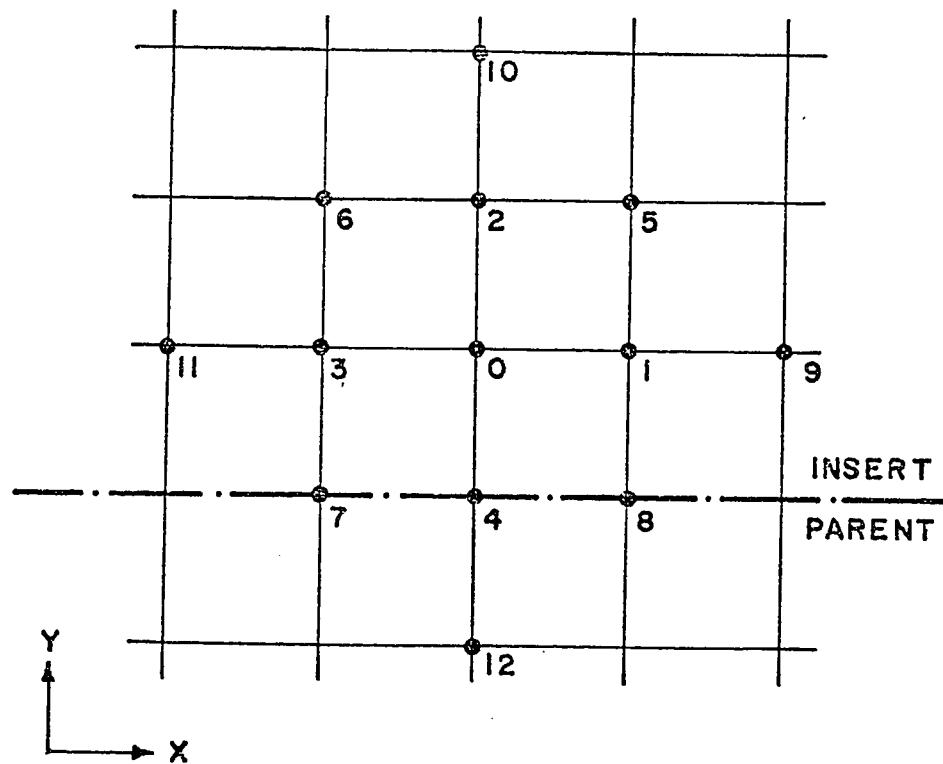


FIG.- 12

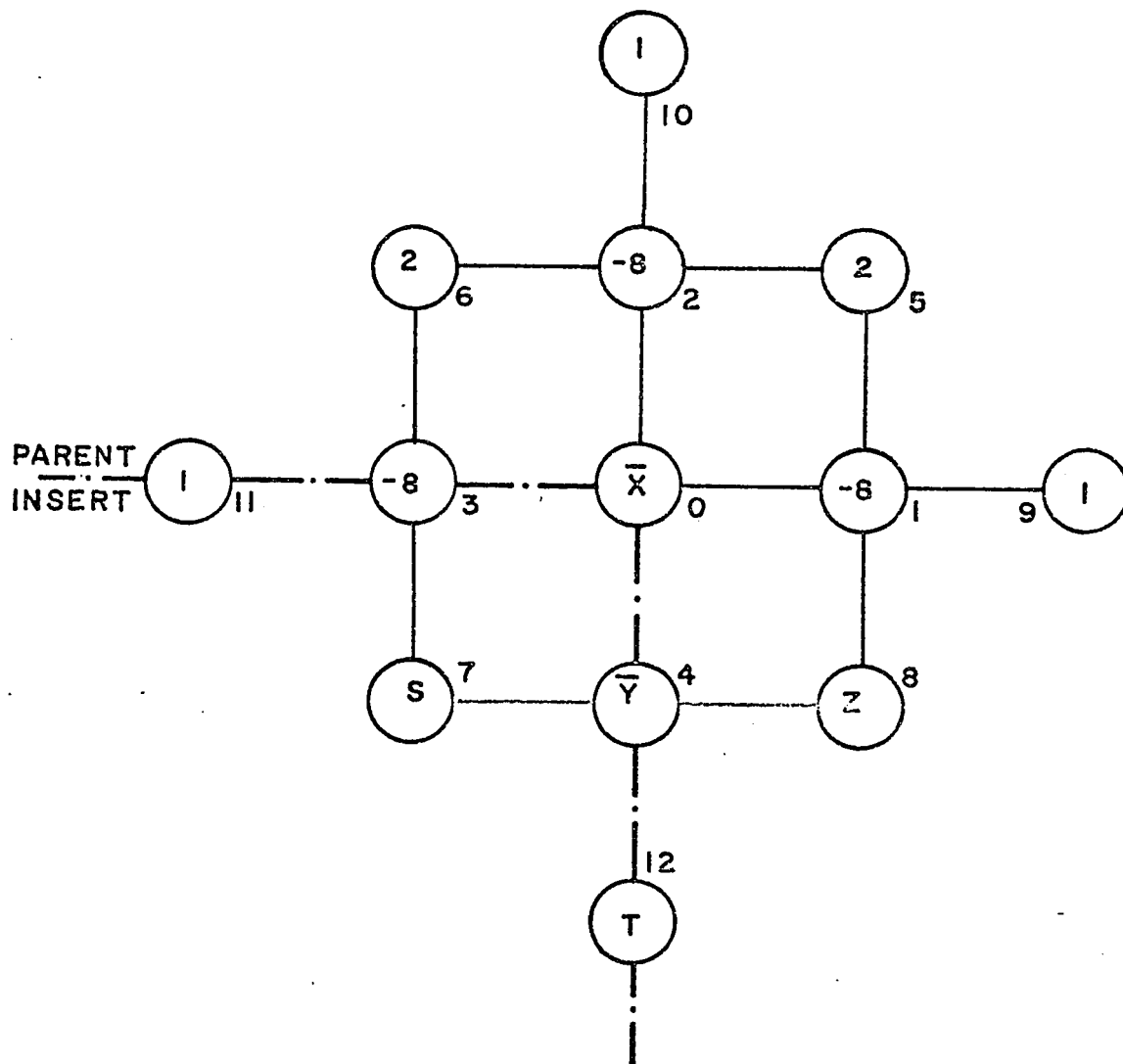


FIG.- 13

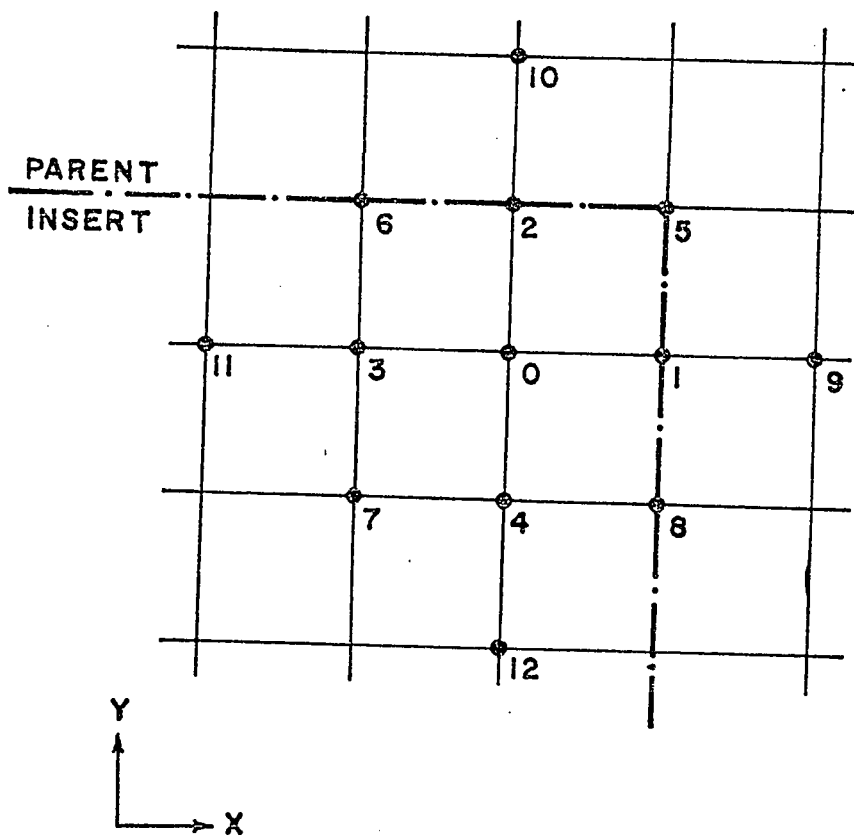


FIG.- 14

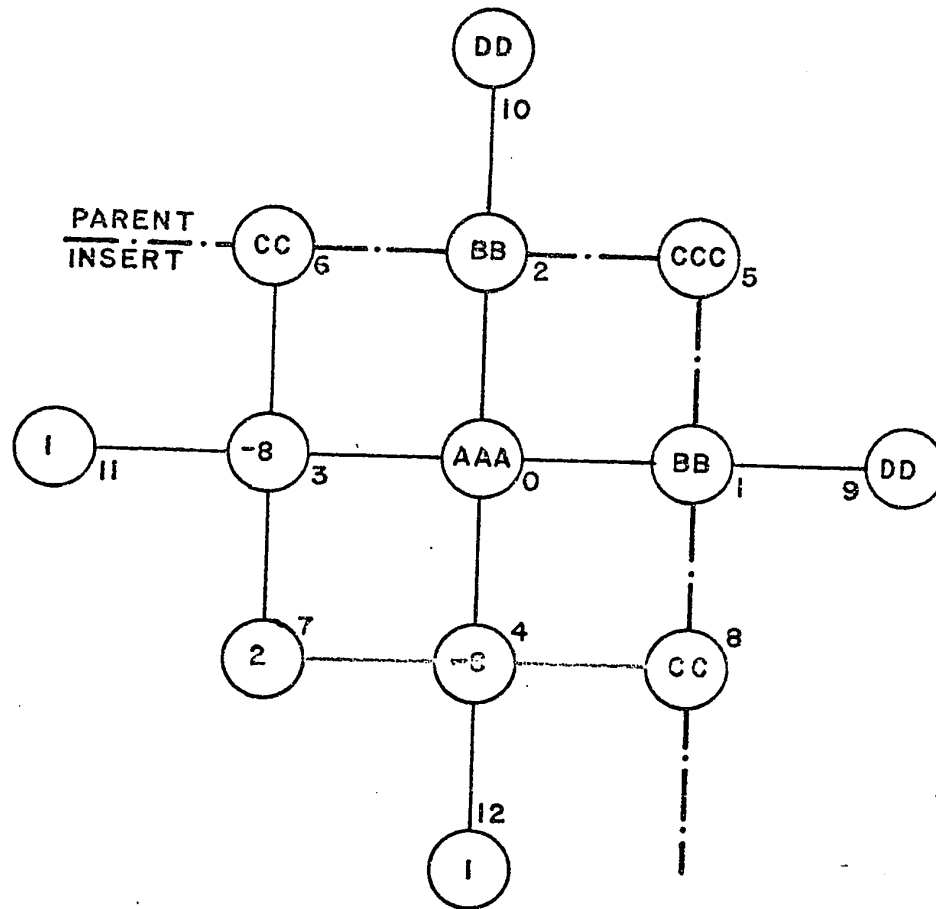


FIG.-15

After the values of ϕ have been attached to all other internal nodes, such as the node 0 in fig. 8, using the pair of relations (2.4.2I), the consistent values of ϕ satisfying the boundary conditions can be evaluated at those internal points such as 1, which lie within one mesh length of the boundary and also at points, like 9, which are fictitious.

Case II

This case shown in fig. 9 is same as before except the orientation is different i.e. the points lie along decreasing value of ϕ . In this case

ϕ_9 and ϕ_1 are given by the following relations

$$\phi_9 = \frac{4 \epsilon K}{(1 + \xi)^2} - \frac{2(1 - \xi) h k}{(1 + \xi)} + \frac{(1 - \xi)^2}{(1 + \xi)^2} \phi_0$$

and

$$\phi_1 = \frac{(1 + 2 \epsilon) K}{(1 + \xi)^2} + \frac{\xi h k}{(1 + \xi)} + \frac{\epsilon^2}{(1 + \epsilon)^2} \phi_0 \quad (2.4.2II)$$

2.4.3. Operator for the Points on the Interface

Referring to fig. 10, the string pattern for the point 0 on the interface can be given as shown in fig. 11.

2.4.4. Operator for the Points one Mesh away from the Interface

Referring to fig. 12, the string pattern of the operator for the points which are one mesh length away from the interface can be given as shown in fig. 13.

The operator for the points below the interface is obtained by reversing the notation appropriately.

2.4.5. Operator for the Point one Mesh away from two Perpendicular Interfaces

Referring to fig. 14, the string pattern of the operator for point 0 can be given as shown in fig. 15.

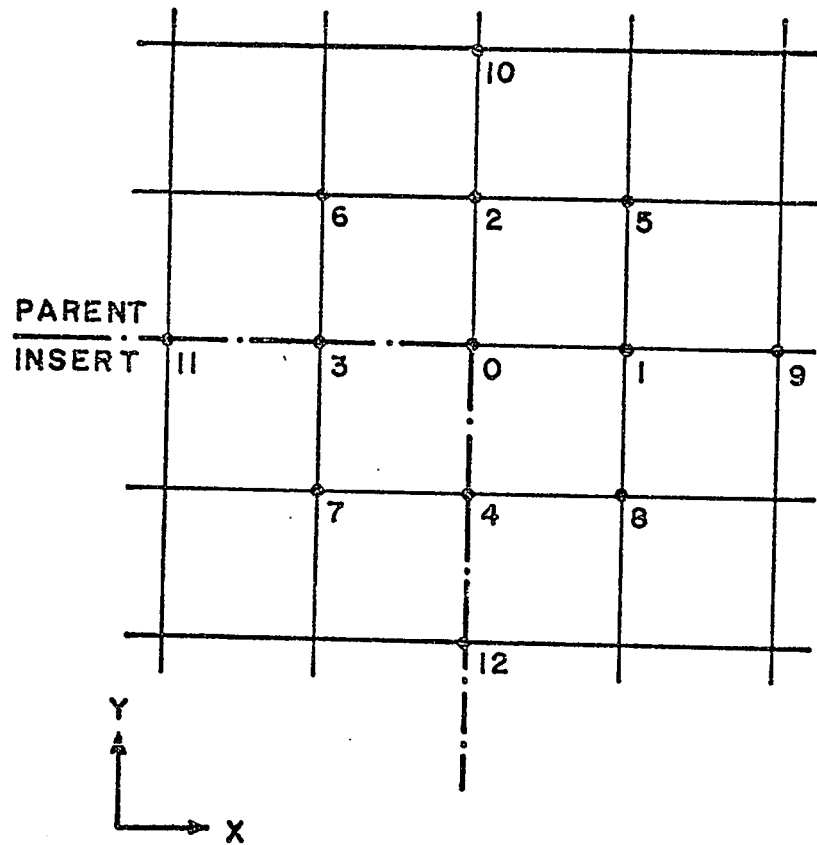


FIG.- 16

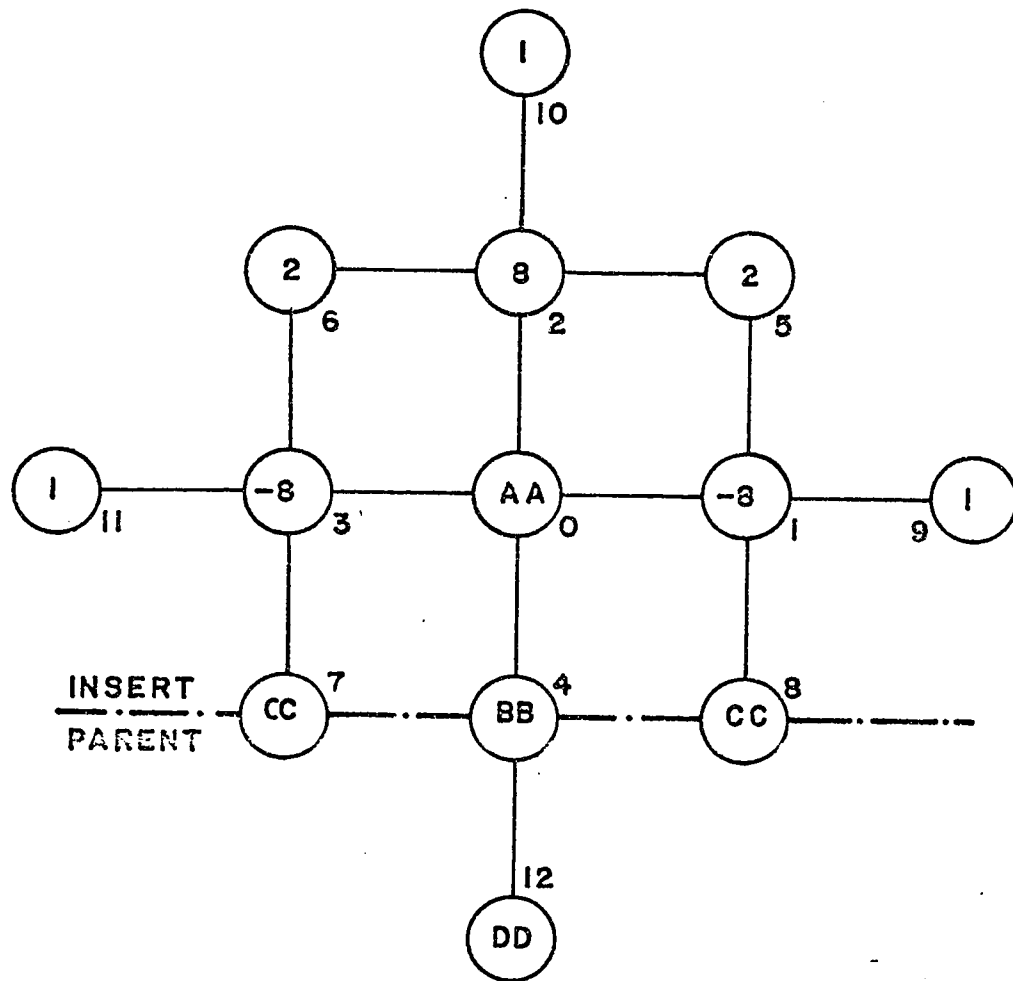


FIG.-17

2.4.6. Operator for the Point at the Intersection of two Perpendicular Interfaces

Referring to fig. 16, the operator for the point 0 at the intersection of two perpendicular interfaces can be given by the string pattern shown in fig. 17.

2.5. EQUATIONS OF BOUNDARY

Referring to fig. 3, the equations of the lines constituting the outside boundary of the plate are as follows:

Case II

$$y_{AB} = x - \frac{1}{\sqrt{2}}$$

$$y_{BC} = -x + \frac{5}{\sqrt{2}}$$

$$y_{CD} = x + \frac{1}{\sqrt{2}}$$

$$y_{AD} = -x + \frac{1}{\sqrt{2}}$$

3. COMPUTATION

To provide a mathematical tool of fairly wide scope, a three step computer program of sufficient flexibility is written for the solution of this problem.

Step 1

1. The first operation consists of formation of a grid on the plate.

In case I, since the center axes of the plate are parallel to the co-ordinate axes, as shown in fig. 2, the square mesh is so formed that the outer points coincide exactly with the boundary.

In case II, the full plate is considered and its axes are inclined at 45° to the co-ordinate axes. For simplification, the whole plate is placed in the first quadrant, so that two of its corners lie on the co-ordinate axes, as shown in fig. 3. All the nodal points are numbered. For convenience, the finite difference mesh is so chosen that the interface coincides with a mesh line. The mesh so formed is antisymmetric.

2. The position of the insert is identified in terms of the nodal points.

3. The ϕ values at the points which fall on the boundary are computed and also the fictitious points which are to be used in the biharmonic operator are written in terms of the real points.

4. The biharmonic equation $\nabla^4 \phi = 0$ is written in finite difference form. This is done for each nodal point, using the appropriate operator depending upon the position of the point in the plate. The set of simultaneous equations thus generated is stored on the tape.

Step 2

This step consists of the actual solution of the previously obtained set of equations. Several methods may be used. In our case, the methods chosen were matrix inversion and Gauss-Siedel iterative technique. The former was employed in case I, when the number of equations was not too

great, while the latter was preferred in case II, in which there were as many as 549 equations. The reason for choosing matrix inversion is that it takes less machine time; it does, however, require a large core storage. On the other hand, the slower Gauss-Siedel method can be programmed with a considerable storage economy, whenever a large proportion of elements are zeros [23].

Step 3

After the stress function values at each nodal point have been determined, this step is used to compute the normal and shear stresses. Once the stresses with respect to the x and y axes are obtained the principal stresses can easily be calculated.

The listings of the programs are given in appendix III.

SCHEMATIC DIAGRAM OF A PHOTOELASTIC BENCH

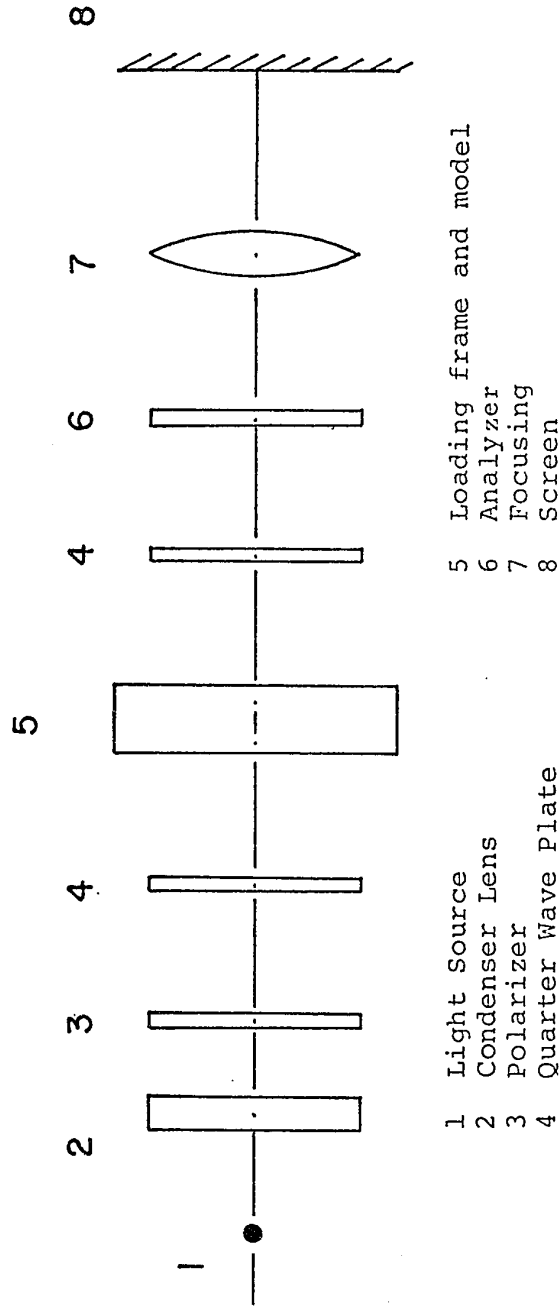


FIG.-18

CHAPTER 4

4. EXPERIMENTAL INVESTIGATION

The object of experimental stress analysis is to deduce the state of stress in a structural element subjected to some specified loading either by observation of the physical changes brought about in it by the applied loads or by measurements made on a model or analogue. The experimental stress analysis is more than just a counterpart of theoretical methods of stress analysis. It maintains direct contact with the true physical characteristics of the problem under consideration.

Although there are usually several possible methods available for carrying out an experimental investigation of stresses, it is important however to select the technique so as to obtain the most satisfactory results with the minimum of time, effort and money.

Under the circumstances, the photoelastic stress analysis appears best suited. The method of photoelasticity has the advantage of giving a complete picture of stress field under investigation. Further it makes use of comparatively small models which in turn require light and relatively inexpensive loading gear.

4.1 THEORY

The photoelastic method is based on the phenomenon of double refraction exhibited by transparent materials when subjected to stress. Loaded models are observed in a beam of polarized light. A simple arrangement is shown in fig. 18. Point light source is used, and a parallel beam is obtained by means of a condenser lens. The polarizer and analyzer are either natural crystals or "Polaroid" discs. The quarter-wave plates are of mica. Their thickness being related to the wavelength of the light source. The model is carried in a loading frame. Light and dark colored bands are produced on the screen and may be photographed for subsequent analysis. An outline of the effects obtained and their interpretation is given below.

If the quarter-wave plates are removed, plane polarized light is produced, and with the axes of the polarizer and analyzer at 90° no light is transmitted

to the screen. When the specimen is loaded, however, some light is passed through, except at points where the direction of one principal stress is in the plane of polarization. The result is a dark band across the model indicating the locus of all points at which the principal stresses are in the same directions. These bands are called isoclinics, and by rotating the polarizer and analyzer together a number of isoclinics can be obtained for various directions of principal stress.

The quarter-wave plates, placed with their axes at 45° to those of the polarizer and analyzer, produce circularly polarized light, and serves to cut out the isoclinics.

Under this arrangement, a color map of lines of equal color appear on the screen. These lines are called isochromatics. Every equal color line represent a constant level line of stress. Beginning at the lower level and progressing to areas of higher level, the color sequence observed will be black, yellow red, blue, yellow, red, green, yellow, red, green etc. .

At a particular point on the model the difference of the principal stresses is given by [24]

$$P_1 - P_2 = \frac{N f_\sigma}{H}$$

H of the material is known and f_σ is determined by calibration. Thus the remaining number to be determined is N . The fringes are related to increasing order as follows:

Along the black fringe	$N = 0$
Along the first fringe (Red - Blue)	$N = 1$
Along the second fringe (first Red - Green)	$N = 2$
Along the third fringe (second Red - Green)	$N = 3$
Along the fourth fringe (third Red - Green)	$N = 4$

4.2 SEPARATION TECHNIQUES

The optical data from the polariscope give only the difference of the principal stresses and their directions. The separate values of each stress can be obtained by different methods and the choice of the method determines the experimental procedures, which must be adopted.

With our polariscope we have the choice of using either shear difference method or oblique incidence method but the shear difference method is preferred for following reasons:

1. In case of oblique incidence method the difficulty can arise where the stress gradient is very steep.
2. In the case of oblique incidence method it is essential that the plane stress conditions be attained. Any stress present in any other plane, which normally is not detected when observed at normal incidence, would greatly affect the result when studied under oblique incidence.

Shear difference method is based on the equations of equilibrium, for the two-dimensional stress state. The equations of equilibrium in the absence of body forces are

$$\frac{\partial P_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} = 0 \quad (4.2.1)$$

$$\frac{\partial P_y}{\partial y} + \frac{\partial \tau_{xy}}{\partial x} = 0 \quad (4.2.2)$$

The solutions of these equations in terms of the normal stresses

P_x and P_y can be obtained from :

$$(P_x)_{x_1} = (P_x)_{x_0} - \int_{x_0}^{x_1} \frac{\partial \tau_{xy}}{\partial y} dx \quad (4.2.3)$$

$$(P_y)_{y_1} = (P_y)_{y_0} - \int_{y_0}^{y_1} \frac{\partial \tau_{xy}}{\partial x} dy \quad (4.2.4)$$

where $(P_x)_{x_0}$ and $(P_y)_{y_0}$ represent the values of P_x and P_y at a given location (for which the stress values are known). Usually these are the values at the free boundary. In the eqns. (4.2.3) and (4.2.4), the value of the integral can be represented by the area under a curve for which the ordinates are the rate of change of shear in a direction normal to the direction of integration and the abscissas are distances along the path of integration.

Equation (4.2.4) can be evaluated numerically by employing finite difference method, as shown below

$$(P_y)_{y_1} = (P_y)_{y_0} - \Delta y \left(\frac{\Delta \tau_{xy}}{\Delta x} \right) \frac{(y_0 + y_1)}{2} \quad (4.2.5)$$

where y , y_1 etc. represent positions along y .

The value of $\Delta \tau_{xy}$ can be computed using the isoclinic and isochromatic data obtained in the following relation.

$$\begin{aligned} \tau_{xy} &= \frac{P_1 - P_2}{2} \cdot \sin 2\theta \\ &= \frac{N f \sigma}{2 H} \cdot \sin 2\theta \end{aligned}$$

where θ is the angle between the x-axis and the direction of P_2 as given by isoclinic parameter.

Hence it is possible to determine P_y at the interior point y_1 . If the procedure is continued stepwise, the value of P_y can be determined across the entire length of the line.

$$P_y | y_2 = P_y | y_1 - \Delta_y \left(\frac{\Delta \tau_{xy}}{\Delta x} \right) \frac{(y_1 + y_2)}{2}$$

$$P_y | y_3 = P_y | y_2 - \Delta_y \left(\frac{\Delta \tau_{xy}}{\Delta x} \right) \frac{(y_2 + y_3)}{2}$$

$$P_y | y_4 = P_y | y_3 - \Delta_y \left(\frac{\Delta \tau_{xy}}{\Delta x} \right) \frac{(y_3 + y_4)}{2}$$

Having obtained P_y along the required line, we proceed further to evaluate P_x using either Mohr's Circle or the analytical expressions, as follows

$$P_x = P_y - (P_1 - P_2) \cos 2\theta \quad (4.2.6)$$

Substituting $(P_1 - P_2)$ from eqn. (4.1.1) into eqn. (4.2.6), we get

$$P_x = P_y - \frac{Nf}{H} \sigma \cdot \cos 2\theta \quad (4.2.7)$$

Now from the first invariant of stress, we have

$$P_x + P_y = P_1 + P_2 \quad (4.2.8)$$

Solving eqns. (4.1.1) and (4.2.8), we get P_1 and P_2 as

$$P_1 = \frac{1}{2} \left(P_x + P_y + \frac{Nf}{H} \sigma \right)$$

$$P_2 = \frac{1}{2} \left(P_x + P_y - \frac{Nf}{H} \sigma \right)$$

In this manner, using the method of shear difference we can separate the stresses along any arbitrary line.

4.3 MATERIALS FOR MODELS

For the success of this investigation, it is necessary that the material should meet the prime characteristics such as complete freedom from internal stresses, excellent transparency, very low time edge effect, higher sensitivity etc. .

PSM 1 and PSM 4 are the trade names of the plastics used in this case which have the above-mentioned characteristics. The elastic properties of the materials are as follows:

(i) PSM-1

This is a hard polyester sheet with a nominal thickness of 0.25 in.

The elastic properties of PSM-1 are

$$E = 340,000 \text{ psi}$$

$$\mu = 0.38$$

$$f_{\sigma} = 36 \text{ psi per fringe per inch}$$

(ii) PSM-4

This is a soft polyurethane sheet,

The elastic properties of PSM-4 are

$$E = 1000 \text{ psi}$$

$$\mu = 0.4$$

$$f_{\sigma} = 3 - 5 \text{ psi per fringe per inch}$$

This is used as an insert.

A special cement has been used to get a bond as free from slippage as possible. The cement has the same elastic properties as PSM-4.

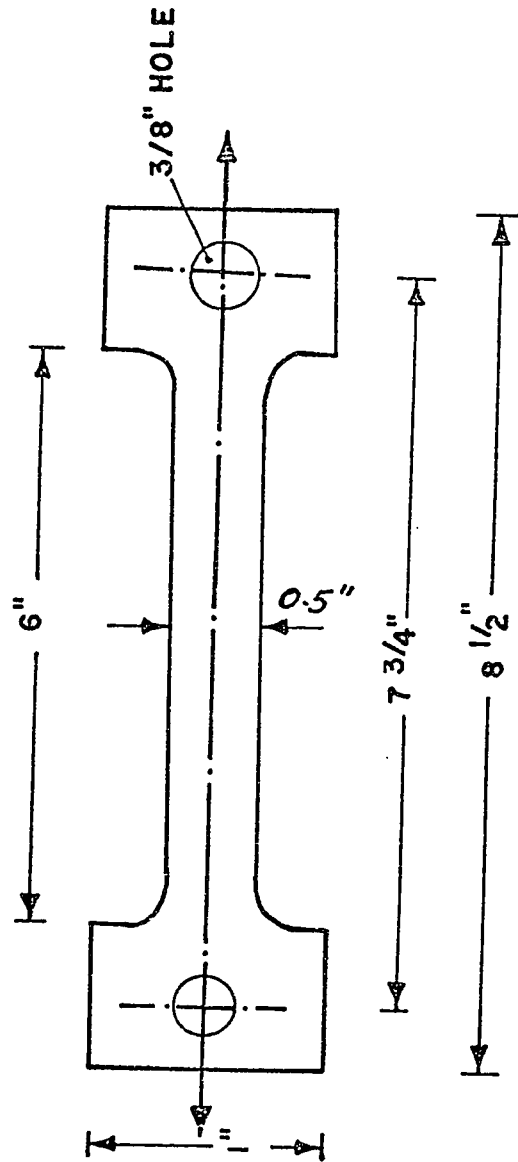


FIG.-19

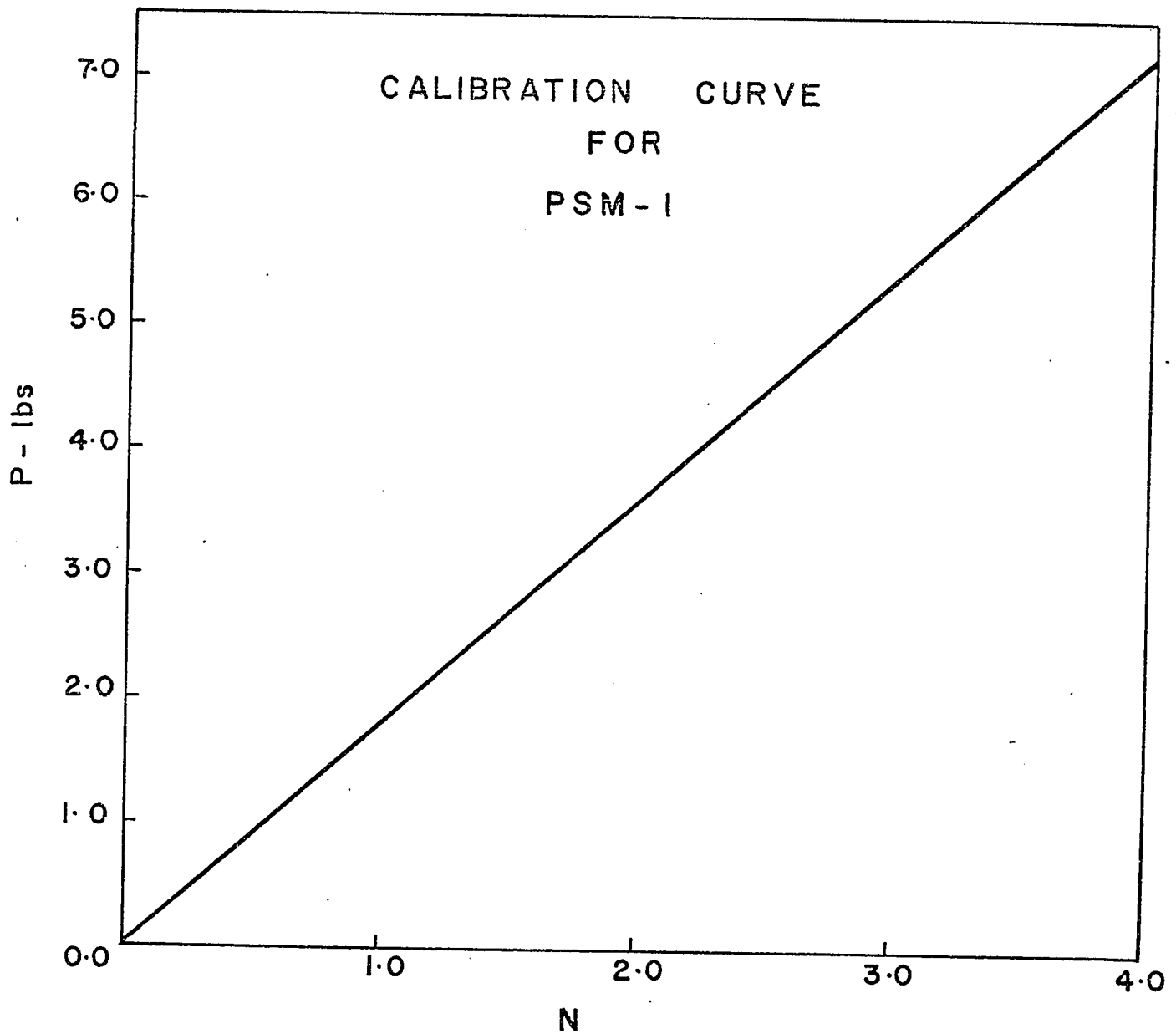


FIG.- 20

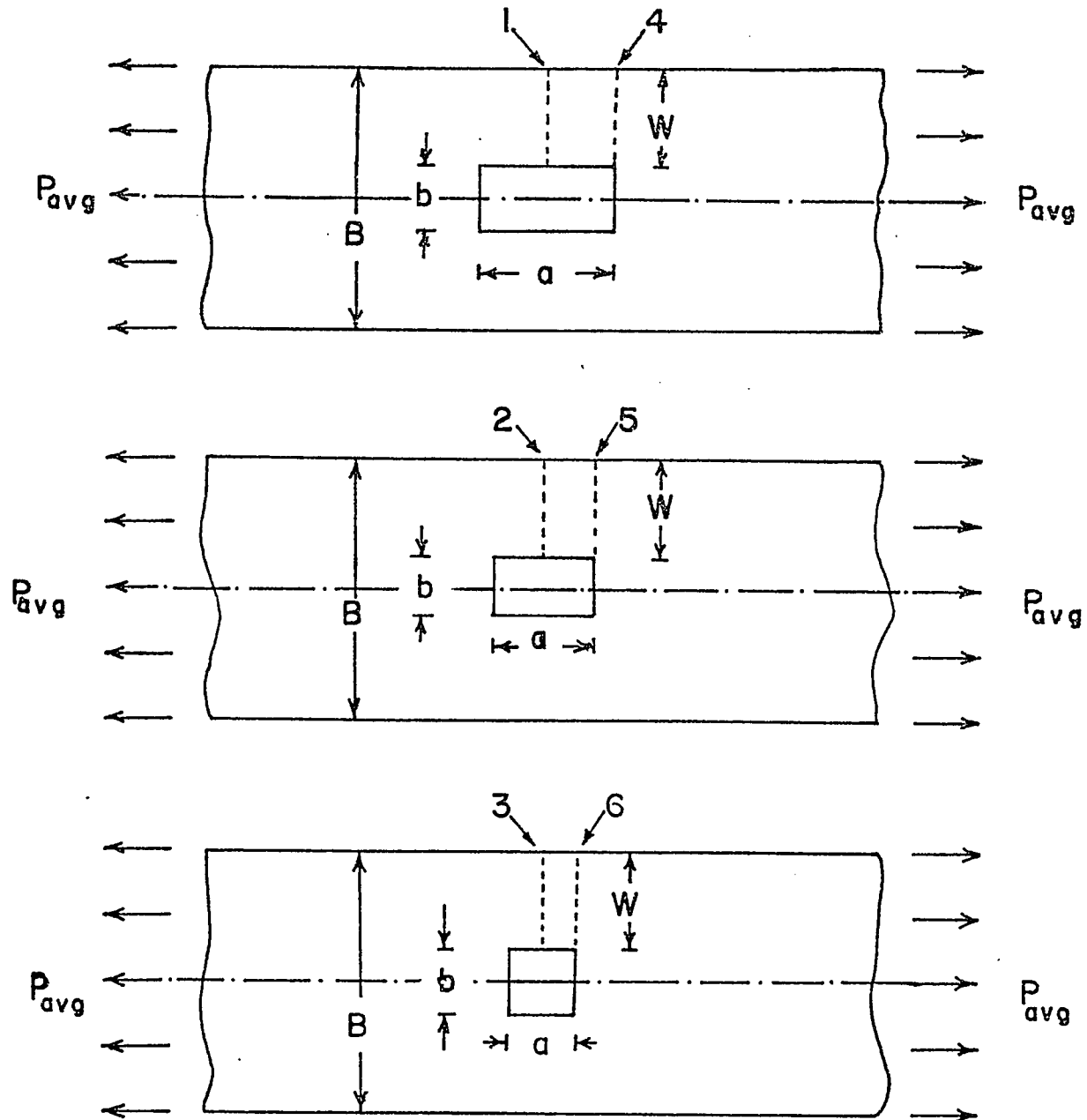


FIG.-21
SHEET-A

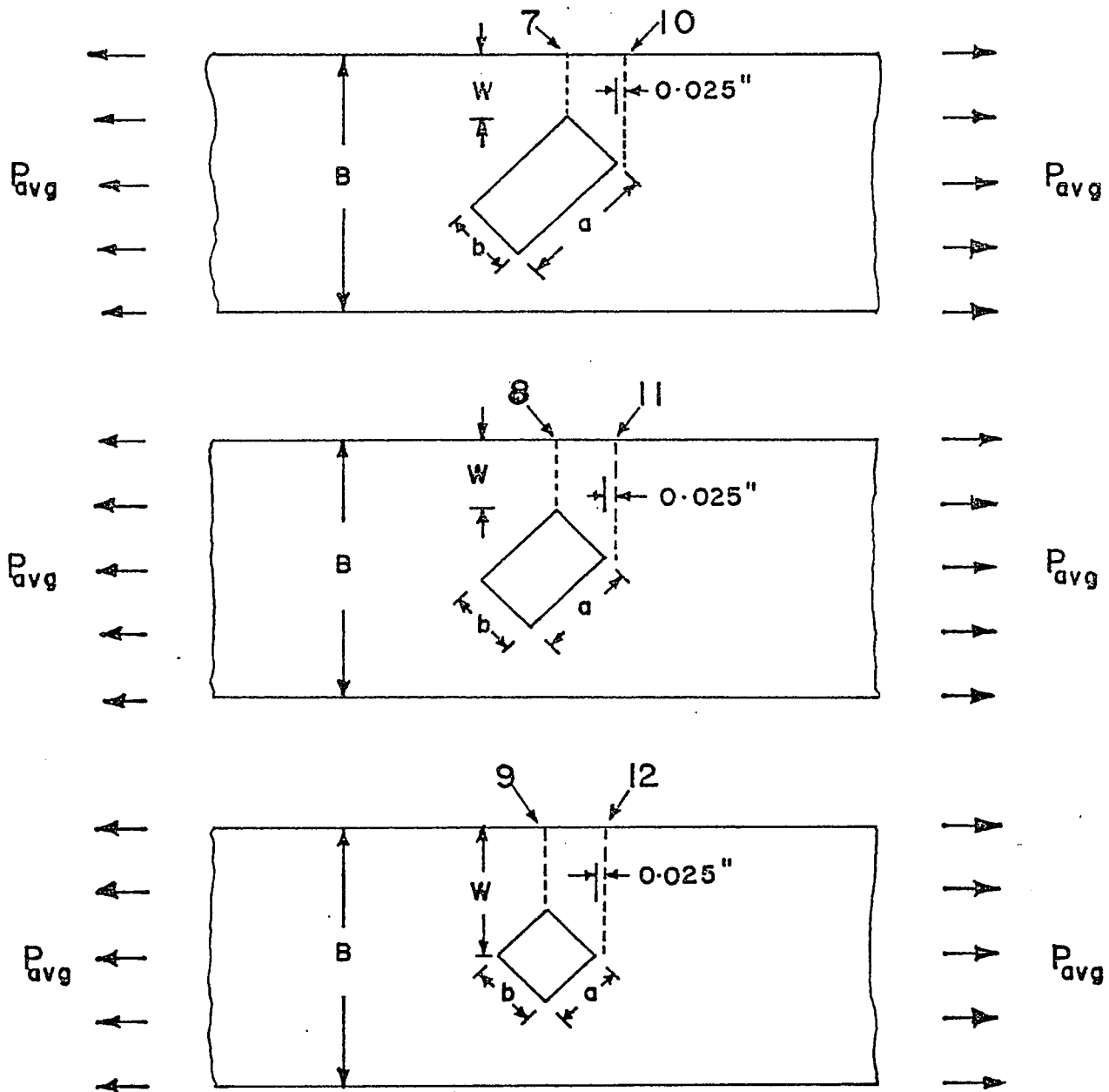


FIG.- 21
SHEET-B

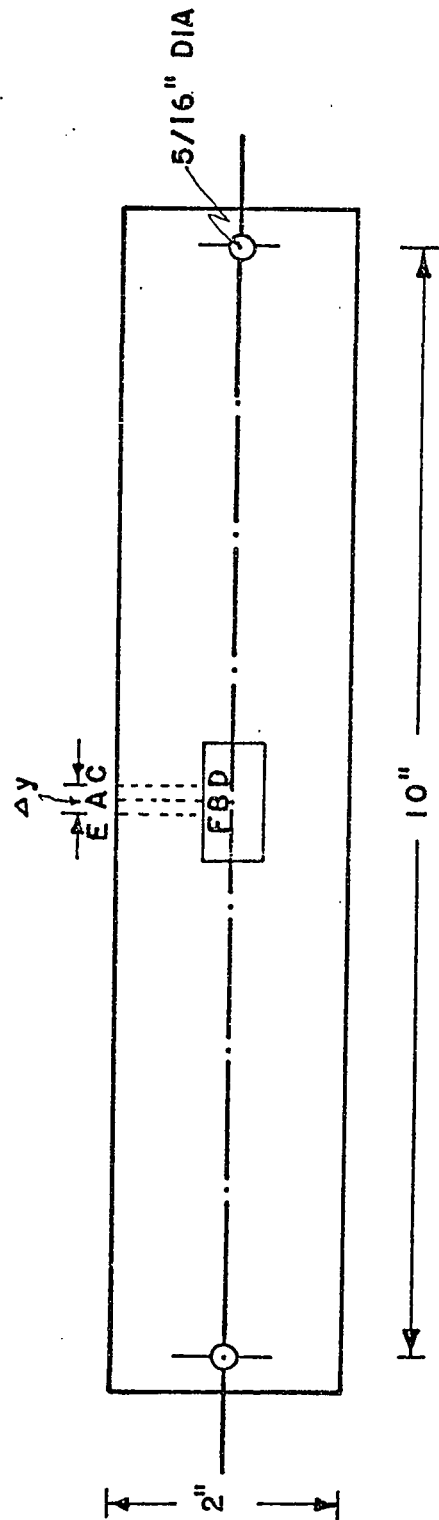


FIG- 22

4.4 CALIBRATION OF MATERIAL

The accuracy of the evaluation of stress distribution depends mainly on material fringe value of the material. Though this value is often provided by the supplier, it is apt to vary due to the effects of time and temperature. Therefore, calibration of the material is necessary in order to obtain accurate results.

A tension bar shown in fig. 19 is employed for the calibration of the material. The calibration model is loaded in steps. The corresponding fringe orders and loads are noted.

The axial stress induced in the necked region of the tensile specimen by load P may be expressed as

$$P_1 = \frac{P}{wH} \quad \text{and} \quad P_2 = 0$$

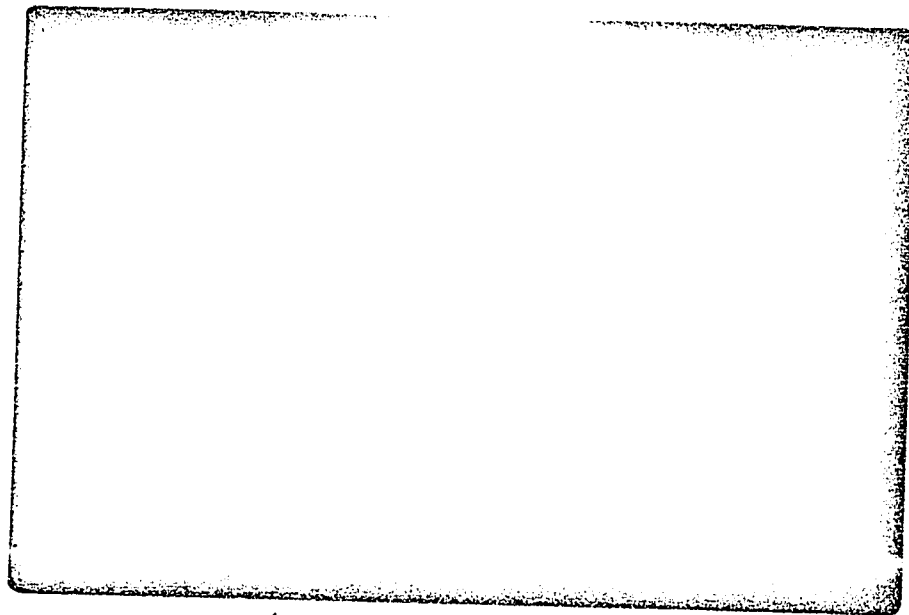
where w is the width at the necked region and H is the thickness of the specimen. Substituting this in eqn. (3.1.1), we get

$$f_{\sigma} = \frac{P}{wH}$$

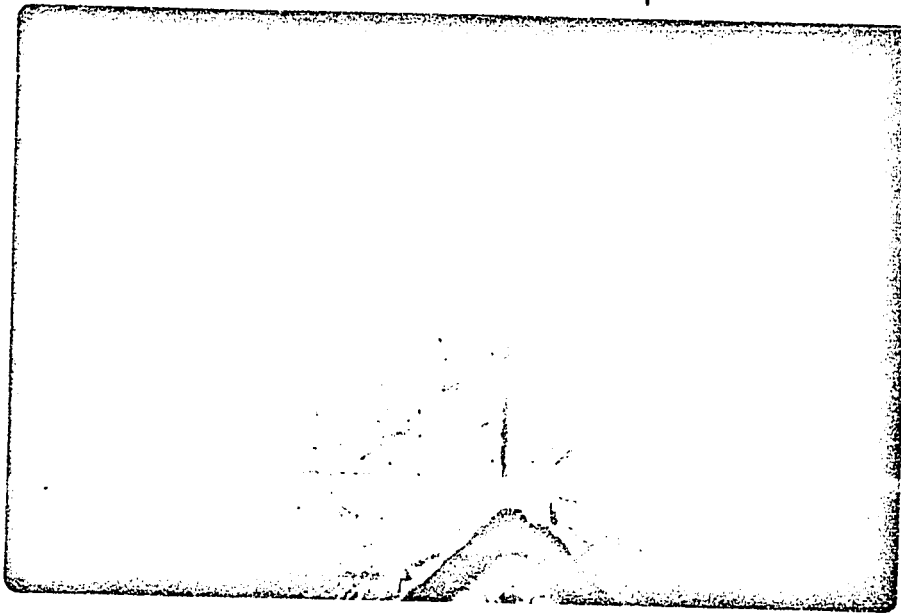
A curve of the load P is plotted as function of N for four different points which give a straight line. The slope of this straight line is used for the value of P/N . Figure 20 shows the calibration curve of material PSM-1.

4.5 EXPERIMENTAL PROCEDURE

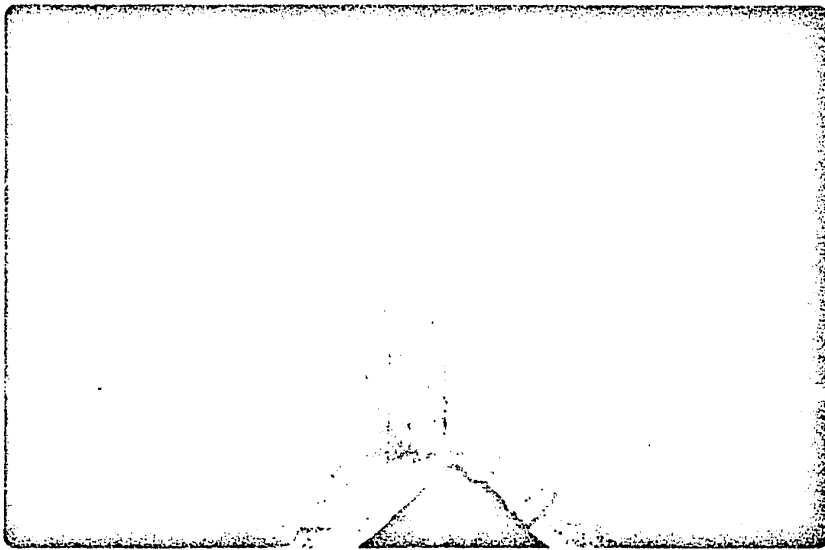
For each specimen, the stress distribution has been studied along two different lines. These lines in case of each specimen are shown in fig. 21. Two more lines are drawn distant $\frac{\Delta x}{2}$ on either side of the line under consideration as shown in fig. 22.



- (a) Constant fringe order across a cross-section
away from the insert.



(b) Isochromatics of specimen with 45°
inclined insert.



(c) Isoclinics of specimen with 45°
inclined insert.

The readings of isoclinics and isochromatics are taken along these lines at various points starting from the boundary and proceeding inwards. A smaller step size is used in the region where the change of stress is rapid.

A few photographs representative of the stress pattern in specimen are given on pages 48 - 50 with appropriate comments.

4.6 COMPUTATION

A computer program based on shear difference method has been written for the calculation of principal stresses and shear stresses from the experimental data. A listing of the program is given in appendix III.

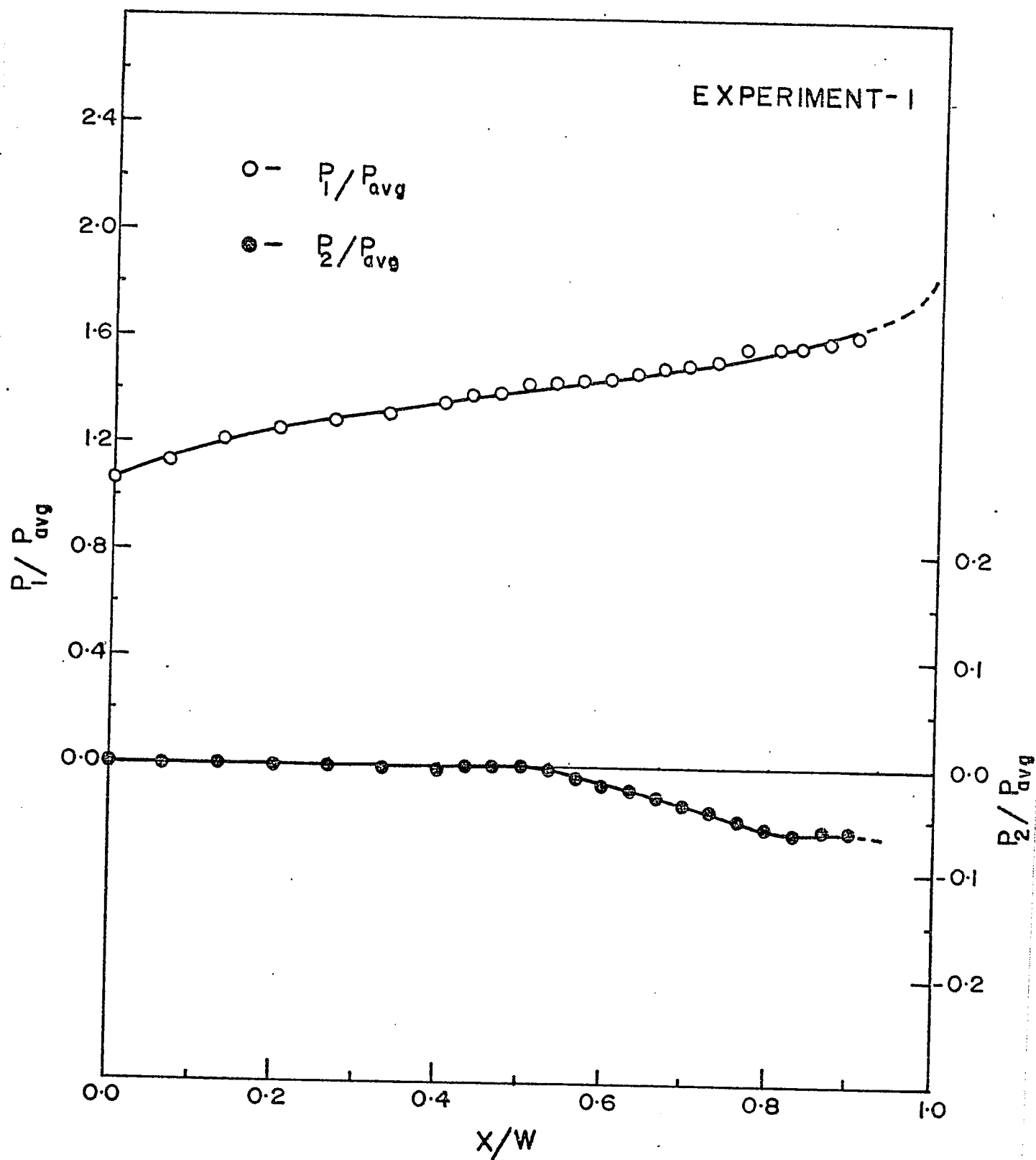


FIG.- 23

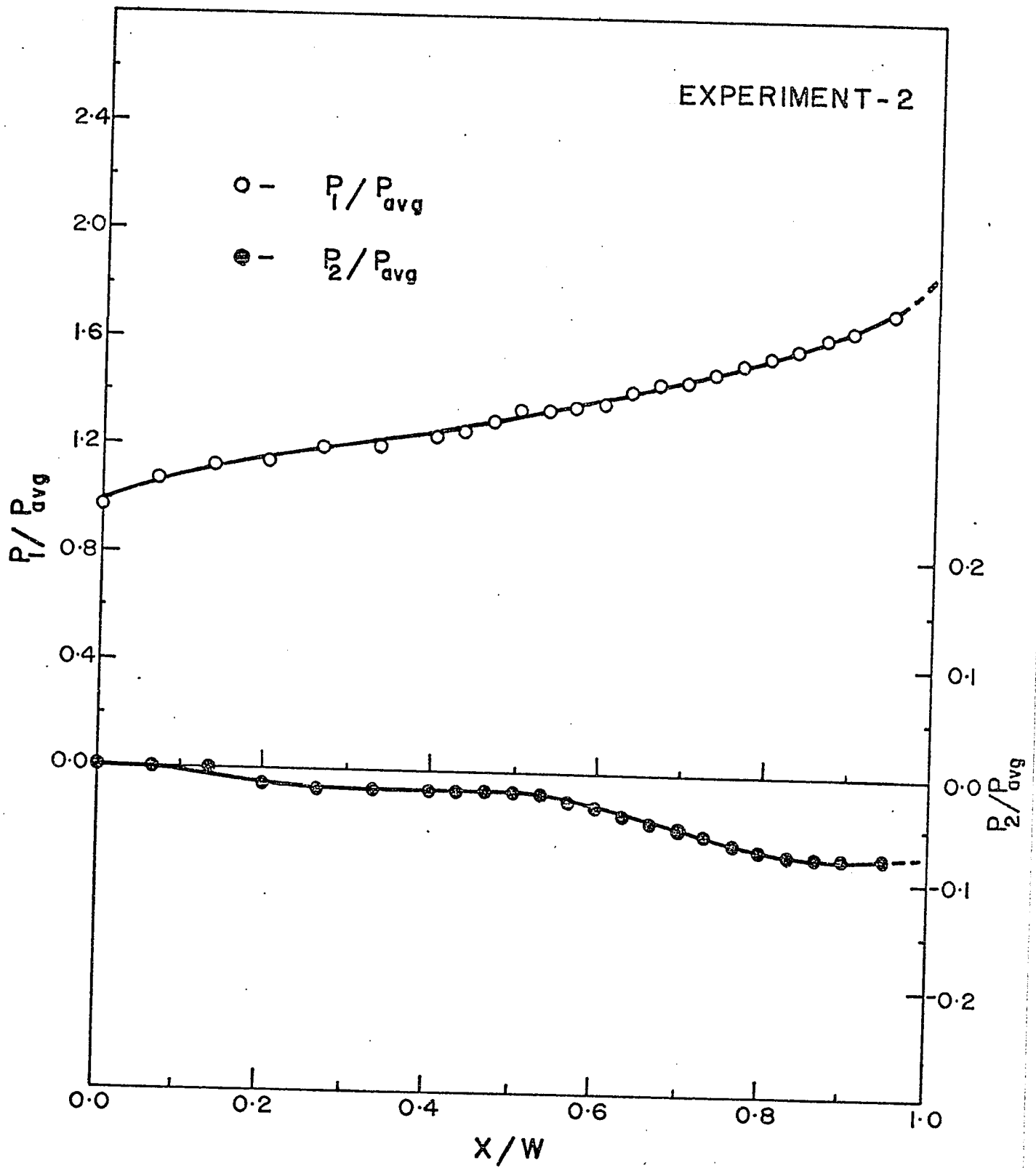


FIG-24

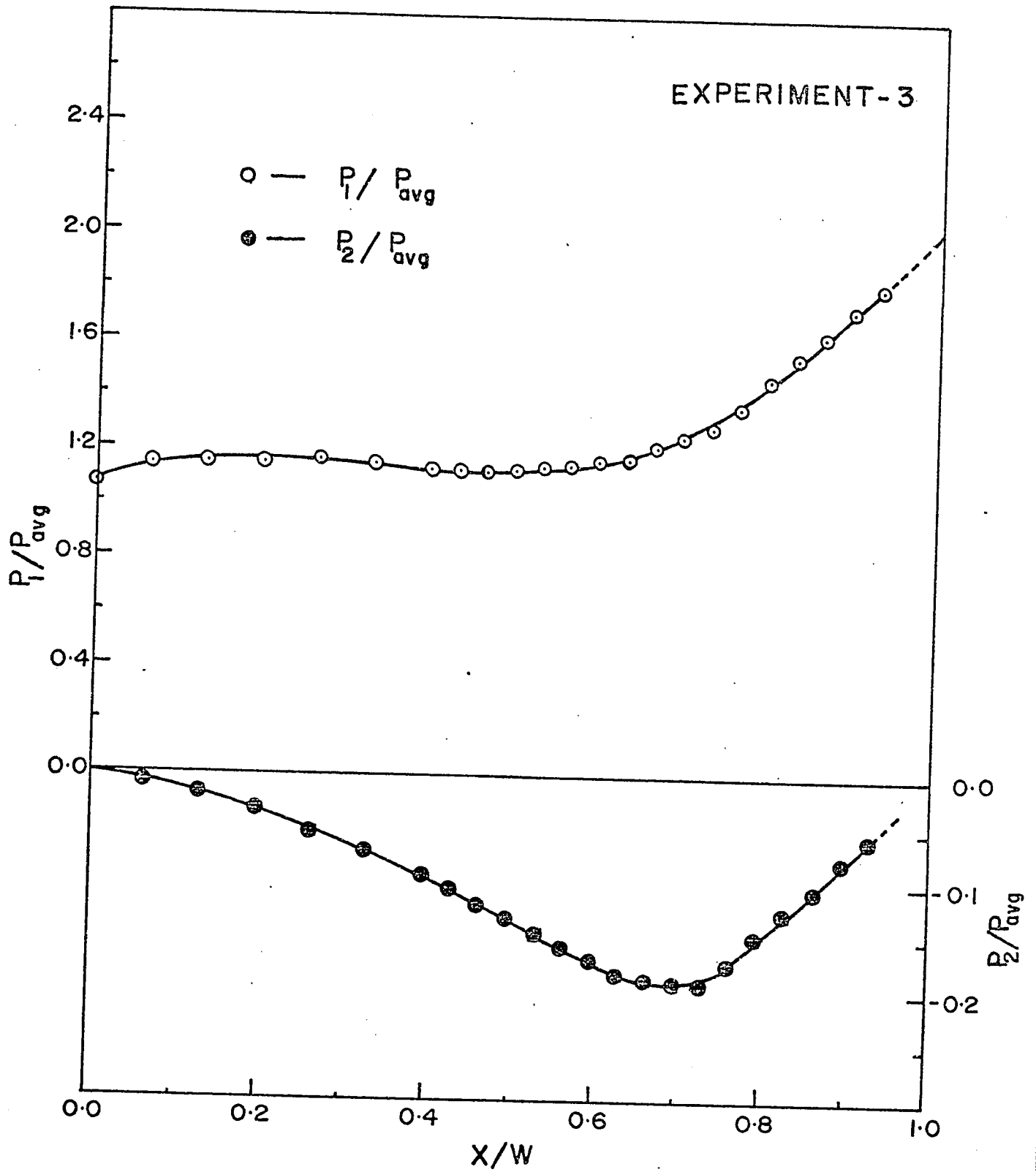


FIG.-25

CHAPTER 5

5. DISCUSSION OF RESULTS AND CONCLUSIONS

The experimental work reported in this thesis consists of twelve experiments performed on six specimens shown in fig. 21. Each specimen is a rectangular plate with a centrally located rectangular insert. The dimensions of the specimens are given in table 1.

The distribution of principal stresses and shear stresses has been studied in each case. The plots are made for P_1/P_{avg} , P_2/P_{avg} and τ_{xy}/P_{avg} against a non-dimensional variable x/w where

P_{avg} = the stress at a distance far away from the insert.

x = the distance of any point along the line from the free boundary.

w = the distance of the insert along the line from the free boundary.

The value $x/w = 0$ represents the free boundary and the value $x/w = 1$ represents the interface between the insert and the parent material.

As would be noticed in figs. 23 - 34, that the experimental curves are not plotted upto the interface. This is due to the following reasons:

Though every care has been taken in the preparation of the specimens, the perfection is hard to achieve. A slight imperfection in the finish of the outer boundary of the specimen is magnified ten times on the screen of the polariscope. Further, dispersion of light takes place along the edge of the specimen and it is difficult to locate the points close to the boundary. The starting point is therefore, shifted away from the boundary whereas in the calculations the distance is taken as $x = 0$. Moreover an interface should be a thin line but in the specimen it shows up

TABLE 1

GROUP	EXP. NO.	DIMENSIONS*						$\frac{P_1 \text{ max.}}{P_{\text{avg}}}$	FRINGE ORDER ⁺
		A	B	a	b	w	θ		
I	1	10	2	1	0.5	0.75	0°	1.86	1.92
	2	10	2	0.75	0.5	0.75	0°	2.00	2.35
	3	10	2	0.5	0.5	0.75	0°	2.04	2.67
II	4	10	2	1	0.5	0.75	0°	2.56	2.78
	5	10	2	0.75	0.5	0.75	0°	2.60	2.84
	6	10	2	0.5	0.5	0.75	0°	2.70	2.92
III	7	10	2	1	0.5	0.47	45°	-	7
	8	10	2	0.75	0.5	0.56	45°	-	5
	9	10	2	0.5	0.5	0.65	45°	-	4
IV	10	10	2	1	0.5	0.82	45°	1.52	2
	11	10	2	0.75	0.5	0.91	45°	1.42	1.84
	12	10	2	0.5	0.5	0.1	45°	1.20	1.3

* All dimensions are in inches.

+ Fringe order at the interface or the corner depending upon the position of the line along which the stresses are being taken at 100 lbs load.

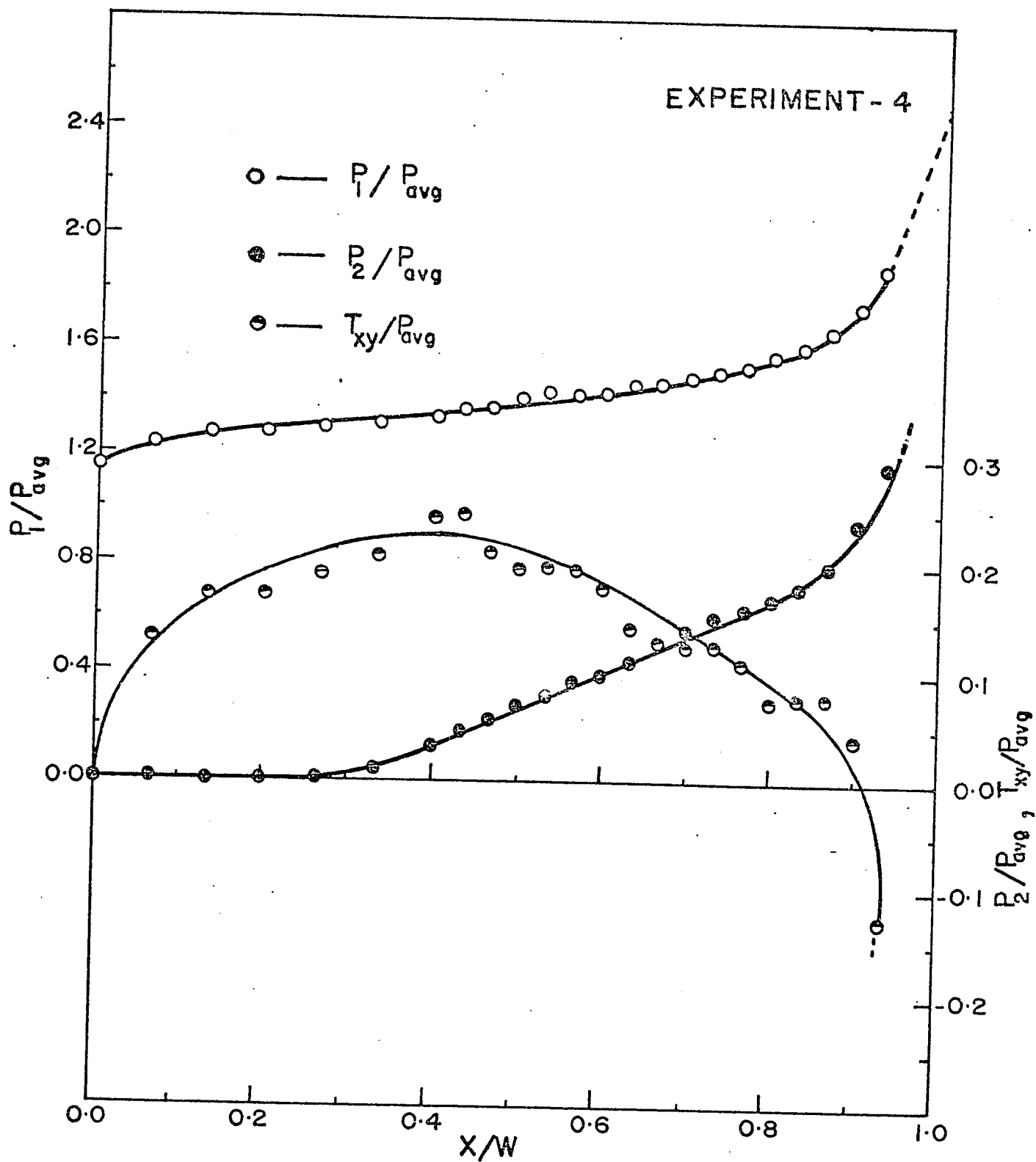


FIG.-26

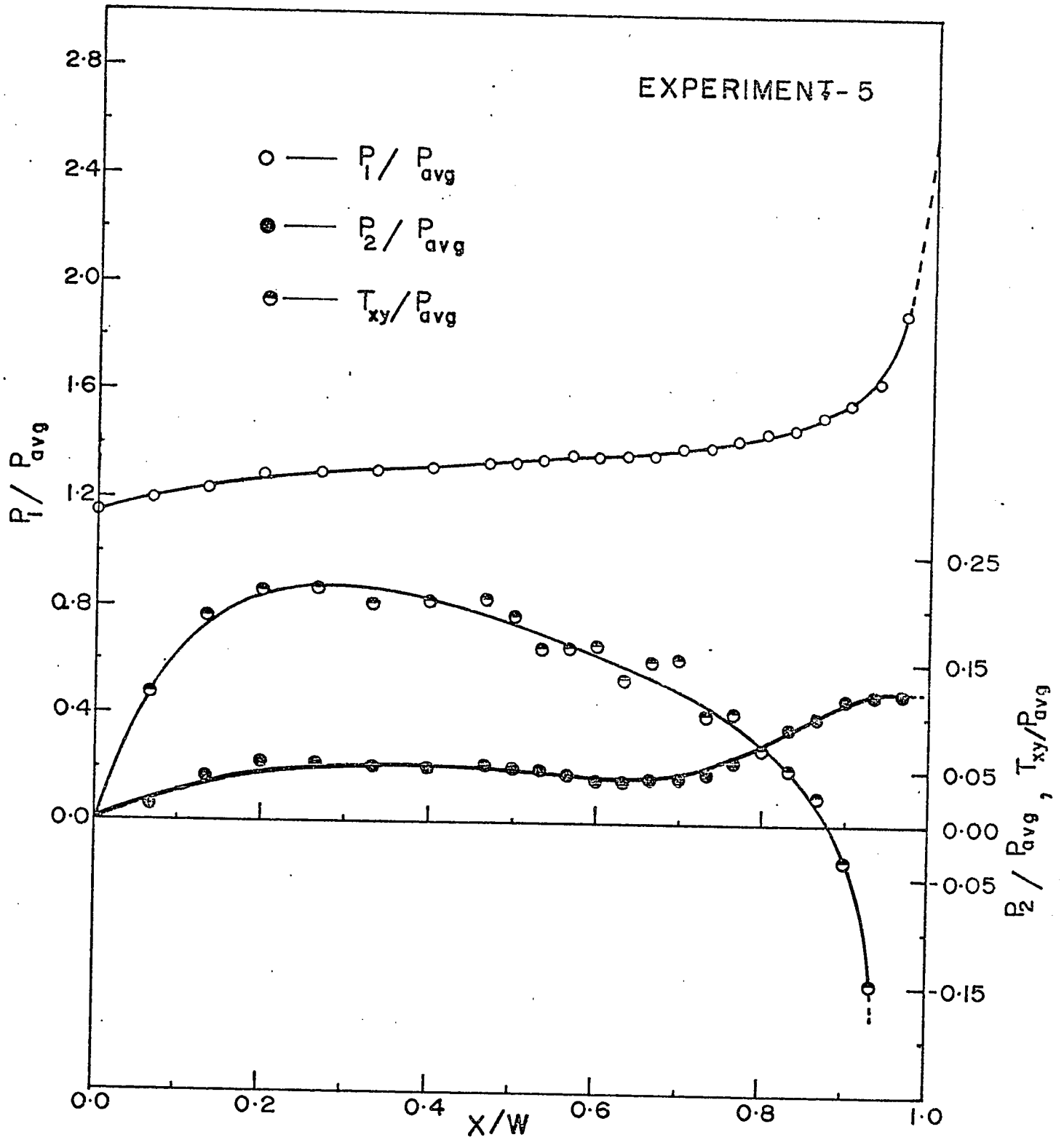


FIG.- 27

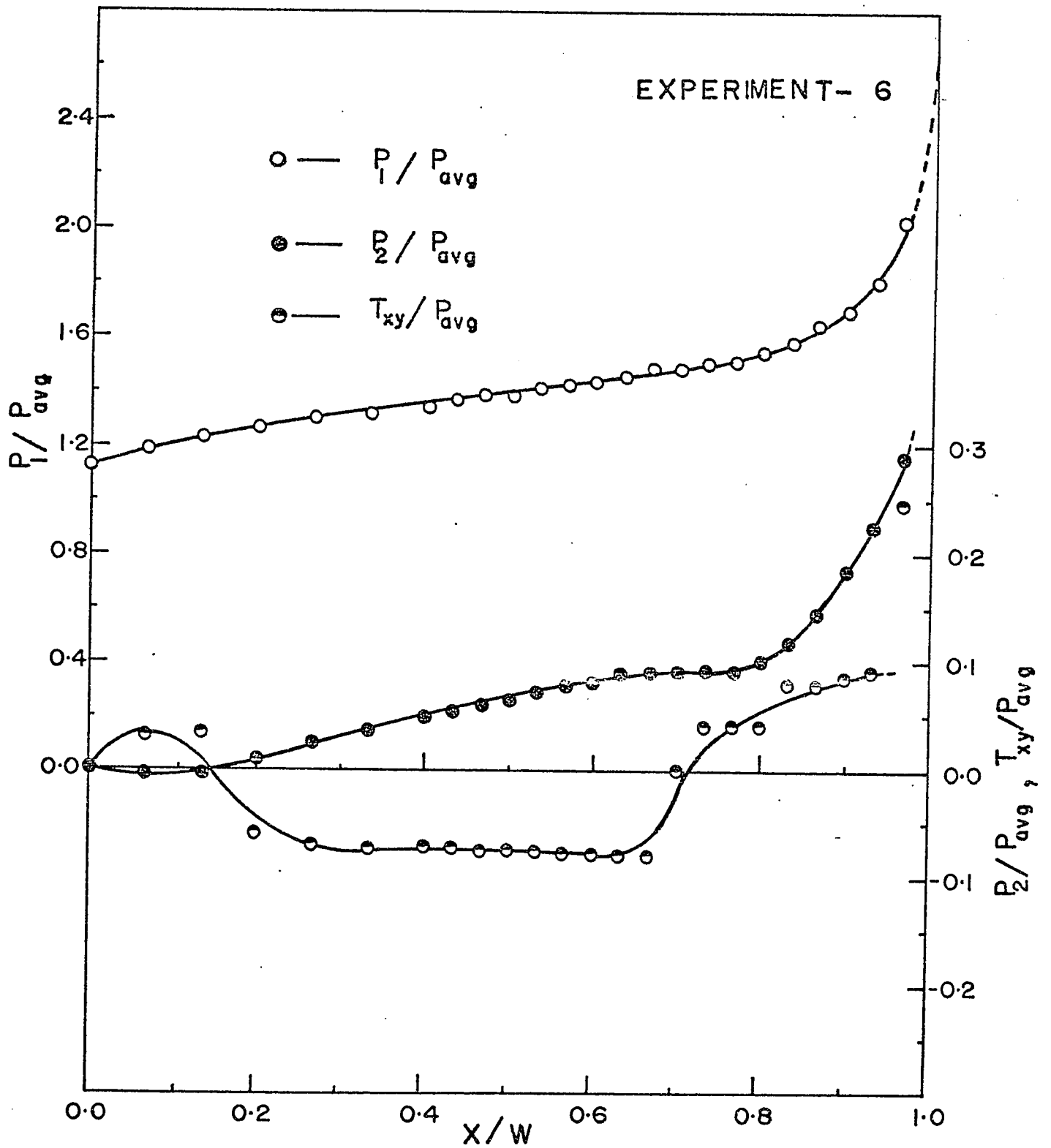


FIG.- 28

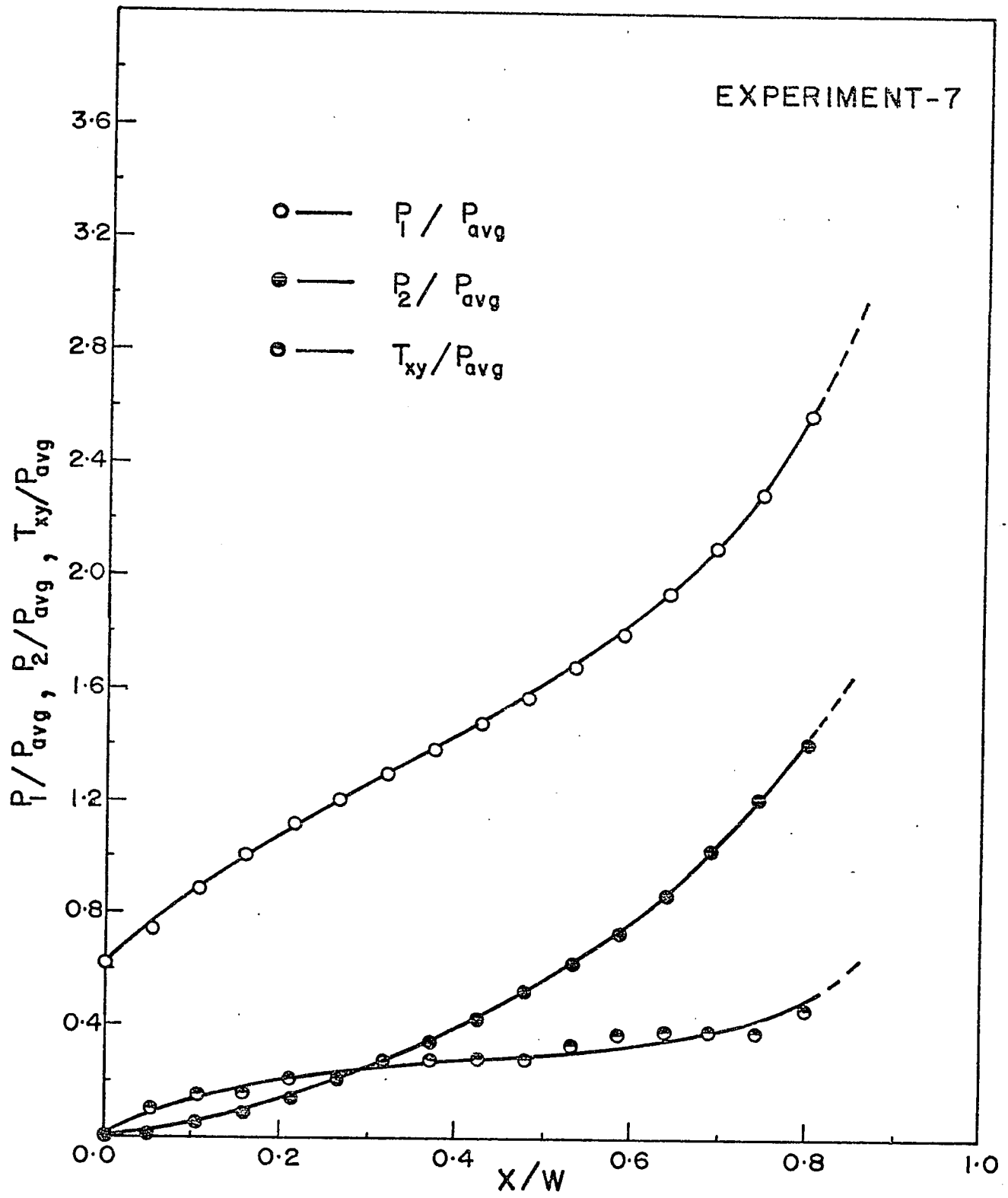


FIG.-29

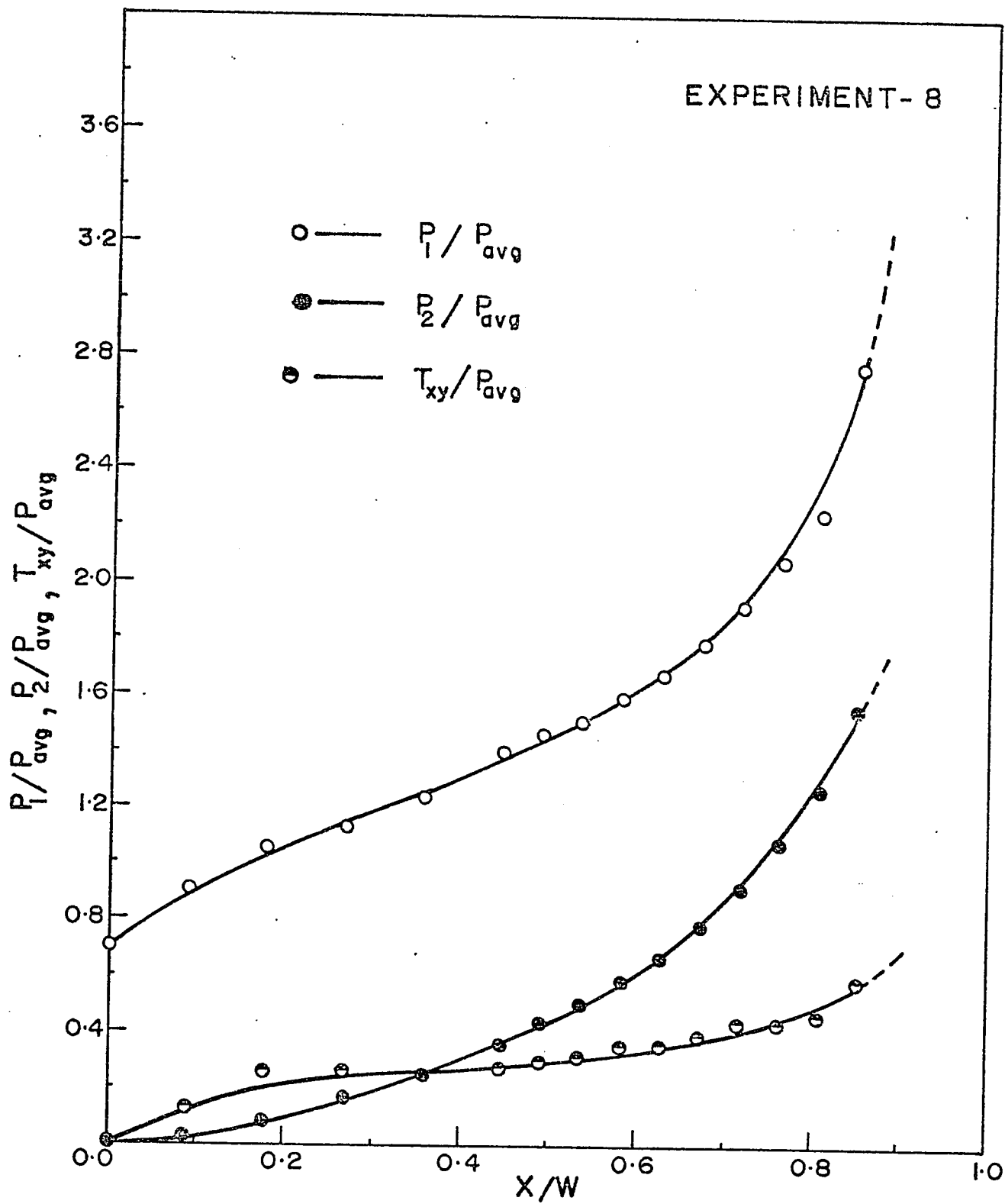


FIG.-30

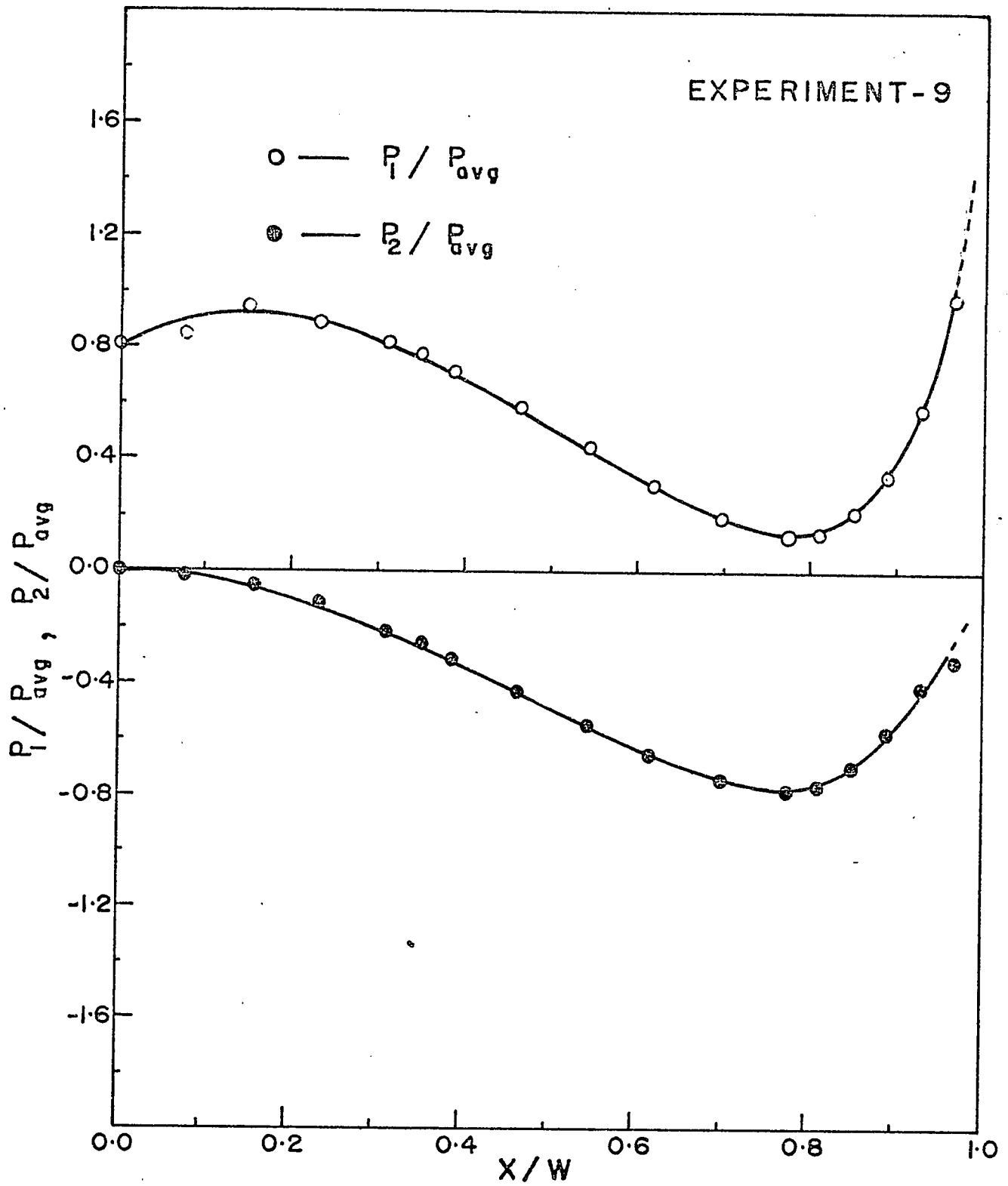


FIG.- 31

as a thick black line. This is due to the presence of the cement between the parent material and the insert even though a 'see through' type cement has been used. Further, due to high concentration of stresses near the interface it is very difficult to identify the fringes clearly. Therefore the readings in this region could not be taken.

In order to have some idea of stress distribution near the interface, a simple extrapolation has been used. This is shown by dotted lines. A comparative idea about the stresses can be had from the fringe order at the point of interest in the specimens under tension.

The twelve experiments have been divided in four groups as shown in table I for a comparative study.

In group I (figs. 23 - 25), the shear stress is zero. This is in agreement with the theory that shear stress is always zero on the line of symmetry. In all the three cases the maximum stress occurs at the interface and the minimum stress occurs at the outer boundary. The stress P_1 at the interface increases with the decrease in the size of the insert and the rate of increase is greater as we approach the interface in the case of smallest insert (case 3). For all the cases, the magnitude of the stress P_2 on the interface is very small.

In group II (figs. 25 - 27), the stress P_1 follows the same pattern as in group I.

In group III (figs. 29 - 31), the maximum stress concentration occurs at the corner in all the three cases but the pattern is slightly different from the other two groups. The stress P_1 at the interface has the largest value in the case of largest insert (case 7) and decreases with the decrease in size of insert. Also the rate of increase of slope decreases with the decrease in size of insert. The minimum value of stress P_1 in the case of smallest insert (case 9) occurs not at the outer boundary but somewhere in the middle. The effect of the corner on the outer boundary is also noted in this group. The shear stress in the case of smallest insert is zero,

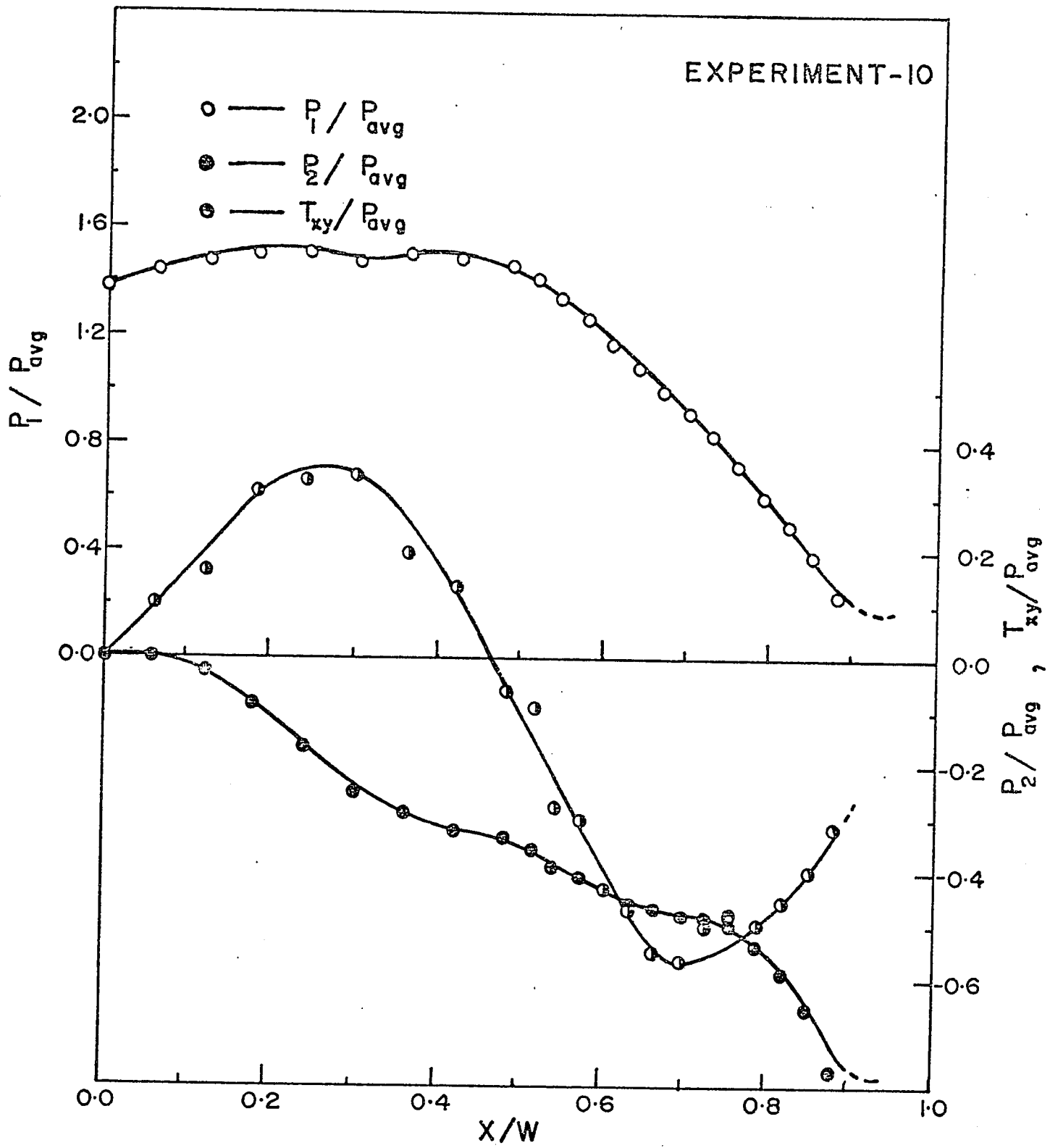


FIG.-32

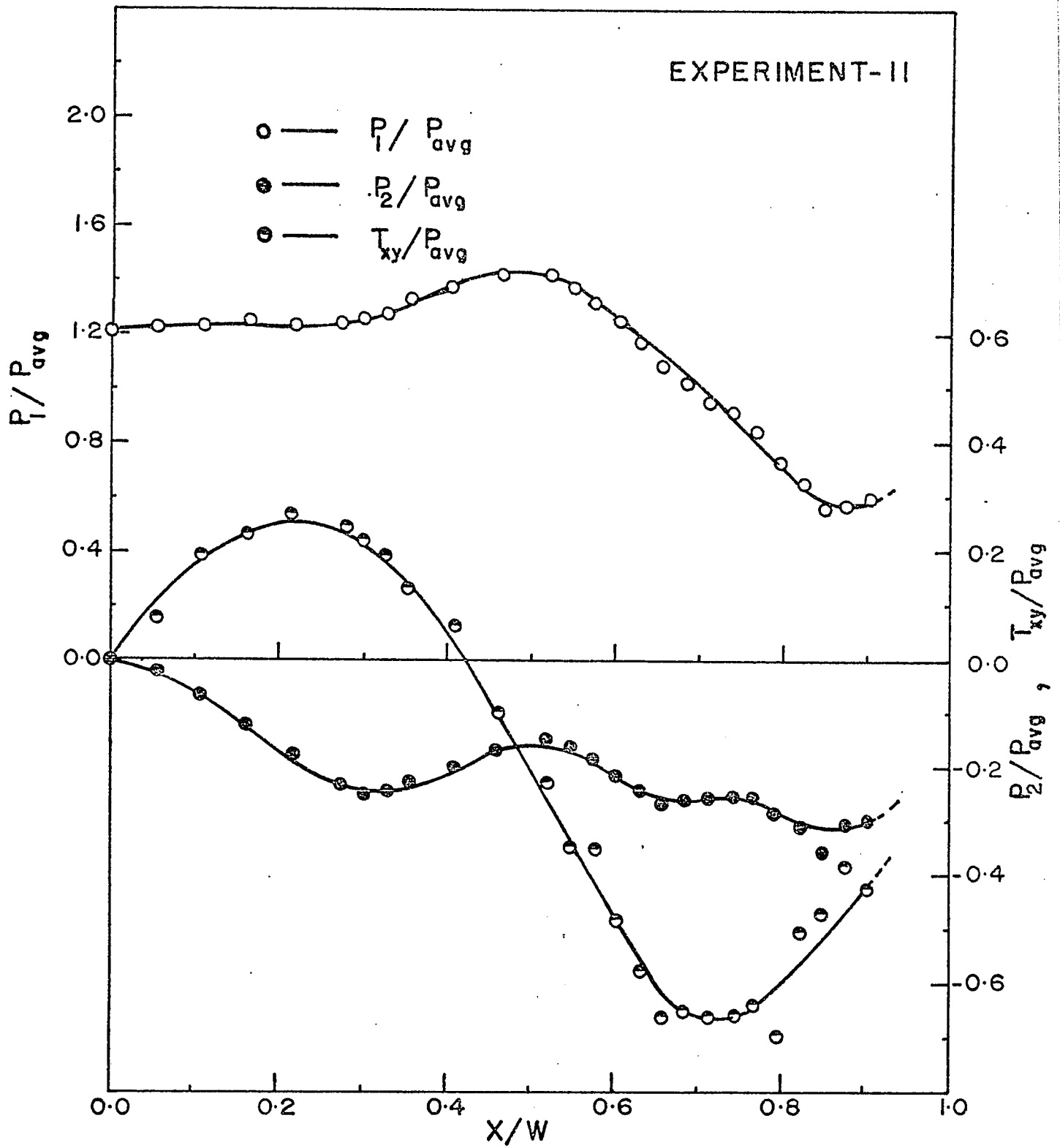


FIG.-33

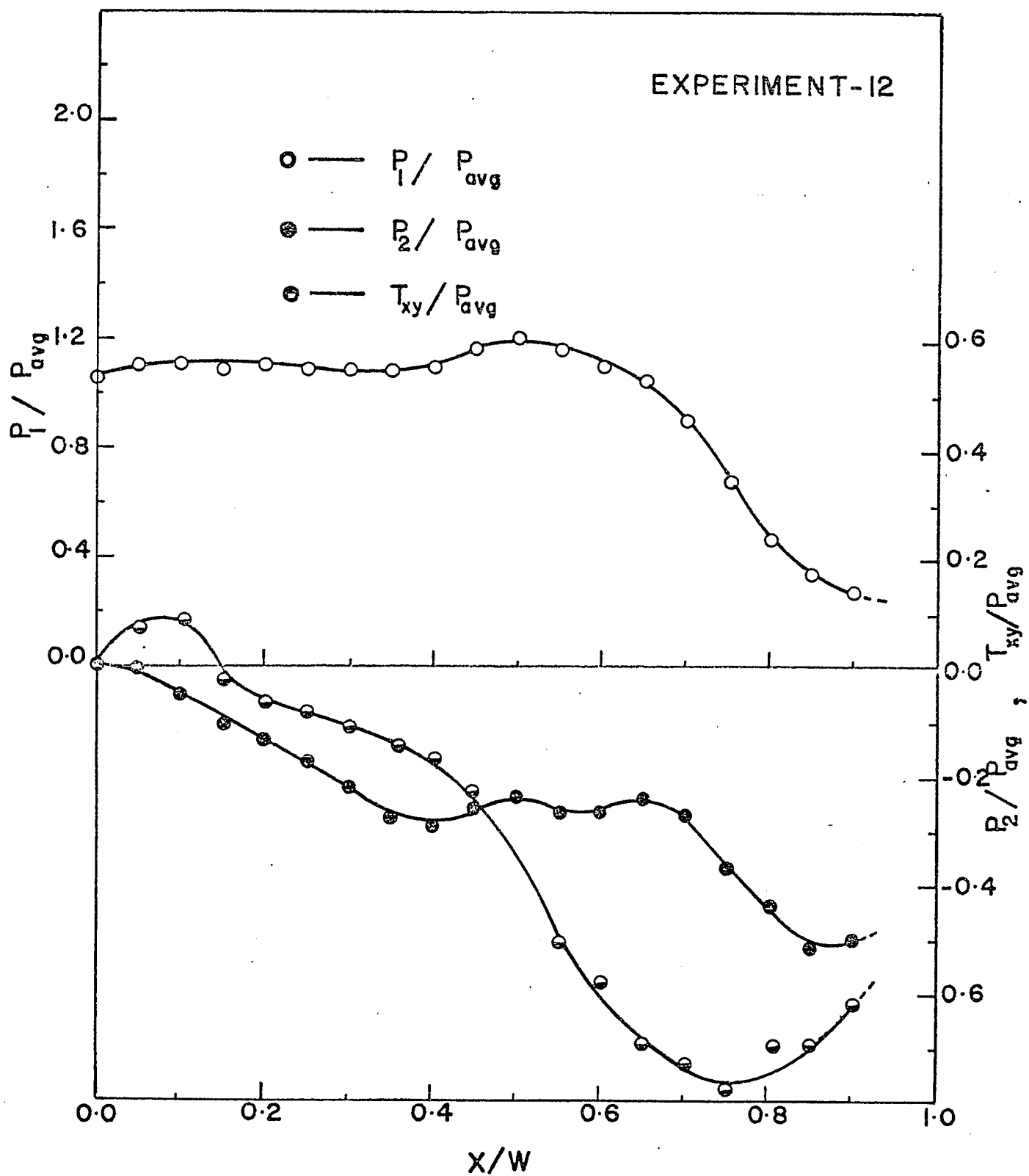


FIG.-34

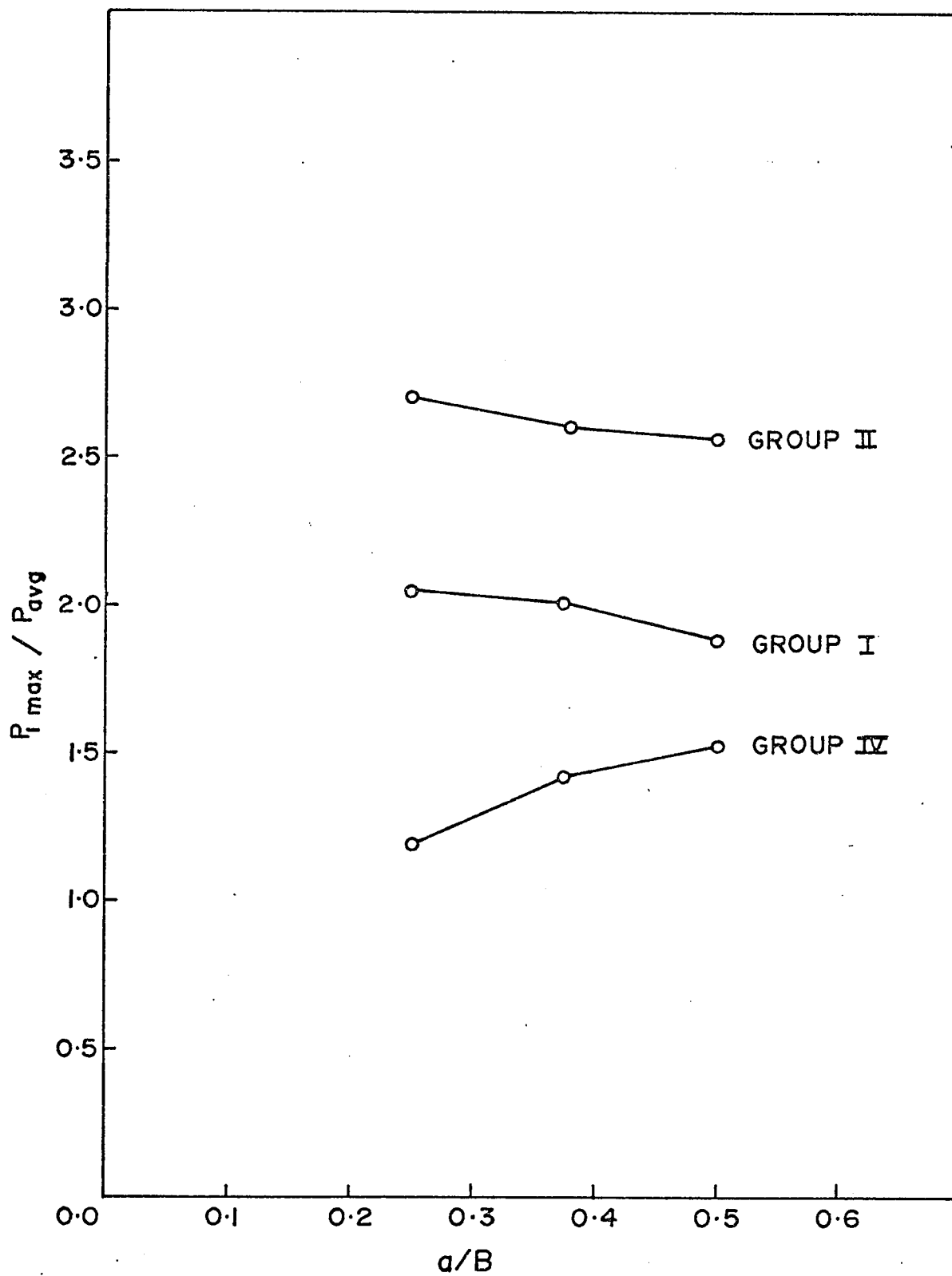


FIG. 35

because of symmetry.

In group IV, (figs. 32 - 34) the line along which the stress distribution is studied is 0.025" away from the corner. In this case, the maximum stress occurs somewhere in the middle and then it drops down rapidly, as we proceed towards the insert. The stress P_2 remains compressive in all three cases. The magnitude of shear stress is quite high as compared to the other groups.

A cross plot between the maximum stress P_1 and the various sizes and orientations of the insert is plotted as shown in fig. 35. From figs. 23 - 35 the following conclusions are made.

1. The maximum stress concentration occurs at the corner closest to the outside boundary.
2. In case of insert with 0° orientation, the stress concentration increases with the decrease in the size of the insert where as in the case of 45° orientation, the stress concentration decreases with the decrease in the size of the insert.
3. The rate of increase of stress P_1 in the region near the interface or corner is much higher than in any other region.
4. The stress P_1 at the outer boundary increases with the increase in distance between the corner and the outer boundary. Since no information is available in this field, it is not possible to compare these results with any other. The theoretical results obtained in this investigation are unsatisfactory and therefore are not presented in this thesis. However an attempt has been made here to lay out the reasons as to why it is so and a few suggestions have been made.

The finite difference approximation as other numerical approximation, is liable to error in region where stresses are subject to very rapid changes such as exist near singularities caused by corners in this case.

Elimination of singularities due to reentrant corners has not yet been successfully treated and the usual way of dealing with the difficulties is to use finer mesh in such regions as suggested by Zienkiewicz [25] . This approach has also been tried but without success. The smallest mesh size tried here is $h = 1/16$ which generates a mesh with 512 nodal points in case of insert with 0° orientation and 549 nodal points in case of insert with 45° orientation. This poses another problem of the solution of a large set of simultaneous equations which has, however, been overcome with the development of a special technique of economical storage and the solution of large matrices [23] , using Gauss Sidel iterative technique. During the numerical calculations, it has been found that the matrix generated is ill conditioned . With a slight alteration in the value of any element of the matrix, the end result changes rapidly and the convergence is also very slow. It has also been noted that the interface conditions obtained in Reference 3 are not valid for piecewise continuous boundaries.

It is therefore suggested that a further study is needed to find a suitable finite difference operator for the re-entrant corner. Once the operator has been found, these programs can be used to determine the values of the stresses in this problem.

APPENDICES

I. 1. DERIVATION OF BOUNDARY CONDITIONS FOR CASE II - 45° ORIENTATION

Referring to fig. 3, the length of the plate is 2L and its breadth is L. The intensity of applied tension along AD and BC is 100 (non dimensional value). The values of $\frac{\partial \phi}{\partial x}$, $\frac{\partial \phi}{\partial y}$ and ϕ along the boundary can be calculated as follows.

Along boundary BC, we have.

$$P_v = 100$$

$$(x, v) = 45^\circ$$

$$\text{and } ds = -\sqrt{2} dx = \sqrt{2} dy$$

Therefore using these values in eqns. (2.2.1) and (2.2.2), we get

$$P_{vx} = 50\sqrt{2}$$

$$P_{vy} = 50\sqrt{2}$$

and substituting in pair of eqns. (2.2.3), we obtain

$$\frac{\partial \phi}{\partial x} = 100x + C_1$$

$$\frac{\partial \phi}{\partial y} = 100y + C_2$$

Now, from eqn. (2.2.4) and using the values obtained above we have

$$\phi_{BC} = 100x^2 + C_1x + 100y^2 + C_2y - 50x^2 - 50y^2 + C_3$$

where C_1 , C_2 and C_3 are constants.

Let $C_1 = C_2 = C_3 = 0$ [see Art 2.2.]

Therefore

$$\phi_{BC} = 50x^2 + 50y^2$$

The same procedure is followed for the boundaries AB, CD and AD.

Now for boundary CD, $P_v = 100$

Therefore from eqns. (2.2.1) and (2.2.2)

$$P_{vx} = 0$$

and

$$P_{vy} = 0$$

The values of $\frac{\partial \phi}{\partial x}$, $\frac{\partial \phi}{\partial y}$ and ϕ should be continuous all along the boundary, therefore

$$\frac{\partial \phi}{\partial x} = 100\sqrt{2}$$

$$\frac{\partial \phi}{\partial y} = 150\sqrt{2}$$

and substituting in eqn. (2.2.4) we have

$$\phi_{CD} = 100\sqrt{2}x + 150\sqrt{2}y + C_4$$

To evaluate C_4 , we equate the values of ϕ_{CD} and ϕ_{BC} at the common point C, Thus,

$$(\phi_{CD})_C = (\phi_{BC})_C$$

or

$$(100\sqrt{2}x + 150\sqrt{2}y + C_4)_C = (50x^2 + 50y^2)_C$$

$$\text{Therefore } C_4 = -325$$

Hence

$$\phi_{CD} = 100\sqrt{2}x + 150\sqrt{2}y - 325$$

Now along boundary AB, we have

$$P_v = 0$$

Therefore, using eqns. (2.2.1) and (2.2.2)

$$P_{vx} = 0$$

$$P_{vy} = 0$$

and using the same reasoning as mentioned in case of boundary CD, we have

$$\left(\frac{\partial \phi}{\partial x}\right)_{AB} = 150\sqrt{2}$$

$$\left(\frac{\partial \phi}{\partial y}\right)_{AB} = 100\sqrt{2}$$

From eqn. (2.2.4), we get

$$\phi_{AB} = 150\sqrt{2}x + 100\sqrt{2}y + C_5$$

Now to evaluate C_5 , put

$$(\phi_{AB})_B = (\phi_{BC})_B$$

Therefore

$$C_5 = -325$$

Hence

$$\phi_{AB} = 150\sqrt{2}x + 100\sqrt{2}y - 325$$

Now for boundary AD, we have

$$P_v = 100$$

$$ds = \sqrt{2} dx = -\sqrt{2} dy$$

Therefore

$$P_{vx} = -50\sqrt{2}$$

$$P_{vy} = -50\sqrt{2}$$

Now using pair of eqns. (2.2.3), we have

$$\frac{\partial \phi}{\partial x} = 100x + a_1$$

$$\frac{\partial \phi}{\partial y} = 100y + a_2$$

where a_1 and a_2 are constants which are evaluated by putting

$$\left(\frac{\partial \phi}{\partial x}\right)_{\text{Along AD at D}} = \left(\frac{\partial \phi}{\partial x}\right)_{\text{Along CD at D}}$$

Therefore

$$a_1 = 100\sqrt{2}$$

Similarly

$$\left(\frac{\partial \phi}{\partial y} \right)_{\text{Along AD at D}} = \left(\frac{\partial \phi}{\partial y} \right)_{\text{Along CD at D}}$$

Therefore

$$a_2 = 100\sqrt{2}$$

Hence

$$\frac{\partial \phi}{\partial x} = 100x + 100\sqrt{2}$$

and

$$\frac{\partial \phi}{\partial y} = 100y + 100\sqrt{2}$$

Now substituting these values in eqn.(2.2.4), we have

$$\phi_{AD} = 50x^2 + 100\sqrt{2}x + 50y^2 + 100\sqrt{2}y + a_3$$

To evaluate the constant a_3 , we use the fact that in order to maintain the continuity of function, ϕ_{AD} and ϕ_{CD} should have the same value at point D i.e.

$$(\phi_{AD})_D = (\phi_{CD})_D$$

Therefore

$$a_3 = -300$$

Check:

The value of ϕ_{AB} and ϕ_{AD} should also be same at the common point A, therefore

$$(\phi_{AD})_A = (\phi_{AB})_A$$

$$\therefore a_3 = -300$$

Hence

$$\phi_{AD} = 50x^2 + 50y^2 + 100\sqrt{2}x + 100\sqrt{2}y - 300$$

II.1 GENERAL FORM OF BIHARMONIC OPERATOR

The size of the mesh is as shown in the fig. 5. The biharmonic equation at point 0 is given by

$$(\nabla^4 \phi)_0 = 0 \equiv \left(\frac{\partial^4 \phi}{\partial x^4}\right)_0 + 2 \left(\frac{\partial^4 \phi}{\partial x^2 \partial y^2}\right)_0 + \left(\frac{\partial^4 \phi}{\partial y^4}\right)_0 \quad (2.1.1a)$$

Now

$$\left(\frac{\partial^4 \phi}{\partial x^4}\right)_0 = \left[\frac{\partial^2}{\partial x^2} \left(\frac{\partial^2 \phi}{\partial x^2}\right)_0 \right]_0 \quad (2.1.2a)$$

and

$$\left(\frac{\partial^2 \phi}{\partial x^2}\right)_0 = 4 \frac{\phi_3 - 2\phi_0 + \phi_1}{\delta^2 (H_2 + H_3)^2}$$

Substituting this in eqn. (2.1.2a), we have

$$\begin{aligned} \left(\frac{\partial^4 \phi}{\partial x^4}\right)_0 &= \frac{4}{\delta^2 (H_2 + H_3)^2} \left[\left(\frac{\partial^2 \phi}{\partial x^2}\right)_1 - 2 \left(\frac{\partial^2 \phi}{\partial x^2}\right)_0 + \left(\frac{\partial^2 \phi}{\partial x^2}\right)_3 \right] \\ &= \frac{4}{\delta^2 (H_2 + H_3)^2} \left[\frac{4}{\delta^2 (H_3 + H_4)^2} (\phi_9 - 2\phi_1 + \phi_0) - \frac{2.4}{\delta^2 (H_2 + H_3)^2} (\phi_1 - 2\phi_0 + \phi_3) \right. \\ &\quad \left. + \frac{4}{\delta^2 (H_1 + H_2)^2} (\phi_0 - 2\phi_3 + \phi_{11}) \right] \quad (2.1.3a) \end{aligned}$$

Similarly $\frac{\partial^4 \phi}{\partial y^4}$ can be written as

$$\begin{aligned} &= \frac{4}{\delta^2 (v_2 + v_3)^2} \left[\frac{4}{\delta^2 (v_3 + v_4)^2} (\phi_{10} - 2\phi_2 + \phi_0) - \frac{2.4}{\delta^2 (v_2 + v_3)^2} (\phi_2 - 2\phi_0 + \phi_4) \right. \\ &\quad \left. + \frac{4}{\delta^2 (v_1 + v_2)^2} (\phi_0 - 2\phi_4 + \phi_{12}) \right] \quad (2.1.4a) \end{aligned}$$

and $\frac{\partial^4 \phi}{\partial x^2 \partial y^2}$

$$= \frac{4}{\delta^2 (v_2 + v_3)^2} \left[\frac{4}{\delta^2 (H_2 + H_3)^2} (\phi_5 - 2\phi_2 + \phi_6) - \frac{2.4}{\delta^2 (H_2 + H_3)^2} (\phi_1 - 2\phi_0 + \phi_3) \right. \\ \left. + \frac{4}{\delta^2 (H_2 + H_3)^2} (\phi_8 - 2\phi_4 + \phi_7) \right] \quad (2.1.5a)$$

Now substituting the values in eqn.(2.1.1a) from eqns.(2.1.3a),(2.1.4a) and (2.1.5a), we get

$$\nabla^4 \phi = \phi_0 [I A] + \phi_1 [I B.] + \phi_2 [I C] + \phi_3 [I D] + \phi_4 [I E] \\ + (\phi_5 + \phi_6 + \phi_7 + \phi_8) [I F] + \phi_9 [I G] + \phi_{10} [I H] \\ + \phi_{11} [I I] + \phi_{12} [I K] \quad (2.1.6a)$$

where

$$I A = \left[\frac{16}{(H_2 + H_3)^2 (H_3 + H_4)^2} + \frac{64}{(H_2 + H_3)^4} + \frac{16}{(H_2 + H_3)^2 (H_1 + H_2)^2} \right. \\ \left. + \frac{16}{(v_2 + v_3)^2 (v_3 + v_4)^2} + \frac{64}{(v_2 + v_3)^4} + \frac{16}{(v_2 + v_3)^2 (v_1 + v_2)^2} + \frac{128}{(v_2 + v_3)^2 (H_2 + H_3)^2} \right]$$

$$I B = \left[- \frac{32}{(H_2 + H_3)^2 (H_3 + H_4)^2} - \frac{32}{(H_2 + H_3)^4} - \frac{64}{(v_2 + v_3)^2 (H_2 + H_3)^2} \right]$$

$$I C = \left[\frac{32}{(v_2 + v_3)^2 (v_3 + v_4)^2} - \frac{32}{(v_2 + v_3)^4} - \frac{64}{(v_2 + v_3)^2 (H_2 + H_3)^2} \right]$$

$$I D = \left[- \frac{32}{(H_2 + H_3)^4} - \frac{32}{(H_2 + H_3)^2 (H_1 + H_2)^2} - \frac{64}{(v_2 + v_3)^2 (H_2 + H_3)^2} \right]$$

$$I E = \left[- \frac{32}{(v_2 + v_3)^4} - \frac{32}{(v_2 + v_3)^2 (v_1 + v_2)^2} - \frac{64}{(v_2 + v_3)^2 (H_2 + H_3)^2} \right]$$

$$IF = \left[\frac{32}{(v_2+v_3)^2 (H_2+H_3)^2} \right]$$

$$IG = \left[\frac{16}{(H_2+H_3)^2 (H_3+H_4)^2} \right]$$

$$IH = \left[\frac{16}{(v_2+v_3)^2 (v_3+v_4)^2} \right]$$

$$II = \left[\frac{16}{(H_2+H_3)^2 (H_1+H_2)^2} \right]$$

$$IK = \left[\frac{16}{(v_2+v_3)^2 (v_1+v_2)^2} \right]$$

When the mesh is square i.e.

$H_1 = H_2 = H_3 = H_4 = v_1 = v_2 = v_3 = v_4$, eqn. (2.1.6 a)
reduces to

$$\nabla^4 \phi = 20 \phi_0 - 8(\phi_1 + \phi_2 + \phi_3 + \phi_4) + 2(\phi_5 + \phi_6 + \phi_7 + \phi_8) \\ + \phi_9 + \phi_{10} + \phi_{11} + \phi_{12} = 0$$

II.2 OPERATOR FOR THE POINTS ON THE INTERFACE

The orientation of the point is shown in fig. 10.

Writing the biharmonic equation in finite difference form at point 0, we have

$$20 \phi_0 - 8(\phi_1 + \phi_2 + \phi_3 + \phi_4) + 2(\phi_5 + \phi_6 + \phi_7 + \phi_8) + \phi_9 + \phi_{10} + \phi_{11} + \phi_{12} = 0$$

in the above equation ϕ_7, ϕ_4, ϕ_8 and ϕ_{12} are fictitious and would have to be replaced by real values.

Using conditions (2.3.8) and (2.3.9) at point 0 and writing in finite difference form, we have

$$\phi_2 - \phi_4 = \bar{\phi}_2 - \bar{\phi}_4 \quad (\text{from 2.3.8}) \quad (2.2.1a)$$

and

$$A_1 [\phi_2 + \phi_4 - 2\phi_0 - B_1 (\phi_1 + \phi_3 - 2\phi_0)] = A_2 [\bar{\phi}_2 + \bar{\phi}_4 - 2\bar{\phi}_0 - B_2 (\bar{\phi}_1 + \bar{\phi}_3 - 2\bar{\phi}_0)]$$

(from 2.3.9)

(2.2.2a)

Now using the condition that on the interface

$$\phi = \bar{\phi}$$

from eqns. (2.2.1a) and (2.2.2a), we get

$$\begin{aligned} \phi_4 \left[\frac{1}{A_1 + A_2} \right] [2A_2 \bar{\phi}_4 - (\phi_1 + \phi_3) \{-A_1 B_1 + A_2 B_2\} - \phi_2 \{A_1 - A_2\} \\ - \{-2A_1 + 2A_1 B_1 + 2A_2 - 2A_2 B_2\}] \end{aligned}$$

Again writing eqns.(2.3.8) and (2.3.9) in finite difference for point 3, we have

$$\phi_6 - \phi_7 = \bar{\phi}_6 - \bar{\phi}_7 \quad (2.2.3a)$$

and

$$A_1 [\phi_6 - 2\phi_3 + \phi_7 - B_1 (\phi_0 - 2\phi_3 + \phi_{11})] = A_2 [\bar{\phi}_6 - 2\bar{\phi}_3 + \bar{\phi}_7 - B_2 (\bar{\phi}_0 - 2\bar{\phi}_3 + \bar{\phi}_{11})] \quad (2.2.4a)$$

From eqns.(2.2.3a) and (2.2.4a) and also using eqn.(2.3.7), we get

$$\begin{aligned} \phi_7 = \left[\frac{1}{A_1 + A_2} \right] [2A_2 \bar{\phi}_7 - (A_1 - A_2) \phi_6 - \phi_3 (2A_1 B_1 - 2A_1 + A_2 - 2A_2 B_2) - \phi_0 \\ (-A_1 B_1 + A_2 B_2) - \phi_{11} (-A_1 B_1 + A_2 B_2)] \end{aligned}$$

Similarly using eqns.(2.3.7), (2.3.8) and (2.3.9) at point 1, we get

$$\begin{aligned} \phi_8 = \left[\frac{1}{A_1 + A_2} \right] [2A_2 \bar{\phi}_8 - (A_1 - A_2) \phi_5 - \phi_1 (-2A_1 + 2A_1 B_1 + A_2 - 2A_2 B_2) \\ - (\phi_9 + \phi_0) \{-A_1 B_1 + A_2 B_2\}] \end{aligned}$$

Now writing eqn. (2.3.10) in finite difference for point 0, we have

$$\begin{aligned}
 & - A_1 [\phi_{10} - 2\phi_2 + 2\phi_4 - \phi_{12} + (2+B_1)(\phi_5 - \phi_8 - 2\phi_2 + 2\phi_4 + \phi_6 - \phi_7)] \\
 & = - A_1 [\bar{\phi}_{10} - 2\bar{\phi}_2 + 2\bar{\phi}_4 - \bar{\phi}_{12} + (2+B_2)(\bar{\phi}_5 - \bar{\phi}_8 - 2\bar{\phi}_2 + 2\bar{\phi}_4 + \bar{\phi}_6 - \bar{\phi}_7)] \\
 & \hspace{25em} (2.2.5a)
 \end{aligned}$$

The condition that at the interface

$$\nabla^4 \phi = \nabla^4 \bar{\phi} = 0$$

can be written in finite difference as follows

$$\begin{aligned}
 20 \phi_0 - 8(\phi_1 + \phi_2 + \phi_3 + \phi_4) + 2(\phi_5 + \phi_6 + \phi_7 + \phi_8) + \phi_9 + \phi_{10} + \phi_{11} + \phi_{12} \\
 = 20 \bar{\phi}_0 - 8(\bar{\phi}_1 + \bar{\phi}_2 + \bar{\phi}_3 + \bar{\phi}_4) + 2(\bar{\phi}_5 + \bar{\phi}_6 + \bar{\phi}_7 + \bar{\phi}_8) + \bar{\phi}_9 + \bar{\phi}_{10} + \bar{\phi}_{11} + \bar{\phi}_{12}
 \end{aligned}$$

Now substituting for $\bar{\phi}_2$, $\bar{\phi}_5$ and $\bar{\phi}_6$ in terms of real values using eqn. (2.3.8) and simplifying we get

$$\bar{\phi}_{10} = -16\phi_4 + 4\phi_7 + 4\phi_8 + \phi_{10} + \phi_{12} + 16\bar{\phi}_4 - 4\bar{\phi}_8 - 4\bar{\phi}_7 - \bar{\phi}_{12}$$

Again in eqn. (2.2.5a) substituting for $\bar{\phi}_2$, $\bar{\phi}_5$, $\bar{\phi}_6$ and $\bar{\phi}_{10}$ in terms of real values and rearranging we get

$$\begin{aligned}
 \frac{1}{A_1 + A_2} \{ \phi_2 [-2A_1 - 2(2+B_1)A_1 + 2A_2 + 2A_2(2+B_2)] + \phi_4 [2A_1 + 2A_1(2+B_1) + 16A_2 - 2A_2 \\
 - 2A_2(2+B_2)] + (\phi_5 + \phi_6) [A_1(2+B_1) - A_2(2+B_2)] + (\phi_7 + \phi_8) [-A_1(2+B_1) - 4A_2 \\
 + (2+B_2)A_2] + \phi_{10}(A_1 - A_2) - \bar{\phi}_4(16A_2) + \bar{\phi}_7(4A_2) + \bar{\phi}_8(4A_2) \} = \phi_{12}
 \end{aligned}$$

Now substituting the values of fictitious points ϕ_4 , ϕ_7 , ϕ_8 and ϕ_{12} in the finite difference form of $\nabla^4 \phi = 0$ and rearranging we get

$$\begin{aligned}
 \phi_0 \left[20 + \frac{1}{(A_1 + A_2)^2} \{ (A_2 B_2 + A_1 B_1) 8(A_2 - A_1) - 6(A_2 B_2 - A_1 B_1)^2 - 4(A_2 - A_1)^2 \} \right] \\
 + (\phi_1 + \phi_3) \left[-8 + \frac{4(A_2 B_2 - A_1 B_1)}{(A_1 + A_2)^2} \{ (A_2 B_2 - A_1 B_1) - (A_2 - A_1) \} \right]
 \end{aligned}$$

$$\begin{aligned}
 & + \phi_2 \left[-8 + \frac{1}{(A_1+A_2)^2} \{ 4A_1(A_2B_2-A_1B_1) + (A_2-A_1)(8A_2+4A_1) \} \right] \\
 & + \phi_4 \left[-8 - \frac{1}{(A_1+A_2)^2} \{ 4A_2(A_2B_2-A_1B_1) + (A_2-A_1)(4A_2+8A_1) \} \right] \\
 & + (\phi_5+\phi_6) \left[2 - \frac{2}{(A_1+A_2)^2} \{ (A_2B_2 - A_1B_1) A_1 + (A_2-A_1)(A_1+A_2) \} \right] \\
 & + (\phi_7+\phi_8) \left[2 + \frac{2}{(A_1+A_2)^2} \{ (A_2B_2 - A_1B_1) A_2 + (A_2-A_1)(A_1+A_2) \} \right] \\
 & + (\phi_9+\phi_{11}) \left\{ 1 - \frac{1}{(A_1+A_2)^2} (A_2B_2 - A_1B_1)^2 \right\} \\
 & + \phi_{10} \left\{ 1 - \frac{A_2-A_1}{A_2+A_1} \right\} + \phi_{12} \left\{ 1 + \frac{A_2-A_1}{A_2+A_1} \right\} = 0
 \end{aligned}$$

or can be written as

$$\begin{aligned}
 & \phi_0 [\bar{A}] + (\phi_1+\phi_3) [\bar{B}] + \phi_2 [\bar{C}] + \phi_4 [\bar{D}] + (\phi_5+\phi_6) [\bar{E}] \\
 & + (\phi_7+\phi_8) [\bar{F}] + (\phi_9+\phi_{11}) [\bar{G}] + \phi_{10} [\bar{H}] + \phi_{12} [\bar{I}] = 0
 \end{aligned}$$

where

$$\begin{aligned}
 \bar{A} &= \left[20 + \frac{1}{(A_1+A_2)^2} \{ (A_2B_2-A_1B_1) 8(A_2-A_1) - 6(A_2B_2-A_1B_1)^2 - 4(A_2-A_1)^2 \} \right] \\
 \bar{B} &= \left[-8 + \frac{4(A_2B_2-A_1B_1)}{(A_1+A_2)^2} \{ (A_2B_2-A_1B_1) - (A_2-A_1) \} \right] \\
 \bar{C} &= \left[-8 + \frac{1}{(A_1+A_2)^2} \{ 4A_1(A_2B_2-A_1B_1) + (A_2-A_1)(8A_2+4A_1) \} \right]
 \end{aligned}$$

$$\bar{D} = \left[-8 - \frac{1}{(A_1 + A_2)^2} \{ 4A_2(A_2B_2 - A_1B_1) + (A_2 - A_1)(4A_2 + 8A_1) \} \right]$$

$$\bar{E} = \left[2 - \frac{2}{(A_1 + A_2)^2} \{ (A_2B_2 - A_1B_1)A_1 + (A_2 - A_1)(A_1 + A_2) \} \right]$$

$$\bar{F} = \left[2 + \frac{2}{(A_1 + A_2)^2} \{ (A_2B_2 - A_1B_1)A_2 + (A_2 - A_1)(A_1 + A_2) \} \right]$$

$$\bar{G} = \left[1 - \frac{1}{(A_1 + A_2)^2} (A_2B_2 - A_1B_1)^2 \right]$$

$$\bar{H} = \left[1 - \frac{A_2 - A_1}{A_2 + A_1} \right]$$

$$\bar{I} = \left[1 + \frac{A_2 - A_1}{A_2 + A_1} \right]$$

The string pattern for this is given in fig. 11.

II.3 OPERATOR FOR THE POINTS ONE MESH AWAY FROM THE INTERFACE

The $\nabla^4 \phi = 0$ in finite difference form at point 0 as shown in fig.12 can be written as

$$20\phi_0 - 8(\phi_1 + \phi_2 + \phi_3 + \phi_4) + 2(\phi_5 + \phi_6 + \phi_7 + \phi_8) + \phi_9 + \phi_{10} + \phi_{11} + \phi_{12} = 0 \quad (2.3.1a)$$

in which ϕ_{12} is a fictitious which have to be replaced by the real value.

Now writing eqns.(2.3.8) and (2.3.9) in finite difference form for points 4, we have

$$\phi_{12} - \phi_0 = \bar{\phi}_{12} - \bar{\phi}_0 \quad (2.3.2a)$$

and

$$A_1 [\phi_{12} - 2\phi_4 + \phi_0 - B_1(\phi_8 - 2\phi_4 + \phi_7)] = A_2 [\bar{\phi}_{12} - 2\bar{\phi}_4 + \bar{\phi}_0 - B_2(\bar{\phi}_8 - 2\bar{\phi}_4 + \bar{\phi}_7)] \quad (2.3.3a)$$

Using eqns. (2.3.2a), (2.3.3a) and the condition

$$\phi = \bar{\phi}$$

At the interface, we get

$$\begin{aligned} \phi_{12} = & \left(1 + \frac{A_2 - A_1}{A_1 + A_2}\right) \bar{\phi}_{12} + \left(\frac{A_2 - A_1}{A_1 + A_2}\right) \phi_0 + \frac{2(A_2 B_2 - A_1 B_1 - A_2 + A_1)}{A_1 + A_2} \phi_4 \\ & - \frac{A_2 B_2 - A_1 B_1}{A_1 + A_2} (\phi_7 + \phi_8) \end{aligned}$$

Substituting ϕ_{12} in eqn. (2.3.1a) and rearranging we get

$$\begin{aligned} \phi_0 \left[20 + \frac{A_2 - A_1}{A_2 + A_1} \right] - 8(\phi_1 + \phi_2 + \phi_3) + \phi_4 \left[-8 + \frac{2}{A_1 + A_2} \{ A_2 B_2 - A_1 B_1 + A_1 - A_2 \} \right] \\ + 2(\phi_5 + \phi_6) - (\phi_7 + \phi_8) \left[2 - \frac{A_2 B_2 - A_1 B_1}{A_1 + A_2} \right] + \phi_9 + \phi_{10} + \phi_{11} \\ + \bar{\phi}_{12} \left[1 + \frac{A_2 - A_1}{A_1 + A_2} \right] = 0 \end{aligned}$$

or this can be written as

$$\begin{aligned} \phi_0 [AA] - (\phi_1 + \phi_2 + \phi_3) [8] + \phi_4 [BB] + (\phi_5 + \phi_6) [2] \\ + (\phi_7 + \phi_8) [CC] + \phi_9 + \phi_{10} + \phi_{11} + \phi_{12} [DD] = 0 \end{aligned}$$

where

$$AA = \left[20 + \frac{A_2 - A_1}{A_2 + A_1} \right]$$

$$BB = \left[-8 + \frac{2}{A_1 + A_2} \{ A_2 B_2 - A_1 B_1 + A_1 - A_2 \} \right]$$

$$CC = \left[2 - \frac{A_2 B_2 - A_1 B_1}{A_1 + A_2} \right]$$

$$DD = \left[1 + \frac{A_2 - A_1}{A_1 + A_2} \right]$$

The string pattern for this is given in fig. 13.

II.4. OPERATOR FOR THE POINT ONE MESH AWAY FROM TWO PERPENDICULAR INTERFACES

The biharmonic equation in the finite difference form at point 0 as shown in fig. 14, can be given as

$$20 \phi_0 - 8(\phi_1 + \phi_2 + \phi_3 + \phi_4) + (2 \phi_5 + \phi_6 + \phi_7 + \phi_8) + \phi_9 + \phi_{10} + \phi_{11} + \phi_{12} = 0$$

(2.3.1a)

Where ϕ_9 and ϕ_{10} are fictitious values. So by using the boundary conditions, these values have to be replaced by real values.

From eqn. (2.3.7), at the interface

$$\phi = \bar{\phi} \quad (2.3.2a)$$

Writing eqns. (2.3.8) and (2.3.9) in the finite difference form at point 2, we have

$$\phi_{10} - \phi_0 = \bar{\phi}_{10} - \bar{\phi}_0 \quad (2.3.3a)$$

and

$$A_1 [\phi_{10} - 2\phi_2 + \phi_0 - B_1(\phi_6 - 2\phi_2 + \phi_5)] = A_2 [\bar{\phi}_{10} - 2\bar{\phi}_2 + \bar{\phi}_0 - B_2(\bar{\phi}_6 - 2\bar{\phi}_2 + \bar{\phi}_5)] \quad (2.3.4a)$$

From eqns. (2.3.2a), (2.3.3a), and (2.3.4a), we get

$$\begin{aligned} \phi_{10} = & \left(1 + \frac{A_2 - A_1}{A_2 + A_1}\right) \bar{\phi}_{10} + \left(\frac{A_2 - A_1}{A_1 + A_2}\right) \phi_0 + \frac{2(A_2 B_2 - A_1 B_1 - A_2 + A_1)}{A_1 + A_2} \phi_2 \\ & - \left(\frac{A_2 B_2 - A_1 B_1}{A_1 + A_2}\right) (\phi_5 + \phi_6) \end{aligned}$$

Similarly, from eqns. (2.3.11), (2.3.12), (2.3.13) and (2.3.14) for the face parallel to y-axis we get

$$\begin{aligned} \phi_9 = & \left(1 + \frac{A_2 - A_1}{A_1 + A_2}\right) \bar{\phi}_9 + \left(\frac{A_2 - A_1}{A_1 + A_2}\right) \phi_0 + \frac{2(A_2 B_2 - A_1 B_1 - A_2 + A_1)}{(A_1 + A_2)} \phi_1 \\ & - \left(\frac{A_2 B_2 - A_1 B_1}{A_1 + B_2}\right) (\phi_5 + \phi_8) \end{aligned}$$

Now substituting the values of ϕ_{10} and ϕ_9 in eqn. (2.3.1a) and rearranging we have

$$\begin{aligned} & \phi_0 \left[20 + \frac{2(A_2 - A_1)}{(A_1 + A_2)} \right] + (\phi_1 + \phi_2) \left[-8 + \frac{2(A_2 B_2 - A_1 B_1 - A_2 + A_1)}{(A_1 + A_2)} \right] \\ & - 8(\phi_3 + \phi_4) + \phi_5 \left[2 - \frac{2(A_2 B_2 - A_1 B_1)}{A_1 + A_2} \right] + (\phi_6 + \phi_8) \left[2 - \frac{2(A_2 B_2 - A_1 B_1)}{A_1 + A_2} \right] \\ & + 2\phi_7 + (\phi_9 + \phi_{10}) \left[1 + \frac{A_2 - A_1}{A_1 + A_2} \right] + \phi_{11} + \phi_{12} = 0 \end{aligned}$$

or

$$\begin{aligned} & \phi_0 [AAA] + (\phi_1 + \phi_2) [BB] - (\phi_3 + \phi_4) 8 + \phi_5 [CCC] + (\phi_6 + \phi_8) [CC] \\ & + 2\phi_7 + (\phi_9 + \phi_{10}) [DD] + \phi_{11} + \phi_{12} = 0 \end{aligned}$$

$$\begin{aligned} \text{where } AAA &= \left[20 + \frac{2(A_2 - A_1)}{(A_1 + A_2)} \right] \\ BB &= \left[-8 + \frac{2(A_2 B_2 - A_1 B_1 - A_2 + A_1)}{(A_1 + A_2)} \right] \\ CC &= \left[2 - \frac{A_2 B_2 - A_1 B_1}{(A_1 + A_2)} \right] \\ CCC &= \left[2 - \frac{2(A_2 B_2 - A_1 B_1)}{(A_1 + A_2)} \right] \\ DD &= \left[1 + \frac{A_2 - A_1}{A_1 + A_2} \right] \end{aligned}$$

The string pattern for this is given in fig. 15.

II.5. OPERATOR FOR THE POINT ONE MESH AWAY FROM TWO PERPENDICULAR INTERFACES

At point 0, as shown in fig. 16, the biharmonic equation in finite difference form can be written as

$$20 \bar{\phi}_0 - 8(\bar{\phi}_1 + \bar{\phi}_2 + \bar{\phi}_3 + \bar{\phi}_4) + 2(\bar{\phi}_5 + \bar{\phi}_6 + \bar{\phi}_7 + \bar{\phi}_8) + \bar{\phi}_9 + \bar{\phi}_{10} + \bar{\phi}_{11} + \bar{\phi}_{12} = 0 \quad (2.3.1a)$$

Where $\bar{\phi}_7$ is a fictitious value which has to be replaced by real value using the interface conditions.

Using eqns. (2.3.11), (2.3.12), (2.3.13) and (2.3.14) in their finite difference form at point 4, we get

$$\begin{aligned} \bar{\phi}_7 = & \left(\frac{A_1 - A_2}{A_1 + A_2} \right) \bar{\phi}_8 + \frac{(2A_2 - 2A_1 + 2A_1B_1 - 2A_2B_2)}{(A_1 + A_2)} \bar{\phi}_4 \\ & + \frac{(A_2B_2 - A_1B_1)}{(A_1 + A_2)} \bar{\phi}_0 + \frac{(A_2B_2 - A_1B_1)}{(A_1 + A_2)} \bar{\phi}_{12} + \frac{2A_1}{A_1 + A_2} \phi_7 \end{aligned} \quad (2.3.2a)$$

Substituting in eqn. (2.3.1a) and rearranging we get

$$\begin{aligned} \bar{\phi}_0 \left[20 + \frac{2(A_2B_2 - A_1B_1)}{A_1 + A_2} \right] - 8(\bar{\phi}_1 + \bar{\phi}_2 + \bar{\phi}_3) + \bar{\phi}_4 \left[-8 + \frac{2(2A_2 - 2A_1 + 2A_1B_1 - 2A_2B_2)}{A_1 + A_2} \right] \\ + 2(\bar{\phi}_5 + \bar{\phi}_6) + \phi_7 \left[\frac{4A_1}{A_1 + A_2} \right] + \phi_8 \left[2 + \frac{2(A_1 - A_2)}{A_1 + A_2} \right] \\ + \bar{\phi}_9 + \bar{\phi}_{10} + \bar{\phi}_{11} + \bar{\phi}_{12} \left[1 + \frac{2(A_2B_2 - A_1B_1)}{A_1 + A_2} \right] = 0 \end{aligned}$$

or

$$\begin{aligned} \bar{\phi}_0 [X] - 8(\bar{\phi}_1 + \bar{\phi}_2 + \bar{\phi}_3) + \bar{\phi}_4 [Y] + 2(\bar{\phi}_5 + \bar{\phi}_6) \\ + \phi_7 [S] + \bar{\phi}_8 [Z] + \bar{\phi}_9 + \bar{\phi}_{10} + \bar{\phi}_{11} + \bar{\phi}_{12} [T] = 0 \end{aligned}$$

where

$$X = \left[20 + \frac{2(A_2 B_2 - A_1 B_1)}{(A_1 + A_2)} \right]$$

$$Y = \left[-8 + \frac{4(A_2 - A_1 + A_1 B_1 - A_2 B_2)}{(A_1 + A_2)} \right]$$

$$Z = \left[2 + \frac{2(A_1 - A_2)}{(A_1 + A_2)} \right]$$

$$S = \left[\frac{4A_1}{A_1 + A_2} \right]$$

$$T = \left[1 + \frac{2(A_2 B_2 - A_1 B_1)}{(A_1 + A_2)} \right]$$

The string pattern for this is given in fig. 16.

III. The Listings of the programs are as follows :

```

*****
THIS PROGRAMME IS THE STEP ONE OF THE THREE STEPS SOLUTION OF
CASE I (C-----INCLINATION).
*****
*****
THIS PROGRAMME GENERATES THE BIHARMONIC EQUATION AT EACH NODAL
POINT USING APPROPRIATE SUBROUTINE.
*****
*****

```

THE VARIABLES ARE DEFINED AS FOLLOWS :

```

M      NO. OF STEPS IN X - DIRECTION
N      NO. OF STEPS IN Y - DIRECTION
MC     LENGTH OF THE INSERT
NC     WIDTH OF THE INSERT
E1,E2  MODULI OF INSERT AND PARENT MATERIAL RESPECTIVELY.
B1,B2  POISSONS RATIO OF INSERT AND PARENT MATERIAL
       RESPECTIVELY.
P(KK)  ROW OF A MATRIX
RT(KK) STEP SIZE
DEL    RIGHT HAND SIDE VECTOR OF THE MATRIX.
BY(J)  PHI VALUE ALONG AB ( REF. FIG.2 )
*****

```

```

IMPLICIT REAL*8(A-H,O-Z)
COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
DIMENSION KR(128)

```

```

READ(1,10)MM,NN,E1,E2,B1,B2
WRITE(3,10)MM,NN,E1,E2,B1,B2
10  FORMAT(2I10,4F15.8)
M=16
N=8
HD=0.125  ODO
HC=5

```

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NC=3

 CALCULATE THE COEFFICIENTS OF THE VARIOUS OPERATORS.

```

11  A1 = 1./E1
    A2 = 1./E2
    AA1 = (A1+A2)**2
    A = (20.+(A2*B2-A1*B1)*8.+(A2-A1)-6.*((A2*B2-A1*B1)**2)-4.*((A2-
1  A1)**2))/AA1
    BI = (-8.+4.*(A2*B2-A1*B1))*((A2*B2-A1*B1)-(A2-A1))/AA1
    CI = (-8.-(4.*A1*(A2*B2-A1*B1)+(A2-A1)*(8.*A2+4.*A1))/AA1)
    DI = (-8.-(4.*A2*(A2*B2-A1*B1)+(A2-A1)*(4.*A2+8.*A1))/AA1)
    E = (2.-2.*((A2*B2-A1*B1)*A1+(A2-A1)*(A1+A2))/AA1)
    F = (2.+2.*((A2*B2-A1*B1)*A2+(A2-A1)*(A1+A2))/AA1)
    G = (1.-((A2*B2-A1*B1)*(A2+A1)))
    H = (1.-((A2-A1)/(A2+A1)))
    TI = (1.+(A2-A1)/(A2+A1))
    AR = A+((-A1*B1+A2*B2)/(A1+A2))
    BR = BI+2.*((-A1+A1*B1+A2-A2*B2)/(A1+A2))
    ER = E+(A1-A2)/(A1+A2)
    FR = F-2.*A2/(A1+A2)
    GR = G+((-A1*B1+A2*B2)/(A1+A2))
    AA1 = (20.+(A2-A1)/(A2+A1))
    BB = (-8.+2.*(A2*B2-A1*B1+A1-A2)/(A1+A2))
    BB1 = (-8.-2.*(A2*B2-A1*B1+A1-A2)/(A1+A2))
    CC = (2.-((A2*B2-A1*B1)/(A1+A2)))
    CC1 = (2.+(A2*B2-A1*B1)/(A1+A2))
    DD = (1.+(A2-A1)/(A1+A2))
    DD1 = (1.-((A2-A1)/(A1+A2)))
    AAA = (20.+2.*(A2-A1)/(A1+A2))
    CCC = (2.-2.*(A2*B2-A1*B1)/(A1+A2))
    XX = (20.-2.*(A1*B1-A2*B2)/(A1+A2))
    YY = (-8.-4.*(A1-A2-A1*B1+A2*B2)/(A1+A2))
    Z = (2.-2.*(A2-A1)/(A1+A2))
    S = (4.*A1/(A1+A2))
    T = (1.-2.*(A1*B1-A2*B2)/(A1+A2))
    WRITE(3,12)A,BI,CI,D,E,F,G,H,TI,AA,AA1,BB,BR1,CC,CC1,DD,DD1,XX,YY,
1  Z,S,T,AAA,CCC
  
```

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12 FORMAT(3X,10F12.5)

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C

THE GENERATION OF MATRIX STARTS FROM HERE.

```
0049 L=0
0047 NU=0
0048 DO 7 J=1,N
0045 DO 7 I=1,M
0050 NU=NU+1
0051 LA(I,J)=NU
0052 CONTINUE
0053 DEL=C.0D0
0054 RY(1)=0.0D0
0055 DO 5 J=2,N
0056 DEL=DEL+HD
0057 BY(J)=50.*(DEL**2)
0058 CONTINUE
0059 DO 9 I=1,M
0060 DO 55 K5=1,NU
0061 P(K5)=0.0D0
0062 B=0.0D0
0063 K=LA(I,J)
0064 IF(J.LT.(NC-1)) GO TO 20
0065 IF(J.EQ.(NC-1)) GO TO 30
0066 IF(J.EQ.NC) GO TO 40
0067 IF(J.EQ.(NC+1)) GO TO 50
0068 CALL STAND
0069 CALL GAIN(I,J)
0070 GO TO 150
0071 IF(I.EQ.(MC-1)) GO TO 70
0072 IF(I.EQ.MC) GO TO 80
0073 IF(I.EQ.(MC+1)) GO TO 90
0074 GO TO 60
0075 IF(I.LT.(MC-1)) GO TO 100
0076 IF(I.EQ.(MC-1)) GO TO 110
0077 IF(I.EQ.MC) GO TO 80
0078 IF(I.EQ.(MC+1)) GO TO 90
0079 GO TO 60
0080
```

```

0081      MAIN
0082      40 IF(I.LT.(MC+1))GO TO 120
0083      IF(I.EQ.MC)GO TO 130
0084      IF(I.EQ.(MC+1))GO TO 90
0085      50 GO TO 80
0086      IF(I.LT.(MC+1))GO TO 140
0087      GO TO 60
0088      CALL FOUR(AA,BB,CC,DD)
0089      CALL GAIN(I,J)
0090      GO TO 150
0091      CALL UNFCUR(A,BI,CI,D,E,F,G,H,II)
0092      CALL GAIN(I,J)
0093      GO TO 150
0094      CALL OFOUR(AA1,BB1,CC1,DD1)
0095      CALL GAIN(I,J)
0096      GO TO 150
0097      CALL TWO(AA,BB,CC,DE)
0098      CALL GAIN(I,J)
0099      GO TO 150
0100      CALL ICFCUR(AAA,BB,CCC,CC,DD)
0101      CALL GAIN(I,J)
0102      GO TO 150
0103      CALL UNWU(A,BI,CI,D,E,F,G,H,II)
0104      CALL GAIN(I,J)
0105      GO TO 150
0106      CALL COFCUR(XX,YY,S,Z,T)
0107      CALL GAIN(I,J)
0108      GO TO 150
0109      CALL OTWU(AA1,BB1,CC1,DD1)
0110      CALL GAIN(I,J)
0111      L=L+1
0112      KR(L)=K
0113      WRITE(7)(P(KK),KK=1,NU)
0114      RT(K)=B
0115      CONTINUE
0116      WRITE(3,500)((KR(KK),RT(KK)),KK=1,NU)
0117      WRITE(7)(RT(KK),KK=1,NU)
0118      FORMAT(4D20.10)
0119      FORMAT(1H1,5X,8(I6,F8.3))
0120      RETURN
0121      END

```

SUBROUTINE GAIN(I,J)

***** THIS SUBROUTINE PUTS THE COEFFICIENTS OF THE VARIOUS OPERATORS IN
APPROPRIATE PLACES. *****

```

10 IMPLICIT REAL*8(A-H,O-Z)
20 COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
30 FX=50.0D0
40 FY=100.0D0*2.0D0*HD
50 LI=LA(I,J)
60 P(LI)=C(1)+P(LI)
70 IF(I.EQ.M) GO TO 10
80 LI=LA(I+1,J)
90 P(LI)=C(2)+P(LI)
100 GO TO 20
110 B=B-BY(J)*C(2)
120 IF(J.EQ.N) GO TO 30
130 LI=LA(I,J+1)
140 P(LI)=C(3)+P(LI)
150 GO TO 40
160 B=B-BX*C(3)
170 IF(I.EQ.1) GO TO 50
180 LI=LA(I-1,J)
190 P(LI)=C(4)+P(LI)
200 GO TO 60
210 LI=LA(I+1,J)
220 P(LI)=C(4)+P(LI)
230 IF(J.EQ.1) GO TO 70
240 LI=LA(I,J-1)
250 P(LI)=C(5)+P(LI)
260 GO TO 80
270 LI=LA(I,J+1)
280 P(LI)=C(5)+P(LI)
290 IF(J.EQ.N) GO TO 100
300 LI=LA(I+1,J+1)
310 P(LI)=C(6)+P(LI)
320 GO TO 110
330 B=B-BY(J+1)*C(6)

```

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URTRAN IV G LEVEL 18

GAIN

DATE = 70278

18/35/47

PAGE 0002

```

0036 GO TO 110
0037 B=B-EX*C(6)
0038 IF(J.EQ.N)GO TO 120
0039 IF(I.EQ.1)GO TO 130
0040 LI=LA(I-1,J+1)
0041 P(LI)=C(7)+P(LI)
0042 GO TO 140
0043 B=B-BX*C(7)
0044 GO TO 140
0045 LI=LA(I+1,J+1)
0046 P(LI)=C(7)+P(LI)
0047 IF(I.EQ.1)GO TO 150
0048 IF(J.EQ.1)GO TO 160
0049 LI=LA(I-1,J-1)
0050 P(LI)=C(8)+P(LI)
0051 GO TO 180
0052 IF(J.EQ.1)GO TO 170
0053 LI=LA(I+1,J-1)
0054 P(LI)=C(8)+P(LI)
0055 GO TO 180
0056 IF(I.EQ.1)GO TO 170
0057 LI=LA(I-1,J+1)
0058 P(LI)=C(8)+P(LI)
0059 GO TO 180
0060 LI=LA(I+1,J+1)
0061 P(LI)=C(8)+P(LI)
0062 IF(J.EQ.1)GO TO 190
0063 IF(I.EQ.M)GO TO 200
0064 LI=LA(I+1,J-1)
0065 P(LI)=C(9)+P(LI)
0066 GO TO 220
0067 IF(I.EQ.M)GO TO 210
0068 LI=LA(I+1,J+1)
0069 P(LI)=C(9)+P(LI)
0070 GO TO 220
0071 IF(J.EQ.1)GO TO 210
0072 B=B-BY(J-1)*C(9)
0073 GO TO 220
0074 B=B-BY(J+1)*C(9)
0075 IF(I.EQ.(M-1))GO TO 230
0076 IF(I.EQ.M)GO TO 240
0077 LI=LA(I+2,J)
0078 P(LI)=C(10)+P(LI)

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18/35/47

DATE = 70278

GAIN

ORTRAN IV G LEVEL 18

```

0079      GU TO 250
0080      B=B-BY(J)*C(10)
0081      GO TO 250
0082      L1=LA(I,J)
0083      P(L1)=P(L1)+C(10)
0084      IF(J.EQ.(N-1))GO TO 260
0085      IF(J.EQ.N)GO TO 270
0086      L1=LA(I,J+2)
0087      P(L1)=P(L1)+C(11)
0088      GO TO 280
0089      B=B-BX*C(11)
0090      GO TO 280
0091      L1=LA(I,J)
0092      P(L1)=P(L1)+C(11)
0093      B=B-FY*C(11)
0094      IF(I.EQ.1)GO TO 290
0095      IF(I.EQ.2)GO TO 300
0096      L1=LA(I-2,J)
0097      P(L1)=C(12)+P(L1)
0098      GO TO 310
0099      L1=LA(I+2,J)
0100      P(L1)=C(12)+P(L1)
0101      GO TO 310
0102      L1=LA(I,J)
0103      P(L1)=P(L1)+C(12)
0104      IF(J.EQ.1)GO TO 320
0105      IF(J.EQ.2)GO TO 330
0106      L1=LA(I,J-2)
0107      P(L1)=C(13)+P(L1)
0108      GO TO 340
0109      L1=LA(I,J+2)
0110      P(L1)=P(L1)+C(13)
0111      GO TO 340
0112      L1=LA(I,J)
0113      P(L1)=P(L1)+C(13)
0114      RETURN
0115      END

```



```

FORTRAN IV G LEVEL 18                                PAGE 0001
0001          C      TWO                               18/35/47
              SUBROUTINE TWO(AA,BB,CC,DD)
              *****
              OPERATOR FOR THE POINT ON THE INSERT , ONE MESH LENGTH
              BELOW THE INTERFACE . REF ART. 2.4.4 *****
              *****
0002          C      IMPLICIT REAL*8(A-H,O-Z)
0003          C      COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
              C
              C(1)=A/.0D0
              C(2)=-8.0D0
              C(3)=BB
              C(4)=-8.0D0
              C(5)=-8.0D0
              C(6)=CC
              C(7)=CC
              C(8)=2.0D0
              C(9)=2.0D0
              C(10)=1.0D0
              C(11)=DD
              C(12)=1.0D0
              C(13)=1.0D0
              RETURN
0018          END

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```

0001      SUBROUTINE FOUR(AA,BB,CC,DD)
C          *****
C          OPERATOR FOR THE POINT ON THE INSERT , ONE MESH LENGTH
C          LEFT OF INTERFACE. REF. ART.--2.4.4
C          *****
C          IMPLICIT REAL*8(A-H,O-Z)
C          COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
C          C(1)=AA
C          C(2)=BB
C          C(3)=-8.0D0
C          C(4)=-8.0D0
C          C(5)=-8.0D0
C          C(6)=CC
C          C(7)=2.0D0
C          C(8)=2.0D0
C          C(9)=CC
C          C(10)=DD
C          C(11)=1.0D0
C          C(12)=1.0D0
C          C(13)=1.0D0
          RETURN
          END
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FORTRAN IV G LEVEL 18          ICFOUR      DATE = 70278      18/35/47      PAGE 0001

0001      SUBROUTINE ICFOUR(AAA,BB,CCC,CC,DD)
C      *****
C      OPERATOR FOR THE POINT ON INSERT ONE MESH LENGTH AWAY FROM THE
C      UPPER RIGHT CORNER OF INTERFACE. REF. ART.--2..4.5
C      *****
0002      IMPLICIT REAL*(A-H,O-Z)
0003      COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
C
C      C(1)=AAA
C      C(2)=38
C      C(3)=BB
C      C(4)=-8.0D0
C      C(5)=-8.0D0
C      C(6)=CCC
C      C(7)=CC
C      C(8)=2.0D0
C      C(9)=CC
C      C(10)=DD
C      C(11)=DD
C      C(12)=1.0D0
C      C(13)=1.0D0
C      RETURN
0018      END

```

FCRTRAN IV G LEVEL 16 OTWD DATE = 70278 18/35/47 PAGE 0001

```

0001 SUBROUTINE OTWO(AAL,BB1,CC1,DD1)
0002 OPERATOR FOR THE POINT ON THE PARENT MATERIAL ONE MESH LENGTH
0003 ABOVE THE INTERFACE. REF. ART.--2.4.4
0004 IMPLICIT REAL*8(A-H,O-Z)
0005 COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
0006 C(1)=AAL
0007 C(2)=-8.0D0
0008 C(3)=-8.0D0
0009 C(4)=-8.0D0
0010 C(5)=BB1
0011 C(6)=2.0D0
0012 C(7)=2.0D0
0013 C(8)=CC1
0014 C(9)=CC1
0015 C(10)=1.0D0
0016 C(11)=1.0D0
0017 C(12)=1.0D0
0018 C(13)=DD1
0019 RETURN
0020 END

```

```

FORTRAN IV 6 LEVEL 18          OFOUR          DATE = 70278          18/35/47          PAGE 0001

0001      SUBROUTINE OFOUR(AA1,BB1,CC1,DD1) .
C      *****
C      OPERATOR FOR THE POINT ON THE PARENT MATERIAL ONE MESH LENGTH
C      RIGHT OF INTERFACE. REF.ART.--2..4.4
C      *****
C      IMPLICIT REAL*(A-H,O-Z)
C      COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
C(1)=AA1
C(2)=-8.0D0
C(3)=-8.0D0
C(4)=BB1
C(5)=-8.0D0
C(6)=2.0D0
C(7)=CC1
C(8)=CC1
C(9)=2.0D0
C(10)=1.0D0
C(11)=1.0D0
C(12)=DD1
C(13)=1.0D0
RETURN
0018  END

```

```

0001      C      SUBROUTINE ONTWO(A,BI,CI,D,E,F,G,H,II)
          C      *****
          C      OPERATOR FOR THE POINT ON THE
          C      UPPER INTERFACE . REF. ART. --2.4.3
          C      *****
0002      C      IMPLICIT REAL*8(A-H,O-Z)
0003      C      COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
          C
          C(1)=A
          C(2)=BI
          C(3)=D
          C(4)=BI
          C(5)=CI
          C(6)=F
          C(7)=E
          C(8)=E
          C(9)=E
          C(10)=C
          C(11)=II
          C(12)=C
          C(13)=H
          RETURN
          END
0004
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0001      C
          C SUBROUTINE ONFOUR(A,BI,CI,D,E,F,G,H,II)
          C OPERATOR FOR THE POINT ON THE
          C RIGHT INTERFACE . REF. ART.--2.4.3
          C *****
          C IMPLICIT REAL*8(A-H,O-Z)
          C COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
          C
          C(1)=A
          C(2)=D
          C(3)=BI
          C(4)=CI
          C(5)=BI
          C(6)=EE
          C(7)=EE
          C(8)=E
          C(9)=F
          C(10)=II
          C(11)=G
          C(12)=H
          C(13)=G
          RETURN
          END
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PAGE 0001

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0001 SUBROUTINE COFOUR(AR,BI,BR,C1,D,E,ER,F,FR,G,GR,H,T,I)
CCCC *****
CCCC OPERATOR FOR THE POINT ON THE
CCCC UPPER RIGHT CORNER OF THE INTERFACE . REF. ART.--2..4.6
CCCC *****
00C2 IMPLICIT REAL*8(A-H,O-Z)
00C3 COMMON P(128),RT(128),C(13),BY(8),B,HD,LA(16,8),M,N
CC C(1)=X
00C4 C(2)=-8.OO0
00C5 C(3)=-8.OO0
00C6 C(4)=-8.OO0
00C7 C(5)=Y
00C8 C(6)=2.OO0
00C9 C(7)=2.OO0
0010 C(8)=S
0011 C(9)=Z
0012 C(10)=1.OO0
0013 C(11)=1.OO0
0014 C(12)=1.OO0
0015 C(13)=T
0016 RETURN
0017 END
0018

```

CCCCCCCCCCCCCCCC

***** THIS PROGRAMME IS THE SECOND STEP IN THE THREE STEPS SOLUTION OF
THE PROBLEM *****

***** THIS SOLVES THE MATRIX, GENERATED BY STEP-1, USING GAUSSIAN
ELIMINATION METHOD. *****

```

0001  IMPLICIT REAL*8(A-H,O-Z)
0002  COMMON Q(128,128)
0003  DIMENSION Q(128),X(128),P(128),PP(128)
0004  READ(1,5)N
0005  DO 45 I=1,N
0006  READ(7)(P(K),K=1,N)
0007  DO 45 J=1,N
0008  Q(I,J)=P(J)
0009  READ(7,END=123)(PP(I),I=1,N)
0010  CONTINUE
0011  DO 75 I=1,N
0012  QQ(I)=PP(I)
0013  L1=1
0014  L=16
0015  WRITE(3,25)L1,L
0016  DO 35 I=1,N
0017  WRITE(3,110)(I,(Q(I,J),J=L1,L))
0018  IF(L.EQ.N)GO TO 20
0019  L1=L+1
0020  L=L+16
0021  IF(L.GT.N)L=N
0022  GO TO 40
0023  CONTINUE
0024  WRITE(3,10)(QQ(I),I=1,N)
0025  FORMAT(8F10.4)
0026  CALL MATINV(N)
0027  DO525 I=1,N
0028  X(I)=0.0
0029  DO525 J=1,N

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0030      X(I)=X(I)+Q(I,J)*Q(J)
0031      WRITE(3,57)((I,X(I)),I=1,N)
0032      WRITE(2,15)((X(I),I=1,N)
0033      15 FORMAT(4E20.10)
0034      15 FORMAT(15)
0035      25 FORMAT(//,5X,' COLUMN NO.',15,3X,15)
0036      57 FORMAT(//,3X,6(I5,F12.6,2X))
0037      110 FORMAT(2X,13,1X,16(F6.2,1X))
0038      RETURN
0039      END

```

```

0001 SUBROUTINE MATINV( N)
0002 IMPLICIT REAL*8(A-F,O-Z)
0003 COMMON A(128,128)
0004 DIMENSION INDEX(128,2)
0005 DO 108 I=1,N
0006 INDEX(I,1)=0
0007 II=0
0008 AMAX=-1.0D0
0009 DO 110 J=1,N
0010 IF (INDEX(I,1)) 110,111,110
0011 DO 112 J=1,N
0012 IF (INDEX(J,1)) 112,113,112
0013 TEMP=DABS(A(I,J))
0014 IF (TEMP-AMAX) 112,112,114
0015 IROW=I
0016 ICOL=J
0017 AMAX=TEMP
0018 CONTINUE
0019 CONTINUE
0020 IF(AMAX) 225,115,116
0021 INDEX (ICOL,1)=IROW
0022 IF (IROW-ICOL) 119,118,119
0023 DO 120 J=1,N
0024 TEMP=A(IROW,J)
0025 A(IROW,J)=A(ICOL,J)
0026 A(ICOL,J)=TEMP
0027 II=II+1
0028 INDEX (II,2)=ICOL
0029 PIVOT=A(ICOL,ICOL)
0030 A(ICOL,ICOL)=1.0D0
0031 PIVOT=1.0D0/PIVOT
0032 DO 121 J=1,N
0033 A(ICOL,J)=A(ICOL,J)*PIVOT
0034 DO 122 I=1,N
0035 IF (I-ICOL) 123,122,123
0036 TEMP=A(I,ICOL)
0037 A(I,ICOL)=0.0D0
0038 DO 124 J=1,N
0039 A(I,J)=A(I,J)-A(ICOL,J)*TEMP
0040 CONTINUE
0041 GO TO 109
0042 ICOL=INDEX(II,2)
0043 IROW=INDEX(ICOL,1)

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18/33/30

DATE = 70278

MATINV

FURTRAN IV G LEVEL 18

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0044 DO 126 I=1,N
0045 TEMP=A(I,I,ROW)
0046 A(I,IROW)=A(I,ICOL)
0047 A(I,ICOL)=TEMP
0048 IF(I=1) I=I-1
0049 126 IF(I) 125,127,125
0050 115 WRITE(3,100)
0051 100 FORMAT(1H0,2X,' ZERO PIVOT')
0052 127 CONTINUE
0053 RETURN
0054 END

```

 THIS PROGRAMME IS THE THIRD STEP IN THE THREE STEPS SOLUTION OF
 THE PROBLEM (CASE 1 --- INSERT INCLINED 0). *****

 THIS PROGRAMME CALCULATES THE DIRECT AND SHEAR STRESSES FROM THE
 PHI VALUES AT THE NODAL POINTS OBTAINED FROM STEP 2 . *****

 THE VARIABLES ARE DEFINED AS FOLLOWS : -----

I NUMBER COUNTER IN Y - DIRECTION
 J NUMBER COUNTER IN X - DIRECTION
 M NO. OF STEPS IN Y - DIRECTION
 N NO. OF STEPS IN X - DIRECTION
 E1 , E2 MODULI OF ELASTICITY OF INSERT AND PARENT MATERIAL
 B1 , B2 RESPECTIVELY .
 C THE POISSONS RATIO OF INSERT AND PARENT MATERIAL
 B1(1,J) PHI VALUE ALONG BC (REF. FIG. 2)
 B2(1,J) THE PHI VALUE OF A FICTITIOUS POINT ON THE INSERT
 IN TERMS OF REAL POINTS
 B3(1,J) THE PHI VALUE OF A FICTITIOUS POINT ON THE PARENT
 MATERIAL IN TERMS OF REAL POINTS.
 PX , PY DIRECT STRESSES
 TXY SHEAR STRESS
 PX1 , PY1 MAJOR AND MINOR PRINCIPAL STRESSES
 PX2 , PY2 MAJOR AND MINOR PRINCIPAL STRESSES AT THE INTERFACE
 ON THE PARENT MATERIAL.

THE INSERT SIZE IS DEFINED AS FOLLOWS :
 LENGTH

LARGEST NC = 5

UUUUUUUUUU

WIDTH

MEDIUM
SMALLEST

$$\frac{43}{22} = 1.9545$$

FOR ALL CASES MC = 3

❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀ ❀

```

0001 IMPLICIT REAL*8 (A-H,O-Z)
0002 DIMENSION A(16,32),BI(16,32),BO(16,32),P(16),PPXX(8),
0003 1PPY(17,32),PX2(8),PY1(4),PY2(4)
0004 READ(1,10)M,N,DEL
0005 WRITE(3,10)M,N,DEL
0006 10 FORMAT(2I5,F10.3)
0007 READ(1,16)EI,E2,B1,B2
0008 WRITE(3,16)EI,E2,B1,B2
0009 16 FORMAT(2D20.8,2F10.6)
0010 MC=3
0011 NC=3
0012 MC1=MC-1
0013 NC1=NC-1
0014 DE=0.0
0015 P(1)=0.0
0016 DO 25 I=2,M
0017 DE=DE+0.125
0018 P(I)=50.*(DE**2)
0019 WRITE(3,26)(P(I),I=1,M)
0020 26 FORMAT(8F10.6)
0021 READ(1,20)((A(I,J),J=1,M),I=1,M)
0022 WRITE(3,20)((A(I,J),J=1,N),I=1,M)
0023 20 FORMAT(4D20.12)
0024 M1=M+1
0025 DO 5 I=1,M1
0026 DO 5 J=1,N
0027 PPXX(I,J)=0.
0028 PPY(I,J)=0.
0029 IXY(I,J)=0.
0030 5 CONTINUE
0031 A1=1./EI
0032 A2=1./E2
0033 Q1=(A2-A1)/(A2+A1)

```

```

0033      O2=((A2*B2-A1*B1+A1-A2)*2.)/(A1+A2)
0034      O3=-((A2*B2-A1*B1)/(A1+A2))
0035      O4=(1+(A2-A1)/(A1+A2))
0036      O01=-((A2-A1)/(A2+A1))
0037      O02=-((A2*B2-A1*B1+A1-A2)*2.)/(A1+A2)
0038      O03=((A2*B2-A1*B1)/(A1+A2))
0039      O04=(1-(A2-A1)/(A1+A2))

```

```

*****
CALCULATION OF PXX
*****

```

```

DO 85 K=1,NC1
PPXX(K)=0.0
PPYY(K)=0.0
I=1
J=1
40 PXX(I,J)=(A(I+1,J)-A(I,J))*2.*DEL
J=J+1
IF(J.LT.(N+1))GOTO40
I=I+1
J=1
60 PXX(I,J)=(A(I+1,J)+A(I-1,J))-2.*A(I,J))*DEL
WRITE(3,81)I,J,A(I+1,J),A(I-1,J),PXX(I,J)
81 FORMAT(5X,15,5X,15,5X,4F10.4)
J=J+1
IF(J.LT.(N+1))GOTO50
I=I+1
IF(I.EQ.MC)GO TO 70
IF(I.LT.M)GO TO 60
GOTO90
J=1
70 BI(I+1,J)=A(I-1,J)*O1+A(I,J)*O2+A(I,J+1)*2.*O3+A(I+1,J)*O4
BI(I-1,J)=A(I+1,J)*O01+A(I,J)*O02+A(I,J+1)*2.*O03+A(I-1,J)*O04
GO TO 80
80 BI(I+1,J)=A(I-1,J)*O1+A(I,J)*O2+(A(I,J+1)+A(I,J-1))*O3+A(I+1,J)*O4
BI(I-1,J)=A(I+1,J)*O01+A(I,J)*O02+(A(I,J+1)+A(I,J-1))*O03+A(I-1,J)*O04
1*O04
80 PXX(I,J)=(BI(I+1,J)+A(I-1,J)-A(I,J)*2.)*DEL
PPXX(J)=(A(I+1,J)+B0(I-1,J)-A(I,J)*2.)*DEL
J=J+1

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C C C C C C C

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0068 IF(J.LT.NC) GO TO 75
0069 GO TO 50
0070 C=50.
0071 J=1
0072 PXX(I,J)=(C+A(I-1,J)-2.*A(I,J))*DEL
0073 PXX(I+1,J)=(A(I,J)*2.+12.5-2.*C)*DEL
0074 J=J+1
0075 IF(J.LT.(N+1))GO TO 100

```

CCCCC

 CALCULATION OF PYY *****

```

0076 J=1
0077 I=1
0078 PYY(I,J)=(A(I,J+1)-A(I,J))*2.*DEL
0079 I=I+1
0080 IF(I.LT.(M+1))GO TO 110
0081 I=1
0082 J=2
0083 PYY(I,J)=(A(I,J+1)+A(I,J-1))-2.*A(I,J))*DEL
0084 J=J+1
0085 IF(J.EQ.NC.AND.I.EQ.1)GO TO 140
0086 IF(J.EQ.NC.AND.I.LT.NC) GO TO 150
0087 IF(J.LT.N)GO TO 130
0088 GO TO 160
0089 BI(I,J+1)=A(I,J-1)*01+A(I,J)*02+A(I+1,J)*2.*03+A(I,J+1)*04
0090 BO(I,J-1)=A(I,J+1)*001+A(I,J)*002+A(I+1,J)*2.*003+A(I,J-1)*004
0091 GO TO 170
0092 BI(I,J+1)=A(I,J-1)*01+A(I,J)*02+(A(I-1,J)+A(I+1,J))*03+A(I,J+1)*04
0093 BO(I,J-1)=A(I,J+1)*001+A(I,J)*002+(A(I-1,J)+A(I+1,J))*003+A(I,J-1)
0094 1*004
0095 PYY(I,J)=(BI(I,J+1)+A(I,J-1))-A(I,J)*2.*DEL
0096 PPYY(I)=(A(I,J+1)+BO(I,J-1))-A(I,J)*2.*DEL
0097 J=J+1
0098 GO TO 130
0099 J=N
0100 PYY(I,J)=(P(I)+A(I,J-1))-2.*A(I,J))*DEL
0101 I=I+1
0102 IF(I.LT.(M+1))GO TO 120

```

C

 CALCULATION OF TXY

```

DO 180 I=1,M1
DO 180 J=1,N
TXY(I,J)=0.0
I=2
J=2
IF(I.EQ.MC.AND.J.EQ.NC) GO TO 200
TXY(I,J)=-(A(I+1,J+1)-A(I-1,J-1)+A(I-1,J+1)-A(I+1,J-1))*DEL/4.
J=J+1
IF(J.LT.N) GO TO 190
J=2
I=I+1
IF(I.LT.M) GO TO 190
GO TO 205
TXY(I,J)=-(A(I+1,J+1)-A(I-1,J-1)+A(I-1,J+1)-A(I+1,J-1))*DEL/4.
GO TO 195
I=M
J=2
TXY(I,J)=-(A(I-1,J-1)-A(I-1,J+1))*DEL/4.
J=J+1
IF(J.LT.N) GO TO 210
J=N
I=2
TXY(I,J)=-(P(I+1)-A(I+1,J-1)+A(I-1,J-1)-P(I-1))*DEL/4.
I=I+1
IF(I.LT.M) GO TO 220
I=M
J=N
TXY(I,J)=-(A(I-1,J-1)-P(I-1))*DEL/4.
WRITE(3,199)
DO 300 I=1,M1
DO 300 J=1,N
PC=(PXX(I,J)+PYY(I,J))/2.
PR=DSQRT(((PXX(I,J)-PYY(I,J))/2. )**2+(TXY(I,J) )**2)
P1(I,J)=PC+PR
P2(I,J)=PC-PR
GO TO 300
CONTINUE
I=3

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0139 DO 700 J=1,NC1
0140 PC=(PPXX(J)+PPYY(I,J))/2.
0141 PR=DSQRT(((PPXX(J)-PPYY(I,J))/2.)**2+((TXY(I,J))**2)
0142 PX1(J)=PC+PR
0143 PX2(J)=PC-PR
0144 CONTINUE
0145 J=NC
0146 DO 705 I=1,MC1
0147 PC=(PXX(I,J)+PPYY(I,J))/2.
0148 PR=DSQRT(((PXX(I,J)-PPYY(I,J))/2.)**2+((TXY(I,J))**2)
0149 PY1(I)=PC+PR
0150 PY2(I)=PC-PR
0151 CONTINUE
0152 705 WRITE(3,246)
0153 246 FORMAT(5X,'RECT. INSERT A/AA=1/4, B/BB=1/8, BB/AA=1/2, STRESS
1=100PSI,')
0154 WRITE(3,249)
0155 249 FORMAT(45X,'STRESS PXX')
0156 WRITE(3,250)((PXX(I,J),J=1,N),I=1,M1)
0157 199 FORMAT(//,1X,16F8.0)
0158 FORMAT(1H1)
0159 WRITE(3,259)
0160 259 FORMAT(//,2X,'STRESS ON OUTSIDE OF INTERFACE')
0161 WRITE(3,260)((PXX(K),K=1,8)
0162 260 FORMAT(//,1X,4F8.0)
0163 WRITE(3,199)
0164 WRITE(3,248)
0165 248 FORMAT(45X,'STRESS PYY')
0166 WRITE(3,250)((PPY(I,J),J=1,N),I=1,M)
0167 WRITE(3,259)
0168 WRITE(3,260)((PPY(K),K=1,MC1)
0169 WRITE(3,199)
0170 WRITE(3,247)
0171 247 FORMAT(45X,'STRESS TXY')
0172 WRITE(3,250)((TXY(I,J),J=1,N),I=1,M)
0173 WRITE(3,199)
0174 K1=1
0175 K=10
0176 755 WRITE(3,716)
0177 716 FORMAT(1H1,20X,'PRINCIPAL STRESSES P1 (LARGER ) AT INTERFACE ')
0178 WRITE(3,15)K1,K
0179 DO 725 I=1,M
0180 725 WRITE(3,35)(I,( P1(I,J),J=K1,K))

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0181 IF(K.EQ.N)GO TO 745
0182 K1=K+1
0183 K=K+10
0184 IF(K.GT.N)K=N
0185 GO TO 755
0186 CONTINUE
0187 WRITE(3,259)
0188 WRITE(3,800) (PX1(J),J=1,8), (PY1(I),I=1,4)
0189 FORMAT(/,2X,'PX1 ',8F 9.0,10X,'PY2 ',2X,4F 9.0)
0190 WRITE(3,199)
0191 K1=1
0192 K=10
0193 WRITE(3,916)
0194 FORMAT(1H1,20X,' PRINCIPAL STRESSES P2 (SMALLER )')
0195 WRITE(3,15)K1,K
0196 DO 925 I=1,M
0197 WRITE(3,35)(I,(P2(I,J),J=K1,K))
0198 IF(K.EQ.N)GO TO 945
0199 K1=K+1
0200 K=K+10
0201 IF(K.GT.N)K=N
0202 GO TO 955
0203 CONTINUE
0204 FORMAT(/,5X,110,10F10.0)
0205 WRITE(3,259)
0206 WRITE(3,801) (PX2(J),J=1,NC1), (PY2(I),I=1,MC1)
0207 FORMAT(/,2X,'PX2 ',8F 9.0,10X,'PY2 ',2X,4F 9.0)
0208 RETURN
0209 END
0210

```

```
0001      COMMON DEL(35), DEL1(35), DX(35,35), DY(35,35), BX(35,35), RY(35,35),  
1      BFX(35,35), BFY(35,35), BOFX(35,35), BOFY(35,35), C(13), LA(35,35),  
2      DEK(35), DEK1(35), P(451), FX(35,35), FFEX(35,35), FY(35,35), FFY(35,35),  
3      M1(500), N1(500), RT(500)
```

```
0002      READ(1,10) MM, NN, E1, E2, B1, B2  
0003      WRITE(3,10) MM, NN, E1, E2, B1, B2  
0004      READ(1,20) (DEL(I), I=1, MM), (DEK1(J), J=1, NN)  
0005      WRITE(1,20) (DEK(I), I=1, MM), (DEK1(J), J=1, NN)  
0006      WRITE(3,20) ((DEL(I), I=1, MM), (DEK1(J), J=1, NN))  
0007      WRITE(5,20) ((DEK(I), I=1, MM), (DEK1(J), J=1, NN))  
0008      FORMAT(2I10, 4F15.8)  
0009      DO 65 I=1, MM  
0010      DO 65 J=1, NN  
0011
```

INITIAL THE VARIABLES.

```
LA(I,J)=0  
DX(I,J)=0.0  
DY(I,J)=0.0  
BX(I,J)=0.0  
BY(I,J)=0.0  
BFX(I,J)=0.0  
BFY(I,J)=0.0  
BOFX(I,J)=0.0  
BOFY(I,J)=0.0  
FFEX(I,J)=0.  
FFY(I,J)=0.  
FFY(I,J)=0.  
65 CONTINUE
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```
0031 BI = (-8.+4.*(A2*B2-A1*B1))*((A2*B2-A1*B1)-(A2-A1))/AA1)
0032 CI = (-8.+4.*(A2*B2-A1*B1))*((A2-A1))*((8.*A2+4.*A1))/AA1)
0033 D = (-8.-4.*(A2*B2-A1*B1))*((A2-A1))*((4.*A2+9.*A1))/AA1)
0034 E = (2.-2.*(A2*B2-A1*B1))*A1+(A2-A1)*(A1+A2))/AA1)
0035 F = (2.+2.*(A2*B2-A1*B1))*A2+(A2-A1)*(A1+A2))/AA1)
0036 G = (1.-((A2*B2-A1*B1))*((A2*B2-A1*B1))/AA1)
0037 H = (1.-((A2-A1))/((A2+A1)))
0038 I = (1.+((A2-A1))/((A2+A1)))
0039 AA = (20.+((A2-A1))/((A2+A1)))
0040 AA1 = (20.-((A2-A1))/((A2+A1)))
0041 BB = (-8.+2.*(A2*B2-A1*B1+A1-A2))/(A1+A2))
0042 BB1 = (-8.-2.*(A2*B2-A1*B1+A1-A2))/(A1+A2))
0043 CC = (2.-((A2*B2-A1*B1))/((A1+A2)))
0044 CC1 = (2.+((A2*B2-A1*B1))/((A1+A2)))
0045 DD = (1.+((A2-A1))/((A1+A2)))
0046 DD1 = (1.-((A2-A1))/((A1+A2)))
0047 AAA = (20.+2.*(A2-A1))/((A1+A2))
0048 CCC = (20.-2.*(A2*B2-A1*B1))/((A1+A2))
0049 XX = (20.-2.*(A1*B1-A2*B2))/(A1+A2))
0050 YY = (-8.-4.*(A1-A2-A1*B1+A2*B2))/(A1+A2))
0051 Z = (2.-2.*(A2-A1))/((A1+A2))
0052 S = (1.-2.*(A1*B1-A2*B2))/(A1+A2))
0053 T = (1.-2.*(A1*B1-A2*B2))/(A1+A2))
0054 WRITE(3,12)A,BI,CI,D,E,F,G,H,I,AA,AA1,BB,BB1,CC,CC1,DD,DDL,XX,YY,
12 FORMAT(3X,10F12.5)
17,S,T,AAA,CCC
DO 451 K=1,N
DO 453 K5=1,N
453 P(K5)=0.0
B=0.0
111 I=M1(K)
J=M1(K)
IF(I.LT.(M1-1)).OR.I.GT.(M0-1)) GO TO 170
IF(J.LT.(M1-1)).OR.J.GT.(M0-1)) GO TO 170
IF(I.LT.(M1+1)).OR.I.GT.(M0-1)) GO TO 180
IF(J.LT.(M1+1)).OR.J.GT.(M0-1)) GO TO 180
POINTST FALL INSIDE THE BOUNDARY.
195 IF(I.GT.(M1+1)).AND.I.LT.(M0-1)) GO TO 200
IF(I.EQ.(M1+1)).AND.J.EQ.(M1+1)) GO TO 230
IF(I.EQ.(M0-1)).AND.J.EQ.(M1+1)) GO TO 250
IF(I.EQ.(M1+1)).AND.J.EQ.(M0-1)) GO TO 260
IF(I.EQ.(M0-2)).AND.J.EQ.(M0-2)) GO TO 225
IF(I.EQ.(M1+1)).AND.J.EQ.(M0-2)) GO TO 215
```

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0072 WRITE(3,498)I,J,N
0073 RETURN
0074 IF(J.EQ.(NO-2)) GO TO 190
0075 IF(J.EQ.(NI+1)) GO TO 210
0076 IF(J.EQ.(NO-1)) GO TO 220
0077 WRITE(3,498)I,J,N
0078 RETURN
0079 CALL GAIN(I,J)
0080 GO TO 450
0081
0082 CALL GAIN(I,J)
0083 GO TO 450
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0085 CALL GAIN(I,J)
0086 GO TO 450
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0088 CALL GAIN(I,J)
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0091 CALL GAIN(I,J)
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0094 CALL GAIN(I,J)
0095 GO TO 450
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0097 CALL GAIN(I,J)
0098 GO TO 450
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0100 CALL GAIN(I,J)
0101 GO TO 450
0102
0103 CALL GAIN(I,J)
0104 GO TO 450
0105

0106 THE POINTS FALL ON THE BOUNDARY OR OUTSIDE THE INSERT BUT WITH
0107 REGULAR MESH.
0108 IF(I.GT.NI.AND.I.LT.NO)GO TO 270
0109 IF(J.GT.NI.AND.J.LT.NO)GO TO 280
0110 IF(I.EQ.NI.AND.J.EQ.NO)GO TO 370
0111 IF(I.EQ.NI.AND.J.EQ.NO)GO TO 380
0112 IF(I.EQ.NO.AND.J.EQ.NO)GO TO 390
0113 IF(I.EQ.NO.AND.J.EQ.NO)GO TO 400
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0113 CALL GAIN(I,J)
0114 GO TO 450
0115 IF(J.EQ.(NI-1)) GO TO 290
0116 IF(J.EQ.(NO+1)) GO TO 300
0117 IF(J.EQ.NI)GO TO 330
0118 IF(J.EQ.NO)GO TO 340
0119 WRITE(3,498)I,J,N
0120 RETURN
0121 IF(I.EQ.(MI-1)) GO TO 310
0122 IF(I.EQ.(MO+1)) GO TO 320
0123 IF(I.EQ.MI)GO TO 350
0124 IF(I.EQ.MO)GO TO 360
0125 WRITE(3,498)I,J,N
0126 RETURN
0127 CALL GAIN(I,J)
0128 GO TO 450
0129 CALL GAIN(I,J)
0130 GO TO 450
0131 CALL GAIN(I,J)
0132 GO TO 450
0133 CALL GAIN(I,J)
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0135 CALL GAIN(I,J)
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0137 CALL GAIN(I,J)
0138 GO TO 450
0139 CALL GAIN(I,J)
0140 GO TO 450
0141 CALL GAIN(I,J)
0142 GO TO 450
0143 CALL GAIN(I,J)
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0145 CALL GAIN(I,J)
0146 GO TO 450
0147 CALL GAIN(I,J)
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0149 CALL GAIN(I,J)
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0151 CALL GAIN(I,J)
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0153 CALL GAIN(I,J)
0154 GO TO 450
0155 CALL GAIN(I,J)

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OGNE(AAL,BB1,CC1,DDL)
 OTWO(AAL,BB1,CC1,DDL)
 OTHREE(AAL,BB1,CC1,DDL)
 OFOUR(AAL,BB1,CC1,DDL)
 ONONE(A,BI,CI,D,E,F,G,H,II)
 ONTWO(A,BI,CI,D,E,F,G,H,II)
 ONTHRE(A,BI,CI,D,E,F,G,H,II)
 ONFOUR(A,BI,CI,D,E,F,G,H,II)
 CCONE(XX,YY,S,Z,T)
 COTWO(XX,YY,S,Z,T)

FORTRAN IV G-LEVEL 18		MAIN	DATE = 70278	18/30/45	PAGE 0007
0156		GO TO 450			
0157	390	CALL GAIN(I,J)			
0158		GO TO 450			
0159					
0160	400	CALL GAIN(I,J)			
0161		GO TO 450			
0162		CALL GNERAL(I,J,B,K)			
0163	170	WRITE(3,499)K			
0164	450	WRITE(7)(P(KK),KK=1,N)			
0165		WRITE(3,501)B			
0166		RT(K)=B			
0167		FORMAT(//,5X,15)			
0168	499	FORMAT(3X,10(F10.4,2X))			
0169	500	FORMAT(5X,3110)			
0170	498	FORMAT(//,5X,F10.4)			
0171	501	CONTINUE			
0172	451	WRITE(7)(RT(KK),KK=1,N)			
0173		RETURN			
0174		END			
0175					

```

0001 SUBROUTINE MESH(MM,NN)
0002 COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
      1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
      2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
      3,MI(500),NI(500),RI(500)

```

C

```

0003 Y=0.0
0004 X1=0.0
0005 Y1=0.0
0006 N=0
0007 DO 60 J=1,35
0008 Y=Y+DELI(J)
0009 X=0.0
0010 DO 60 I=1,34
0011 X=X+DELI(I)
0012 EE=1./SQRT(2.)
0013 IF(X.GT.EE)GO TO 70
0014 IF(Y.GT.EE)GO TO 130
0015 Y1=-X+EE
0016 X1=-Y+EE
0017 IF(Y1.GT.Y)GO TO 60
0018 IF((Y-Y1).GT.DELI(J))GOTO40
0019 IF((X-X1).GT.DEL(I))GO TO 90
0020 CALL BOONE(I,J,X,X1,Y,Y1)
0021 GO TO 60
0022 CALL YONE(I,J,X,Y,Y1)
0023 GO TO 60
0024 IF((X-X1).LT.DEL(I))GO TO 100
0025 130 Y1=X+EE
0026 X1=Y-EE
0027 IF(Y1.LT.Y)GO TO 60
0028 IF((Y1-Y).GT.DELI(J+1))GO TO 50
0029 IF((X-X1).GT.DEL(I))GO TO 55
0030 CALL BTWO(I,J,X,X1,Y,Y1)
0031 GO TO 60
0032 CALL YTWO(I,J,X,Y,Y1)
0033 GO TO 60
0034 CALL XONE(I,J,X,X1,Y)
0035 GO TO 60
0036 50 IF((X-X1).GT.DEL(I))GO TO 80
0037 CALL XTWO(I,J,X,X1,Y)
0038 GO TO 60

```

```

0039 70 Y1=X-EE
0040 X1=Y+EE
0041 IF(Y1.GT.Y)GO TO 60
0042 IF((Y-Y1).GT.DEL(I,J))GO TO 110
0043 IF((X1-X).GT.DEL(I+1))GO TO 120
0044 CALL BCTHRE(I,J,X,X1,Y,Y1)
0045 GO TO 60
0046 CALL YTHRE(I,J,X,Y,Y1)
0047 GO TO 60
0048 IF((X1-X).GT.DEL(I+1))GO TO 140
0049 CALL XTHREE(I,J,X,X1,Y)
0050 GO TO 60
0051 IF(X.LT.(2./SQRT(2.)))GO TO 130
0052 EEL=5.*EE
0053 Y1=-X+EEI
0054 X1=-Y+EEI
0055 IF(Y1.LT.Y)GO TO 60
0056 IF((Y1-Y).GT.DEL(I,J+1))GO TO 150
0057 IF((X1-X).GT.DEL(I+1))GO TO 160
0058 CALL BCFOUR(I,J,X,X1,Y,Y1)
0059 GO TO 60
0060 CALL YFOUR(I,J,X,Y,Y1)
0061 GO TO 60
0062 IF((X1-X).GT.DEL(I+1))GO TO 80
0063 CALL XFOUR(I,J,X,X1,Y)
0064 GO TO 60
0065 N=N+1
0066 M1(N)=I
0067 N1(N)=J
0068 LA(I,J)=N
0069 CONTINUE
0070 WRITE(3,81)N
0071 WRITE(3,87)
0072 WRITE(3,88)((DX(I,J),I=1,MM),J=1,NN)
0073 WRITE(3,87)
0074 WRITE(3,88)((DY(I,J),I=1,MM),J=1,NN)
0075 WRITE(3,87)
0076 WRITE(3,88)((BX(I,J),I=1,MM),J=1,NN)
0077 WRITE(3,87)
0078 WRITE(3,88)((BY(I,J),I=1,MM),J=1,NN)
0079 WRITE(3,87)
0080 WRITE(3,88)((BFX(I,J),I=1,MM),J=1,NN)
0081 WRITE(3,87)

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ORTRAN IV G LEVEL 18

MESH

DATE = 70278

18/30/45

PAGE 0003

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0082 WRITE(3,88)((BFY(I,J),I=1,MM),J=1,NN)
0083 WRITE(3,87)
0084 WRITE(3,88)((BOFX(I,J),I=1,MM),J=1,NN)
0085 WRITE(3,87)
0086 WRITE(3,88)((BOFY(I,J),I=1,MM),J=1,NN)
0087 WRITE(3,87)
0088 WRITE(3,89)((LA(I,J),I=1,MM),J=1,NN)
0089 WRITE(3,87)
0090 FORMAT(///)
0091 FORMAT(5X,11F11.5)
0092 FORMAT(5X,21I5)
0093 FORMAT(5X,15)
0094 RETURN
0095
0096
```

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END

```

0001      SUBROUTINE XONE (I,J,X,X1,Y)
C      *****
C      THIS SUBROUTINE IS USED ALONG LINE AD USING POINTS IN X DIRECTION.
C      REF. -ART.2.4.2
C      *****
0002      COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFX(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C
C      DD=X-X1
C      IF(I.EQ.2)GO TO 10
C      DU=DEL(I)-DD
C      GO TO 20
C      DU=.0625-DD
C      DELTA=0.06250
C      DX(I,J)=DU/DELTA
C      SLOX=100.0*X1 + 100.0*SQR(2.0)
C      BX(I-1,J)=50.0*(X1**2)+50.0*(Y*Y)
C      1+100.0*SQR(2.0) *X1 + 100.0*SQR(2.0) *Y-300.0
C      D1 = 1+DX(I,J)
C      D2 = D1**2
C      D3 = 1-DX(I,J)
C      BFX(I,J) = (1+2.*DX(I,J))*BX(I-1,J)/D2+DD*SLOX/D1
C      BOFX(I-1,J) = 4.*DX(I,J)*BX(I-1,J)/D2-2.*DU*SLOX/D1
C      FFX(I+1,J)=DX(I,J)**2/D2
C      FFX(I+1,J)=D3**2/D2
C      RETURN
C      END

```

```

000001  ORTRAN IV G-LEVEL 18          YONE      DATE = 70278          18/30/45          PAGE 0001
0001    SUBROUTINE YONE (I,J,X,Y,Y1)
0002    ***** THIS SUBROUTINE IS USED ALONG LINE AD USING POINTS IN Y DIRECTION. *****
0003    REF -ART.2.4.2
0004    ***** COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
0005    1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
0006    2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
0007    *****
0008    DD=Y-Y1
0009    IF(J.EQ.2)GO TO 10
0010    DU=DEL1(J)-DD
0011    GO TO 20
0012    DU=.0625-DD
0013    DELTA=0.06250
0014    DY(I,J)=DD/DELTA
0015    SLOY=100.0*Y1+100.0*SQR(T2.0)
0016    BY(I,J-1)= 50.0*(X*X)+50.0*(Y1**2) +100.0*SQR(T2.0)*X
0017    1+100.*SQR(T2.0)*Y1 -300.0
0018    D1 = 1+DY(I,J)
0019    D2 = D1**2
0020    D3 = 1-DY(I,J)
0021    BFX(I,J) = (1+2.*DY(I,J))*BY(I,J-1)/D2+DD*SLOY/D1
0022    BOFY(I,J-1) = 4.*DY(I,J)*BY(I,J-1)/D2-2.*DU*SLOY/D1
0023    FY(I,J+1)=DY(I,J)**2/D2
0024    FFY(I,J+1)=D3**2/D2
0025    RETURN
0026    END

```

```

FORTRAN IV G LEVEL 18
0001 SUBROUTINE BOONE (I,J,X,X1,Y,Y1)
0002 COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
0003 C
0004 C
0005 C
0006 C
CALL
CALL
RETURN
END
YONE (I,J,X,Y,Y1)
XONE (I,J,X,X1,Y)
DATE = 70278
18/30/45
PAGE 0001

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FORTRAN IV G-LEVEL 18          XTWO          DATE = 70278          18/30/45          PAGE 0001
0001      SUBROUTINE XTWO (I,J,X,X1,Y )
C      *****
C      THIS SUBROUTINE IS USED ALONG LINE CD USING POINTS IN X DIRECTION.
C      -ART.2..4.2
C      *****
0002      COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),RX(35,35),BY(35,35),
1BFX(35,35),BFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C
DD=X-X1
IF(1.EQ.2)GO TO 10
DU=DEL(I)-DD
GO TO 20
DU=.0625-DD
DELTA=0.06250
DX(I,J)=DD/DELTA
SLOX = 100.0*SQRT(2.0)
BX (I-1,J) = 100.0*SQRT(2.0)*X1 + 150.0 *SQRT(2.0)*Y -325.0
D1 = 1+DX(I,J)
D2 = D1**2
D3 = 1-DX(I,J)
BFX(I,J) = (1+2.*DX(I,J))*BX(I-1,J)/D2+DD*SLOX/D1
BOFX(I-1,J)=4.*DX(I,J)*BX(I-1,J)/D2-2.*DU*SLOX/D1
FX(I+1,J)=DX(I,J)**2/D2
FFX(I+1,J)=D3**2/D2
RETURN
END
0003
0004
0005
0006
0007
0008
0009
0010
0011
0012
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0020

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0001      FORTRAN IV G LEVEL 18          YTDW          DATE = 70278          18/30/45          PAGE 0001
0001      SUBROUTINE YTDW (I,J,X,Y,Y1)
0001      *****
0001      THIS SUBROUTINE IS USED ALONG LINE CD USING POINTS IN Y DIRECTION.
0001      REF. -ART. 2.4.2
0001      *****
0001      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
0001      1BFX(35,35),B=FY(35,35),BQFY(35,35),C(13),LA(35,35),
0001      2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
0001      *****
0001      DD=Y1-Y
0001      IF(J.EQ.2)GO TO 10
0001      DU=DELI(J+1)-DD
0001      GU TO 20
0001      DU=.0625-DD
0001      DELTA=0.06250
0001      DY(I,J+1)=DD/DELTA
0001      SLOY = 150.0 *SQRT (2.0)
0001      BY (I,J+1) = 100.0 *SQRT (2.0)*X+ 150.0*SQRT (2.0) *Y1 -325.0
0001      D1=1+DY(I,J+1)
0001      D2=D1**2
0001      D3=1-DY(I,J+1)
0001      BIFY(I,J)=(1.+2.*DY(I,J+1))*BY(I,J+1)/D2-DD*SLOY/D1
0001      BCFY(I,J+1)=4.*DY(I,J+1)*BY(I,J+1)*2/D2
0001      FY(I,J-1)=DY(I,J+1)**2/D2
0001      FFY(I,J-1)=D3**2/D2
0001      RETURN
0001      END
0002
0003
0004
0005
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FORTRAN IV G LEVEL 18

PAGE 0001

18/30/45

DATE = 70279

BOTWO

```
0001      C      SUBROUTINE BOTWO (I,J,X,X1,Y,Y1)
0002      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
      1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
      2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
      C
      CALL      YIWO (I,J,X,X1,Y,Y1)
      CALL      XTWO (I,J,X,X1,Y)
      RETURN
      END
0003
0004      10
0005      20
0006
```



```

0001      SUBROUTINE YTHRE (I,J,X,Y,Y1)
C          *****
C          THIS SUBROUTINE IS USED ALONG LINE AB USING POINTS IN Y DIRECTION.
C          REF. -ART.2..4.2
C          *****
0002      COMMON DEL(35),DEK1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFX(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C
C          DD=Y-Y1
C          IF(J.EQ.2)GO TO 10
C          DU=DEL1(J)-DD
C          GO TO 20
C          DU=.0625-DD
C          DELTA=0.06250
C          DY(I,J)=DU/DELTA
C          SLOY = 100.*SQRT(2.)
C          BY(I,J-1) = 150.*SQRT(2.)*X+100.*SQRT(2.)*Y1-325.
C          D1=1.+DY(I,J)
C          D2=D1**2
C          D3=1.-DY(I,J)
C          BFX(I,J)=(1.+2.*DY(I,J))*BY(I,J-1)/D2+DD*DD*SLOY/D1
C          BOFY(I,J-1)=4.*DY(I,J)*BY(I,J-1)/D2-2.*DU*SLOY/D1
C          FFX(I,J+1)=DY(I,J)**2/D2
C          FFY(I,J+1)=D3**2/D2
C          RETURN
C          END

```

FORTRAN IV G LEVEL 18 BOTHRE DATE = 70278 18/30/45 PAGE 0001

0001

 SUBROUTINE BOTHRE (I,J,X,X1,Y,Y1)

0002

 COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
 1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
 2DEK(35),DEK1(35),P(500),FX(451),FFX(451),FY(451),FFY(451)

0003

 YTHRE (I,J,X,X,Y,Y1)

0004

 CALL

0005

 CALL

0006

 RETURN

 10

 20

 END

```

0001      SUBROUTINE XFOUR(I,J,X,X1,Y)
C          *****
C          THIS SUBROUTINE IS USED ALONG LINE BC USING POINTS IN X DIRECTION.
C          REF. -ART.2..4.2
C          *****
0002      COMMON DEL(35),DEK1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FEX(35,35),FY(35,35),FFY(35,35)
C          *****
0003      DD=X1-X
0004      DU=DEL(I+1)-DD
0005      DELTA=0.00250
0006      DX(I+1,J)=DD/DELTA
0007      SLOX=100.*X1
0008      BX(I+1,J)=50.*X1**2.+50.*Y**2.
0009      D1 = 1+DX(I+1,J)
0010      D2 = D1**2
0011      D3 = 1-DX(I+1,J)
0012      BOFX(I,J) = (1.+2.*DX(I+1,J))*BX(I+1,J)/D2-DD*SLOX/D1
0013      BOFY(I+1,J) = 4.*DX(I+1,J)*BX(I+1,J)/D2+2.*DU*SLOX/D1
0014      FEX(I-1,J)=DX(I+1,J)**2/D2
0015      FEX(I-1,J)=D3**2/D2
0016      RETURN
0017      END

```

```

0001      SUBROUTINE YFOUR(I,J,X,Y,Y1)
C          *****
C          THIS SUBROUTINE IS USED ALONG LINE BC USING POINTS IN Y DIRECTION.
C          REF. -ART.2.4.2
C          *****
C          COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C

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```

0003      DD=Y1-Y
0004      DU=DEL1(J+1)-DD
0005      DELTA=0.06250
0006      DY(I,J+1)=DU/DELTA
0007      SLODY=100.*Y1
0008      BY(I,J+1)=50.*X**2.+50.*Y1**2.
0009      D1=1.+DY(I,J+1)
0010      D2 = D1**2
0011      D3 = 1. - DY(I,J+1)
0012      BFY(I,J) = (1.+2.*DY(I,J+1))*BY(I,J+1)/D2-DD*SLODY/D1
0013      BOFY(I,J+1) = 4.*DY(I,J+1)*BY(I,J+1)/D2+2.*DU*SLODY/D1
0014      FY(I,J-1)=DY(I,J+1)**2/D2
0015      FFY(I,J-1)=D3**2/D2
0016      RETURN
0017      END

```

```

0001      SUBROUTINE BOFOUR(I,J,X,X1,Y,Y1)
C
C      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),RY(35,35),
1BEX(35,35),BFY(35,35),BOEX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C
C      CALL
10 CALL      XFOUR(I,J,X,X1,Y)
20 RETURN    YFOUR(I,J,X,X1,Y1)
0003      END
0004
0005
0006

```


1000

SUBROUTINE STAND

STANDARD OPERATOR WITH SQUAKE MESH. REF. ART.--2..4.1

STANDARD OPERATOR WITH SQUARE MESH. REF. ART.--2..4.1

0002

CJMMON DEL(35),DELI(35),OX(35,35),OY(35,35),BX(35,35),BY(35,35),
IBFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

1BFX(35,35),BFY(35,35),BOFX(35,35),C(13),IA(35,35),

2DEK(35),CEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

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C
C(1)=2C:
C(2)=-8:
C(3)=-8:
C(4)=-8:
C(5)=-8:
C(6)=2:
C(7)=2:
C(8)=2:
C(9)=2:
C(10)=1:
C(11)=1:
C(12)=1:
C(13)=1:
      RETURN
      END

```

 $C(1) = 20.$

•

0.8 = 0.8

$$C(5) = 18$$

2000

22
11
—
—
—
—

$$C(9) = 2.$$

• 111011

$$C(11)=1, C(12)=1,$$
$$C(13)=1.$$
RETURN
END

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FORTRAN IV G LEVEL 13

ONE

DATE = 70278

18/30/45

PAGE 0001

0001

SUBROUTINE ONE(AA,BB,CC,DD)

C
C
C
C
C

OPERATOR FOR THE POINT ON THE INSERT, ONE MESH LENGTH
ABOVE THE INTERFACE. REF ART.--2..4.4

0002

COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFX(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

C
C

0003 C(1)=AA
0004 C(2)=-8.
0005 C(3)=-8.
0006 C(4)=-8.
0007 C(5)=BB
0008 C(6)=2.
0009 C(7)=2.
0010 C(8)=CC
0011 C(9)=CC
0012 C(10)=1.
0013 C(11)=1.
0014 C(12)=1.
0015 C(13)=DD
0016 RETURN
0017 END

FORTRAN IV G LEVEL

SUBROUTINE TWO(AA,BB,CC,DD)

 OPERATOR FOR THE POINT ON THE INSERT, ONE MESH LENGTH
 BELOW THE INTERFACE. REF ART. 2.4.4

COMMON DEL(35), DEL1(35), DX(35,35), DY(35,35), BX(35,35), BY(35,35),
 LBFX(35,35), BFX(35,35), BOFY(35,35), C(13), LA(35,35),
 ZDEK(35), DEK1(35), P(451), FX(35,35), FFY(35,35), FY(35,35), FFY(35,35)

C(1)=AA
 C(2)=-8.
 C(3)=BB
 C(4)=-8.
 C(5)=-8.
 C(6)=CC
 C(7)=CC
 C(8)=2.
 C(9)=2.
 C(10)=1.
 C(11)=DD
 C(12)=1.
 C(13)=1.
 RETURN
 END

0001

SUBROUTINE THREE(AA,BB,CC,DD)

C
C
C
C
C
C

OPERATOR FOR THE POINT ON THE INSERT, ONE MESH LENGTH
RIGHT OF INTERFACE. REF. ART. --2.4.4

0002

COMMON DEL(35),DEK1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FFY(35,35)

C
C

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0007
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0009
0010
0011
0012
0013
0014
0015
0016
0017

C(1)=AA
C(2)=-8.
C(3)=-8.
C(4)=BB
C(5)=-8.
C(6)=2.
C(7)=CC
C(8)=CC
C(9)=2.
C(10)=1.
C(11)=1.
C(12)=DD
C(13)=1.
RETURN
END

SUBROUTINE FOUR(AA,BB,CC,DD)

```
*****  
OPERATOR FOR THE POINT ON THE INSERT , ONE MESH LENGTH  
LEFT OF INTERFACE. REF. ART.--2..4.4
```

```
COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEKI(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
```

```

C(1)=AA      .
C(2)=BB      .
C(3)=-8      .
C(4)=-8      .
C(5)=-8      .
C(6)=CC      .
C(7)=CC      .
C(8)=2       .
C(9)=CC      .
C(10)=DD     .
C(11)=1      .
C(12)=1      .
C(13)=1      .
RETURN
END

```

FORTRAN IV G LEVEL 18

PAGE 0001

18/30/45

DATE = 70278

ICONE

0001

SUBROUTINE ICONE(AAA,BB,CCC,CC,DD)

C

OPERATOR FOR THE POINT ON INSERT ONE MESH LENGTH AWAY FROM THE
LOWER CORNER OF INTERFACE. REF. ART.--2.4.5

C

C

C

C

0002

COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

C

0003

C(1)=AAA

0004

C(2)=-8.

0005

C(3)=-8.

0006

C(4)=BB

0007

C(5)=BB

0008

C(6)=2.

0009

C(7)=CC

0010

C(8)=CCC

0011

C(9)=CC

0012

C(10)=1.

0013

C(11)=1.

0014

C(12)=DD

0015

C(13)=DD

0016

RETURN

0017

END


```

FORTRAN IV G LEVEL 18          ICFOUR          DATE = 70278          18/30/45          PAGE 0001
0001      SUBROUTINE ICFOUR(AAA,BB,CCC,CC,DD)
C      *****
C      OPERATOR FOR THE POINT ON INSERT ONE MESH LENGTH AWAY FROM THE
C      UPPER RIGHT CORNER OF INTERFACE. REF. ART.--2.4.5
C      *****
0002      COMMON DEL(35),DEK1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BDFX(35,35),BDFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C
0003      C(1)=AAA
0004      C(2)=BB
0005      C(3)=BB
0006      C(4)=-8.
0007      C(5)=-8.
0008      C(6)=CCC
0009      C(7)=CC
0010      C(8)=2.
0011      C(9)=CC
0012      C(10)=DD
0013      C(11)=DD
0014      C(12)=1.
0015      C(13)=1.
0016      RETURN
0017      END

```

SUBROUTINE CONE(AA1,BB1,CC1,DD1)

 OPERATOR FOR THE POINT ON THE PARENT MATERIAL ONE MESH LENGTH
 BELOW THE INTERFACE. REF. ART.2.4.4

COMMON DEL(35), DELL(35), DX(35,35), DY(35,35), BX(35,35), BY(35,35),
LBFX(35,35), BFX(35,35), BOFY(35,35), C(13), LA(35,35),
ZDEK(35), DEK1(35), P(451), FX(35,35), FFX(35,35), FY(35,35), FFY(35,35)

```

C(1)=AA1
C(2)=-8
C(3)=BB1
C(4)=-9
C(5)=-8
C(6)=CC1
C(7)=CC1
C(8)=2
C(9)=2
C(10)=1
C(11)=DD1
C(12)=1
C(13)=1
C(13)=1
RETURN
END

```

[illegible]

0001 SUBROUTINE OTWO(AA1,BB1,CCL,DDL)

 OPERATOR FOR THE POINT ON THE PARENT MATERIAL ONE MESH LENGTH
 ABOVE THE INTERFACE. REF. ART.--2..4.4

0002 COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
 LBEX(35,35),BFX(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
 ZDEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

C(1)=AA1
 C(2)=-8.
 C(3)=-8.
 C(4)=-8.
 C(5)=BB1
 C(6)=2.
 C(7)=2.
 C(8)=CCL
 C(9)=CCL
 C(10)=1.
 C(11)=1.
 C(12)=1.
 C(13)=DDL
 RETURN
 END

0003
 0004
 0005
 0006
 0007
 0008
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FCRTRAN IV C LEVEL 18 QTHREE DATE = 70278 18/30/45 PAGE 0001

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C C C C C C C C C C
SUBROUTINE OTHREE(AA1,BB1,CC1,DD1)
** ** ** ** **
OPERATOR FOR THE POINT ON THE PARENT MATERIAL ONE MESH LENGTH
LEFT OF THE INTERFACE. REF. ART.-- 2..4..4
** ** ** **
COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFX(35,35),BOFX(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

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00003      AAI
00004      BBI
00005      -8.
00006      -8.
00007      -8.
00008      CCI
00009      2.
00010      2.
00011      DD1
00012      1.
00013      1.
00014      1.
00015      1.
00016      RETURN
00017      END
00018      CCI

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FURTRAN IV G LEVEL 18 QFOUR DATE = 70278 18/30/45 PAGE 0001

```
0001      SUBROUTINE QFOUR(AA1,BB1,CC1,DD1)
C          *****
C          OPERATOR FOR THE POINT ON THE PARENT MATERIAL ONE MESH LENGTH
C          RIGHT OF INTERFACE. REF.ART.--2..4.4
C          *****
0002      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEKI(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C          C(1)=AA1
C          C(2)=-8.
C          C(3)=-8.
C          C(4)=BB1
C          C(5)=-8.
C          C(6)=2.
C          C(7)=CC1
C          C(8)=CC1
C          C(9)=2.
C          C(10)=1.
C          C(11)=1.
C          C(12)=DD1
C          C(13)=1.
          RETURN
          END
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FCRTRAN IV G LEVEL 18
DATE = 70278
18/30/45
PAGE 0001

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C SUBROUTINE ONONE(A,B,C,D,E,F,G,H,I)
C *****
C OPERATOR FOR THE POINT JN THE
C LOWER INTERFACE. REF. ART. 2..4.3
C *****
COMMON DEL(35),DEK1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
IBEX(35,35),BFX(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
ZDEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
```

COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

```

C(1)=A
C(2)=B
C(3)=C1
C(4)=B
C(5)=D
C(6)=E
C(7)=E
C(8)=F
C(9)=F
C(10)=G
C(11)=H
C(12)=G
C(13)=I
RETURN
END

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[illegible]

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FORTRAN IV G LEVEL 16          ONTWO      DATE = 70278      18/30/45      PAGE 0001

0001      SUBROUTINE ONTWO(A,B,C1,D,F,G,H,TI)
C      *****
C      OPERATOR FOR THE POINT ON THE
C      UPPER INTERFACE . REF. ART.--2..4.3
C      *****
0002      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),RX(35,35),RY(35,35),
1BEX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C      *****
C      C(1)=A
C      C(2)=B
C      C(3)=D
C      C(4)=B
C      C(5)=C1
C      C(6)=F
C      C(7)=F
C      C(8)=E
C      C(9)=E
C      C(10)=G
C      C(11)=TI
C      C(12)=G
C      C(13)=H
RETURN
END

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18/30/45

DATE = 70278

ONFOUR

FORTRAN IV G LEVEL 18

SUBROUTINE ONFOUR(A,B,C1,D,E,F,G,H,I1)

OPERATOR FOR THE POINT ON THE

RIGHT INTERFACE . REF. ART. --2.4.3

COMMON DEL(35),DEK1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)

C(1)=A
C(2)=D
C(3)=B
C(4)=C1
C(5)=B
C(6)=F
C(7)=E
C(8)=E
C(9)=F
C(10)=I
C(11)=G
C(12)=H
C(13)=G
RETURN
END

CCCCC
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FORTRAN IV G LEVEL 18          COONE          DATE = 70278          18/30/45          PAGE 0001

0001      SUBROUTINE COONE(X,Y,S,Z,T)
C          *****
C          OPERATOR FOR THE POINT ON THE
C          LOWER LEFT CORNER OF THE INTERFACE .REF ART.--2..4.6
C          *****
0002      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
        LBFX(35,35),BFX(35,35),BOFY(35,35),C(13),LA(35,35),
        ZDEK(35),DEKI(35),P(451),FX(35,35),FFX(35,35),FXY(35,35),FFY(35,35)
C          *****
C          C(1)=X
C          C(2)=-8.
C          C(3)=Y
C          C(4)=-8.
C          C(5)=-8.
C          C(6)=S
C          C(7)=Z
C          C(8)=Z.
C          C(9)=Z.
C          C(10)=I.
C          C(11)=T.
C          C(12)=1.
C          C(13)=1.
        RETURN
0003      END

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0001      SUBROUTINE COTHR(X,Y,S,Z,T)
C          *****
C          OPERATOR FOR THE POINT ON THE
C          LOWER RIGHT CORNER OF THE INTERFACE. REF. ART --2..4.6
C          *****
C          COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEKI(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C          *****

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C(1)=X
C(2)=-8.
C(3)=-8.
C(4)=Y
C(5)=-8.
C(6)=2.
C(7)=S
C(8)=Z
C(9)=2.
C(10)=1.
C(11)=1.
C(12)=T
C(13)=1.
RETURN
END

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FORTRAN IV G LEVEL 18      COFOUR      DATE = 70278      18/30/45      PAGE 0001

0001      SUBROUTINE COFOUR(X,Y,S,Z,1)
C          *****
C          OPERATOR FOR THE POINT ON THE
C          UPPER RIGHT CORNER OF THE INTERFACE . REF. ART .--2..4.6
C          *****
C          COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BEX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),C(13),LA(35,35),
2DEK(35),DEKL(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
C          *****
C          C(1)=X
C          C(2)=-8.
C          C(3)=-8.
C          C(4)=-8.
C          C(5)=Y
C          C(6)=2.
C          C(7)=2.
C          C(8)=S
C          C(9)=Z
C          C(10)=1.
C          C(11)=1.
C          C(12)=1.
C          C(13)=1.
          RETURN
          END

```

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0001 SUBROUTINE GNERAL(I,J,B,N)
0002
0003 THIS SUBROUTINE USES THE GENERAL OPERATOR . RFF. ART. --2..4.1
0004
0005 COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
0006 1BFX(35,35),BFX(35,35),BOFY(35,35),C(13),LA(35,35),
0007 2DEK(35),DEK1(35),P(451),FX(35,35),FFX(35,35),FY(35,35),FFY(35,35)
0008
0009 H3=DEK(I+1)
0010 V3=DEK1(J+1)
0011 H2=DEK(I)
0012 V2=DEK1(J)
0013 H4=DEK(I+2)
0014 V4=DEK1(J+2)
0015 H1=DEK(I-1)
0016 V1=DEK1(J-1)
0017 F1=(H2+H3)**2.
0018 F2=(H3+H4)**2.
0019 F3=(H1+H2)**2.
0020 F4=(V2+V3)**2.
0021 F5=(V3+V4)**2.
0022 F6=(V1+V2)**2.
0023 F7=(H2+H3)**4.
0024 F8=(V2+V3)**4.
0025 PHO=16./(F1*F2)+64./F7+16./(F1*F3)+16./(F4*F5)+64./F8+16./(F4*F6)+
0026 128./(F4*F1)
0027 PH1=-32./(F1*F2)-32./F7-64./(F4*F1)
0028 PH2=-32./(F4*F5)-32./F8-64./(F4*F1)
0029 PH3=-32./(F7)-32./(F1*F3)-64./(F4*F1)
0030 PH4=-32./F8-32./(F4*F6)-64./(F4*F1)
0031 PH5=32./(F4*F1)
0032 PH9=16./(F1*F2)
0033 PH10=16./(F4*F5)
0034 PH11=16./(F1*F3)
0035 PH12=16./(F4*F6)
0036 P(N)=PHO
0037 IF (LA(I+1,J)) 140, 130, 140
0038 L1=LA(I+1,J)
0039 P(L1)=P(L1)+PHI
0040 GO TO 150
0041
0042 140
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0034 B = B - BFX(I+1, J) * PH1
0035 L1 = LA(I, J)
0036 P(L1) = P(L1) + FX(I, J) * PH1
0037 IF ( LA(I, J+1) ) 170, 160, 170
0038 B = B - BFX(I, J+1) * PH2
0039 L1 = LA(I, J)
0040 P(L1) = P(L1) + FY(I, J) * PH2
0041 GO TO 180
0042 L1 = LA(I, J+1)
0043 P(L1) = P(L1) + PH2
0044 IF ( LA(I-1, J) ) 200, 190, 200
0045 B = B - BFX(I-1, J) * PH3
0046 L1 = LA(I, J)
0047 P(L1) = P(L1) + FX(I, J) * PH3
0048 GO TO 210
0049 L1 = LA(I-1, J)
0050 P(L1) = P(L1) + PH3
0051 IF ( LA(I, J-1) ) 230, 220, 230
0052 B = B - BFX(I, J-1) * PH4
0053 L1 = LA(I, J)
0054 P(L1) = P(L1) + FY(I, J) * PH4
0055 GO TO 240
0056 L1 = LA(I, J-1)
0057 P(L1) = P(L1) + PH4
0058 IF ( LA(I+1, J+1) ) 250, 260, 250
0059 L1 = LA(I+1, J+1)
0060 P(L1) = P(L1) + PH5
0061 GO TO 290
0062 IF (I.EQ.33.AND.J.EQ.23) GO TO 275
0063 IF (I.EQ.33.AND.J.EQ.24) GO TO 285
0064 IF (I.EQ.23.AND.J.EQ.33) GO TO 295
0065 IF (I.EQ.24.AND.J.EQ.33) GO TO 305
0066 IF (BFX(I+1, J+1) ) 270, 280, 270
0067 B = B - BFX(I+1, J+1) * PH5
0068 L1 = LA(I, J+1)
0069 P(L1) = P(L1) + FX(I, J+1) * PH5
0070 GO TO 290
0071 B = B - BFX(I+1, J+1) * PH5
0072 L1 = LA(I-1, J+1)
0073 P(L1) = P(L1) + FFX(I-1, J+1) * PH5
0074 GO TO 290
0075 B = B - BFX(I+1, J+1) * PH5
0076 L1 = LA(I+1, J)

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0077 P(L1)=P(L1)+FY(I+1,J)*PH5
0078 GO TO 290
0079 305 B=B-BQFY(I+1,J+1)*PH5
0080 L1=LA(I+1,J-1)
0081 P(L1)=P(L1)+FFY(I+1,J-1)*PH5
0082 GO TO 290
0083 270 B=B-(BFX(I+1,J+1)+BFY(I+1,J+1))*Q.5*PH5
0084 L1=LA(I,J+1)
0085 P(L1)=P(L1)+0.5*FX(I,J+1)*PH5
0086 L1=LA(I+1,J)
0087 P(L1)=P(L1)+0.5*FY(I+1,J)*PH5
0088 GO TO 290
0089 280 B=B-(BQFX(I+1,J+1)+BQFY(I+1,J+1))*Q.5*PH5
0090 L1=LA(I-1,J+1)
0091 P(L1)=P(L1)+.5*FFX(I-1,J+1)*PH5
0092 L1=LA(I+1,J-1)
0093 P(L1)=P(L1)+.5*FFY(I+1,J-1)*PH5
0094 IF(LA(I-1,J+1))300,310,300
0095 L1=LA(I-1,J+1)
0096 P(L1)=P(L1)+PH5
0097 GO TO 340
0098 310 IF(I.EQ.23.AND.J.EQ.33)GO TO 304
0099 IF(I.EQ.24.AND.J.EQ.33)GO TO 315
0100 IF(I.EQ.3.AND.J.EQ.12)GO TO 325
0101 IF(I.EQ.3.AND.J.EQ.13)GO TO 335
0102 IF(BFX(I-1,J+1))320,330,320
0103 B=B-BQFY(I-1,J+1)*PH5
0104 L1=LA(I-1,J-1)
0105 P(L1)=P(L1)+FFY(I-1,J-1)*PH5
0106 GO TO 340
0107 315 B=B-BFY(I-1,J+1)*PH5
0108 L1=LA(I-1,J)
0109 P(L1)=P(L1)+FY(I-1,J)*PH5
0110 GO TO 340
0111 325 B=B-BFX(I-1,J+1)*PH5
0112 L1=LA(I,J+1)
0113 P(L1)=P(L1)+FX(I,J+1)*PH5
0114 GO TO 340
0115 335 B=B-BQFX(I-1,J+1)*PH5
0116 L1=LA(I+1,J+1)
0117 P(L1)=P(L1)+FFX(I+1,J+1)*PH5
0118 GO TO 340
0119 320 B=B-(BFX(I-1,J+1)+BFY(I-1,J+1))*Q.5*PH5

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01200 L1=LA(I-1,J)
01210 P(L1)=P(L1)+.5*FY(I-1,J)*PH5
01220 L1=LA(I,J+1)
01230 P(L1)=P(L1)+.5*FX(I,J+1)*PH5
01240 GO TO 340
01250 B=B-(BOFX(I-1,J+1)+BOFY(I-1,J+1))*0.5*PH5
01260 L1=LA(I-1,J-1)
01270 P(L1)=P(L1)+.5*FFY(I-1,J-1)*PH5
01280 L1=LA(I+1,J+1)
01290 P(L1)=P(L1)+.5*FFX(I+1,J+1)*PH5
01300 IF(LA(I-1,J-1))350,360,350
01310 L1=LA(I-1,J-1)
01320 P(L1)=P(L1)+PH5
01330 GO TO 390
01340 IF(I.EQ.3.AND.J.EQ.12)GO TO 365
01350 IF(I.EQ.3.AND.J.EQ.13)GO TO 375
01360 IF(I.EQ.12.AND.J.EQ.3)GO TO 385
01370 IF(I.EQ.13.AND.J.EQ.3)GO TO 395
01380 IF(BOFX(I-1,J-1))370,380,370
01390 B=B-BOFX(I-1,J-1)*PH5
01400 L1=LA(I,J-1)
01410 P(L1)=P(L1)+FX(I,J-1)*PH5
01420 GO TO 390
01430 B=B-BOFX(I-1,J-1)*PH5
01440 L1=LA(I+1,J-1)
01450 P(L1)=P(L1)+FFX(I+1,J-1)*PH5
01460 GO TO 390
01470 B=B-BOFY(I-1,J-1)*PH5
01480 L1=LA(I-1,J+1)
01490 P(L1)=P(L1)+FFY(I-1,J+1)*PH5
01500 GO TO 390
01510 B=B-BOFY(I-1,J-1)*PH5
01520 L1=LA(I-1,J)
01530 P(L1)=P(L1)+FY(I-1,J)*PH5
01540 GO TO 390
01550 B=B-(HFX(I-1,J-1)+BFY(I-1,J-1))*0.5*PH5
01560 L1=LA(I,J-1)
01570 P(L1)=P(L1)+.5*FX(I,J-1)*PH5
01580 L1=LA(I-1,J)
01590 P(L1)=P(L1)+.5*FY(I-1,J)*PH5
01600 GO TO 390
01610 B=B-(BOFX(I-1,J-1)+BOFY(I-1,J-1))*0.5*PH5
01620 L1=LA(I+1,J-1)

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0163 P(L1)=P(L1)+.5*FFX(I+1,J-1)*PH5
0164 L1=LA(I-1,J+1)
0165 P(L1)=P(L1)+.5*FFY(I-1,J+1)*PH5
0166 IF(LA(I+1,J-1))400,410,400
0167 L1=LA(I+1,J-1)
0168 P(L1)=P(L1)+PH5
0169 GO TO 440
0170 IF(I.EQ.12.AND.J.EQ.3)GO TO 415
0171 IF(I.EQ.13.AND.J.EQ.3)GO TO 425
0172 IF(I.EQ.33.AND.J.EQ.23)GO TO 435
0173 IF(I.EQ.33.AND.J.EQ.24)GO TO 445
0174 IF(BFX(I+1,J-1))420,430,420
0175 B=B-BFY(I+1,J)*PH5
0176 L1=LA(I+1,J)
0177 P(L1)=P(L1)+FY(I+1,J)*PH5
0178 GO TO 440
0179 B=B-BFY(I+1,J-1)*PH5
0180 L1=LA(I+1,J+1)
0181 P(L1)=P(L1)+FFY(I+1,J+1)*PH5
0182 GO TO 440
0183 B=B-BFX(I+1,J-1)*PH5
0184 L1=LA(I-1,J-1)
0185 P(L1)=P(L1)+FFX(I-1,J-1)*PH5
0186 GO TO 440
0187 B=B-BFX(I+1,J-1)*PH5
0188 L1=LA(I,J-1)
0189 P(L1)=P(L1)+FX(I,J-1)*PH5
0190 GO TO 440
0191 B=B-(BFX(I+1,J-1)+BFY(I+1,J-1))*0.5*PH5
0192 L1=LA(I+1,J)
0193 P(L1)=P(L1)+.5*FY(I+1,J)*PH5
0194 L1=LA(I,J-1)
0195 P(L1)=P(L1)+.5*FX(I,J-1)*PH5
0196 GO TO 440
0197 B=B-(BFX(I+1,J-1)+BFY(I+1,J-1))*0.5*PH5
0198 L1=LA(I+1,J+1)
0199 P(L1)=P(L1)+.5*FFY(I+1,J+1)*PH5
0200 L1=LA(I-1,J-1)
0201 P(L1)=P(L1)+.5*FFX(I-1,J-1)*PH5
0202 IF(LA(I+2,J))450,460,450
0203 L1=LA(I+2,J)
0204 P(L1)=P(L1)+PH9
0205 GO TO 490

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0206 IF ( BFX (I+2,J)) 470,490,470
0207 B = B -BFX ( I+2 , J) * PH9
0208 LI = LA(I+1,J)
0209 P(LI)=P(LI)+FX(I+1,J)*PH9
0210 GO TO 490
0211 B = B - BCFX ( I+2,J) *PH9
0212 LI = LA(I,J)
0213 P(LI)=P(LI)+FFX(I,J)*PH9
0214 IF(LA(I,J+2))500,510,500
0215 LI = LA(I,J+2)
0216 P(LI) = P(LI) + PH10
0217 GO TO 540
0218 IF(BFY(I,J+2))520,530,520
0219 B = B -BFY(I,J+2)*PH10
0220 LI = LA(I,J+1)
0221 P(LI)=P(LI)+FY(I,J+1)*PH10
0222 GO TO 540
0223 B = B -BOFY(I,J+2)*PH10
0224 LI = LA(I,J)
0225 P(LI)=P(LI)+FFY(I,J)*PH10
0226 IF(LA(I-2,J))550,560,550
0227 LI = LA(I-2,J)
0228 P(LI) = P(LI)+PH11
0229 GO TO 590
0230 IF(BFX(I-2,J))570,580,570
0231 B = B -BFX(I-2,J)*PH11
0232 LI = LA(I-1,J)
0233 P(LI)=P(LI)+FX(I-1,J)*PH11
0234 GO TO 590
0235 B = B -BOFX(I-2,J)*PH11
0236 LI = LA(I,J)
0237 P(LI)=P(LI)+FFX(I,J)*PH11
0238 IF(LA(I,J-2))600,610,600
0239 LI = LA(I,J-2)
0240 P(LI) = P(LI)+PH12
0241 GO TO 640
0242 IF(BFY(I,J-2))620,630,620
0243 B = B -BFY(I,J-2)*PH12
0244 LI = LA(I,J-1)
0245 P(LI)=P(LI)+FY(I,J-1)*PH12
0246 GO TO 640
0247 B = B -BOFY(I,J-2)*PH12
0248 LI = LA(I,J)

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PAGE 0007

18/30/45

DATE = 70278

GNERAL

FORTRAN IV G LEVEL 18

0249 P(L1)=P(L1)+FFY(I,J)*PH12
0250 640 RETURN
0251 END

THIS IS THE SECOND STEP OF THE THREE STEPS PROGRAM FOR THE
SOLUTION OF THE PROBLEM

THIS PROGRAM IS USED FOR THE SOLUTION OF SET OF SIMULTANEOUS
EQUATIONS GENERATED BY STEP 1.

THIS IS BASED ON THE GAUSS - SIEDEL ITERATIVE TECHNIQUE.
THIS PROGRAM IS MOST USEFUL WHERE THE LARGE MATRICES ARE
ENCOUNTERED.

THIS METHOD PERFORMS OPERATIONS ONLY ON NON - ZERO ELEMENTS.

THE VARIABLES USED ARE DEFINED AS FOLLOWS .

NPS NO. OF ROWS
MAXIT TOLERANCE LIMIT
J ITERATION LIMIT
I COLUMN COUNTER
P(J) ROW COUNTER
KK ELEMENTS OF ANY ROW
K COUNTER FOR TOTAL NO. OF NON-ZERO ELEMENTS EXCEPT
C(I) DIAGONAL ELEMENTS IN MATRIX
D(KK) COUNTER FOR TOTAL NO. OF NON-ZERO ELEMENTS EXCEPT
M(KK) DIAGONAL ELEMENTS IN EACH ROW
NN(I) ALL THE NON-ZERO ELEMENTS OF THE MATRIX
PP(I) JTH POSITION FOR THE NON-ZERO ELEMENTS OF MATRIX
RIGHT HAND SIDE VECTOR

IMPLICIT REAL*8(A-H,O,Q-Z)

```

0002 DIMENSION P(512),PP(512),PPP(512),D(7000),C(512),X(512),B(512),
      IM(7000),NN(512)
      READ N,EPSILON AND THE MAXIMUM NUMBER OF ITERATIONS.
      READ(1,20)N,EPS,MAXIT
      WRITE(3,20)N,EPS,MAXIT
      FORMAT(14,E10.3,14)
20

```

```

*****
THIS PART OF THE PROGRAM STORES THE MATRIX IN ECONOMICAL FORM
BY SICKING ONLY NON-ZERO ELEMENTS
*****

```

```

KK=0
DO 130 I=1,N
  K=0
  READ(7)(P(J),J=1,N)
  DO 130 J=1,N
    IF (I-J) 60,50,60
    IF (P(J)) 120,130,120
    KK=KK+1
  D(KK)=P(J)
  K=K+1
  NN(I)=K
  M(KK)=J
  GO TO 130
C(1)=P(J)
CONTINUE
WRITE(3,160)
FORMAT(1,5X,C(1),7X,NN(1))
WRITE(3,170)((I,C(1),NN(I)),I=1,N)
FORMAT(5X,4(15,5X,F10.5,18))
WRITE(3,162)((D(KL),M(KL)),KL=1,KK)
FORMAT(5X,8(F8.3,4X,13))
WRITE(3,165)KK
FORMAT(1,110)
READ(7)(PP(I),I=1,N)
INITIAL VALUES OF X(I).
READ(1,132)(PPP(I),I=1,N)
WRITE(3,157)((I,PPP(I)),I=1,N)
FORMAT(4E20.10)
132 DO 751=1,N

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0034 X{1}=PPP{1}
0035 B{1}=PP{1}

C C C C C C C C
** THE GAUSS SEIDEL ITERATION STARTS FROM HERE **
** ** ** ** **
0036 ITER=1
0037 BIG=0.0
0038 LL=0
0039 K2=1
0040 DO 105 I=1,N
0041 SUM=0.0
0042 LL=LL+NN(I)
0043 DO 100 KK=K2,LL
0044 L=1(KK)
0045 SUM=SUM+D(KK)*X(L)
0046 TEMP=(B(I)-SUM)/C(I)
0047 K2=LL+1

C C C
AT END OF SWEEP OF THIS STATEMENT HAS PUT LARGEST RESIDUAL IN BIG.

0048 RESID=DABS(TEMP-X(I))
0049 IF(RESID.GT.BIG) BIG=RESID
0050 X(I)=TEMP

C C C C C C C C
ONE SWEEP HAS NOW BEEN COMPLETED- PRINT VARIABLES.
IF LARGEST RESIDUAL IS LESS THAN EPSILON THE PROCESS HAS CONVERGED

0051 IF(BIG.LT.EPS)GO TO 190
0052 IF ITERATION COUNTER EXCEEDS MAXIMUM ALLOWABLE, STOP ITERATING.
0053 IF(ITER.GT.MAXIT)GO TO 190
0054 ITER=ITER+1
0055 GO TO 99
0056 WRITE(3,57)((I,X(I)),I=1,N)
0057 WRITE(3,111)ITER
0058 FORMAT(5X,15)
0059 FORMAT(/,3X,6(15,F12.6,2X))
WRITE(2,15)( X(I),I=1,N)
```

DRTRAN IV G LEVEL 18

0060 15 FORMAT(4E20.10)
0061 RETURN
0062 END

MAIN

DATE = 70278

18/33/39

PAGE 0004

***** STEP OF THREE STEPS PROGRAM FOR THE SOLUTION OF
THIS IS THIRD. INCLINED INSERT)
CASE 2 (45 DEG. INCLINED INSERT)

```

** THIS PROGRAM CALCULATES THE VA LUES OF STRESS FUNCTION
** OBTAINED FROM STEP-2.

```

THE VARIABLES ARE DEFINED AS FOLLOWS--

I	NUMBER COUNTER IN X DIRECTION.
J	NUMBER COUNTER IN Y DIRECTION.
MM	NUMBER OF STEPS ALONG X DIRECTION.
NN	NUMBER OF STEPS ALONG Y DIRECTION.
DEL(I)	STEP SIZE ALONG X DIRECTION.
DEL(J)	STEP SIZE ALONG Y DIRECTION.
DEK(I)	RATIO OF ACTUAL TO STANDARD STEP SIZE ALONG X DIRECTION.
DEK(J)	RATIO OF ACTUAL TO STANDARD STEP SIZE ALONG Y DIRECTION.
E1,E2	THE MODULII OF ELASTICITY OF INSERT AND PARENT MATERIAL RESPECTIVELY.
B1,B2	THE POISSON RATIO OF INSERT AND PARENT MATERIAL RESPECTIVELY.
B1,B2	THE DISTANCE OF THE NODAL POINT FROM THE CO-ORDINATE AXIS.
X,Y	THE NUMBER OF NODAL POINTS.
LA(I,J)	THE DISTANCES IN X AND Y DIRECTIONS RESPECTIVELY FROM THE BOUNDARY OF THE PLATE OF THE POINTS LESS THAN ONE LENGTH AWAY.
DX(I,J)	I-TH POSITION OF THE NODAL POINTS.
DY(I,J)	J-TH POSITION OF THE NODAL POINTS.
M1	ROW OF THE CO-EFFICIENT MATRIX.
N1	THE RIGHT HAND VECTOR.
P(KK)	THE PHI VALUE OF THE FICTITIOUS POINT ON THE
B1(I,J)	

INSERT IN TERM OF REAL VALUES OF THE FICTIOUS POINT ON THE PARENT
BO(I,J) THE PHI VALUE OF THE FICTIOUS POINT ON THE PARENT
MATERIAL IN TERM OF REAL POINTS.
PP(I,J) VALUE OF STRESS FUNCTION AT THE NODAL POINTS
PX, PY DIRECT STRESSES ON THE INTERFACE IN THE PARENT MATERIAL
PPX, PPY DIRECT STRESSES
TX, TY SHEAR STRESS
P1, P2 MAJOR AND MINOR PRINCIPAL STRESSES

THE INSERT SIZE IS DEFINED AS FOLLOWS :

LENGTH LARGEST MI = 14 MO = 22
MEDIUM MI = 15 MO = 21
SMALLEST MI = 16 MO = 20
WIDTH FOR ALL CASES NI = 16 NO = 20

COMMON DEL(35), DEL1(35), DX(35,35), DY(35,35), BX(35,35), BY(35,35),
1BFX(35,35), BFX(35,35), BOFX(35,35), BOFY(35,35), DEK(35), DEK1(35),
2LA(35,35), KA(35,35), P(451), FX(35,35), FFY(35,35), FY(35,35),
3FFY(35,35), PP(35,35)

10 READ(1,10)MM,NN,E1,E2,B1,B2
WRITE(3,10)MM,NN,E1,E2,B1,B2
READ(1,20) (DEL(I), I=1,MM), (DEL1(J), J=1,NN)
READ(1,20) (DEK(I), I=1,MM), (DEK1(J), J=1,NN)
FORMAT(2110,4F15.6)
20 WRITE(3,20) ((DEL(I), I=1,MM), (DEL1(J), J=1,NN))
WRITE(3,20) ((DEK(I), I=1,MM), (DEK1(J), J=1,NN))
FORMAT(8F10.6)

INITIALIZE THE VARIABLES.

Y=0.0
X1=0.0

```

Y1=0.0
N=0
DO 65 I=1,MM
DO 65 J=1,NN
LA(I,J)=C
KA(I,J)=0
DX(I,J)=0.0
DY(I,J)=0.0
BX(I,J)=0.0
BY(I,J)=C.0
BFX(I,J)=0.0
BFY(I,J)=0.0
BOFX(I,J)=0.0
BOFY(I,J)=0.0
PX(I,J)=0.0
FEX(I,J)=0.0
FY(I,J)=0.0
FFY(I,J)=0.0

```

65

CONTINUE

```

*****
DRAW THE MESH.
NUMBER THE NODAL POINTS.
CALCULATE THE FICTITIOUS POINTS FALLING LESS THAN ONE LENGTH
AWAY FROM THE BOUNDARY, IN TERMS OF REAL VALUES USING SUBROUTINES
XONE , YONE , BOONE , XTWO , YTWO , BOTWO , XTHREE , YTHREE ,
BCTHREE , XFOUR , YFOUR , BCFOUR .
*****
DO 60 J=1,35
Y=Y+DELI(J)
X=X+0.0
DO 60 I=1,34
X=X+DEL(I)
EE=1./SQRT(2.)
IF(X.GT.EE)GO TO 70
IF(Y.GT.EE)GO TO 130
Y1=-X+EE
X1=-Y+EE
IF(Y1.GT.Y)GO TO 60
IF((Y-Y1).GT.DELI(J))GOTO40
IF((X-X1).GT.DEL(I))GO TO 90
CALL BOJNE(I,J,X,X1,Y,Y1)
*****

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3046 KA(I,J)=1
3047 GO TO 60
3048 90 CALL YONE(I,J,X,Y,Y1)
3049 KA(I,J)=1
3050 GO TO 60
3051 40 IF((X-X1).LT.DEL(I))GO TO 100
3052 130 Y1=X+EE
3053 X1=Y-EE
3054 IF(Y1.LT.Y)GO TO 60
3055 IF((Y1-Y).GT.DEL(I+1))GO TO 50
3056 IF((X-X1).GT.DEL(I))GO TO 55
3057 CALL BCTWO(I,J,X,X1,Y,Y1)
3058 KA(I,J)=1
3059 GO TO 60
3060 55 CALL YTWO(I,J,X,Y,Y1)
3061 KA(I,J)=1
3062 GO TO 60
3063 100 CALL XONE(I,J,X,X1,Y)
3064 KA(I,J)=1
3065 GO TO 60
3066 50 IF((X-X1).GT.DEL(I))GO TO 80
3067 CALL XTWO(I,J,X,X1,Y)
3068 KA(I,J)=1
3069 GO TO 60
3070 70 Y1=X-EE
3071 X1=Y+EE
3072 IF(Y1.GT.Y)GO TO 60
3073 IF((Y-Y1).GT.DEL(J))GO TO 110
3074 IF((X1-X).GT.DEL(I+1))GO TO 120
3075 CALL BOTHRE(I,J,X,X1,Y,Y1)
3076 KA(I,J)=1
3077 GO TO 60
3078 120 CALL YTHRE(I,J,X,Y,Y1)
3079 KA(I,J)=1
3080 GO TO 60
3081 110 IF((X1-X).GT.DEL(I+1))GO TO 140
3082 CALL XTHREE(I,J,X,X1,Y)
3083 KA(I,J)=1
3084 GO TO 60
3085 140 IF(X.LT.(2./SORT(2.)))GO TO 130
3086 EE1=5.*EE
3087 Y1=-X+EE1
3088 X1=-Y+EE1

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0089 IF(Y1-LI,Y)GO TO 60
0090 IF((Y1-Y).GT.DEL1(J+1))GO TO 150
0091 IF((X1-X).GT.DEL(I+1))GO TO 160
0092 CALL BOFOUR(I,J,X,X1,Y,Y1)
0093 KA(I,J)=1
0094 GO TO 60
0095 YFOUR(I,J,X,Y,Y1)
0096 CALL KA(I,J)=1
0097 GO TO 60
0098 IF((X1-X).GT.DEL(I+1))GO TO 80
0099 CALL XFOUR(I,J,X,X1,Y)
0100 KA(I,J)=1
0101 GO TO 60
0102 N=N+1
0103 LA(I,J)=N
0104 CONTINUE
0105 READ(1,35)(P(I),I=1,N)
0106 WRITE(3,25)(P(I),I=1,N)
0107 FORMAT(5X,10F12.7)
0108 FORMAT(4E20.10)
0109 WRITE(3,81)N
0110 WRITE(3,87)
0111 FORMAT(//)
0112 FORMAT(5X,11F11.5)
0113 FORMAT(5X,21I5)
0114 FORMAT(5X,I5)

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CCCCC

CALCULATION OF STRESSES STARTS FROM HERE.

```

0115 CALL PHI
0116 WRITE(3,88)((PP(I,J),I=1,MM),J=1,NN)
0117 CALL STRESS(MM,NN,E1,E2,B1,B2,N)
0118 RETURN
0119 END

```



```

0027 TIXXYY(I,J)=0.
0028 MI=14
0029 MO=22
0030 NI=16
0031 NO=20
0032 31 CONTINUE
0033 MM1=MM-1
0034 NN1=NN-1
0035 CALCULATION OF PXX STARTS FROM HERE.
0036 DO 10 J=2,NN1
0037 DO 10 I=2,MM1
0038 IF(J.EQ.2)GO TO121
0039 DZ=DELI(J)
0040 GO TO 141
0041 D2=.0625
0042 D1=DELI(J+1)
0043 IF(LA(I,J)) 20,10,20
0044 L1=LA(I,J)
0045 F1=P(L1)
0046 GO TO 70
0047 IF(LA(I,J+1)) 90,80,90
0048 L1=LA(I,J+1)
0049 F2=P(L1)
0050 GO TO 160
0051 F2=PP(I,J+1)
0052 IF(LA(I,J-1)) 180, 170,180
0053 L1=LA(I,J-1)
0054 F3=P(L1)
0055 GO TO 250
0056 F3=PP(I,J-1)
0057 IF(I.GT.MI).AND.(I.LT.NO)) GO TO 270
0058 PX(I,J) = (F2-2.*F1+F3)*4./(D1+D2)*.2
0059 GO TO 10
0060 L1=LA(I-1,J)
0061 F4=P(L1)
0062 L1=LA(I+1,J)
0063 F5=P(L1)
0064 IF(J.EQ.16) GO TO 281
0065 IF(J.EQ.20) GO TO 291
0066 GO TO 260
0067 B1(I,J-1) = F2*01+F1*02+(F4+F5)*03+F3*04
0068 B0(I,J+1) = F3*001+F1*002+(F4+F5)*003+F2*004
WRITE(3,7)01,02,03,04,001,002,003,004,81(I,J-1),80(I,J+1)

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0069 7      FORMAT(/,5X,5E20.8)
0070      DELTA=(D1+D2)*#2.
0071      PX(I,J) = (F2-2.*F1+BI(I,J-1))*4./DELTA
0072      PPX(I,J) = (B0(I,J+1)-2.*F1+F3)*4./DELTA
0073      GO TO 10
0074      291      81(I,J+1) = F3*Q1+F1*Q2+(F4+F5)*Q3+F2*Q4
0075      80(I,J-1) = F2*Q01+F1*Q02+(F4+F5)*Q03+F3*Q04
0076      DELTA=(D1+D2)*#2.
0077      PX(I,J) = (BI(I,J+1)-2.*F1+F3)*4./DELTA
0078      PPX(I,J) = (B0(I,J-1)-2.*F1+B0(I,J-1))*4./DELTA
0079      CONTINUE
0080      CALCULATION OF PYY STARTS FROM HERE.
0081      DO 280 J = 2,NN1
0082      DO 280 I = 2,MM1
0083      IF (I.EQ.2) GO TO 21
0084      D2=DEL(I)
0085      GO TO 32
0086      D2=.0625
0087      D1=DEL(I+1)
0088      IF (LA(I,J)) 300,280,300
0089      L1 = LA(I,J)
0090      F1 = P(L1)
0091      GO TO 340
0092      IF (LA(I+1,J)) 360,350,360
0093      L1 = LA(I+1,J)
0094      F2 = P(L1)
0095      GO TO 430
0096      F2=PP(I+1,J)
0097      IF (LA(I-1,J)) 450,440,450
0098      L1 = LA(I-1,J)
0099      F3 = P(L1)
0100      GO TO 520
0101      F3=PP(I-1,J)
0102      IF (J.GT.NI.AND.J.LT.NO) GO TO 540
0103      DELTA=(D1+D2)*#2.
0104      PY(I,J)=(F2-2.*F1+F3)*4./DELTA
0105      GO TO 280
0106      L1=LA(I,J-1)
0107      F4=P(L1)
0108      L1=LA(I,J+1)
0109      F5=P(L1)
0110      IF (I.EQ.16) GO TO 550
      IF (I.EQ.20) GO TO 560

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18/32/15

DATE = 70278

STRESS

18

FORTRAN IV G LEVEL

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0111 GO TO 530
0112 31(I-1,J)=F2*Q1+F1*Q2+(F4+F5)*Q3+F3*Q4
0113 B0(I+1,J)=F3*Q01+F1*Q02+(F4+F5)*Q03+F2*Q04
0114 DELTA=(D1+D2)**2.
0115 PYP(I,J)=(F2-2.*F1+B1(I-1,J))*4./DELTA
0116 PYP(I,J)=(B0(I+1,J)-2.*F1+F3)*4./DELTA
0117 GO TO 280
0118 31(I+1,J) = F3*Q1+ F1*Q2 + (F4 + F5)*Q3 + F2*Q4
0119 B0(I-1,J) = F2*Q01 + F1*Q02 + (F4 + F5)*Q03 + F3*Q04
0120 DELTA=(D1+D2)**2.
0121 PYP(I,J)=(B1(I+1,J)-2.*F1+F3)*4./DELTA
0122 PYP(I,J) = (F2 - 2.0*F1 + B0(I-1,J))*4./DELTA
0123 CONTINUE
0124 CALCULATION OF TXY STARTS FROM HERE.
0125 DO 580 J=2,NN1
0126 DO 580 I=2,MM1
0127 IF(I.EQ.2)GO TO221
0128 D2=DEL(I)
0129 GO TO 231
0130 D2=.0625
0131 IF(J.EQ.2)GO TO241
0132 D4=DEL1(J)
0133 GO TO 252
0134 D4=.0625
0135 D1=DEL(I+1)
0136 D3=DEL1(J+1)
0137 DELTA=(D1+D2)*(D3+D4)
0138 IF(LA(I,J))590,580,590
0139 IF(LA(I+1,J+1))600,610,600
0140 L1=LA(I+1,J+1)
0141 F1=P(L1)
0142 GO TO 700
0143 F1=PP(I+1,J+1)
0144 IF(LA(I-1,J+1))720,710,720
0145 L1=LA(I-1,J+1)
0146 F2=P(L1)
0147 GO TO 790
0148 F2=PP(I-1,J+1)
0149 IF(LA(I-1,J-1))810,800,810
0150 L1=LA(I-1,J-1)
0151 F3=P(L1)
0152 GO TO 88C
0153 F3=PP(I-1,J-1)

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153 IF(LA(I+1,J-1))900,890,900
154 L1=LA(I+1,J-1)
155 F4=P(L1)
156 GO TO 97C
157
158 IF(I.EQ.M1.AND.J.EQ.N1)GO TO 980
159 IF(I.EQ.MO.AND.J.EQ.N1)GO TO 990
160 IF(I.EQ.M1.AND.J.EQ.NO)GO TO 1000
161 IF(I.EQ.MO.AND.J.EQ.NO)GO TO 1100
162 TXY(I,J)=- (F1-F2+F3-F4)/DELTA
163 GO TO 580
164
165 TXY(I,J)=- (B0(I+1,J+1)-F2+F3-F4)/DELTA
166 GO TO 580
167
168 TXY(I,J)=- (F1-B0(I-1,J+1)+F3-F4)/DELTA
169 GO TO 580
170
171 TXY(I,J)=- (F1-F2+F3-B0(I+1,J-1))/DELTA
172 GO TO 580
173
174 TXY(I,J)=- (F1-F2+B0(I-1,J-1)-F4)/DELTA
175
176 DO 1124J=1,NN
177 DO 1124I=1,MM
178 PXX(I,J)=(PX(I,J)+PY(I,J))*0.5+TXY(I,J)
179 IF(PPX(I,J).EQ.0.)GO TO 581
180 PPX(I,J)=(PPX(I,J)+PXX(I,J))*0.5-TXY(I,J)
181 PPY(I,J)=(PX(I,J)+PY(I,J))*0.5- TXY(I,J)
182 IF(PPY(I,J).EQ.0.)GO TO 582
183 PPY(I,J)=(PPY(I,J)+PPY(I,J))*0.5
184 CONTINUE
185 WRITE(3,1123)
186 WRITE(3,1122)(( PX(I,J),I=1,MM),J=1,NN)
187 WRITE(3,1123)
188 WRITE(3,1122)(( PPX(I,J),I=1,MM),J=1,NN)
189 WRITE(3,1123)
190 WRITE(3,1122)(( PY(I,J),I=1,MM),J=1,NN)
191 WRITE(3,1123)
192 WRITE(3,1122)(( PPY(I,J),I=1,MM),J=1,NN)
193 WRITE(3,1123)
194 WRITE(3,1122)(( TXY(I,J),I=1,MM),J=1,NN)
195 WRITE(3,1123)
196 WRITE(3,1122)(( TTX(I,J),I=1,MM),J=1,NN)
197 DO 1300J=1,NN
198 DO 1300I=1,MM

```

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0196 PXX(I,J)=0.
0197 PYY(I,J)=0.
0198 PPXX(I,J)=0.
0199 PPRY(I,J)=0.
0200 PC=(PX(I,J)+PY(I,J))/2.
0201 PR=SQR(((PX(I,J)-PY(I,J))/2.)*2+TX(I,J)**2)
0202 PXX(I,J)=PC+PR
0203 PYY(I,J)=PC-PR
0204 CONTINUE
0205 DO 1400 J=16,20,4
0206 DO 1400 I=17,19
0207 PC=(PPX(I,J)+PPY(I,J))/2.
0208 PR=SQR(((PPX(I,J)-PPY(I,J))/2.)*2+TX(I,J)**2)
0209 PXX(I,J)=PC+PR
0210 PYY(I,J)=PC-PR
0211 CONTINUE
0212 DO 1500 I=16,20,4
0213 DO 1500 J=17,19
0214 PC=(PX(I,J)+PPY(I,J))/2.
0215 PR=SQR(((PX(I,J)-PPY(I,J))/2.)*2+TX(I,J)**2)
0216 PXX(I,J)=PC+PR
0217 PYY(I,J)=PC-PR
0218 CONTINUE
0219 WRITE(3,1123)
0220 K1=1
0221 K=10
0222 WRITE(3,16)
0223 FORMAT(1H1,20X,'STRESSES IN THE DIRECTION OF LOAD.')
0224 WRITE(3,15)K1,K
0225 DO 25 J=1,NN
0226 WRITE(3,35)(J,(PXX(I,J),I=K1,K))
0227 IF(K.EQ.MM)GO TO 45
0228 K1=K+1
0229 K=K+10
0230 IF(K.GT.MM)K=MM
0231 GO TO 55
0232 CONTINUE
0233 K1=1
0234 K=10
0235 WRITE(3,114)
0236 FORMAT(1H1,25X,'INTERFACE STRESSES IN THE DIRECTION OF LOAD')
0237 WRITE(3,15)K1,K
0238 DO 125 J=1,NN

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0239 WRITE(3,35)(J,(PPXX(I,J),I=K1,K))
0240 IF(K.EQ.MM)GO TO 135
0241 K1=K+1
0242 K=K+10
0243 IF(K.GT.MM)K=MM
0244 GO TO 155
0245 CONTINUE
0246 K1=1
0247 K=10
0248 WRITE(3,216)
0249 FORMAT(1H1,20X,'STRESSES IN THE DIRECTION PARPENDICULAR TO LOAD.')
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216

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0250 WRITE(3,15)K1,K
0251 DO 225 J=1,NN
0252 WRITE(3,35)(J,(PPY(I,J),I=K1,K))
0253 IF(K.EQ.MM)GO TO245
0254 K1=K+1
0255 K=K+10
0256 IF(K.GT.MM)K=MM
0257 GO TO 255
0258 CONTINUE
0259 K1=1
0260 K=10
0261 WRITE(3,316)
0262 FORMAT(1H1,20X,'INTERFACES STRESSES PARPENDICULAR TO THE LOAD.')
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316

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0263 WRITE(3,15)K1,K
0264 DO 325 J=1,NN
0265 WRITE(3,35)(J,(PPY(I,J),I=K1,K))
0266 IF(K.EQ.MM)GO TO345
0267 K1=K+1
0268 K=K+10
0269 IF(K.GT.MM)K=MM
0270 GO TO 355
0271 CONTINUE
0272 K1=1
0273 K=10
0274 WRITE(3,416)
0275 FORMAT(1H1,20X,'SHEAR STRESSES.')
```

416

```

0276 WRITE(3,15)K1,K
0277 DO 425 J=1,NN
0278 WRITE(3,35)(J,(TXYY(I,J),I=K1,K))
0279 IF(K.EQ.MM)GO TO445
0280 K1=K+1
0281 K=K+10
```

```

0282 IF(K.GT.MM)K=MM
0283 GO TO 455
0284 CONTINUE
0285 K1=1
0286 K=10
0287 WRITE(3,556)
0288 FORMAT(1H1,20X,'PRINCIPAL STRESSES P1 (LARGER)')
0289 WRITE(3,15)K1,K
0290 DO 557 J=1,NN
0291 WRITE(3,35)(J,(PXX(I,J),I=K1,K))
0292 IF(K.EQ.MM)GO TO 559
0293 K1=K+1
0294 K=K+10
0295 IF(K.GT.MM)K=MM
0296 GO TO 555
0297 CONTINUE
0298 K1=1
0299 K=10
0300 WRITE(3,716)
0301 FORMAT(1H1,20X,'PRINCIPAL STRESSES P1 (LARGER) AT INTERFACE ')
0302 WRITE(3,15)K1,K
0303 DO 725 J=1,NN
0304 WRITE(3,35)(J,(PPXX(I,J),I=K1,K))
0305 IF(K.EQ.MM)GO TO 745
0306 K1=K+1
0307 K=K+10
0308 IF(K.GT.MM)K=MM
0309 GO TO 755
0310 CONTINUE
0311 K1=1
0312 K=10
0313 WRITE(3,916)
0314 FORMAT(1H1,20X,'PRINCIPAL STRESSES P2 (SMALLER)')
0315 WRITE(3,15)K1,K
0316 DO 925 J=1,NN
0317 WRITE(3,35)(J,(PYY(I,J),I=K1,K))
0318 IF(K.EQ.MM)GO TO 945
0319 K1=K+1
0320 K=K+10
0321 IF(K.GT.MM)K=MM
0322 GO TO 955
0323 CONTINUE
0324 K1=1

```

ORTRAN IV G LEVEL 18

STRESS

DATE = 70278

18/32/15

PAGE 0009

```

0325 K=10
0326 WRITE(3,1016)
0327 FORMAT(1H1,20X,' PRINCIPAL STRESSES P2(SMALLER) AT INTERFACE')
0328 WRITE(3,15)K1,K
0329 DO1025 J=1,NN
0330 1025 WRITE(3,35)(J,PPYY(I,J),I=K1,K)
0331 IF(K.EQ.MM)GO TO 1045
0332 K1=K+1
0333 K=K+10
0334 IF(K.GT.MM)K=MM
0335 GO TO 1055
0336 CONTINUE
0337 1045 FORMAT(5X,2I10)
0338 35 FORMAT(/,5X,I10,10F10.0)
0339 1122 FORMAT(5X,10(F10.0,2X))
0340 1123 FORMAT(///)
0341 RETURN
0342 END

```

```

0001      SUBROUTINE XONE (I,J,X,X1,Y)
C      *****
C      THIS SUBROUTINE IS USED ALONG LINE AD USING POINTS IN X DIRECTION.
C      REF. -ART.2..4.2
C      *****
C      COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFX(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEK1(35),
2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
3FFY(35,35),PP(35,35)
C
0002      DD=X-X1
0003      IF(I.EQ.2)GO TO 10
0004      DU=DEL(I)-DD
0005      GO TO 20
0006      10 DU=.0625-DD
0007      DX(I,J)=CD*16
0008      SLOX=100.0*X1 + 100.0*SQRT(2.0)
0009      BX(I-1,J)=50.0*(X1**2)+50.0*(Y*Y)
0010      1+100.0*SQRT(2.0) *X1 + 100.0*SQRT(2.0) *Y-300.0
0011      D1 = 1+DX(I,J)
0012      D2 = D1**2
0013      D3 = 1-DX(I,J)
0014      BFX(I,J) = (1+2.*DX(I,J))*BX(I-1,J)/D2+DD*SLOX/D1
0015      BOFX(I-1,J) = 4.*DX(I,J)*EX(I-1,J)/D2-2.*DU*SLOX/D1
0016      FX(I+1,J)=DX(I,J)**2/D2
0017      FFX(I+1,J)=D3**2/D2
0018      RETURN
0019      END

```

ORTRAN IV G LEVEL 18

XONE

DATE = 70278

18/32/15

PAGE 0002

MBCL LOCATION
L 0
3A84
9A38
C8A4

SYMBOL
DELL
BEX
DEK1
FFX

COMMON BLOCK /

LOCATION
8C
4DA8
9AC4
DBC8

SYMBOL
DX
BFY
LA
FY

MAP SIZE
LOCATION
118
60CC
9B50
EEEC

SYMBOL
DY
BOFX
KA
FFY

LOCATION
143C
73E0
AE74
10210

SYMBOL
BX
BOFY
P
PP

LOCAT
276
871
C19
1153

MBCL LOCATION
RT E0

SYMBOL

SUBPROGRAMS CALLED
LOCATION

SYMBOL

LOCATION

SYMBOL

LOCATION

SYMBOL

LOCAT

MBCL LOCATION
FC
104
118

SYMBOL
X
SLOX

SCALAR MAP
LOCATION
F4
108

SYMBOL
X1
Y

LOCATION
F8
10C

SYMBOL
I
D1

LOCATION
FC
110

SYMBOL
DU
D2

LOCAT
10
11

OPTIONS IN EFFECT ID, EBCDIC, SOURCE, NOLIST, NODECK, LOAD, MAP
OPTIONS IN EFFECT NAME = XONE, LINECNT = 45
STATISTICS SOURCE STATEMENTS = 19, PROGRAM SIZE = 1256
STATISTICS NO DIAGNOSTICS GENERATED

ORTRAN IV C LEVEL 18

YONE

DATE = 70278

18/32/15

PAGE 0002

SYMBOL	LOCATION	SYMBOL	LOCATION	COMMON BLOCK /	SYMBOL	MAP	SIZE	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION
DEL1	3A84	DEL1	8C	DEL1	8C	DEL1	118	12858	DEL1	118	DEL1	118	DEL1	118
DEK1	9A38	DEK1	4DA8	DEK1	4DA8	DEK1	60CC		DEK1	60CC	DEK1	60CC	DEK1	60CC
FFX	C6A4	FFX	9AC4	FFX	9AC4	FFX	9B50		FFX	9B50	FFX	9B50	FFX	9B50
			DBC8		DBC8		EEEC			EEEC		EEEC		EEEC

SYMBOL	LOCATION	SYMBOL	LOCATION	SUBPROGRAMS CALLED	SYMBOL	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION
EO	EO	EO	EO	EO	EO	EO	EO	EO	EO	EO	EO	EO	EO	EO

SYMBOL	LOCATION	SYMBOL	LOCATION	SCALAR MAP	SYMBOL	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION	SYMBOL	LOCATION
FC	FC	FC	FC	FC	FC	FC	FC	FC	FC	FC	FC	FC	FC	FC
104	104	104	104	104	104	104	104	104	104	104	104	104	104	104
118	118	118	118	118	118	118	118	118	118	118	118	118	118	118

CPTIONS IN EFFECT ID,EBCDIC,SOURCE,NOLIST,NODECK,LOAD,MAP
CPTIONS IN EFFECT NAME = YONE
STATISTICS SOURCE STATEMENTS = 19, PROGRAM SIZE = 1276
STATISTICS NO DIAGNOSTICS GENERATED

```

0001 SUBROUTINE BOONE (I,J,X,XI,Y,YI)
      C
0002 COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
      1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEKI(35),
      2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
      3FFY(35,35),PP(35,35)

```

```

0003 CALL XONE (I,J,X,XI,Y,YI)
0004 CALL XONE (I,J,X,XI,Y)
0005 RETURN
0006 END

```

```

0001      SUBROUTINE XTWO (I,J,X,X1,Y )
C          *****
C          THIS SUBROUTINE IS USED ALONG LINE CD USING POINTS IN X DIRECTION.
C          REF. -ART.2..4.2
C          *****
C          COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFX(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEK1(35),
2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
3FFY(35,35),PP(35,35)
C
0002      DD=X-X1
0003      IF(I.EQ.2)GO TO 10
0004      DU=DEL(I)-DD
0005      GO TO 20
0006      DU=.0625-DD
0007      DX(I,J)=DD*16
0008      SLOX = 100.0*SQRT(2.0)
0009      BX (I-1,J) = 100.0*SQRT(2.0)*X1 + 150.0 *SQRT(2.0)*Y -325.0
0010      D1 = 1+DX(I,J)
0011      D2 = D1**2
0012      D3 = 1-DX(I,J)
0013      BFX(I,J) = (1+2.*DX(I,J))*BX(I-1,J)/D2+DD*SLOX/D1
0014      BOFX(I-1,J)=4.*DX(I,J)*BX(I-1,J)/D2-2.*DU*SLOX/D1
0015      FFX(I+1,J)=DX(I,J)**2/D2
0016      FFX(I+1,J)=D3**2/D2
0017      RETURN
0018      END
0019

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FORTRAN IV G LEVEL 19

PAGE 0001

18/32/15

DATE = 70278

BOTWO

SUBROUTINE BOTWO (I,J,X,X1,Y,Y1)

COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEK1(35),
2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
3FFY(35,35),PP(35,35)

C

YTWO (I,J,X,X1,Y,Y1)
XTWO (I,J,X,X1,Y)

CALL
CALL
RETURN
END

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FORTRAN IV G LEVEL 18      BJTHRE      DATE = 70278      18/32/15      PAGE 0001
0001      SUBROUTINE BOTHRE (I,J,X,X1,Y,Y1)
C
C      COMMON DEL(35),DEL1(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEK1(35),
2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
3FFY(35,35),PP(35,35)
C
C      CALL      YTHRE (I,J,X,X,Y,Y1)
0003      CALL      XTHREE(I,J,X,X1,Y)
0004      RETURN
0005      END
0006

```

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0001      SUBROUTINE YTHRE (I,J,X,Y,Y1)
C          *****
C          THIS SUBROUTINE IS USED ALONG LINE AB USING POINTS IN Y DIRECTION.
C          REF. -ART.2..4.2
C          *****
0002      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
16FX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEK1(35),
2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
3FFY(35,35),PP(35,35)
C
C          DD=Y-Y1
C          IF(J.EQ.2)GO TO10
C          DU=DELI(J)-DD
C          GO TO 20
10      DY(I,J)=DD*16
20      SLOY = 100.*SQRT(2.)
      BY(I,J-1) = 150.*SQRT(2.)*X+100.*SQRT(2.)*Y1-325.
      D1=1.+DY(I,J)
      D2=D1**2
      D3=1.-DY(I,J)
      BOFY(I,J)=(1.+2.*DY(I,J))*BY(I,J-1)/D2+DD*DD*SLOY/D1
      FY(I,J+1)=DY(I,J)**2/D2
      FFY(I,J+1)=D3**2/D2
      RETURN
      END

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FORTRAN IV G LEVEL 18          BOFOUR          DATE = 70278          18/32/15          PAGF 0001
0001      SUBROUTINE BOFOUR(I,J,X,X1,Y,Y1)
C
C
COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),RY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEK1(35),
2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
3FFY(35,35),PP(35,35)
C
C      CALL          XFOUR(I,J,X,X1,Y)
C      CALL          YFOUR(I,J,X,X1,Y1)
10  CALL
20  RETURN
END
0003
0004
0005
0006

```

SUBROUTINE YFOUR(I,J,X,Y,Y1)

THIS SUBROUTINE IS USED ALONG LINE BC USING POINTS IN Y DIRECTION.
REF. -ART.2..4.2

REF. -ART.2.4.2

COMMON DEL(35), DEL1(35), DX(35,35), DY(35,35), DX(35,35), BY(35,35),
LBFX(35,35), BFX(35,35), BOFX(35,35), BOFY(35,35), DEK(35), DEK1(35),
2LA(35,35), KA(35,35), P(451), FX(35,35), FFX(35,35), FY(35,35),
3FFY(35,35), PP(35,35)

$$00=Y1-Y$$
$$DU = DE L I (J+1) - DD$$
$$DY(I, J+1) = DD * I6$$
$$\text{SLO}Y=100 \cdot *YI$$

```
BY(I,J+1)=50.*X**2.+50.*Y1**2.
```

$$DI = I + DY(I, J+1)$$

02 03 04 05 06 07 08 09 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100 101 102 103 104 105 106 107 108 109 110 111 112 113 114 115 116 117 118 119 120 121 122 123 124 125 126 127 128 129 130 131 132 133 134 135 136 137 138 139 140 141 142 143 144 145 146 147 148 149 150 151 152 153 154 155 156 157 158 159 160 161 162 163 164 165 166 167 168 169 170 171 172 173 174 175 176 177 178 179 180 181 182 183 184 185 186 187 188 189 190 191 192 193 194 195 196 197 198 199 200 201 202 203 204 205 206 207 208 209 210 211 212 213 214 215 216 217 218 219 220 221 222 223 224 225 226 227 228 229 230 231 232 233 234 235 236 237 238 239 240 241 242 243 244 245 246 247 248 249 250 251 252 253 254 255 256 257 258 259 260 261 262 263 264 265 266 267 268 269 270 271 272 273 274 275 276 277 278 279 280 281 282 283 284 285 286 287 288 289 290 291 292 293 294 295 296 297 298 299 300 301 302 303 304 305 306 307 308 309 310 311 312 313 314 315 316 317 318 319 320 321 322 323 324 325 326 327 328 329 330 331 332 333 334 335 336 337 338 339 340 341 342 343 344 345 346 347 348 349 350 351 352 353 354 355 356 357 358 359 360 361 362 363 364 365 366 367 368 369 370 371 372 373 374 375 376 377 378 379 380 381 382 383 384 385 386 387 388 389 390 391 392 393 394 395 396 397 398 399 400 401 402 403 404 405 406 407 408 409 410 411 412 413 414 415 416 417 418 419 420 421 422 423 424 425 426 427 428 429 430 431 432 433 434 435 436 437 438 439 440 441 442 443 444 445 446 447 448 449 450 451 452 453 454 455 456 457 458 459 460 461 462 463 464 465 466 467 468 469 470 471 472 473 474 475 476 477 478 479 480 481 482 483 484 485 486 487 488 489 490 491 492 493 494 495 496 497 498 499 500 501 502 503 504 505 506 507 508 509 510 511 512 513 514 515 516 517 518 519 520 521 522 523 524 525 526 527 528 529 530 531 532 533 534 535 536 537 538 539 540 541 542 543 544 545 546 547 548 549 550 551 552 553 554 555 556 557 558 559 560 561 562 563 564 565 566 567 568 569 570 571 572 573 574 575 576 577 578 579 580 581 582 583 584 585 586 587 588 589 590 591 592 593 594 595 596 597 598 599 600 601 602 603 604 605 606 607 608 609 610 611 612 613 614 615 616 617 618 619 620 621 622 623 624 625 626 627 628 629 630 631 632 633 634 635 636 637 638 639 640 641 642 643 644 645 646 647 648 649 650 651 652 653 654 655 656 657 658 659 660 661 662 663 664 665 666 667 668 669 670 671 672 673 674 675 676 677 678 679 680 681 682 683 684 685 686 687 688 689 690 691 692 693 694 695 696 697 698 699 700 701 702 703 704 705 706 707 708 709 710 711 712 713 714 715 716 717 718 719 720 721 722 723 724 725 726 727 728 729 730 731 732 733 734 735 736 737 738 739 740 741 742 743 744 745 746 747 748 749 750 751 752 753 754 755 756 757 758 759 760 761 762 763 764 765 766 767 768 769 770 771 772 773 774 775 776 777 778 779 780 781 782 783 784 785 786 787 788 789 790 791 792 793 794 795 796 797 798 799 800 801 802 803 804 805 806 807 808 809 810 811 812 813 814 815 816 817 818 819 820 821 822 823 824 825 826 827 828 829 830 831 832 833 834 835 836 837 838 839 840 841 842 843 844 845 846 847 848 849 850 851 852 853 854 855 856 857 858 859 860 861 862 863 864 865 866 867 868 869 870 871 872 873 874 875 876 877 878 879 880 881 882 883 884 885 886 887 888 889 890 891 892 893 894 895 896 897 898 899 900 901 902 903 904 905 906 907 908 909 910 911 912 913 914 915 916 917 918 919 920 921 922 923 924 925 926 927 928 929 930 931 932 933 934 935 936 937 938 939 940 941 942 943 944 945 946 947 948 949 950 951 952 953 954 955 956 957 958 959 960 961 962 963 964 965 966 967 968 969 970 971 972 973 974 975 976 977 978 979 980 981 982 983 984 985 986 987 988 989 990 991 992 993 994 995 996 997 998 999 1000 1001 1002 1003 1004 1005 1006 1007 1008 1009 1010 1011 1012 1013 1014 1015 1016 1017 1018 1019 1020 1021 1022 1023 1024 1025 1026 1027 1028 1029 1030 1031 1032 1033 1034 1035 1036 1037 1038 1039

$$\text{REY}(i,j) = (U(i,j) - U(i,j+1) + U(i,j+2) - U(i,j+3) + \dots)$$
$$BQ EY(I, J+1) = 4 * DY(I, J+1) * BY(I, J+1) / DZ + 2 * BY(I, J+1) * B(I, J+1) / DZ - DZ * SLUY / D1$$

```
DO I=1,J,I/=+1,NDIV,J+1
EY(I,J-1)=DY(I,J+1)**2/D2
```

FFY(I, J-1)=D3**2/D2

RETURN

END

```

0001      SUBROUTINE XFOUR(I,J,X,X1,Y)
C
C
C      THIS SUBROUTINE IS USED ALONG LINE 8C USING POINTS IN X DIRECTION.
C      REF. -ART.2..4.2
C
0002      COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEKI(35),
2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
3FFY(35,35),PP(35,35)
C
C
C      DD=X1-X
C      DU=DEL(I+1)-DD
C      DX(I+1,J)=DD*16
C      SLOX=100.*X1
C      BX(I+1,J)=50.*X1**2.+50.*Y**2.
C      D1 = 1+DX(I+1,J)
C      D2 = D1**2
C      D3 = 1-DX(I+1,J)
C      BFX(I,J) = (1.+2.*DX(I+1,J))*BX(I+1,J)/D2-DD*SLOX/D1
C      BOFX(I+1,J) = 4.*DX(I+1,J)*BX(I+1,J)/D2+2.*DU*SLOX/D1
C      FX(I-1,J)=DX(I+1,J)**2/D2
C      FFX(I-1,J)=D3**2/D2
C      RETURN
C      END

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0001

SUBROUTINE PHI

CCCCCCCC

 THIS SUBROUTINE PUTS THE VALUES OF PHI OF THE NODAL POINTS LESS
 THAN ONE MESH AWAY FROM THE BOUNDARY IN ORDER

0002

COMMON DEL(35),DELI(35),DX(35,35),DY(35,35),BX(35,35),BY(35,35),
 1BFX(35,35),BFY(35,35),BOFX(35,35),BOFY(35,35),DEK(35),DEK1(35),
 2LA(35,35),KA(35,35),P(451),FX(35,35),FFX(35,35),FY(35,35),
 3FFY(35,35),PP(35,35)

DO 10 J = 3,12
 DO 10 I = 3,12
 IF (KA(I,J)) 20,10,20
 L1 = LA(I+1,J)
 L2 = LA(I,J+1)
 L3 = LA(I-1,J+2)
 PP(I,J) = (BFX(I,J)+BFY(I,J)+FX(I+1,J)*P(L1)+FY(I,J+1)*P(L2)

20

1)*#0.5
 IF (I.EQ.3.AND.J.EQ.11) GO TO 10
 PP(I-1,J) = (BOFX(I-1,J) + BOFY(I-1,J) + FFX(I+1,J) * P(L1)
 1+FFY(I-1,J+2)*P(L3))*0.5

0010

0011

10

CONTINUE
 DO 30 J = 13,33
 DO 30 I = 3,23
 IF (KA(I,J)) 40,30,40

40

L1 = LA(I+1,J)
 L2 = LA(I,J-1)
 L3 = LA(I-1,J-2)
 PP(I,J) = (BFX(I,J) + BFY(I,J) + FX(I+1,J) *P(L1) +
 1FY(I,J-1) * P(L2))*0.5
 IF (I.EQ.3.AND.J.EQ.14) GO TO 30
 PP(I-1,J) = (BOFX(I-1,J) + BOFY(I-1,J) + FFX(I+1,J) *
 1P(L1)+FFY(I-1,J-2)*P(L3))*0.5

0020

0021

30

CONTINUE
 DO 50 J = 24,33
 DO 50 I = 24,33
 IF (KA(I,J)) 60,50,60
 L1 = LA(I-1,J)
 L2 = LA(I,J-1)
 L3 = LA(I+1,J-2)

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0029 PP(I , J ) = (BFX(I , J ) + BFY(I , J ) + FX(I-1,J ) *
0030 1P(L1) + FY(I , J-1) ) * P(L2)) * 0.5
0031 PP(I+1,J ) = ( BDFX(I+1,J ) + BDFY(I+1,J ) + BDFX(I-1,J ) *
0032 1P(L1)+FFY(I+1,J-2)*P(L3))*0.5
0033 50 CONTINUE
0034 DO 70 J=3,23
0035 DO 70 I=13,33
0036 IF(KA(I,J))80,70,80
0037 L1=LA(I-1,J)
0038 L2=LA(I,J+1)
0039 L3=LA(I+1,J+2)
0040 PP(I,J)=(BFX(I,J)+BFY(I,J)+FX(I-1,J)*P(L1)+FY(I,J+1)*P(L2))*0.5
0041 IF(I.EQ.33 .AND.J.EQ.22)GO TO 70
0042 PP(I+1,J)=(BDFX(I+1,J)+BDFY(I+1,J)+FFX(I-1,J)*P(L1)+FFY(I+1,J+2)
0043 1*P(L3))*0.5
0044 70 CONTINUE
0045 DO 90 J=2,3
0046 DO 90 I=11,14
0047 IF(KA(I,J))100,90,100
0048 L2=LA(I,J+1)
0049 IF(I.EQ.11.AND.J.EQ.3)GO TO 95
0050 IF(I.EQ.14.AND.J.EQ.3)GO TO 95
0051 PP(I,J)=BFY(I,J)+FY(I,J+1)*P(L2)
0052 PP(I,J-1)=BDFY(I,J-1)+FFY(I,J+1)*P(L2)
0053 90 CONTINUE
0054 DO 110 J=11,14
0055 DO 110 I=2,3
0056 IF(KA(I,J))120,110,120
0057 L1=LA(I+1,J)
0058 IF(I.EQ.3.AND.J.EQ.11)GO TO 130
0059 IF(I.EQ.3.AND.J.EQ.14)GO TO 130
0060 PP(I,J)=BFX(I,J)+FX(I+1,J)*P(L1)
0061 PP(I-1,J)=BDFX(I-1,J)+FFX(I+1,J)*P(L1)
0062 110 CONTINUE
0063 DO 140 J=22,25
0064 DO 140 I=33,34
0065 IF(KA(I,J))150,140,150
0066 L1=LA(I-1,J)
0067 IF(I.EQ.33.AND.J.EQ.22)GO TO 160
0068 PP(I,J)=BFX(I,J)+FX(I-1,J)*P(L1)
0069 PP(I+1,J)=BDFX(I+1,J)+FFX(I-1,J)*P(L1)
0070 140 CONTINUE

```

FORTRAN IV G LEVEL 13

PHI

DATE = 70278

18/32/15

PAGE 0003

```
0069      DO 170 J=33,34
0070      DO 170 I=22,25
0071      IF (KA(I,J)) 180,170,180
0072      L2=LA(I,J-1)
0073      IF (I.EQ.22.AND.J.EQ.33) GO TO 190
0074      IF (I.EQ.25.AND.J.EQ.33) GO TO 190
0075      PP(I,J)=BFFY(I,J)+FY(I,J-1)*P(L2)
0076      PP(I,J+1)=BFFY(I,J+1)+FFY(I,J-1)*P(L2)
0077      170 CONTINUE
0078      RETURN
0079      END
```

THIS PROGRAM CALCULATES THE PRINCIPAL STRESSES AND SHEAR STRESSES
FOR THE EXPERIMENTAL DATA, USING SHEAR DIFFERENCE METHOD.

THE VARIABLES ARE DEFINED AS FOLLOWS.

P NL APPLIED OF TENSILE LOAD.

X NL DISTANCE FROM THE FREE BOUNDARY.

B NL WIDTH OF THE SPECIMEN.

H NL THICKNESS OF THE SPECIMEN.

DL=0.75 DISTANCE OF THE INTERFACE FROM THE FREE BOUNDARY.

DL=1.-(3.75/SQRT(2.))

DL=1.-(3.75/SQRT(2.))

DL=1.-(3.75/SQRT(2.))

DL=1.-(3.75/SQRT(2.))

DL=1.

REFER. FIG. -22

AN, ANN FRINGE ORDER ALONG LINE AB.

AN1, ANN1 FRINGE ORDER ALONG LINE CD.

AN2, ANN2 FRINGE ORDER ALONG LINE EF.

C STRESS OPTIC COEFFICIENT.

DELX DISTANCE BETWEEN LINE CD AND EF.

DIMENSION PXX(30), PYY(30), TAB(30), TCD(30), TEF(30), P1(30), P2(30),

LDIST(30)

WRITE(3,160)

DL=1.

P=125.

NL=19

AN=2. -27./180.

AN1=2. -30./180.

AN2=2. -26./180.

X=0.0

0001

0002

0003

0004

0005

0006

0007

0008

0009

```

0010 B1=2.*2.*DL
0011 DIST(1)=0.0
0012 B=2.
0013 H=0.254
0014 DELY=.05
0015 A=(B-B1)*H
0016 A1=B*H
0017 PNET=P/A
0018 PNET1=P/A1
0019 PI=3.1415927
0020 H1=2.*H
0021 THETA=0.0
0022 THETA1=0.0
0023 THETA2=0.0
0024 THETA1=THETA1*PI/180.
0025 THETA2=THETA2*PI/180.
0026 C=36.
0027 PXX(1)=0.0
0028 PYY(1)=(AN*C)/H
0029 TAB(1)=(AN1*C*SIN(2.*THETA1))/H1
0030 TCD(1)=(AN2*C*SIN(2.*THETA2))/H1
0031 TEF(1)=0.0
0032 CONFAC=PY(1)/PNET
0033 CONFAL=PY(1)/PNET1
0034 WRITE(3,150)PNET,PNET1,PXX(1),PYY(1),TAB(1),TCD(1),CONFAC,CONFAL,X
0035 I=1
0036 WRITE(3,400)
0037 READ(1,100)DELX,ANL,ANN1,THETA1,AN,ANN,THETA,AN2,ANN2,THETA2
0038 X=X+DELX
0039 I=I+1
0040 DIST(1)=X/DL
0041 IL=I-1
0042 THETA=THETA*PI/180.
0043 THETA1=THETA1*PI/180.
0044 THETA2=THETA2*PI/180.
0045 TAB(1)=((AN1+ANN1/180.)*C*SIN(2.*THETA1))/H1
0046 TCD(1)=((AN2+ANN2/180.)*C*SIN(2.*THETA2))/H1
0047 TEF(1)=((AN+ANN/180.)*C*SIN(2.*THETA))/H1
0048 TABX=(TAB(1)+TAB(IL))/2.
0049 TCDX=(TCD(1)+TCD(IL))/2.
0050 DETXY=TABX-TCDX
0051 PXX(1)=PXX(1)-(DETXY*DELX/DELY)
0052 PYY(1)=PXX(1)+((AN+ANN/180.)*C*COS(2.*THETA))/H

```

```

0053 AB=PXX(I)+PYY(I)
0054 CD=((AN+ANN/180.)*C)/H
0055 P1(I)=(AB+CD)/2.
0056 P2(I)=(AB-CD)/2.
0057 CONFAC=P1(I)/PNET
0058 CONFAL=P1(I)/PNET1
0059 WRTTE(3,101),CONFAL,
0060 IF(I-NL)200,300,300
0061 300 FAC=PNET1/100.
0062 WRTTE(3,45)
0063 P1(I)=PYY(I)/FAC
0064 P2(I)=PXX(I)/FAC
0065 DO 70 I=2,NL
0066 P1(I)=P1(I)/FAC
0067 P2(I)=P2(I)/FAC
0068 70 CONTINUE
0069 160 WRTTE(3,46)((I,DIST(I),P1(I),P2(I),TEF(I)),I=1,NL)
0070 160 FORMAT(1H1,10X,PNET1,10X,PXX(1),8X,PYY(1),8X,TA
0071 150 1B(1),10X,TCD(1),8X,CONFAC,8X,CONFAL,8X,X)
0072 150 FORMAT(/,1X,8F15.4,F9.3)
0073 400 FORMAT(1H1,2X,I,12X,CONFAC1,9X,P1(I),9X,P2(I),
0074 19X,PXX(I),9X,PYY(I),9X,CONFAC,9X,P1-P2,5X,X)
0075 100 FORMAT(10F8.4)
0076 101 FORMAT(/,1X,12,8(F13.4,2X))
0077 45 FORMAT(1H1,12X,I,15X,DIST(I),22X,P1(I),19X,P2(I),16X,
0078 1SHEAR STRESS)
46 FORMAT(/,10X,I3,15X,F10.6,15X,F10.2,15X,F10.2)
RETURN
END

```

REFERENCES

1. Paterson, R.E., "Stress Concentration Design Factors," John Wiley and Sons, Inc., New York.
2. Howland, R.C.J., "On The Stresses In The Neighborhood Of A Circular Hole In A Strip Under Tension," Phil. Trans. Roy. Soc. (London) A, Vol. 229 (1929-30), pp. 67.
3. Howland, R.C.J., and Stevenson, A., "Biharmonic Analysis In A Perforated Strip", Phil. Trans. Roy. Soc. London A232, 1933, pp. 155.
4. Isida, M., "On The Tension Of An Infinite Strip With An Eccentric Circular Hole (In Japanese), Tran. J.S.M.E. Vol.19, 1953, pp. 100.
5. Isida, M., "Form Factors Of A Strip With An Elliptic Hole In Tension and Bending", Sci. Pap. Fac. Engng., Tokushima Univ. Vol. 4, 1953 pp. 70.
6. Howland, R.C.J., "Stresses In A Plate Containing An Infinite Row Of Circular Holes", Proc. Roy. Soc. London A, Vol.148, 1935, pp. 471.
7. Ling, C.B., "On The Stresses In A Plate Containing Two Circular Holes", J. Appl. Phys. Vol.19, January 1948, pp. 77.
8. Durelli, A.J., Parks, V.J. and Feng, H.C., "Stresses Around An Elliptical Hole In A Finite Plate Subjected To Axial Loading", J. Appl. Mech. March 1962, pp. 102.
9. Atsumi, A., "On The Stresses In A Strip Under Tension And Containing Two Equal Circular Holes Placed Longitudinally", J. Appl. Mech. Vol. 23, 1956, pp. 555.
10. Nisida, M., and Hirai, N., "Stress Concentration In Strip With Rectangular Hole Of Rounded-Off Corners", Proc. Japan Nat. Congr. Appl. Mech. Vol. 12, 1962, pp 63.

11. Savin, G.N., "Stress Concentration Around Holes", Govt. Printing Office, Leningrad (1951), pp. 456.
12. Isida, M. "On The Tension Of An Infinite Strip Containing A Square Hole With Rounded Corners", Bull. J.S.M.E., Vol.3, N 10, May 1960, pp. 254.
13. Wittrick, W.H., "Stress Concentration In A Family Of Uniformly Reinforced Square Holes With Rounded Corners", Aeronautical Quarterly, Vol.13, August 1962, pp. 223.
14. Hicks, R., "Reinforced Elliptical Holes In Stressed Plates", J. Roy. Aeronaut. Soc. Vol. 61, 1967, pp. 688.
15. Wittrick, W.H., "Analysis Of Stress Concentrations At Reinforced Holes In Infinite Sheets", Aeronaut. Quarterly, Vol.11, N 3, August 1960, pp. 233.
16. Goodier, J.N., "Concentration Of Stress Around Spherical And Cylindrical Inclusions And Flaws", Trans. A.S.M.E., Vol.5, 1932, pp. 30.
17. Donnell, L.H., "Stress Concentrations Due To Elliptical Discontinuities In Plates Under Edge Forces," Von Karman Anniversary Volume, California Institute Of Tech., Pasadena, 1941, pp. 293.
18. Thibodeau, W.E. And Wood, L.A., "Photo Elastic Determination Of Stresses Around Circular Inclusion", National Bureau Of Standard Journal of Research, Vol. 20, N 3, Paper 1083, March 1938, pp. 393.
19. Edmonds, D.V. And Beevers, C.J., "The Effect Of Inclusions On The Stress Distribution in Solids", J. Of Materials Sci. 3 (1968) pp. 457.
20. Timoshenko, S. and Goodier, J.N., "Theory Of Elasticity", McGraw-Hill Book Company, Inc., New York.

21. Allen, D.N. Deg. "Relaxation Methods", McGraw-Hill Book Company, Inc., New York:
22. Zienkiewicz, O.C. And Gerstner, R.W., "A Stress Function Approach To Interface And Mixed Boundary Condition Problem", Intern. J. Mech. Sci., 2, 93 (1960).
23. Malhotra, R., Mirza, S. And Feingold, A., "Computational Techniques For Economical Storage Of Large Sparsed And Banded Matrices", Proceedings Of The Fourth DECUS Conference, Montreal (1970).
24. Dally, W.J., "Experimental Stress Analysis", McGraw-Hill Book Company, New York.
25. Zienkiewicz, O.C. And Holister, G.S., "Stress Analysis," John Wiley and Sons Limited, 1965.