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THE PREDICTION OF CHF IN CROSS FLOW

by

RA-MIN TAIN

A thesis
presented to the University of Ottawa
on December, 1989
in partial fulfillment of the
requirements for the degree of
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in
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Ottawa-Carleton Institute for
Mechanical and Aeronautical Engineering



Ra-Min Tain, Ottawa, Canada, 1990



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UNIVERSITÉ D'OTTAWA
UNIVERSITY OF OTTAWA

Abstract

A semi-empirical model to predict the Critical Heat Flux (CHF) in cross-flow boiling is presented in this work. This model, using an empirical correlation to calculate the vapor sheet thickness, is able to predict the CHF in subcooled or saturated region. The empirical correlation presented by Ungar and Eichhorn (1988) to calculate the vapor sheet thickness is modified by this work to include the quality effect. The RMS error for this semi-empirical correlation against the 271 experimental data obtained from various investigations is around 22%.

A correction factor resulting from the investigation of the relationship for CHF's between cross-flow and axial-flow boiling is obtained from this work. During this investigation, an empirical correlation to predict the cross-flow CHF is also discovered. The correction factor method and the empirical correlation have wider ranges of mass flux effect than the semi-empirical correlation does. The RMS error for the correction factor method and the empirical correlation against the 177 experimental data are 16.2% and 13.4%, respectively. Because these prediction methods are in dimensionless forms, they can be applied to water as well as the non-aqueous fluids.

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Nomenclature

a	Empirical value as defined in Eq. (4.27)	
A_1^*	Coefficient used in Eq. (5.1)	
A_2^*	Coefficient used in Eq. (5.1)	
A_3^*	Coefficient used in Eq. (5.1)	
A_h	Area of heated surface,	m^2
A_j	Cross-sectional area of the vapor jets leaving the cylinder,	m^2
A_m	Mean flow area as defined in Eq. (5.4),	m^2
A_v	Cross-sectional area of the vapor stems,	m^2
A_∞	Cross-sectional area of control volume or flow channel,	m^2
b	Empirical value as defined in Eq. (4.28)	
Bo	Boiling number, or $Bo = q_{max}/(Gh_{fg})$.	
Bo_{axial}	Boiling number for axial flow	
Bo_{cross}	Boiling number for cross flow	
c	Empirical value as defined in Eq. (4.21) or Eq. (4.29)	
C_2	Empirical constant used in Eq. (3.31)	
D	Outside diameter for cross flow; inside diameter for axial flow,	m
D'	Length of control volume as shown in Fig. 5.3,	m
E_c	Eckert number	
f	Frequency of vapor departure or bubble break-off,	Hz
$\bar{f}$	Fraction of heat transferred directly to vapor	
Fr	Froude number, $U_\infty/(gD)^{1/2}$ or $U/(gD)^{1/2}$	
g	Acceleration due to gravity,	$m \cdot s^{-2}$
G	Mass flux,	$kg \cdot m^{-2} \cdot s^{-1}$
G_c	Gravity influence parameter as defined in Eq. (3.27)	
G_D	Mass flow rate of deposition of entrained liquid droplets,	$kg \cdot s^{-1}$
G_E	Mass flow rate of the entrainment of liquid droplets,	$kg \cdot s^{-1}$
G_{LF}	Liquid film flow rate along the heated tube,	$kg \cdot s^{-1}$

h_f	Enthalpy of saturated liquid,	$J \cdot kg^{-1}$
h_g	Enthalpy of saturated vapor,	$J \cdot kg^{-1}$
h_{fg}	Enthalpy of vaporization,	$J \cdot kg^{-1}$
h_l	Enthalpy other than at saturated condition,	$J \cdot kg^{-1}$
K	Empirical constant as defined in Eq. (3.29)	
$KCHF$	Correction factor as defined in Eq. (7.1)	
K_q	Blockage factor,	D/w
L	Length of heated cylinder in cross flow,	m
L_h	Length of heated tube in axial flow,	m
m	Empirical constant as defined Eq. (3.30)	
$MESC$	Mechanical energy stability criterion	
$\dot{m}_g$	Mass flow rate of vapor at the outlet of control volume,	$kg \cdot s^{-1}$
$\dot{m}_l$	Mass flow rate of liquid at the outlet of control volume,	$kg \cdot s^{-1}$
$\dot{m}_\infty$	Mass flow rate of liquid at the inlet of control volume,	$kg \cdot s^{-1}$
q	Heat flow rate,	W
$\dot{q}$	Heat flux,	$W \cdot m^{-2}$
Q	Volumetric rate of channel flow,	$m^3 \cdot s^{-1}$
$q_{CHF,FB}$	CHF for flow boiling as defined in Eq. (3.31),	$W \cdot m^{-2}$
$q_{CHF,PB}$	CHF for pool boiling as defined in Eq. (4.1),	$W \cdot m^{-2}$
q_{co}	CHF in saturated boiling as defined in Eq. (3.8) and Eq. (3.12),	$W \cdot m^{-2}$
$q_{co,z}$	q_{co} predicted by Zuber as defined in Eq. (2.1),	$W \cdot m^{-2}$
q_{max}	When $\dot{q}$ is at CHF,	$W \cdot m^{-2}$
$q_{max,axial}$	CHF occurs in axial flow,	$W \cdot m^{-2}$
$q_{max,cross}$	CHF occurs in cross flow,	$W \cdot m^{-2}$
r	Density ratio, or ρ_f/ρ_g	
r'	Instantaneous radius of bubble,	m
R	Radius of the cylinder,	m
R'	Dimensionless radius of cylinder as defined in Eq. (3.9)	
s	Instantaneous height of bubble center,	m

St	Strouhal number	
t	Time,	s
T_{sat}	Saturated temperature of liquid,	$^{\circ}C$
T_w	Temperature at the heated wall,	$^{\circ}C$
U	Liquid velocity corrected for profile and blockage,	$m \cdot s^{-1}$
U_g	Vapor velocity relative to liquid flow,	$m \cdot s^{-1}$
U_H	Helmholtz unstable value of U_g ,	$m \cdot s^{-1}$
u_{TP}	Homogeneous flow velocity as defined in Eq. (3.33),	$m \cdot s^{-1}$
U_{∞}	Velocity of bulk liquid or liquid velocity at the inlet of control volume.	$m \cdot s^{-1}$
V	Instantaneous volume of bubble as defined in Eq. (3.6),	m^3
w	Channel width of flow,	m
W	Width of control volume,	m
We_f	Liquid Weber number	
We_g	Vapor Weber number	
x	Equilibrium quality of flow	
x'	The arc length from the front stagnation point of cylinder,	m
X_d	Diagonal pitch in tube bundles,	m
X_{dd}	Ratio of diagonal pitch to tube diameter, or X_d/D	
X_l	Longitudinal pitch in tube bundles,	m
X_{ll}	Ratio of longitudinal pitch to tube diameter, or X_l/D	
X_t	Transverse pitch in tube bundles,	m
X_{tt}	Ratio of transverse pitch to tube diameter, or X_t/D	

Greek Symbols

α	Vapor sheet thickness divided by $2R$
α'	Vapor sheet thickness as defined in Eq. (4.18)
α^*	Effective dimensionless vapor wake thickness as used in Eq. (4.19)
α_v	Void fraction

γ	$ \partial(\rho_f/\rho_g)/\partial P _{saturation}$ used in Ahmad's second scaling factor	
δ	Thickness of vapor blanket used in Eq. (3.20) and Fig. 4-7,	m
δ_c	Critical thickness of liquid film as defined in Eq. (3.4),	m
Δh_{sub}	inlet subcooling,	$J \cdot kg^{-1}$
Δt_{sat}	Temperature of wall above liquid saturation temperature,	$^{\circ}C$
ϵ	Thermal diffusivity of liquid	
κ	Thermal conductivity of liquid,	$J \cdot s^{-1} \cdot m^{-1} \cdot ^{\circ}C^{-1}$
λ	Wavelength,	m
λ_c	Taylor critical wavelength,	m
λ'_D	most susceptible Taylor wavelength,	m
λ_H	Helmholtz wavelength,	m
μ_f	Liquid viscosity,	$kg \cdot m^{-1} \cdot s^{-1}$
μ_g	Vapor viscosity,	$kg \cdot m^{-1} \cdot s^{-1}$
ρ, ρ_{TP}	Homogeneous fluid density as defined in Eq. (3.32),	$kg \cdot m^{-3}$
ρ_f	Saturated liquid density,	$kg \cdot m^{-3}$
ρ_g	Saturated vapor density,	$kg \cdot m^{-3}$
σ	Surface tension,	$N \cdot m^{-1}$
τ_d	Hovering period of bubble,	s
v	Volumetric growth rate of bubble as defined in Eq. (3.2),	$m^3 \cdot s^{-1}$
ϕ	Dimensionless CHF as defined in Eq. (4.10)	
Ψ_k	Katto's scaling factor as in Table 6.3	
Ψ_{σ}	Ahmad's first scaling factor as shown in Table 6.3	
Ψ_{γ}	Ahmad's second scaling factor as shown in Table 6.3	

Chapter 1

Introduction

Boiling heat transfer is a very efficient means of energy transport. Numerous engineering applications have led to investigations of various aspects of boiling heat transfer, especially the mechanism of transition from nucleate to film boiling. Boiling heat transfer for flow across a cylinder or bank of cylinders has received little attention in thermal-hydraulics research. The departure from nucleate boiling (DNB), or the dryout threshold, is one of the main limits that characterizes the boiling curve of the whole flow field. Very little work has been done on the present problem in which the flow is normal to the axis of the cylinder. For pool boiling, the critical heat flux, or CHF, mechanism occurring on a horizontal single cylinder is reasonably well understood, and the geometry of a single cylinder in cross-flow boiling also provides a typical approach to the classical theory for the CHF. Besides, the single rod is the building block of a rod bundle, and more information might provide a base for attacking that far more complex problem.

For boiling inside a vertical tube, many correlations have been developed for the prediction of critical heat flux. Most of them are only applicable for a limited range of flow parameters. Groeneveld's (1986a, 1986b) CHF table look-up method is one of the methods providing wide ranges of system parameters. This method predicts not only water CHF but also the CHF for non-aqueous fluids, and can be used for a geometry other than a single vertical tube (e.g. bundle geometry). Assessment of various published correlations undertaken by ESDU (1986) shows that this tabular method is one of the most reliable methods of all published methods.

In the present study, the objective of this investigation is to develop a prediction

method of CHF for cross-flow boiling. The CHF for cross flow predicted by relating it at the CHF for axial flow forms the initial idea of this study. It is obvious that the flow patterns of cross flow and axial flow are very different. The mechanisms of the CHF occurrences for both cases are not close to each other. Therefore, it is difficult to determine the relationship of CHF's between cross flow and axial flow from the theoretical point of view. However, most CHF prediction methods for cross flow and axial flow are empirical or semi-empirical. They are developed from the observation of parametric trends and some of these parameters are used in both cases. Thus the relationship of CHF's between cross flow and axial flow determined on the basis of using common parameters would be a possible and an appropriate approach. If it is possible, the investigation of the CHF relationship between cross flow and axial flow can lead to a correction factor to relate the CHF's in both cases. However, this approach may not be the best method. The semi-empirical and empirical methods are also investigated in this study.

The data base used to obtain the CHF for axial flow is based on Groeneveld's CHF look-up table, and the sources of experimental data for cross flow are obtained from Vliet (1962), Kezios and Lo (1958), Yilmaz and Westwater (1980), Diesselhorst et al. (1977), Pourdashti (1984), Hasan et al. (1981), Min (1975), Broussard and Westwater (1985), and Ungar and Eichhorn (1988).

The experimental observations of CHF trends from the literature about pool boiling, axial-flow boiling and cross-flow boiling are discussed in Chapter 2. In Chapter 3, the two best known analytical models for cross-flow CHF are introduced. The empirical correlations for cross-flow CHF are also reviewed in this chapter. The semi-empirical model to predict the CHF in cross-flow boiling by employing Lienhard and Eichhorn's (1976) assumption in their model and the hydrodynamic model to predict the vapor sheet thickness in Ungar and Eichhorn (1988) is presented in Chapter 4. The experimental CHF data for cross-flow boiling used in this semi-empirical analysis are illustrated in Appendix A.

Chapter 5 defines the characterization of velocity and diameter in cross-flow boiling for use by the following chapters. In Chapter 6, the prediction methods for CHF in axial flow are reviewed. Groeneveld's table look-up method is discussed together with the water and non-aqueous CHF tables shown in this chapter. The programs for predicting the water CHF and the CHF of non-aqueous fluids by Groeneveld's table look-up method are presented in Appendix B and Appendix C,

respectively. A correction factor relating the cross-flow CHF with the CHF in axial-flow boiling is presented in Chapter 7. An empirical correlation for cross-flow CHF developed by correlating the experimental cross-flow CHF data is also presented in this chapter including the comparison of the accuracies of both prediction methods. The experimental cross-flow CHF data analyzed in Chapter 7 are illustrated in Appendix A. In Chapter 8, the discussion includes the parameter range and limitation of the semi-empirical model, correction factor method and the empirical correlation, and the conclusion contains the recommendation of the prediction method for cross-flow CHF.

Chapter 2

Literature Survey of Experimental Observations for CHF

2.1 Pool Boiling from A Flat Horizontal Plate

Pool boiling is defined as boiling from a heated surface submerged in a large volume of stagnant liquid which may be at or below its boiling point. The results of investigations into heat transfer rates in pool boiling are usually plotted on a graph of surface heat flux against heater wall temperature (T_w) or wall superheat ($T_w - T_{sat}$). For example, a curve for the boiling of water at atmospheric pressure is shown in Figure 2.1. The present study concentrates its interest on the CHF point, i.e., point D in Fig. 2.1, including the CHF occurring in flow boiling forced convection.

It is now generally accepted that the critical heat flux in pool boiling occurs as a result of a hydrodynamic flow pattern transition close to the heating surface. The mechanism is one in which insufficient liquid is able to reach the heating surface due to the rate at which vapor is leaving the surface. Zuber's (1958) model is most widely accepted, which predicts the CHF for pool boiling on an infinite, upward-facing, horizontal, and flat plate. Zuber idealized the flow pattern as a square array of vapor jets leaving the heated surface with a spacing corresponding to the most rapidly growing Taylor instability wavelength λ_c . The CHF therefore occurs when the vapor-liquid interface of the escape passage becomes unstable due to

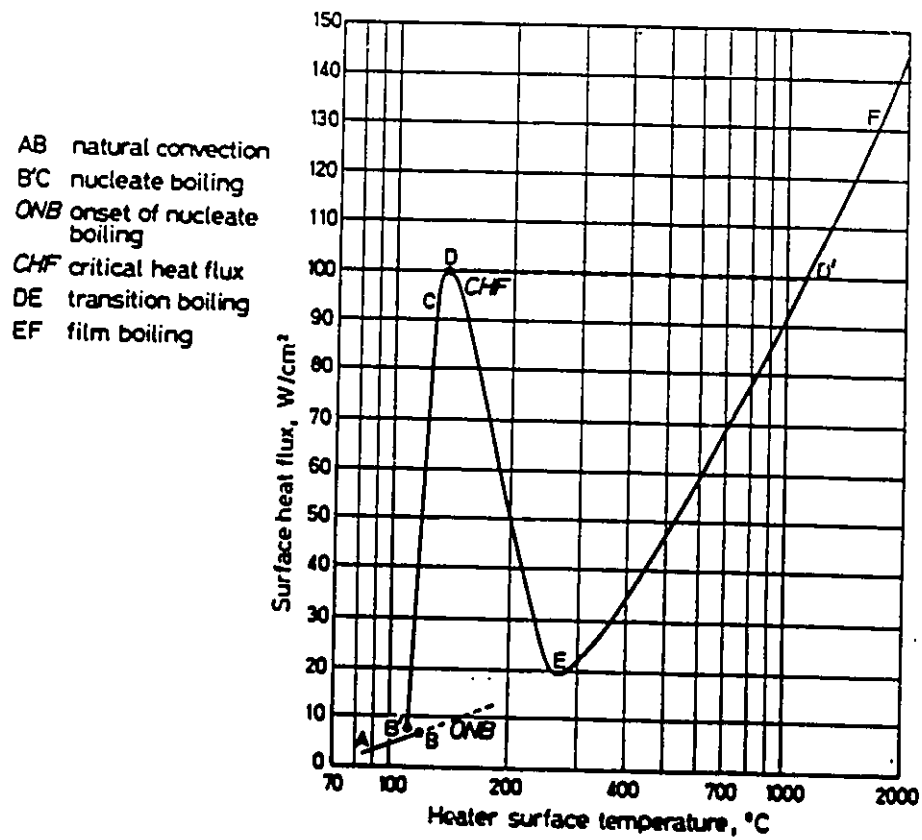


Figure 2.1: Pool boiling curve for water at atmospheric pressure

Helmholtz instability. However, the magnitude of the Helmholtz critical wavelength λ_H is unknown, so it is determined by assuming $\lambda_H = \pi\lambda_c/2$. In addition, a few approximations such as $3/\sqrt{2\pi} \cong 1$ are made to give the CHF as

$$q_{max} = \frac{\pi}{24} h_{fg} \sqrt{\rho_g} \sqrt{\sigma g (\rho_f - \rho_g)} \quad (2.1)$$

Lienhard et al. (1973) continued Zuber's investigation with the same test conditions: pool boiling of an infinite, upward-facing, horizontal, and flat plate at atmospheric pressure. The CHF in this case was found 14 percent higher than Zuber's prediction. Lienhard and Hasan (1979a) extended the usefulness of the *mechanical energy stability criterion* adapted by Lienhard and Eichhorn (1976) to replace the need for a knowledge of Helmholtz wavelength, which is needed in the hydrodynamic descriptions of pool boiling. The geometries investigated include the infinite horizontal flat plate, the cylinder and the sphere. The result for an infinite horizontal flat plate agrees with Zuber's assumed value of the jet radius.

2.2 Pool Boiling from A Horizontal Cylinder

The geometry of a horizontal cylinder is of the most interest for present study. Yet most correlations and models of predicting CHF for a horizontal cylinder are adapted from the results which are used for flat plate pool boiling. Lienhard and Hasan (1979b) and Katto and Haramura (1983b) have derived very typical approaches for this geometry.

2.3 Axial-flow Boiling

CHF observations for axial flow have been conducted by numerous investigators. Many correlations have been used to predict the CHF for different limited ranges of applications. Groeneveld and Snoek (1982) concluded the commonly observed asymptotic trends, which can be attributed by :

1. CHF approaches the pool boiling CHF at low flows.
2. CHF approaches zero for quality and void fraction approaching unity.

3. CHF cannot be less than zero, not even for negative flow(down-flow).
4. CHF reaches a maximum within the approximate pressure range of 3-4 MPa.
5. CHF approaches zero when pressure approaches critical pressure.

Effects such as flux distribution, subcooling, or spacers which are correlated as parametric trends by different observations have been tabulated by Groeneveld and Snoek (1982) and illustrated in Table 2.1.

2.4 Cross-flow Boiling

Various investigations of the heat transfer rate for two-phase flow boiling across a tube or bundle have been undertaken with various specific conditions. Some of the experimentally observed trends will be discussed in the survey below.

Beecher (1948) investigated natural convection, saturated boiling of water at atmospheric pressure from a 0.61 *mm* platinum wire, and forced convection from a 1.2 *mm* OD. stainless-steel tube. The experiment showed that the CHF was inversely proportional to the 0.15 power of diameter over the entire velocity range from 0.36 to 3 m/s. Vliet and Leppert (1964a and b) also studied forced convection boiling from wires and tubes from 0.25 to 4.8 *mm* OD. placed normal to a stream of slightly subcooled water at velocities from 0.36 to 2.9 m/s, as well as the measurement for water at atmospheric pressure over a range of subcooling from 1.7 to 55.6 deg. C and velocity ranging from 0.15 to 3.4 m/s. They found that the CHF for nearly saturated water at atmospheric pressure was proportional to the 0.5 power of the fluid velocity and -0.15 power of the diameter.

Kezios and Lo (1958) investigated the effects of velocity and large values of subcooling on the CHF for water flowing horizontally and normal to 3.2 *mm* rods, and stated that the CHF was proportional to the 0.8 power of subcooling and 0.2 power of the mass flux. McKee and Bell (1969) showed by experiment that when the diameter of the cylinder is increased beyond a certain value under the condition of fixed bulk liquid velocity, a rapid reduction of CHF takes place. The test conditions for their measurement were: 6.4-18 *mm* OD. heater diameter, 1.7 to 7.8 deg. C subcooled water for 124.2 kPa abs. and approach velocities of 1.0 and 1.66 m/s.

Table 2.1: Parametric trends for CHF in axial flow

Effect	Correction factor	References	Comments
Spacing device	$\frac{\text{CHF with grids}}{\text{CHF without grids}} = 1 + 0.1 \exp \left(- \frac{L_{sp}}{40D} \right)$ $\frac{\text{CHF with grids}}{\text{CHF without grids}} = 1 + \phi \times \left(\frac{G}{1 - \phi} \right)^{0.6} D^{0.28} \exp \left(- \frac{\phi L_{sp}}{D \eta} \right)$ ϕ, ϕ are spacer and flow regime constants X = pressure loss coefficient of spacer L_{sp} = spacer pitch	Saolin and Polyakov (1978) Tong (1975)	Constants 0.1 and 40 must be dependent on spacer geometry. Based on experiments using PWR-type bundles and grids. M.D. increases in CHF of over 200% have been observed for short spacer pitch. Spacers also improve post CHF heat transfer
Diameter	$\frac{\text{CHF}(D)}{\text{CHF}(D_0)} = \left(\frac{D}{D_0} \right)^{1/3}$ $\frac{\text{CHF}(D)}{\text{CHF}(D_0)} = 1 + \frac{q_0}{\text{CHF}(D_0 \approx 0.008)} \left(\frac{0.008}{D} \right)^{1/3} - 1$ $\frac{\text{CHF}(D)}{\text{CHF}(D_0)} = \left(\frac{D_0}{D} \right)^n \quad n = 0.4, \text{ if } D > 10 \text{ mm}$ $\quad \quad \quad n = 0.6, \text{ if } D < 10 \text{ mm}$ $\frac{\text{CHF}(D)}{\text{CHF}(D_0)} = f(D)$	Doroshchuk et al. (1975) Doroshchuk and Lanteman (1970) Blasi et al. (1987) Decker (1987)	Valid for $4 < D < 16 \text{ mm}$, for $\Delta T_{\text{sub}} < 75^\circ\text{C}$ and burnout of the first kind (not for very high qualities). Valid for positive qualities only; $q_0 = f(P)$. From Blasi's correlation. Ranges: $D = 0.003\text{--}0.0375 \text{ m}$, $P = 270\text{--}14,000 \text{ kPa}$, $G = 100\text{--}8000 \text{ kg m}^{-2} \text{ s}^{-1}$, $\frac{1}{1 + \rho_l/\rho_g} < X < 1.0$ From Decker's correlation. Ranges: $D = 0.004\text{--}0.025 \text{ m}$, $P = 270\text{--}10,000 \text{ kPa}$, $G = 120\text{--}3400 \text{ kg m}^{-2} \text{ s}^{-1}$, $X = 0\text{--}0.3$
Flow	$\frac{\text{CHF}(G)}{\text{CHF}(G_0)} = \left(\frac{G}{G_0} \right)^{1/3}$ $\frac{\text{CHF}(G)}{\text{CHF}(G_0)} = \left(\frac{G}{1000 (\text{kg m}^{-2} \text{ s}^{-1})} \right)^{0.68} \frac{P}{P_0} - 1.2 \quad X = 0.3$	Doroshchuk et al. (1975) Decker (1987) Griffl and Bonilla (1983)	Derived from Doroshchuk's correlation. Large-tube data base. Ranges: $D = 0.004\text{--}0.016 \text{ m}$, $P = 290\text{--}13,600 \text{ kPa}$, $G < 2000 \text{ kg m}^{-2} \text{ s}^{-1}$, $\Delta T_{\text{sub}} < 50^\circ\text{C}$ From Decker's correlation. Large-tube data base. Ranges: $X = 0\text{--}0.6$, $G = 120\text{--}3400 \text{ kg m}^{-2} \text{ s}^{-1}$, $P = 270\text{--}10,000 \text{ kPa}$ Tube data covered: $308 < G < 10,000 \text{ kg m}^{-2} \text{ s}^{-1}$, $3.5 < P < 10.3 \text{ MPa}$, and negative X ($0 < \Delta T_{\text{sub}} < 65^\circ\text{C}$).

Table 2.1 (cont'd)

Effect	Correction factor	References	Comments
Pressure	$\frac{CHF(P)}{CHF(P_0)} = \left(\frac{1 - P/P_{cr}}{1 - P_0/P_{cr}} \right)^{0.1} \frac{CHF_{PB}(P)}{CHF_{PB}(P_0)}$ where $CHF_{PB} = 0.14 h_{fg} \rho_l^{1/2} \left[\log(\rho_l / \rho_g) \right]^{1/4}$ $\frac{CHF(P)}{CHF(P_0)} = \left(\frac{P_0}{P} \right)^{0.3234}$ $\frac{CHF(P)}{CHF(P_0)} = \frac{CHF_{PB}(P)}{CHF_{PB}(P_0)}$	Tolubinsky et al. (1977) Levitan and Lantieri (1977) Candle and Bengtson (1978) Griffith et al. (1977)	From Levitan's correlation based on a large number of annuli data. Ranges: $P = 5000-20,000$ kPa, $X < 0.3$ $G = 230-3000$ kg m⁻² s⁻¹ From RELAP 4-MOD 7 correlation. Based on large-bundle data base. Ranges: $P = 690-15,200$ kPa, $X = 0.1-1.0$ $G = 100-4100$ kg m⁻² s⁻¹ From Griffith-Zuber correlation for some void fraction. Valid for low flows only.
Adjacent Unheated Surface	$\frac{CHF_{adj \text{ unheated surf}}}{CHF_{ref}} = f(D_{he}, D_{hu}, X, G, P)$	Tong (1972a)	Adjacent unheated surface robs liquid that would otherwise be available for cooling of the heated surface
Void and transients (low flow)	$\frac{CHF}{CHF_{PB}} = (1 - \alpha)$	Griffith et al. (1977)	At low flow (also during CCF) flow quality becomes meaningless; α becomes a much better parameter to describe the vapor presence.
Void and transients (blowdown)	$\frac{CHF}{CHF_{W-3}(X=0)} = 1.33(0.98 - \alpha)^{1/2} + \frac{q_{\text{atom}}}{CHF_{W-3}(X=0)}$	Hsu and Becker (1977)	From Hsu-Becker correlation. Increases CHF due to transient effect. Reference CHF (CHF at $X = 0$, W-3 correlation) is fairly arbitrary. Recommended for high pressure, blowdown transients.
Axial flux distribution	$\frac{CHF_{\text{non-UM}}}{CHF_{\text{UM}}} = \bar{r} = f(\Delta P_D, G, X)$ CHF occurs when $q_{0LA} > CHF_{0H}$	Tong (1972a)	Upstream APD has a weak effect at highly subcooled conditions (local conditions dominate) but becomes important at higher enthalpies.
Flux spike	$\frac{CHF_{\text{with spike}}}{CHF_{\text{ref}}} = f(G, X, \frac{q_{sp}}{q_{av}}, \frac{L_{sp}}{D})$	Groeneveld (1981a)	In the annular flow regime, use boiling length average heat flux instead of local heat flux.
		Groeneveld (1975)	Short flux spikes can significantly affect CHF in the subcooled region but usually have negligible effect in annular flow regime.

Table 2.1 (cont'd)

Local enthalpy ($x < 0$)	$\frac{\text{CHF}(\Delta T_{\text{sub}})}{\text{CHF}(\Delta T_{\text{sub}} = 0)} = 1 + B \Delta T_{\text{sub}}$ $B = 0.1 \left(\frac{p_t}{p_g} \right)^{0.75} \left(\frac{G}{h_{fg}} \right)$	Ivey and Morris (1962)	Ivey's correction is based only on pool boiling results and agrees with water data at $p < 1800$ kPa.
	$\frac{P(\text{MPa})}{B(\frac{1}{\text{G}^2})} \left \begin{array}{c c c c c} 20 & 35-70 & 100 & 130 & \\ \hline 0.008 & 0.035 & 0.033 & 0.030 & \end{array} \right $ $B = 0.1 \left(\frac{p_t}{p_g} \right)^{0.75} \left(\frac{G}{h_{fg}} \right)$	Povarnin (see Tong, 1972a)	Povarnin's correction is based on small-diameter tubing (2-3 mm) and $p = 2000 - 20,100$ kPa.
Local enthalpy ($x > 0$)	$\frac{\text{CHF}(x)}{\text{CHF}(x_0)} = \frac{x_{\text{max}} - x}{x_{\text{max}} - x_0} \quad x_{\text{max}} = 1.0$ For very low G : $x_{\text{max}} = 1 - \frac{4 \text{ MVR}}{G \text{ Dh}_y}$ $\text{MVR} = \text{minimum wetting rate}$ $\frac{\text{CHF}(x)}{\text{CHF}(x = 0)} = e^{-1.5x \left(\frac{G}{1000} \right)^{-1.2x}}$	Groeneveld (1981b)	Only for high-quality. x_0 is highest quality where CHF is known. MVR value reported by Norman and McIntyre (1960).
	$\frac{\text{CHF}(x)}{\text{CHF}(x = 0)} = e^{-1.5x \left(\frac{G}{1000} \right)^{-1.2x}}$	Doroshchuk et al. (1975)	From Doroshchuk's correlation. Ranges: $p = 2900-15,600$ kPa, $\Delta T_{\text{sub}} < 50^\circ\text{C}$, $G < 2000 \text{ kg m}^{-2} \text{ s}^{-1}$, $D = 0.004-0.016$ m. Durnout of the first kind only.
Cross section geometry	$\frac{\text{CHF}_{\text{annulus}}}{\text{CHF}_{\text{tube}} (D = 8 \text{ mm})} = K_x K_p K_D$ $\frac{P(\text{MPa})}{K_p} \left \begin{array}{c c} 1 & 1.3/P/9.8 \\ \hline 1 & 1.3/P/9.8 \end{array} \right \frac{\text{Dh}_y(\text{mm})}{K_D} \left \begin{array}{c c} 0.8 & 0.2 \\ \hline 0.8 & 0.2 \end{array} \right \left(\frac{\text{Dh}_y}{D_t} \right)^{0.2} \left(\frac{64}{D_t \text{ Dh}_y} \right)^{0.2}$ $K_x = \min(1, e^{-2x})$ CHF bundle = CHF annulus having equivalent D_t , D_o equiv. $D_t = d$ (= rod diameter) equiv. $D_o = [d(d + \text{Dh}_{\text{ext}})]^{1/2}$, $\text{Dh}_{\text{ext}} = \frac{4A}{\pi d} \frac{q_{\text{max}}}{q_{\text{avg}}}$	Levitan and Lantsean (1977)	Based on large number of annuli data and on tube CHF correlation from Doroshchuk et al. (1975) which was equally based on a large-tube data bank.
		Barnett (1968)	Barnett obtained excellent agreement using the equivalent annulus and his annulus CHF correlation.

*For definition and units of all symbols, see Nomenclature.

Broussard and Westwater (1985) studied the diameter and velocity effects of CHF for cross flow boiling by conducting the experiment with Freon-113 fluid and horizontal copper tubes of diameter 1.59-7.94 *mm* for pool boiling and 3.18-12.7 *mm* for flow boiling. The test fluid at atmospheric pressure flowed upward, normal to the heaters, at approach velocities of 2.4, 4.0 and 6.0 m/s. The experiment showed that, at a fixed velocity and wall temperature, smaller tubes resulted in higher heat flux values, except at low wall temperatures. The CHF was proportional to the 0.44 power of velocity and inversely proportional to the 0.28 power of diameter. Yilmaz and Westwater (1980) also made measurements of heat transfer to Freon-113 at near atmospheric pressure, boiling outside a 6.5 *mm* diameter horizontally steam-heated copper tube. Tests included pool boiling and also forced flow vertically upward at velocities of 2.4, 4.0 and 6.8 m/s. The CHF was proportional to the square root of velocity.

Hasan et al. (1982), Cumo et al. (1980) and Kezios et al. (1961) studied the CHF of cross flow by conducting the measurements of flow normal to the bundle. Hasan et al. (1982) observed the burn-out of horizontally heated cylinders in up-flow fluids, isopropanol and methanol, adjacent to one or two parallel unheated cylinders. The static pressure at the elevation of the rod bundle varied between 101 and 104 kPa and the diameter varied from 0.51 to 1.65 *mm*. They concluded that a parallel neutral cylinder, in the neighborhood of a cylindrical heater, has only a minor influence on CHF unless it is directly upstream with its centerline less than 4 diameters away from the heater. Cumo et al. (1980) investigated the influence of quality on CHF in flow boiling, employing a 9-tube bundle with cross flow of Freon-12. The diameter of each tube of the 9-tube bundle was 13.6 *mm* and the pressure was 7 times higher than the atmospheric pressure. This test condition was much different from those of other investigations mentioned above. Rather surprisingly, Cumo et al. observed that the CHF rises with quality. Kezios et al. (1961) reported the critical heat flux for a heated 3.2 *mm* diameter rod in a matrix of unheated rods. They concluded that critical heat flux in arrays is higher than in single rods, but there is a question of the proper characteristic velocity to use for comparison.

Chapter 3

Prediction Methods for CHF in Cross Flow

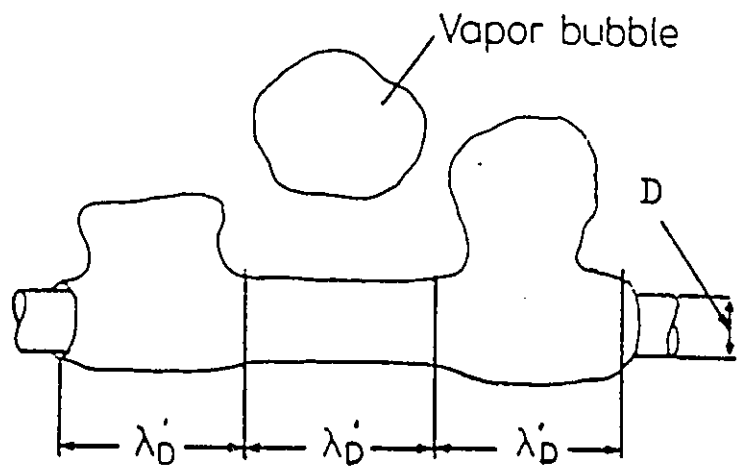
3.1 Analytical Models

3.1.1 Katto and Haramura's model (1983)

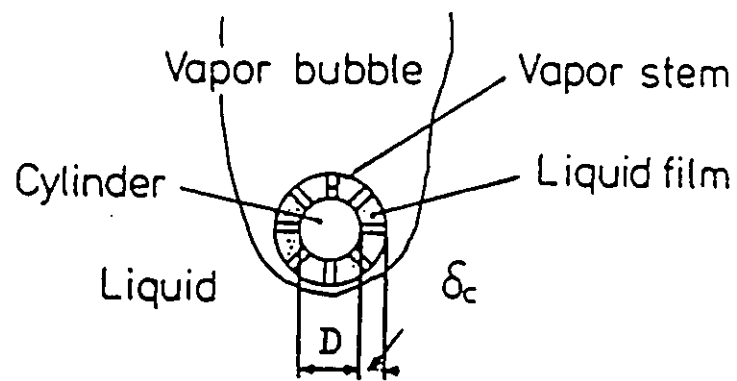
Katto and Haramura (1983a, 1983b) presented a hydrodynamic model of CHF applicable to both pool and forced convection boiling on submerged bodies in saturated liquid. The geometries of submerged bodies included a horizontal and infinitely flat plate, a horizontal and infinitely long cylinder, an upward-facing and horizontally small disk, a flat plate parallel to the flow, and a cylinder normal to the flow. In Katto and Haramura's hydrodynamic model (1983a, 1983b), for liquid flowing upward and normal to the cylinder at low velocities a flow pattern as illustrated in Figure 3.1 appears: a bubble-like flow, where the vapor generated at the cylinder wall forms a massive vapor bubble at approximately equal intervals. Each bubble departs from the cylinder when it has grown sufficiently. Thus, CHF can be determined from the energy balance at the interface of the bubble and liquid film around the cylinder.

In Fig. 3.1a, the λ'_D was given as

$$\lambda'_D = 2\sqrt{3}\pi \sqrt{\frac{\sigma}{g(\rho_f - \rho_g)} \frac{1}{1 + \frac{2\sigma}{D^2 g(\rho_f - \rho_g)}}} \quad (3.1)$$



(a)



(b)

Figure 3.1: Pool boiling at high heat fluxes on a horizontal cylinder

which is the most susceptible (or dangerous) wavelength of Taylor instability and has been given by Lienhard and Wong (1964) for a horizontal, cylindrical vapor-liquid interface in a stationary bulk liquid. Fig. 3.1b shows a liquid film wetting the cylinder surface, and surrounded by a vapor bubble in which there are many columnar vapor stems in the liquid film when the film is sufficiently thick, so that the heat transferred from the cylindrical wall is absorbed by latent heat at the part of the vapor-liquid interface of the vapor stem near the heated surface. When the thickness of liquid film reduces considerably, evaporation takes place at the outer surface of the liquid film. Therefore, the volumetric growth rate v of a vapor bubble surrounding the cylinder via the heat balance is

$$v = \frac{\pi D \lambda'_D \ddot{q}}{\rho_g h_{fg}} \quad (3.2)$$

The hovering period of the bubble attached to the solid wall is assumed as a mean value of τ_d . Haramura and Katto (1983a) postulated that the CHF appears at the end of a hovering period when the liquid film evaporates out and is not fed with liquid from the bulk region during the hovering period, for which the energy balance is given by

$$\tau_d \ddot{q} A_h = \rho_f \delta_c \left(A_h - A_v + A_h \frac{\delta_c}{D} \right) h_{fg} \quad (3.3)$$

where A_h is the area of the heated surface, A_v the cross-sectional area of the vapor stems, and δ_c the initial thickness of the liquid film. The δ_c is also the critical thickness of a liquid film capable of stabilizing itself on a solid wall based on the Helmholtz instability of the vapor-liquid interface of vapor stems and is given by

$$\delta_c = \frac{\pi}{2} \sigma \frac{\rho_f + \rho_g}{\rho_f \rho_g} \left(\frac{A_v}{A_h} \right)^2 \left(\frac{\rho_g h_{fg}}{\ddot{q}} \right)^2 \quad (3.4)$$

where A_v/A_h is a weak function of ρ_g/ρ_f and yields $A_v/A_h = 0.0584(\rho_g/\rho_f)^{0.2}$.

To solve for τ_d , Katto and Haramura (1983b) postulated a theoretical analysis of the bubble behavior such as illustrated in Fig. 3.2. In Fig. 3.2, a spherical bubble, being nourished with vapor from a fixed point P, is moving upward due to the buoyant force. For this bubble, the equation of motion is written as

$$\frac{d}{dt} \left[\left(\frac{11}{16} \rho_f + \rho_g \right) V \left(\frac{ds}{dt} - U_\infty \right) \right] = (\rho_f - \rho_g) V g \quad (3.5)$$

where 11/16 is the theoretical volumetric ratio of the accompanying liquid to a moving spherical bubble derived by Katto and Yokoya (1976), V the bubble volume

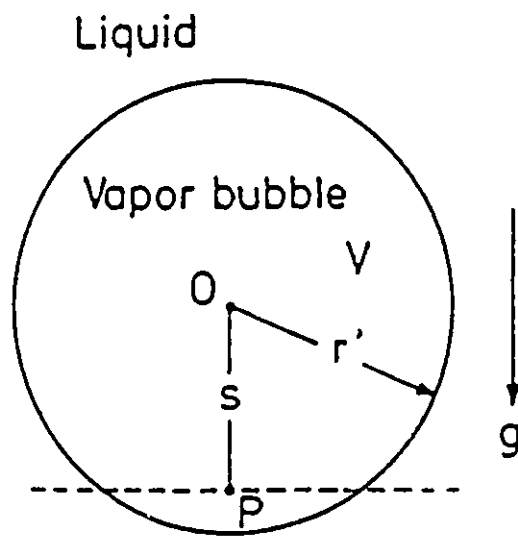


Figure 3.2: A model of a vapor bubble growing through the feed of vapor as well as rising due to buoyancy

at the time t after the onset of bubble growth, s the height of the bubble center measured from the fixed point P, and U_∞ the vertical upward velocity of bulk liquid. Note that $ds/dt - U_\infty$ is the relative velocity of the bubble center to the bulk liquid.

In equation (3.5), V is evaluated by

$$V = vt \quad (3.6)$$

Thus, solving equation (3.5) together with equation (3.6) for $s(t)$, and taking into account the initial condition $s = 0$ at $t = 0$ as well as the departure condition $s = r' = [3V/(4\pi)]^{1/3}$ at $t = \tau_d$, yields

$$\tau_d = \left(\frac{3}{4\pi}\right)^{1/5} \left[\frac{4(\xi\rho_f + \rho_g)}{g(\rho_f - \rho_g)}\right]^{3/5} v^{1/5} \left[1 + \frac{4(\xi\rho_f + \rho_g)U_\infty}{\tau_d g(\rho_f - \rho_g)}\right]^{3/5} \quad (3.7)$$

where

$$\xi = 11/16$$

Now, ignoring A_v/A_h as compared with unity in equation (3.3), since $A_v/A_h < 0.0584$, and solving equations (3.1)-(3.4) and (3.7) for the critical heat flux $\bar{q}$, and finally writing the $\bar{q}$ as q_{co} results in

$$\begin{aligned} \frac{q_{co}}{q_{co,z}} = & \left(\frac{\sqrt{3}}{R'}\right)^{1/16} \left[1 - \frac{1}{2R'^2}\right]^{1/32} \left[1 + 0.156 \left(\frac{\rho_g}{\rho_f}\right)^{0.4} \left(1 + \frac{\rho_g}{\rho_f}\right) \frac{1}{R'} \frac{1}{(q_{co}/q_{co,z})^2}\right]^{5/16} \\ & \left[1 + 1.68 \frac{\frac{11}{16} + \frac{\rho_g}{\rho_f}}{\left(\frac{\rho_g}{\rho_f}\right)^{0.4} \left(1 + \frac{\rho_g}{\rho_f}\right)} \frac{\left(\frac{q_{co}}{q_{co,z}}\right)^3}{1 + 0.156 \left(\frac{\rho_g}{\rho_f}\right)^{0.4} \left(1 + \frac{\rho_g}{\rho_f}\right) \frac{1}{R'} \frac{1}{(q_{co}/q_{co,z})^2}} u'\right]^{1/16} \end{aligned} \quad (3.8)$$

where

$$R' = \frac{D/2}{\sqrt{\frac{\sigma}{g(\rho_f - \rho_g)}}} \quad (3.9)$$

$$u' = \frac{U_\infty}{\left[\frac{\sigma g(\rho_f - \rho_g)}{\rho_g^2}\right]^{1/4}} \quad (3.10)$$

where U_∞ is the bulk mean velocity.

In the case of CHF with sheet-like vapor escape flow, if the bulk liquid velocity is sufficiently high, CHF is hardly affected by gravity, and the analysis of CHF in Katto and Haramura's model (1983b) is summarized as follow: *A liquid film flowing over a cylinder wall (see Fig. 3.3) receives saturated liquid constantly at the front stagnation point with δ_c in film thickness from the bulk flow of velocity U_∞ , which*

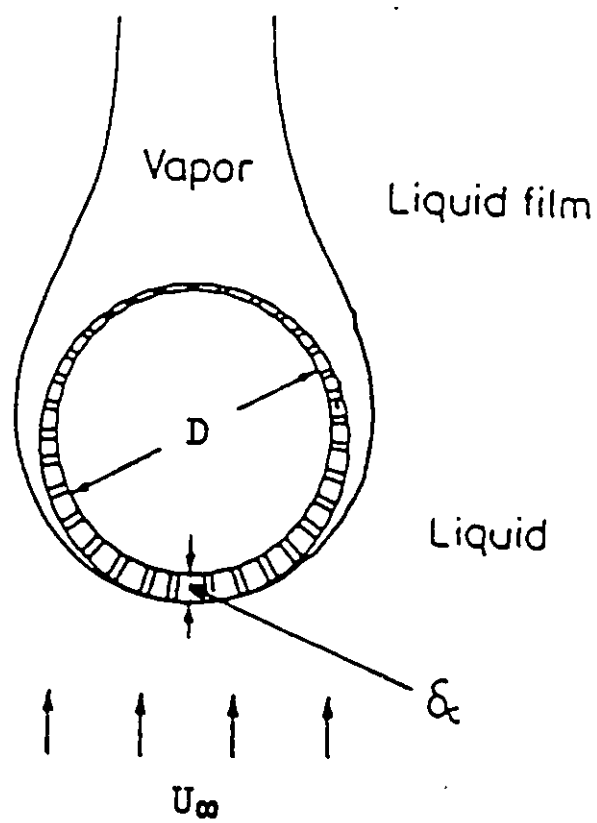


Figure 3.3: Nucleate boiling on a cylinder in a cross flow with a sheet-like escape flow of vapor at high heat fluxes

causes CHF when the liquid film flow dries up at the rear stagnation point, and the heat balance is written as

$$\frac{\ddot{q}\pi D}{2} = \rho_f \delta_c U_\infty h_{fg} \quad (3.11)$$

Solving equations (3.4), (3.11) and $A_v/A_h = 0.0584(\rho_g/\rho_f)^{0.2}$ for the critical heat flux $\ddot{q}$, and writing $\ddot{q}$ as q_{co} results in

$$\frac{q_{co}}{G h_{fg}} = 0.151 \left(\frac{\rho_g}{\rho_f} \right)^{0.467} \left(1 + \frac{\rho_g}{\rho_f} \right)^{1/3} \left(\frac{\sigma \rho_f}{G^2 D} \right)^{1/3} \quad (3.12)$$

where

$$G = U_\infty \rho_f \quad \text{the mass velocity of bulk liquid flow.}$$

3.1.2 Lienhard and Eichhorn's model (1976)

Lienhard et al. (1970, 1973, 1973, 1964) extended Zuber's model to include finite bodies such as illustrated in Fig. 3.4. Furthermore, Lienhard et al. (1979a, 1979b, 1979, 1976) studied the CHF in forced convection boiling (see Fig. 3.5) as well as in pool boiling by replacing hydrodynamic instability analyses with a concept of *mechanical energy stability criterion*, which considers a balance between the kinetic energy of the vapor flow and the surface energy necessary to develop the liquid interface.

Lienhard and Eichhorn (1976) postulated that the heat generated from the heated cylinder can be accounted for by removal of vapor. Therefore, when the CHF occurs, the energy balance should be

$$q_{max} = \rho_g h_{fg} U_\infty \frac{A_j}{A_h} \left(1 + \frac{U_g}{U_\infty} \right) \quad (3.13)$$

where U_g is vapor velocity relative to the velocity of incoming flow, A_j and A_h the cross-sectional area of the vapor jets leaving the cylinder and the area of heated surface. For the case of a heated cylinder, they assumed the width of the vapor sheet is equal to $2\alpha R$. Then

$$\frac{A_j}{A_h} = \frac{2\alpha RL}{2\pi RL} = \frac{\alpha}{\pi}$$

where L is the length of heated cylinder. Therefore the equation (3.13) becomes

$$\frac{\pi q_{max}}{\rho_g h_{fg} U_\infty} = \alpha + \alpha \left(\frac{U_g}{U_\infty} \right) \quad (3.14)$$

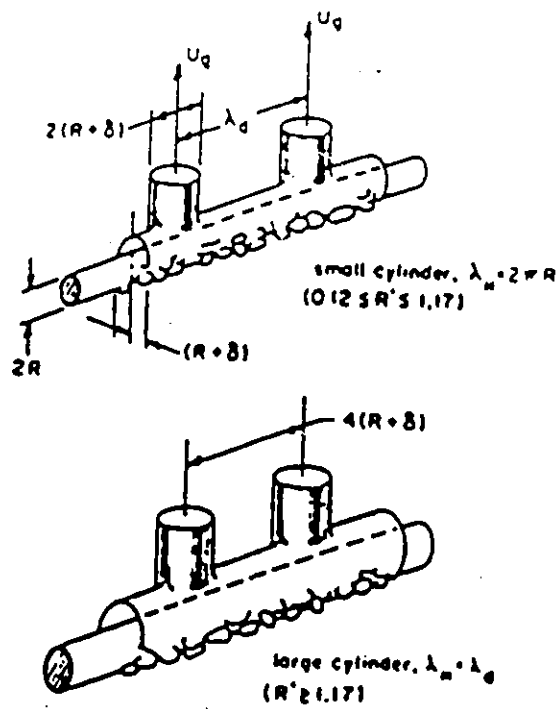


Figure 3.4: Liquid-vapor interface during pool boiling from cylinder

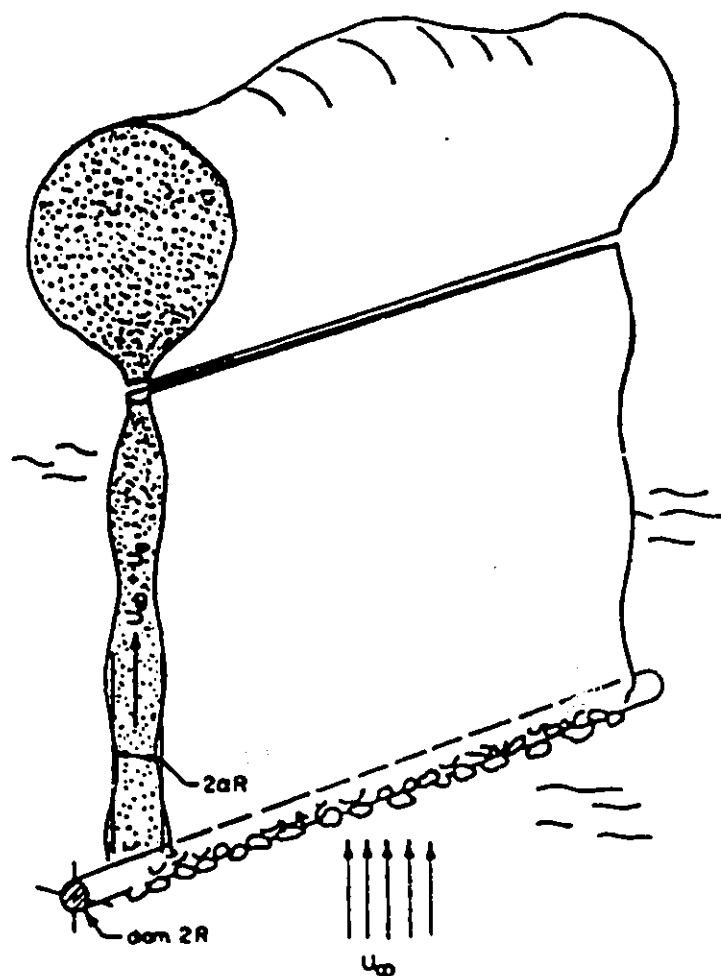


Figure 3.5: Vapor removal by the action of two-dimensional jets before burn-out occurs

When the U_g reaches the Helmholtz unstable velocity, U_H , the vapor sheet will collapse and cause the CHF. At this moment, the U_H is given by

$$\frac{U_H}{U_\infty} = \frac{2\pi}{U_\infty} \left(\frac{3\sigma R}{\rho_g \lambda_H^2} \right)^{1/2} = 2\pi \left(\frac{3}{2} \right)^{1/2} \left(\frac{1}{We_g} \right)^{1/2} \frac{2R}{\lambda_H} \quad (3.15)$$

Equation (3.15) cannot be solved because the term $2R/\lambda_H$ is unknown. Nevertheless Lienhard and Eichhorn used the model of *mechanical energy stability criterion* or *MESC* to resolve this problem, which should be satisfied at burn-out: *A little above the cylinder the liquid is again moving at U_∞ . Hence Fig. 3.6 shows a control volume which moves upward with velocity, U_∞ , relative to the cylinder. The top of the vapor sheet, which is observed to maintain a fixed average distance above the cylinder, will move downward in the control volume at an average speed U_∞ . The detached bubbles have negligible surface area per unit volume in comparison with the sheet. Therefore the control volume loses surface energy at a rate equal to $\rho_g U_g (2\alpha R) U_g^2 / 2$.*

Then according to the mechanical energy balance along the control volume, which yields:

$$-2U_\infty \sigma + \frac{\rho_g 2\alpha R U_g^3}{2} = 0 \quad (3.16)$$

or

$$\frac{U_g}{U_\infty} = \frac{4^{1/3}}{(\alpha We_g)^{1/3}} \quad (3.17)$$

Thus, Lienhard and Eichhorn (1976) simplified this problem to correlate an appropriate value for α instead of calculating the U_H used in equation (3.15). Once the α is known, the CHF can be obtained. Substituting equation (3.17) into equation (3.14), gives

$$\phi = \alpha + \alpha \left(\frac{4}{\alpha We_g} \right)^{1/3} \quad (3.18)$$

where $\phi = \pi q_{max} / (\rho_g h_{fg} U_\infty)$. So, now the rest of the work is to find out the appropriate value of α .

For low velocity of flow, the liquid momentum, $\rho_f U_\infty^2$, does not significantly exceed the vapor momentum, $\rho_g U_g^2$. Hence the vapor can just flow straight up and the liquid cannot curve around to the back of the cylinder. In this case, α is probably close to unity and equation (3.18) yields

$$\phi = 1 + \left(\frac{4}{We_g} \right)^{1/3} \quad (3.19)$$

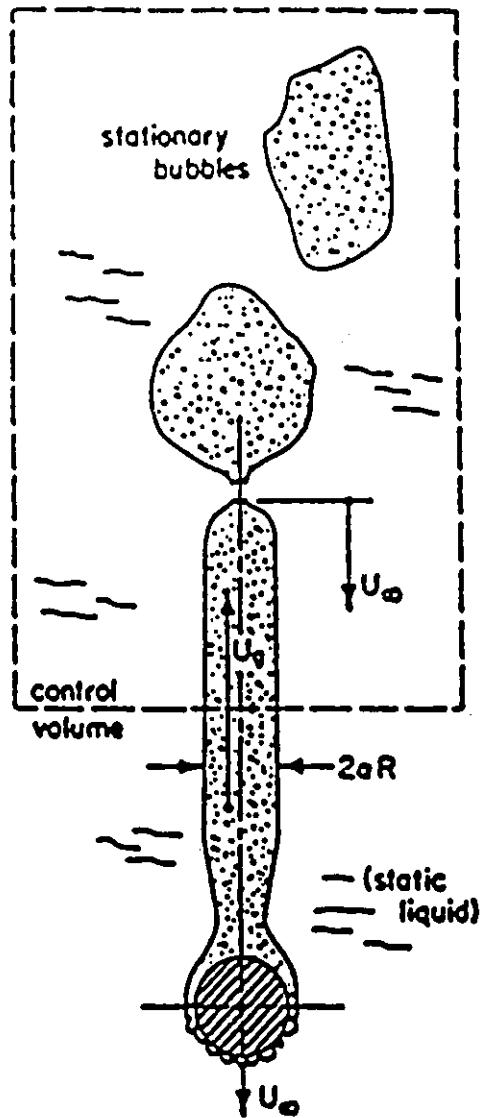


Figure 3.6: Control volume for kinetic/surface energy balance used in Lienhard and Eichhorn's MESC model

for low U_∞ .

In the high flow condition, the liquid flow can be treated as a potential flow and the vapor moves without slip between the liquid and cylinder. Thus the interface will reach a velocity near $2U_\infty$ as illustrated in Fig. 3.7 for the horizontal diametral plane. Because the wake behavior has the advantage of suppressing vapor separation behind the cylinder, Bernoulli's equation can be used to describe both the liquid and vapor flow from station (1) to (2) as indicated in Fig. 3.7. Combining the two Bernoulli equations and the continuity equation shown in Fig. 3.7 to solve for α and using equation (3.17) to eliminate U_g/U_∞ at CHF, gives

$$1 + \left(\frac{4}{\alpha W e_g} \right)^{1/3} = \frac{\left(\frac{3\rho_f}{\rho_g} \right)^{1/2}}{\left[\alpha^2 \left(\frac{R}{\delta} \right)^2 - 1 \right]^{1/2}} \quad (3.20)$$

where δ is the vapor blanket thickness shown in Fig. 3.7. Since the terms 1 and -1 in equation (3.20) are negligible compared to their accompanying terms, equation (3.20) gives

$$\alpha = \text{constant} \left(\frac{\rho_f}{\rho_g} \right)^{3/4} \quad (3.21)$$

where the constant should be determined on the basis of experimental results and yields

$$\alpha = \frac{\left(\frac{\rho_f}{\rho_g} \right)^{3/4}}{169} \quad (3.22)$$

Finally the equation (3.18) results in:

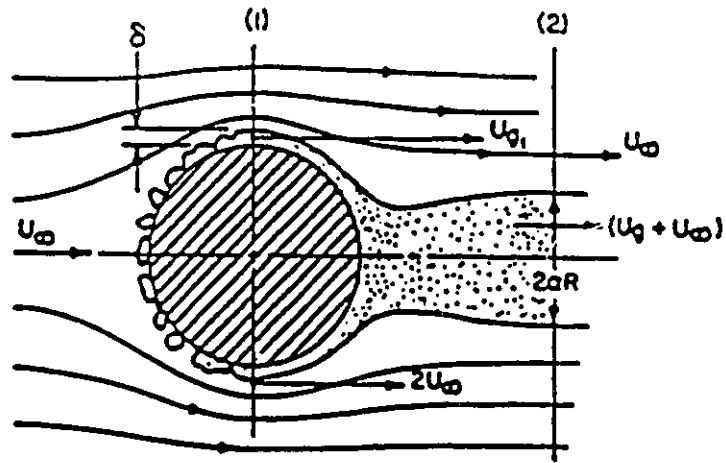
$$\frac{\pi q_{max}}{\rho_g h_{fg} U_\infty} = \frac{1}{169} \left(\frac{\rho_f}{\rho_g} \right)^{3/4} + \frac{1}{19.2} \left(\frac{\rho_f}{\rho_g} \right)^{1/2} \frac{1}{W e_g^{1/3}} \quad (3.23)$$

for high U_∞ .

3.1.3 Vliet and Leppert's correlation (1964a)

Vliet (1962) and Vliet and Leppert (1964a) measured the CHF for a slightly subcooled water stream flowing normal to wires and tubes of 0.25 to 4.8 mm OD. The velocity range of water varies from 0.36 to 2.9 m/s. They, therefore, presented a model for predicting the CHF in a cross flow of saturated liquid:

$$q_{co} = \frac{1.083 \kappa \Delta t_{sat}}{\bar{f} \sqrt{\epsilon}} \sqrt{\frac{U}{D}} \quad (3.24)$$



Bernouli equation:

$$\text{liquid:} \quad \frac{P_1}{\rho_l} + \frac{(2U_\infty)^2}{2} = \frac{U_\infty^2}{2}$$

$$\text{vapor:} \quad \frac{P_1}{\rho_g} + \frac{(U_{g1} + U_\infty)^2}{2} = \frac{(U_g + U_\infty)^2}{2}$$

$$\text{Continuity:} \quad 2\delta(U_{g1} + U_\infty) = 2\alpha R(U_g + U_\infty)$$

Figure 3.7: Potential flow model for the evaluation of α

where κ is the thermal conductivity of liquid, Δt_{sat} the superheat of heated wall, $\bar{f}$ the fraction of heat transferred directly to vapor while bubbles are on the heated surface, ϵ the thermal diffusivity of liquid, U the mean channel velocity corrected by blockage and velocity profile. Equation (3.24) is not practical for use because of the unknown parameter $\bar{f}$.

3.2 Empirical Approach

3.2.1 Yilmaz and Westwater's correlation (1980)

The CHF measurements were undertaken in Freon-113 flowing normal to a horizontally steam-heated copper tube of 6.5 mm diameter at near atmospheric pressure and at velocities of 2.4, 4.0, and 6.8 m/s. The CHF, therefore, was correlated using the dimensionless groups:

$$\frac{q_{max}}{\rho_g h_{fg} U_\infty} = 0.369 We_g^{-0.26} \quad (3.25)$$

where $We_g = \rho_g D U_\infty^2 / \sigma$.

Note that the velocities used in Yilmaz and Westwater's experiments refer to the free stream velocity in the test section rather than that of Vliet and Leppert's (1964a) definition. Rearranging equation (3.25) gives the following expression

$$q_{max} \propto U_\infty^{0.48}$$

3.2.2 Hasan et al.'s correlation (1981)

Hasan et al. (1981) observed the CHF during the up-flow and down-flow of isopropanol and methanol over horizontal cylinders, beyond the influence of gravity. The experiments were conducted with 0.5-1.5 mm OD. wires placed in the up-flow stream at velocity varied from 0.261 to 2.435 m/s, as well as the 0.5-1.63 mm OD. wires placed in the down-flow stream at the velocity range of 0.197-2.57 m/s. They, therefore, presented a correlation to predict the CHF:

$$\frac{\pi q_{max}}{\rho_g h_{fg} |U_\infty|} = 0.000919 \frac{\rho_f}{\rho_g} \left(1 + \frac{16.3}{We_f^{1/3}} \right) \quad (3.26)$$

where We_f is the Weber number, $\rho_f U_\infty^2 D / \sigma$, and the liquid velocity is relative to cylinder instead of that corrected by the blockage factor.

The criterion for the gravity influence, then, was given:

$$G_c = 10$$

where

$$G_c = \frac{U_\infty}{\left(\frac{q\sigma}{\rho_f}\right)^{1/4}} \quad (3.27)$$

For $|G_c| > 10$, the influence of gravity on the CHF was assumed to be inconsequential.

3.2.3 Katto's correlation (1987)

Katto et al. (1987) presented an empirical correlation by employing the results of his CHF experiment and those of others investigators. Using water, Freon-113 and Freon-12, CHF experiments were conducted with a 1 mm OD. diameter uniformly heated cylindrical heater in a forced cross flow of $U_\infty = 0.59 - 7.1$ m/s and a wide range of $\rho_g/\rho_f = 0.000624 - 0.306$. The correlation for CHF, then, was presented:

$$\frac{q_{co}}{Gh_{fg}} = K \left(\frac{\sigma \rho_f}{G^2 D} \right)^m \quad (3.28)$$

where

$$K = 0.00588 + 0.5 \left(\frac{\rho_g}{\rho_f} \right)^{1.11} \quad (3.29)$$

and

$$m = 0.42 \left(\frac{\rho_g}{\rho_f} \right)^{0.0428} \quad (3.30)$$

Note that the velocity used in their experiments was referred to the bulk liquid flow, and the wide range of ρ_g/ρ_f implies that the test conditions might not be at atmospheric pressure.

3.2.4 Cumo et al.'s correlation (1979)

A measurement of CHF for fluid flowing normal to the tube bundles was investigated by Cumo et al. in 1979. A sketch of test section employed in a Freon-12

loop is described in Fig. 3.8. There are different aspects for which actual situations differ from those of other investigations:

1. multi-tube bundles through which boiling develops in a rather complex flow pattern instead of single tube.
2. tube diameters of the order of centimeters, instead of the order of millimeters.
3. two-phase inlet flow, instead of single phase flow.

The specific mass flow, G , is computed as a mean value through the bundle area and grouped around values of 41, 66, 91, and 116 kg/m²s, while the quality x is referred to the inlet value up to 30%. The pressure through the test was constant at the value of 785 kPa, which corresponds to an equivalent pressure, for water, 4426 kPa. The central tube is the only tube heated among the 9-tube bundle. Cumo et al., then, presented a correlation to predict the CHF by employing their experimental results and based on Zuber's correlation:

$$\frac{q_{CHF,FB}}{q_{CHF,PB}} = \left(1 + \frac{C_2 h_{fg} \rho_g G^2}{q_{CHF,PB}^2 \rho_{TP}} \right)^{1/2} \quad (3.31)$$

where

$$\rho_{TP} = \frac{1}{\left(\frac{x}{\rho_g} + \frac{1-x}{\rho_f} \right)} \quad (3.32)$$

$$u_{TP} = G \left(\frac{x}{\rho_g} + \frac{1-x}{\rho_g} \right) \quad (3.33)$$

and $q_{CHF,PB}$ is equal to the q_{max} illustrated in equation (2.1). A regression analysis through the data yields $C_2 = 0.002$.

3.2.5 Hwang and Yao's correlation (1984)

Hwang and Yao (1984) studied the effects of quality, flow velocity and channel blockage ratio on the boiling heat transfer and the CHF by placing a DC heated cylinder of 19.1 mm OD. horizontally in a rectangular channel with vertical upward Freon-113 flow at near atmospheric pressure. The flow velocity is varied in a range from 0.003 to 0.235 m/s, and the upstream flow quality is in a range from 0 to

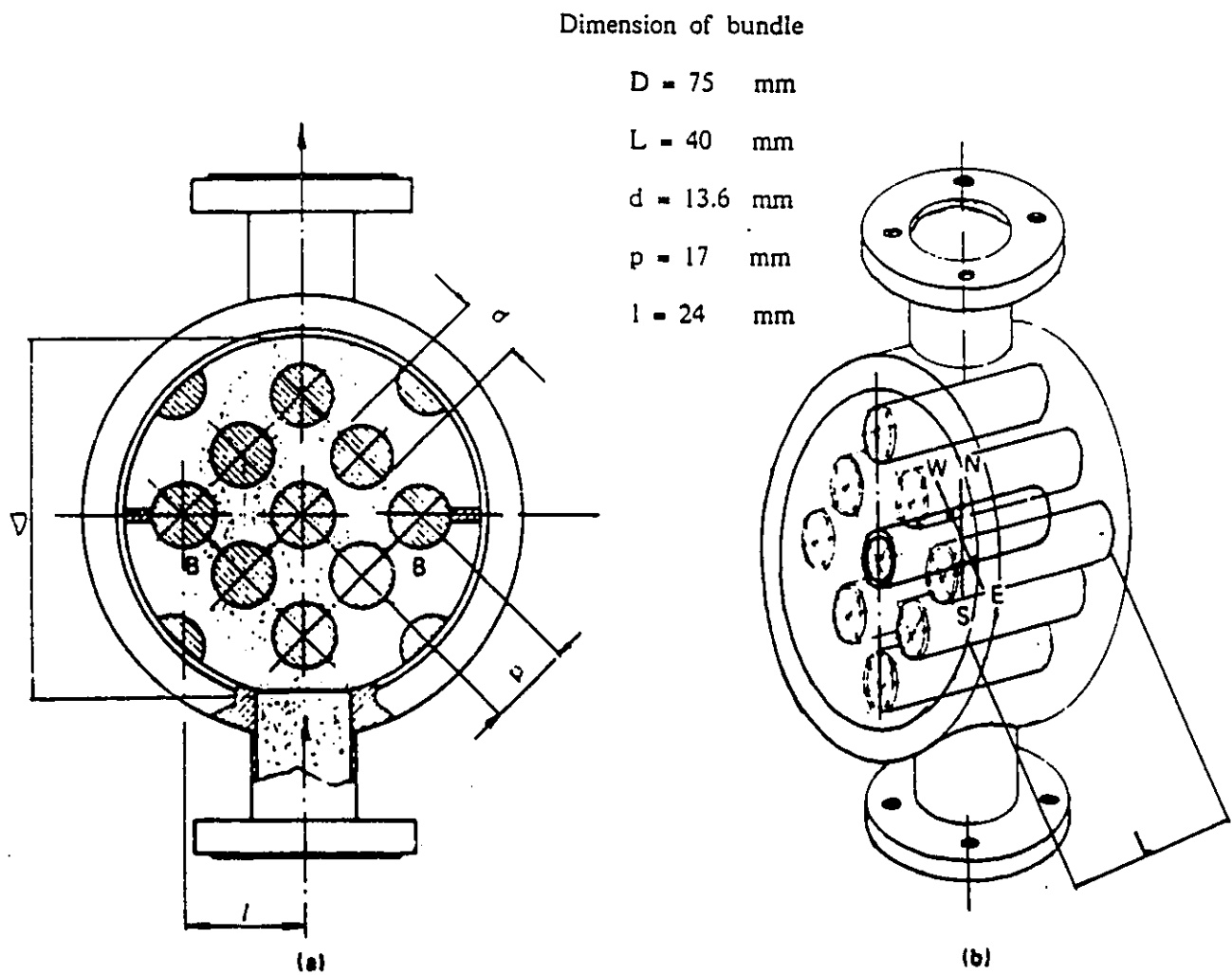


Figure 3.8: The visualized test section of Cumo's experiment

0.143. An empirical correlation for CHF, then, was presented:

$$\frac{q_{max}}{\rho_g h_{fg} U_\infty} = 0.594 W e_g^{-0.476} (1-x)^{0.952} (R^*)^{0.36} \frac{1 - \left(\frac{D}{w}\right)^{0.86}}{2.25 + \left(\frac{\rho_2}{\rho_f}\right)^{0.15}} \quad (3.34)$$

where R^* is equivalent to R' expressed in equation (3.9). Hwang and Yao's empirical CHF correlation shows that the CHF decreases with increasing quality and increasing blockage ratio (D/w), whereas CHF increases weakly with increasing velocity. The main difference between Hwang and Yao's correlation and the other correlations mentioned above is that Hwang and Yao's correlation considered the quality effect as well as the blockage effect in the flow. However, the use of this empirical correlation is limited by the availability for the narrow range of velocity.

Chapter 4

Analysis of CHF for Cross Flow

4.1 Nature of Mass-energy Balance

4.1.1 General

In Chapter 3, Two analytical models of CHF for cross-flow boiling have been discussed. Their applications are restricted in region for the flow at saturated liquid condition. If the flow has quality effect, the energy equation across the heated surface should be reconsidered. In order to solve the energy equation for the flow has quality effect, the assumptions used in Lienhard and Eichhorn's *MESC* model have been adapted rather than Katto and Haramura's hydrodynamic model. The reason to choose Lienhard and Eichhorn's model is that, unlike Katto and Haramura's model, Lienhard and Eichhorn avoided solving the hydrodynamic unstable problem (Taylor instability or Helmholtz instability) and replaced it by studying the mechanical energy balance between the core of vapor sheet and the interface of liquid and vapor phases. The phenomenon of the hydrodynamic instability is not well understood so far.

4.1.2 Flow in saturated liquid condition

Before setting up the control volume for present analysis, some assumptions of the flow conditions have been made:

- (1) vapor removal is an action of a two-dimensional flow pattern
- (2) thickness of vapor sheet is defined as $2\alpha R$, which is identical to Lienhard and Eichhorn's definition.
- (3) saturated fluid flows across the heated cylinder

Then, a stationary control volume to describe these assumptions is illustrated in Fig. 4.1 and the equations of mass and energy balances for satisfying the flow condition are listed as follows:

For mass balance

$$\dot{m}_\infty = \dot{m}_l + \dot{m}_g \quad (4.1)$$

and for energy balance

$$\dot{m}_\infty h_f + \frac{1}{2} \dot{m}_\infty U_\infty^2 + \ddot{q} 2\pi R L = \dot{m}_l h_f + \frac{1}{2} \dot{m}_l U_l^2 + \dot{m}_g h_g + \frac{1}{2} \dot{m}_g (U_\infty + U_g)^2 \quad (4.2)$$

where $\dot{m}_\infty$ mass flow rate of fluid at the inlet of the control volume, i.e,

$$\dot{m}_\infty = \rho_f U_\infty A_\infty$$

$\dot{m}_l$ mass flow rate of liquid at the outlet of the control volume, i.e., $\dot{m}_l = \rho_f U_l A_\infty (1 - \alpha_v)$

$\dot{m}_g$ mass flow rate of vapor at the outlet of the control volume, i.e.,

$\dot{m}_g = \rho_g (U_\infty + U_g) A_\infty \alpha_v$, where A_∞ is the cross-sectional area of the control volume, or WL , α_v the void fraction in the control volume, i.e., $2\alpha R/W$, U_∞ and U_l liquid velocities at the inlet and outlet of the control volume, and U_g the vapor velocity relative to liquid flow.

h_f saturated enthalpy of liquid.

and

h_g saturated enthalpy of vapor, or $h_g = h_f + h_{fg}$.

Substituting the expressions of $\dot{m}_\infty$, $\dot{m}_l$ and $\dot{m}_g$ into equations (4.1) and (4.2), equation (4.1) yields

$$\rho_f U_\infty = \rho_f U_l (1 - \alpha_v) + \rho_g (U_\infty + U_g) \alpha_v \quad (4.3)$$

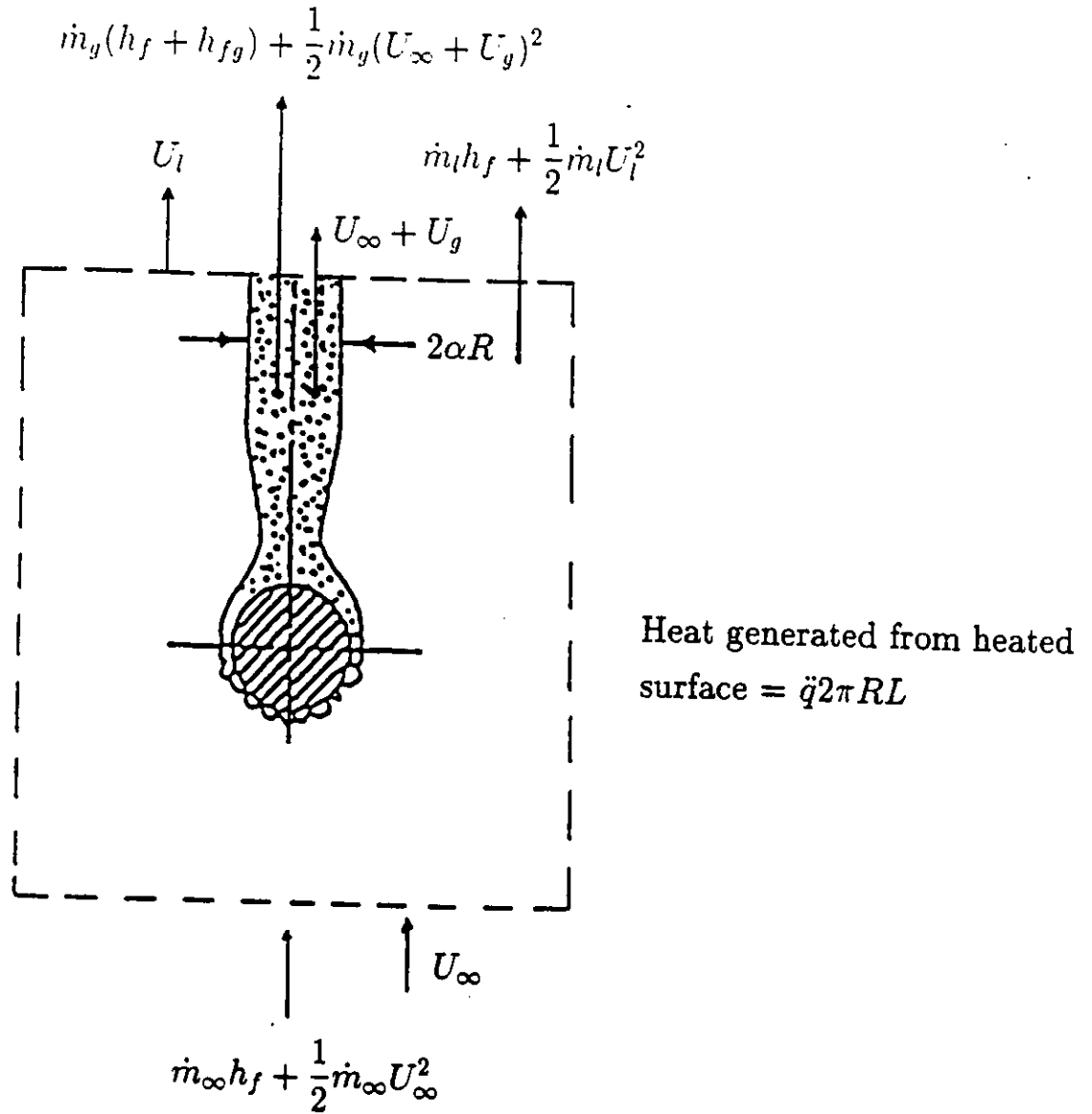


Figure 4.1: Control volume for mass-energy balance at saturated flow condition for present study

and equation (4.2) becomes

$$\begin{aligned} \frac{\ddot{q}2\pi RL}{A_\infty} = & \rho_f U_l (1 - \alpha_v) h_f + \frac{1}{2} \rho_f U_l (1 - \alpha_v) U_l^2 + \rho_g (U_\infty + U_g) \alpha_v h_f \\ & + \rho_g (U_\infty + U_g) \alpha_v h_{fg} + \frac{1}{2} \rho_g (U_\infty + U_g) \alpha_v (U_\infty + U_g)^2 - \rho_f U_\infty h_f \\ & - \frac{1}{2} \rho_f U_\infty U_\infty^2 \end{aligned} \quad (4.4)$$

In the last two terms of equation (4.4), components of $\rho_f U_\infty$ can be substituted by equation (4.3). If A_∞ is replaced by WL , equation (4.4) yields

$$\begin{aligned} \frac{\ddot{q}2\pi RL}{WL} = & \rho_f U_l (1 - \alpha_v) h_f + \frac{1}{2} \rho_f U_l (1 - \alpha_v) U_l^2 + \rho_g (U_\infty + U_g) \alpha_v h_f \\ & + \rho_g (U_\infty + U_g) \alpha_v h_{fg} + \frac{1}{2} \rho_g (U_\infty + U_g) \alpha_v (U_\infty + U_g)^2 - \rho_f U_l (1 - \alpha_v) h_f \\ & - \rho_g (U_\infty + U_g) \alpha_v h_f - \frac{1}{2} [\rho_f U_l (1 - \alpha_v) + \rho_g (U_\infty + U_g) \alpha_v] U_\infty^2 \end{aligned} \quad (4.5)$$

Equation (4.5) can be simplified because some pairs of the terms can be eliminated or combined with each other. It, then, results in

$$\begin{aligned} \frac{\ddot{q}2\pi R}{W} = & \frac{1}{2} \rho_f U_l (1 - \alpha_v) (U_l^2 - U_\infty^2) + \rho_g (U_\infty + U_g) \alpha_v h_{fg} \\ & + \frac{1}{2} \rho_g (U_\infty + U_g) \alpha_v [(U_\infty + U_g)^2 - U_\infty^2] \end{aligned} \quad (4.6)$$

From Lienhard and Eichhorn's model, they assumed the downstream velocity of heated cylinder remains the same as that of the incoming flow. This implies that U_l is equal to U_∞ . In the present case, it is convenient to adopt their assumption so that the first term at the right hand side of equation (4.6) can be cancelled out and yields

$$\frac{\ddot{q}2\pi R}{W \alpha_v} = \rho_g (U_\infty + U_g) h_{fg} + \frac{1}{2} \rho_g (U_\infty + U_g) [U_\infty^2 + 2U_\infty U_g + U_g^2 - U_\infty^2] \quad (4.7)$$

Because $\alpha_v = 2\alpha R/W$, equation (4.7) again can be simplified to

$$\frac{\pi \ddot{q}}{\alpha} = \rho_g h_{fg} (U_\infty + U_g) \left[1 + \frac{1}{2h_{fg}} (2U_\infty U_g + U_g^2) \right] \quad (4.8)$$

Again rearranging the equation (4.8), we finally obtain

$$\frac{\pi \ddot{q}}{\rho_g U_\infty h_{fg}} = \alpha \left(1 + \frac{U_g}{U_\infty} \right) \left[1 + \frac{U_\infty^2}{2h_{fg}} \left(\frac{U_g}{U_\infty} \right) \left(2 + \frac{U_g}{U_\infty} \right) \right]$$

or

$$\phi = \alpha \left(1 + \frac{U_g}{U_\infty} \right) \left[1 + \frac{1}{2} E_c \left(\frac{U_g}{U_\infty} \right) \left(2 + \frac{U_g}{U_\infty} \right) \right] \quad (4.9)$$

where

$$\phi = \frac{\pi \ddot{q}}{\rho_g U_\infty h_{fg}} \quad (4.10)$$

$$E_c = \frac{U_\infty^2}{h_{fg}}$$

E_c is the Eckert number during phase change.

Compared with equation (3.14), equation (4.9) is identical to equation (3.14) except the additional term, $E_c(U_g/U_\infty)(2 + U_g/U_\infty)/2$. From the present analysis with low-flow velocity this term is much smaller in comparison with unity because E_c is near the value at the order of -6. Even in the high-flow region, the influence by this term is still very small. In other words, we obtained a form close to equation (3.14).

4.1.3 Flow other than saturated liquid condition

The use of equations (3.14) and (4.9) are limited by the flow condition for which they can apply. In some cases, the flow at the inlet control volume may not be at saturated condition. Thus, in this section, an attempt to extend the analysis of mass-energy balance for boiling in cross-flow at which the flow is in other than a saturated liquid condition was also conducted. For this case, the third item of the previous assumptions should be modified to

(3) the equilibrium quality, x , of the flow may not be zero.

and an additional assumption is

(4) the variation of equilibrium quality in the bulk liquid across the heated cylinder is negligible.

According to the modified assumptions, some parameters denoted in the previous control volume should be corrected to satisfy the new flow condition as illustrated in Fig. 4.2. Thus the mass and energy balance equations can be rewritten as follows:

For mass balance

$$\dot{m}_\infty = \dot{m}_l + \dot{m}_g \quad (4.11)$$

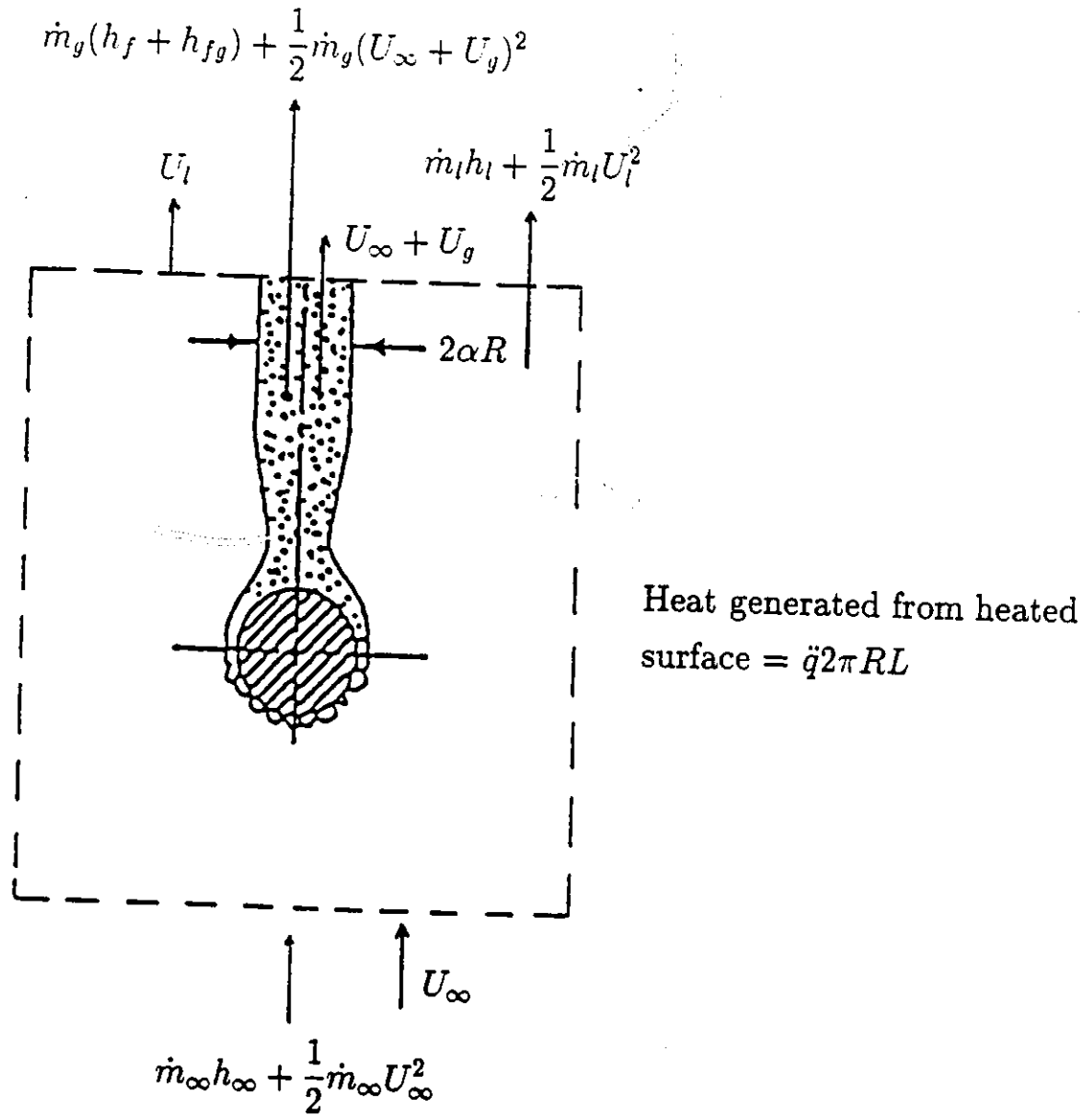


Figure 4.2: Control volume for mass-energy balance at $x \neq 0$ for present study

where

$$\begin{aligned}\dot{m}_\infty &= \bar{\rho} U_\infty A_\infty \\ \dot{m}_l &= \bar{\rho} U_l A_\infty (1 - \alpha_v) \\ \dot{m}_g &= \rho_g (U_\infty + U_g) A_\infty \alpha_v\end{aligned}$$

$\bar{\rho}$ is the fluid density referring to the corresponding quality.

and

ρ_g is the vapor density.

and for the energy balance

$$\dot{m}_\infty h_\infty + \frac{1}{2} \dot{m}_\infty U_\infty^2 + \ddot{q} 2\pi R L = \dot{m}_l h_l + \frac{1}{2} \dot{m}_l U_l^2 + \dot{m}_g h_g + \frac{1}{2} \dot{m}_g (U_\infty + U_g)^2$$

or

$$\ddot{q} 2\pi R L = \dot{m}_l h_l + \frac{1}{2} \dot{m}_l U_l^2 + \dot{m}_g h_g + \frac{1}{2} \dot{m}_g (U_\infty + U_g)^2 - \dot{m}_\infty h_\infty - \frac{1}{2} \dot{m}_\infty U_\infty^2 \quad (4.12)$$

where

$$\begin{aligned}h_l &= h_\infty = h_f + x h_{fg} \\ h_g &= h_f + h_{fg}\end{aligned}$$

Substituting $\dot{m}_\infty$, $\dot{m}_l$ and $\dot{m}_g$ into equations (4.11) and (4.12) with some manipulations for simplification, it yields

$$\bar{\rho} U_\infty = \bar{\rho} U_l (1 - \alpha_v) + \rho_g (U_\infty + U_g) \alpha_v \quad (4.13)$$

and

$$\begin{aligned}\frac{\ddot{q} 2\pi R L}{A_\infty} &= \bar{\rho} U_l (1 - \alpha_v) (h_f + x h_{fg}) + \frac{1}{2} \bar{\rho} U_l (1 - \alpha_v) U_l^2 + \rho_g (U_\infty + U_g) \alpha_v (h_f + h_{fg}) \\ &\quad + \frac{1}{2} \rho_g (U_\infty + U_g) \alpha_v (U_\infty + U_g)^2 - \bar{\rho} U_\infty (h_f + x h_{fg}) - \frac{1}{2} \bar{\rho} U_\infty U_\infty^2\end{aligned} \quad (4.14)$$

Combining equations (4.13) and (4.14) and following the similar manner of manipulation as in saturated liquid conditions, a new expression for CHF at flow conditions with a quality effect can be obtained

$$\phi = \alpha \left(1 + \frac{U_g}{U_\infty} \right) \left[1 + \frac{1}{2} \frac{U_\infty^2}{h_{fg}} \frac{U_g}{U_\infty} \left(2 + \frac{U_g}{U_\infty} \right) - x \right]$$

or

$$\phi = \alpha \left(1 + \frac{U_g}{U_\infty} \right) \left[1 + \frac{1}{2} E_c \frac{U_g}{U_\infty} \left(2 + \frac{U_g}{U_\infty} \right) - x \right] \quad (4.15)$$

Equation (4.15) is very similar to equation (4.9) except the difference of an additional term, x , in equation (4.15). If quality is zero, equation (4.15) is identical to equation (4.9). In other words, equation (4.9) is a special case of equation (4.15).

4.2 Evaluation of U_g/U_∞ and α

The objective of Lienhard and Eichhorn's *MESC* model is to solve the ratio of U_g over U_∞ . Up to now, this model has been upgraded in several versions by later researchers. In present study, only the original version and the hydrodynamic prediction model presented by Ungar and Eichhorn (1988) are discussed.

4.2.1 Lienhard and Eichhorn's *MESC* model

In section 3.1.2, we have discussed the original *MESC* model postulated by Lienhard and Eichhorn (1976). The term U_g/U_∞ can be expressed by equation (3.17)

$$\frac{U_g}{U_\infty} = \left(\frac{4}{\alpha We_g} \right)^{1/3}$$

From the flow condition, We_g can be calculated according to $We_g = D\rho_g U_\infty^2/\sigma$, so the only unknown in equation (3.17) is α . Lienhard and Eichhorn, then, correlated the value of α for various fluids. The expression of empirical α is shown in equation (3.22) and a list of α 's for different fluids is illustrated in Table 4.1. Note that the α is correlated on the basis of experimental results in saturated condition and is valid only for the region of high flow velocity, i.e., two-dimensional flow pattern of vapor. For the low flow velocity, they assumed that α is equal to unity.

4.2.2 Ungar and Eichhorn's hydrodynamic prediction model

Ungar and Eichhorn (1988) conducted experiments to observe the frequency of bubble break-off and the thickness of the vapor sheet for upward cross-flow boiling in order to evaluate the ratio of U_g over U_∞ . They used various fluids heated by DC cylindrical heaters ranging from 0.38 to 2.12 mm in diameter. Their observation showed that the wake thickness may vary with the distance from the heater and bubble break-off from the wake in low speed up-flow is periodic. They, therefore, used α' in equation (3.14) instead of α to distinguish it from that used in Lienhard and Eichhorn's *MESC* model and modified the expression of U_g/U_∞ in equation (3.17).

Table 4.1: Empirical constant α for different test fluids obtained by Lienhard and Eichhorn (1976)

Liquid	ρ_f/ρ_g	α
water	1459.2	1.39
methanol	686.4	0.79
isopropanol	302.8	0.43
Freon-113	169	0.28

From their hydrodynamic study, CHF for upward cross flow is determined by the frequency of bubble break-off, the most rapidly growing wavelength in the wake, the wake thickness and the vapor velocity in the wake at CHF. The bubble break-off frequency is determined from the Strouhal number, which is defined as

$$St = \frac{fD}{U_\infty} = \frac{D}{\lambda} \quad (4.16)$$

where f is the frequency of vapor departure, D the heater diameter, λ the wavelength of the wave in the wake. The Strouhal number is also correlated on the basis of experimental results as

$$St = 0.947(Fr + 0.614)^{-1.5} \quad (4.17)$$

where Fr is the Froude number, $U_\infty/(gD)^{1/2}$. Since Hagerty and Shea (1955) have analyzed the stability of such a thin vapor sheet, their result was used by Kheyran-dish and Lienhard (1985) to determine the most rapidly growing wavelength, or Helmholtz wavelength, λ_H ,

$$\frac{\lambda_H}{D} = 2\pi \frac{U_\infty}{U_g} \sqrt{\frac{3\alpha'}{2We_g}} \quad (4.18)$$

Ungar and Eichhorn's experiments showed that waves move at the same velocity as the liquid. Therefore, at CHF, $f = U_\infty/\lambda_H$ where λ in equation (4.16) is replaced by λ_H . Thus, equation (4.18) yields

$$\frac{U_g}{U_\infty} = 2\pi St \sqrt{\frac{3\alpha''}{2We_g}} \quad (4.19)$$

Ungar and Eichhorn used an effective wake thickness, α'' , in equation (4.19) because the actual wake thickness changes with the distance from the heater. The effective wake thickness is the thickness that determines the most rapidly growing wavelength in the wake. The α'' , then, is correlated and plotted against the Froude number for various fluids on the basis of their experimental results as well as the results obtained by Min (1975), Lienhard and Eichhorn (1976), Hasan et al. (1981), and Vliet and Leppert (1964). Only those data for $G_c \leq 8$ are included in their analysis, where G_c is expressed by equation (3.27). The final form of α'' , then, is correlated as

$$\alpha'' = 0.28Fr^c \quad (4.20)$$

where

$$c = 0.137r^{0.264} \quad (4.21)$$

$$r = \frac{\rho_f}{\rho_g}$$

4.2.3 Application of U_g/U_∞

Since U_g/U_∞ has been obtained by Lienhard and Eichhorn (1976) and Ungar and Eichhorn (1988) respectively, U_g/U_∞ is now ready to be substituted back into equation (3.14). The result obtained by Lienhard and Eichhorn has been shown in equation (3.18) and its alternative form is

$$\phi = \left[1 + \left(\frac{4}{\alpha W e_g} \right)^{1/3} \right] \alpha \quad (4.22)$$

On the other hand, the result derived by Ungar and Eichhorn yields

$$\phi = \left(1 + 2\pi St \sqrt{\frac{3\alpha^*}{2W e_g}} \right) \alpha^* \quad (4.23)$$

In addition, the term of U_g/U_∞ used in equation (4.9) and (4.15), which was obtained from present study, has the same meaning as that used in Lienhard and Eichhorn's *MESC* model and Ungar and Eichhorn's hydrodynamic model. Therefore, equation (4.9) is able to predict CHF for cross flow by employing their derivations of U_g/U_∞ . However, equation (4.15) cannot be used so far because the thickness of the vapor sheet at flow other than saturated condition hasn't been correlated. Thus, more work has to be done on this study. In fact, the measurement of thickness for vapor sheet is just a method to attack this problem for solving U_g/U_∞ . However, the other approach can be attempted as long as the U_g/U_∞ can be derived easily and accurately.

4.3 Derivation of Effective Thickness of Vapor Sheet for The Flow with Quality Effect

Due to the lack of α^* which can be used in the flow at other than saturated liquid condition, efforts to find the α^* for flows with quality effect have been made in the present study. Compared with the α^* derived by Ungar and Eichhorn, the α^* obtained from the present study has an additional independent variable, that is, quality. Recall that, in equation (4.20), α^* is expressed as $\alpha^* = f(Fr)$, where the exponential variable of Fr is a function of density ratio only. However, in the present study α^* is postulated as $\alpha^* = f(Fr, 1 - x)$ and both the exponential

variables of Fr and $(1 - x)$ are also a function of density ratio. Thus, the general expression of α^* is written as

$$\alpha^* = b Fr^c (1 - x)^a \quad (4.24)$$

where a , b and c are empirical constants obtained by a least square method for the individual test fluid.

4.3.1 Calculations of α^* in present study

Instead of equation (3.14), equation (4.15) is used to calculate the values of α^* because of the consideration of the quality effect. However, the term of $\frac{1}{2}E_c(U_g/U_\infty)[2 + (U_g/U_\infty)]$ in equation (4.15) is negligible because E_c results in a very small value in the low flow region. Thus, equation (4.15) can be reduced to

$$\phi = \alpha^* \left(1 + \frac{U_g}{U_\infty}\right) (1 - x) \quad (4.25)$$

where U_g/U_∞ is employed from equation (4.19) and the Strouhal number (St) used in this equation is determined by equation (4.17) rather than equation (4.16). Ungar and Eichhorn obtained a very good result from correlating the Strouhal number based on experimental data of various fluids. It is shown in Figs. 4.3 – 4.5. One question could arise as to whether it is reasonable for the use of Strouhal number determined by equation (4.17) in the flow condition with quality effect. However, reviewing from equation (4.17) and the definition of Froude number, $Fr = U_\infty/(gD)^{1/2}$, the characteristic of the correlation for Strouhal number contains no fluid properties. As a result, Ungar and Eichhorn concluded that the frequency of bubble break-off in low speed up-flow is determined only by the cylinder diameter, the acceleration of gravity, and the speed of the flow. In addition, equation (4.17) was developed from experimental data for various fluids which have widely dissimilar physical properties. Thus, equation (4.17) is valid for any liquid in low speed up-flow and the flow condition with or without quality effect.

Now since the doubt for using equation (4.17) has been cleared, equations (4.19) and (4.25) can be combined and yield

$$\phi = \alpha^* \left(1 + 2\pi St \sqrt{\frac{3\alpha^*}{2We_g}}\right) (1 - x) \quad (4.26)$$

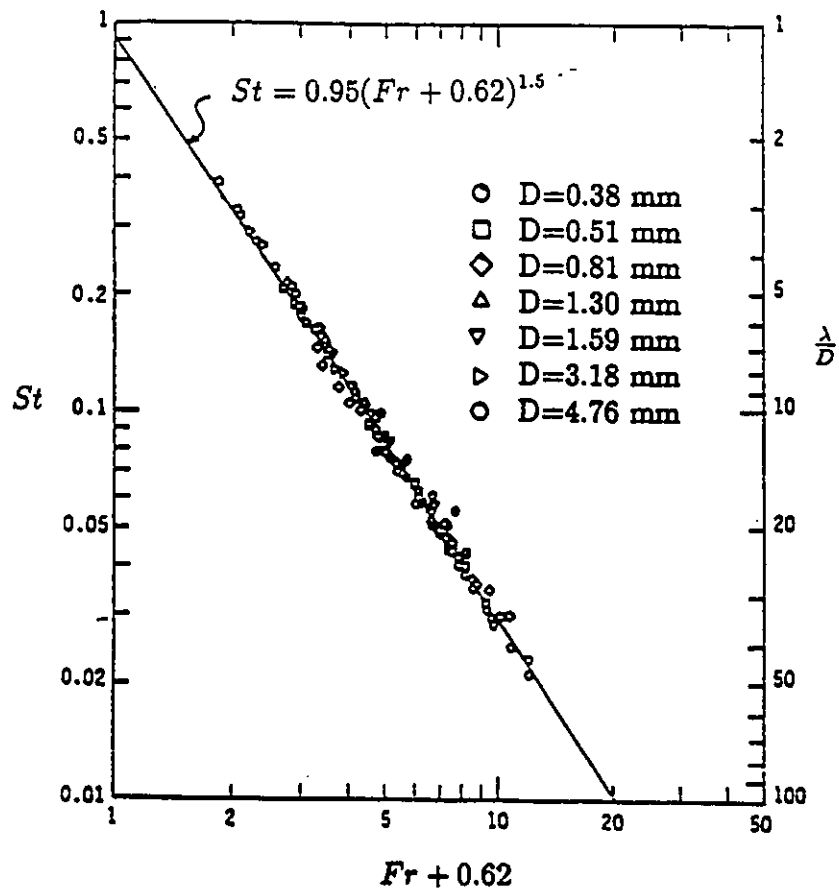


Figure 4.3: Strouhal number vs. Froude number obtained by Ungar and Eichhorn (1988) - acetone

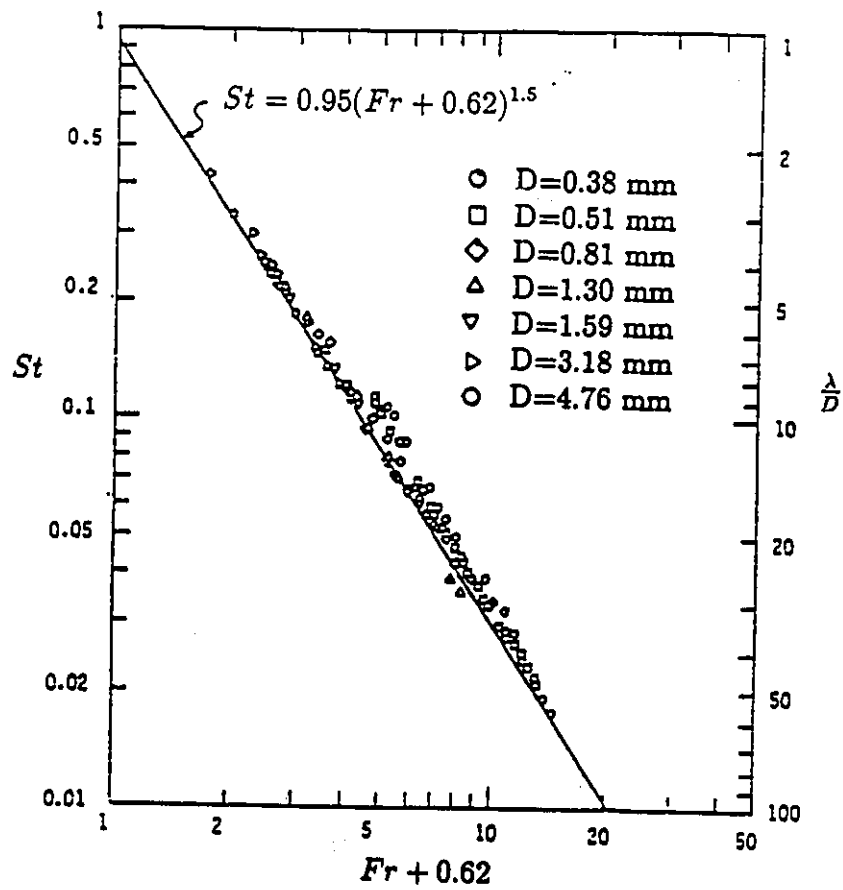


Figure 4.4: Strouhal number vs. Froude number obtained by Ungar and Eichhorn (1988) - isopropanol

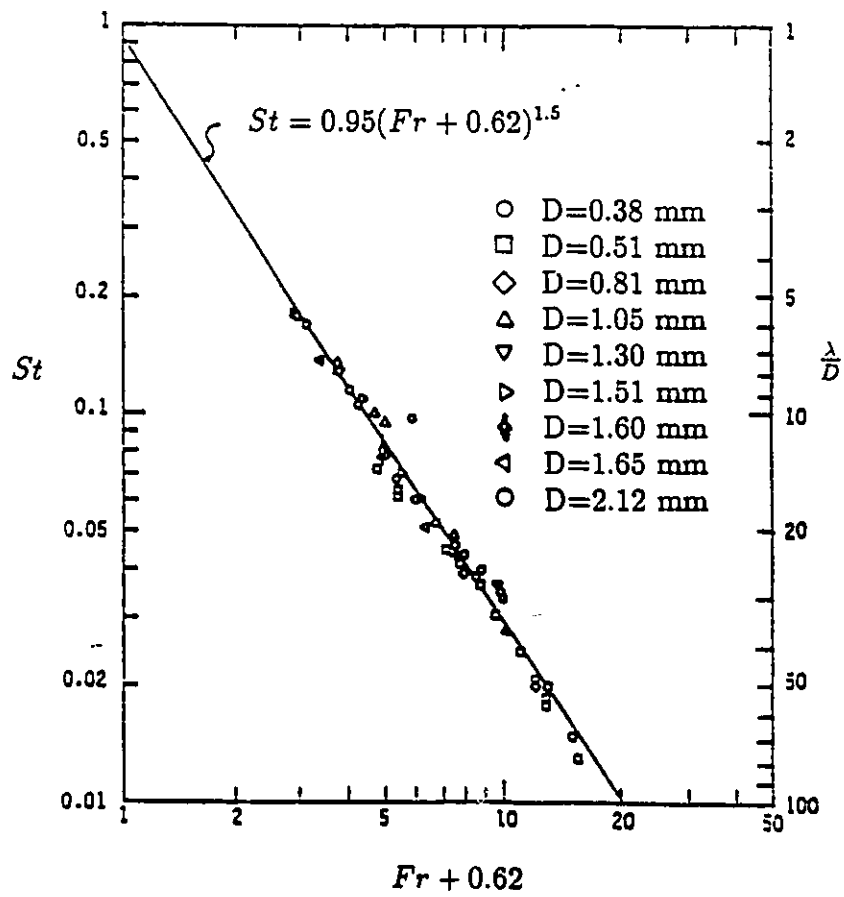


Figure 4.5: Strouhal number vs. Froude number obtained by Ungar and Eichhorn (1988) - water

The sources of experimental results used to evaluate α^* in the present study are listed in Table 4.2. Before correlating the calculated α^* to the form as illustrated in equation (4.24), whether the trends for α^* versus Fr and $(1 - x)$ agree with the postulated equation have been observed first. Figures 4.6 – 4.13 show that the calculated values of α^* for all test fluids increase with increasing Fr . Also, Figures 4.14 – 4.16 show that the α^* increases when $(1 - x)$ increases for the results obtained by Vliet (1962), Pourdashti (1984) and Min (1975). For correlating the α^* to the form of equation (4.24), each test fluid results in different values of a , b and c and they are listed in Table 4.3.

4.3.2 Correlations of Exponential Variables— a , b and c

As shown in Table 4.3, each value of the density ratio corresponds to a set of values a , b and c . Like Ungar and Eichhorn's correlation shown in equation (4.21), c can be expressed as a function of density ratio. However unlike Ungar and Eichhorn's result, in which b is constant along the range of density ratio and equivalent to 0.28 as shown in equation (4.20), the b obtained from present study is not constant with respect to different density ratios. Figs. 4.17 – 4.19 show the distributions of the values of c , a and b listed in Table 4.3 against the density ratio. Efforts have been made to correlate these three exponential values as a function of density ratio. They yield

$$a = 0.0186r^{0.825} \quad (4.27)$$

$$b = 0.341 - 5.7 \times 10^{-5}r \quad (4.28)$$

$$c = 0.086r^{0.329} \quad (4.29)$$

Substituting the above results into equation (4.24), results in

$$\alpha^* = (0.341 - 5.7 \times 10^{-5}r)Fr^{(0.086r^{0.329})}(1 - x)^{0.0186r^{0.825}} \quad (4.30)$$

4.4 Assessment of Recommended Prediction Method and Discussion

Since equation (4.30) has illustrated the correlation for α^* , equation (4.26) is ready to predict the non-dimensional CHF, ϕ , while the dimensional prediction of

Table 4.2: Sources of CHF data for upward across single cylinder

Investigator	Number of data	cylinder diameter (mm)	density ratio	Froude number	quality	fluid
Pourdashti	40	6.35	50~100	0.24~1.14	-0.11~0.11	Freon-113
Hasan et al.	19	0.5~1.5	334.7	2.3~11.6	0	isopropanol
Ungar and Eichhorn	15	1.59~4.76	341.8	2.0~7.6	0	isopropanol
Ungar and Eichhorn	45	0.38~2.12	1553	2.2~12.3	0	water
Min	40	0.51~1.65	615	1.8~9.3	0	methanol
Min	62	0.51~1.02	1647	1.8~9.3	-0.02~0	water
Vliet	81	0.25~3.2	1460	1~7	-0.1~0	water
Diesselhorst et al.	31	3~7.5	1609	0.5~2.3	0	water

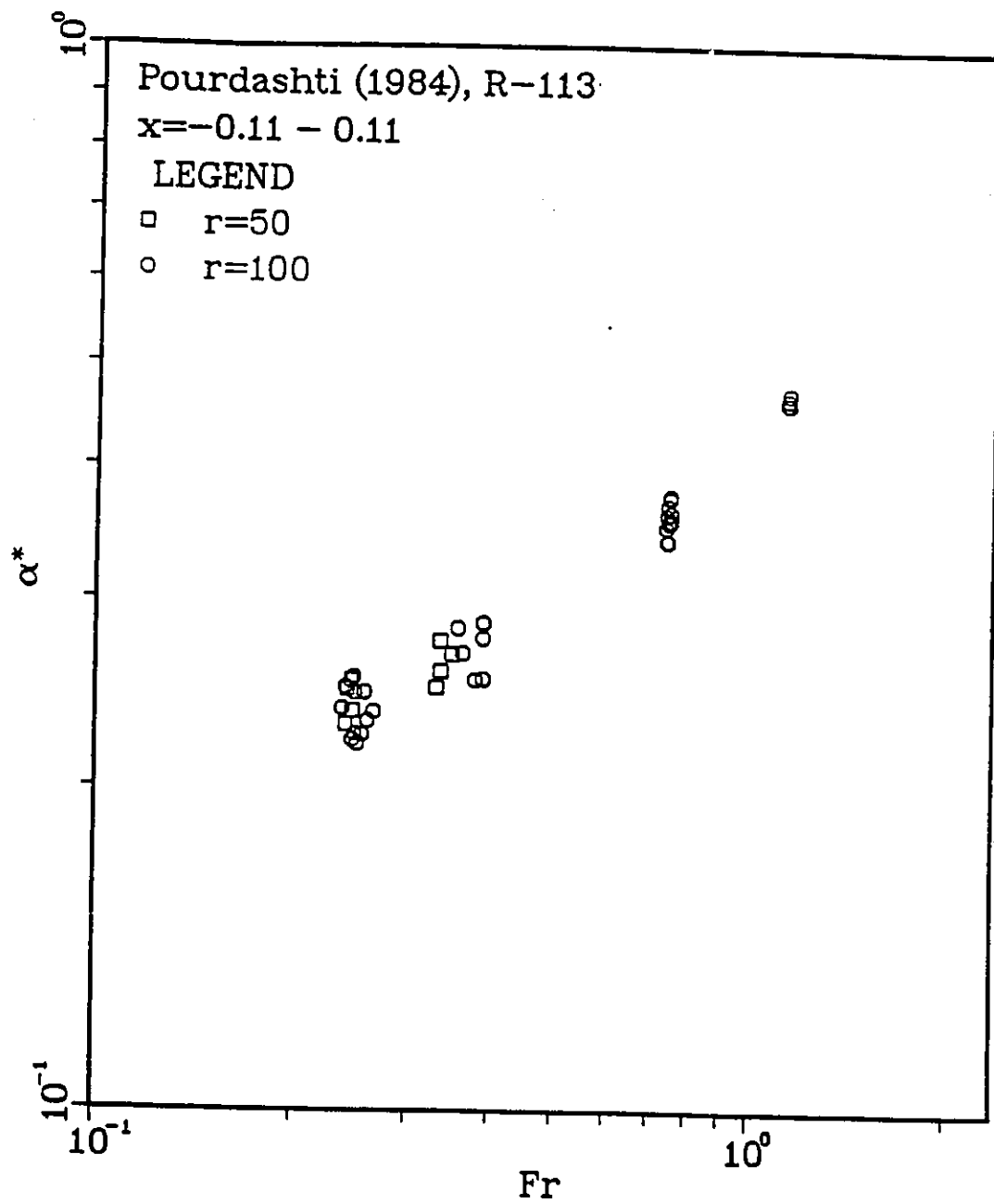


Figure 4.6: Effective vapor sheet thickness vs. Froude number

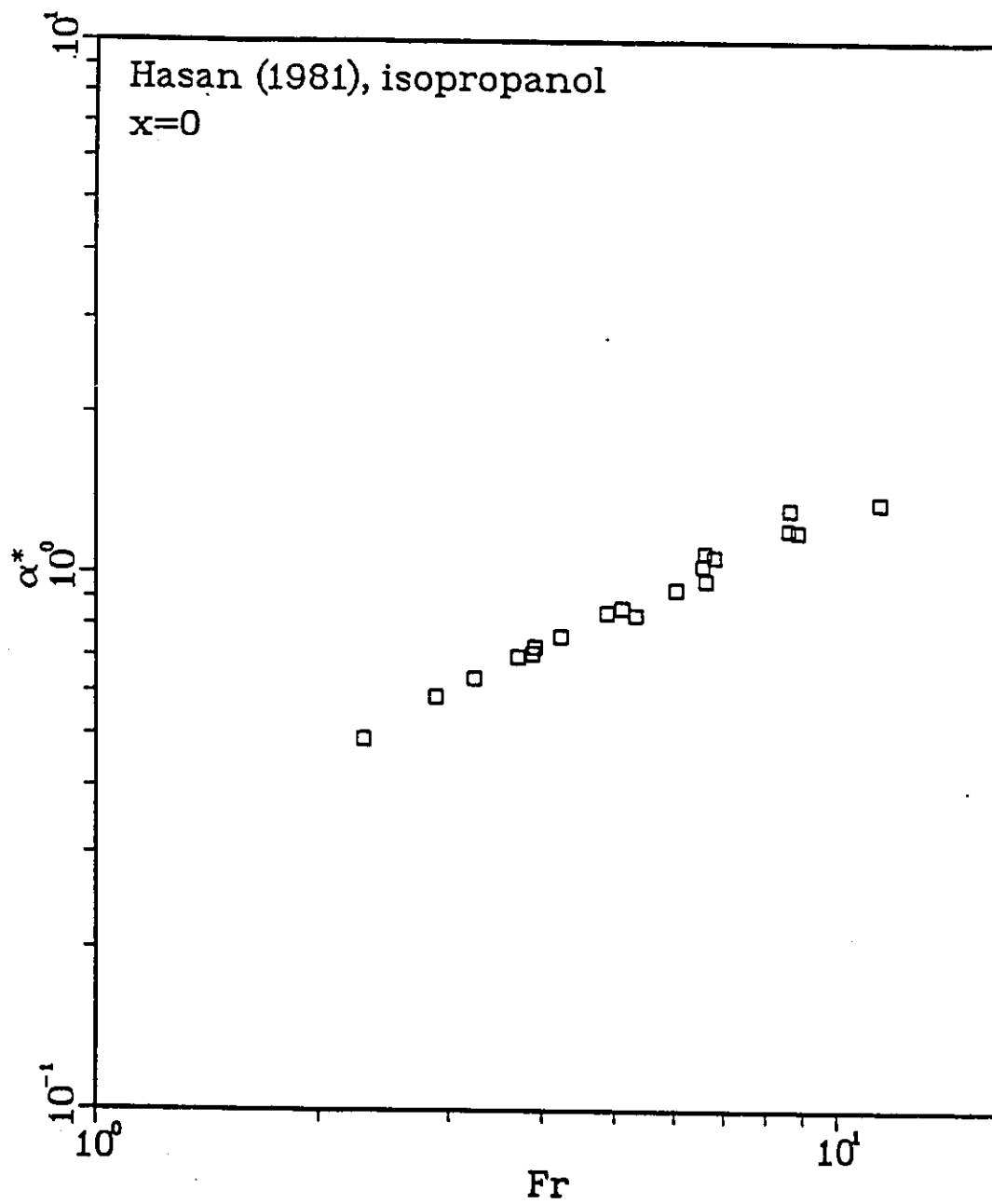


Figure 4.7: Effective vapor sheet thickness vs. Froude number

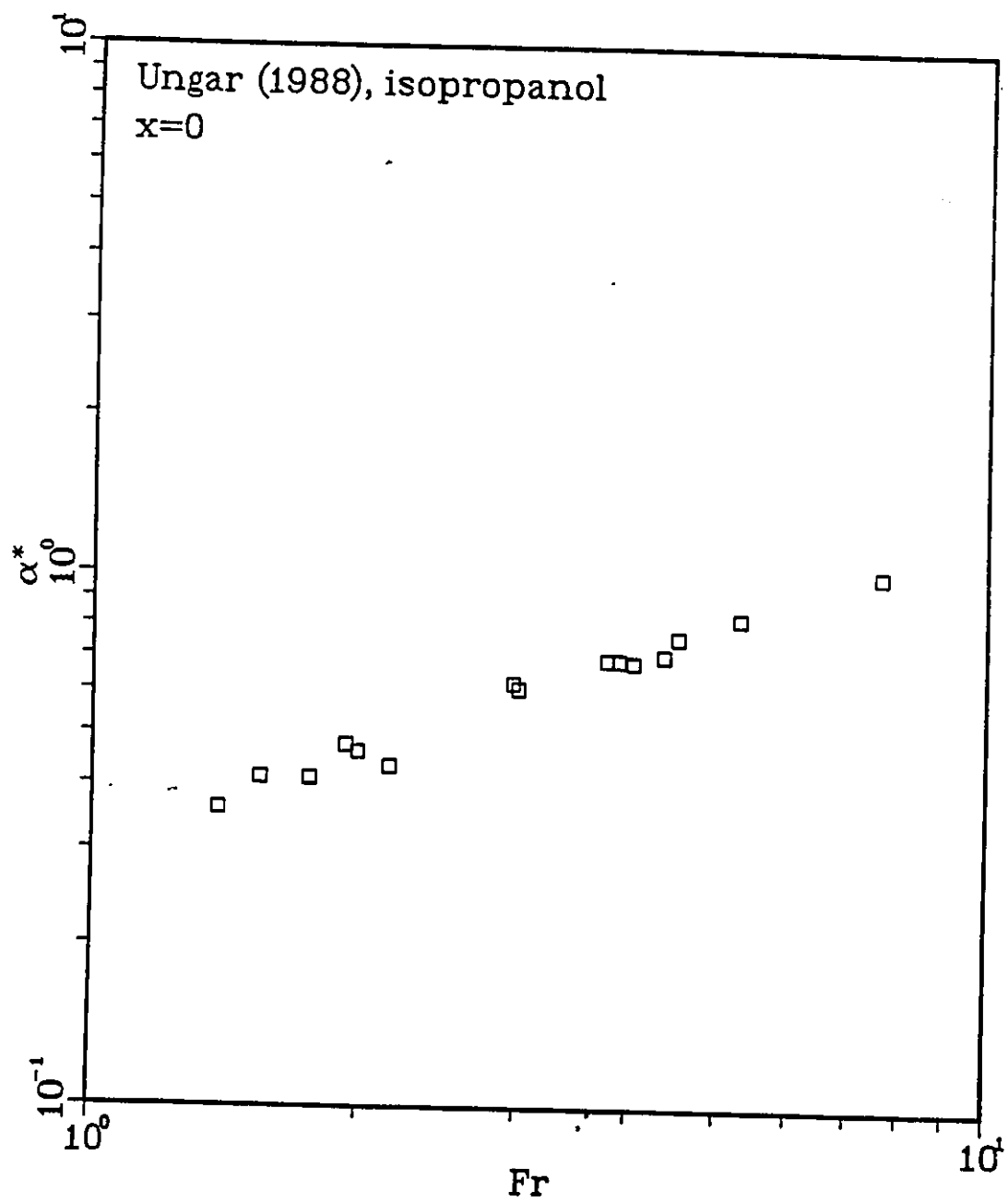


Figure 4.8: Effective vapor sheet thickness vs. Froude number

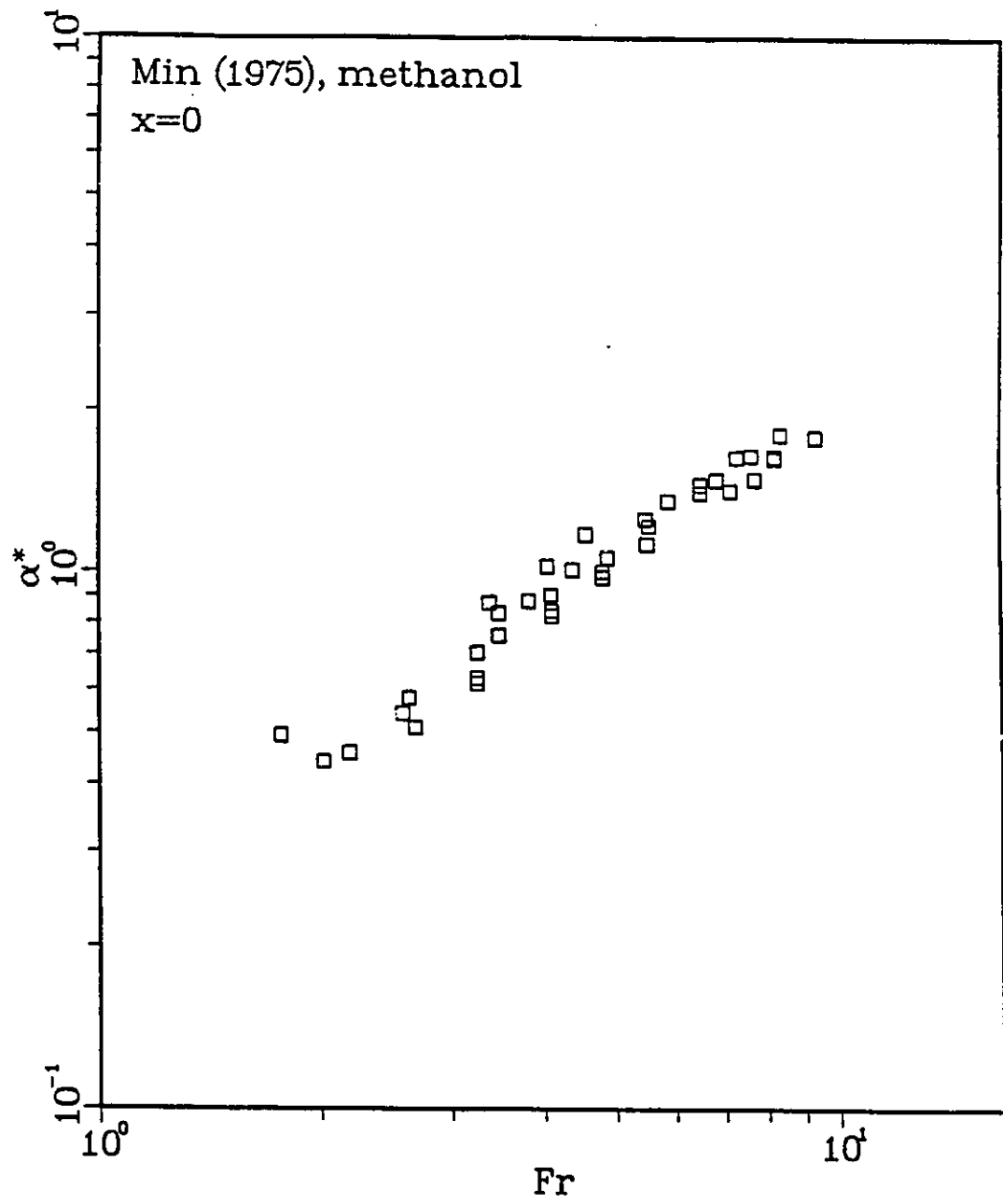


Figure 4.9: Effective vapor sheet thickness vs. Froude number

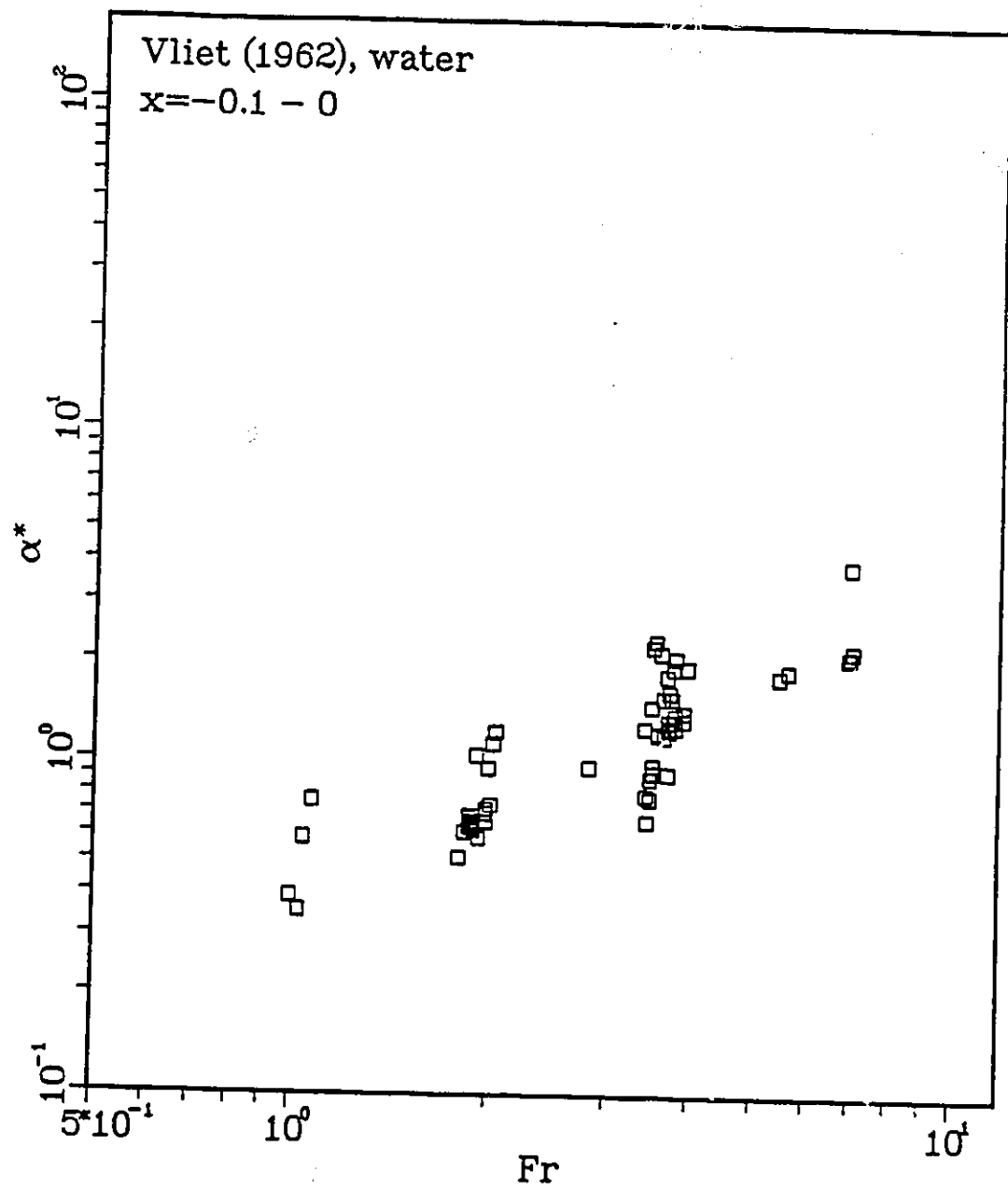


Figure 4.10: Effective vapor sheet thickness vs. Froude number

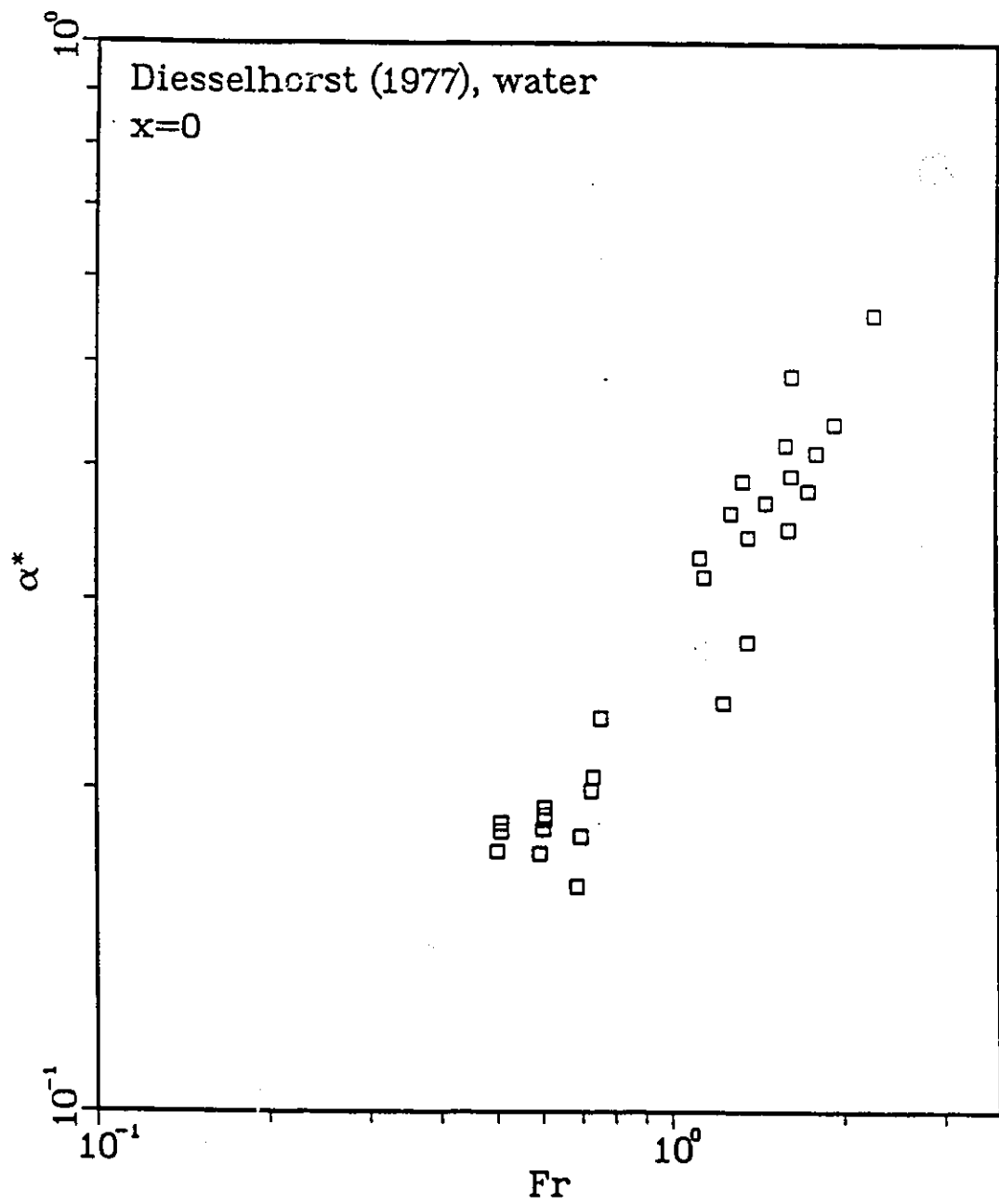
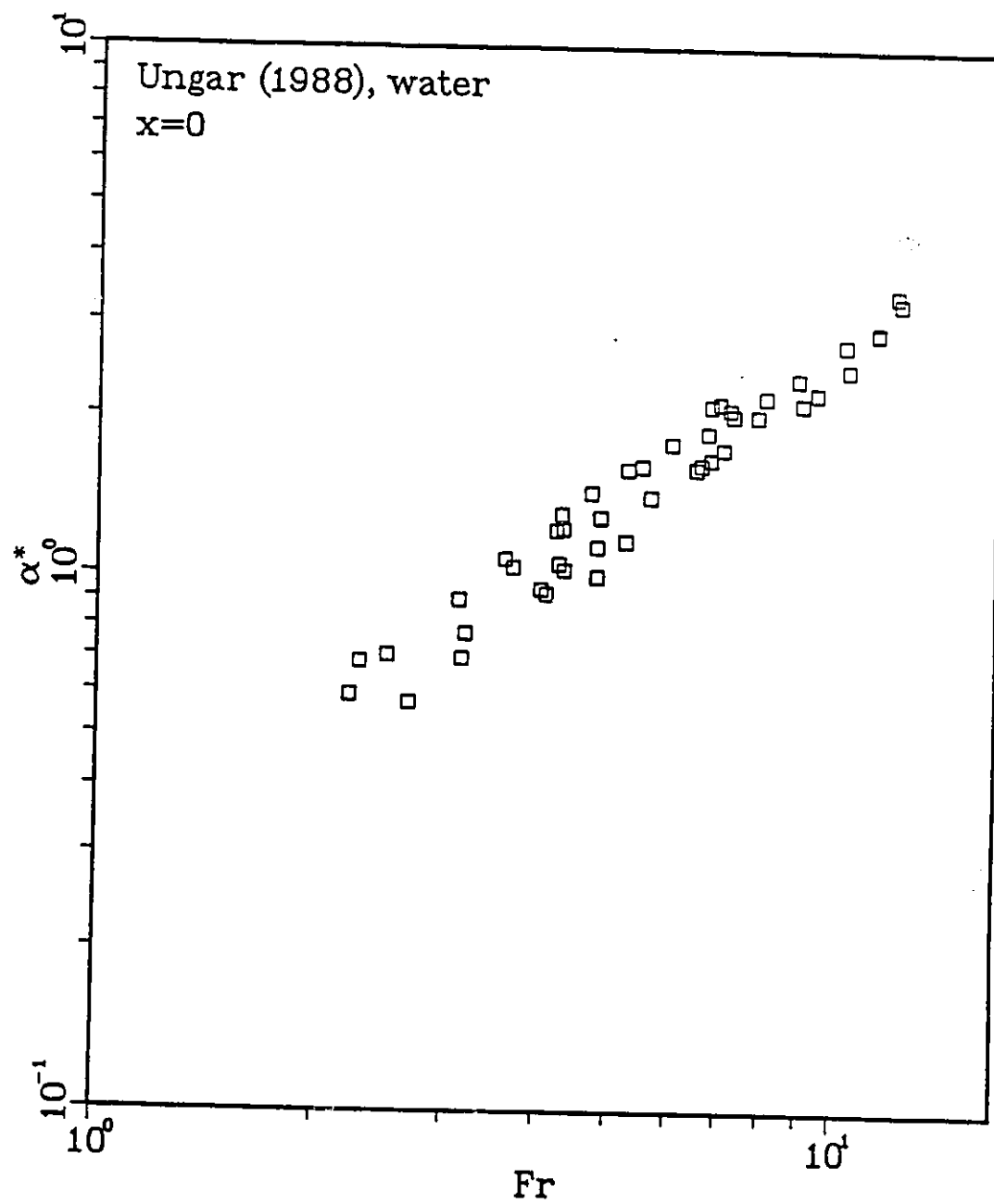


Figure 4.11: Effective vapor sheet thickness vs. Froude number



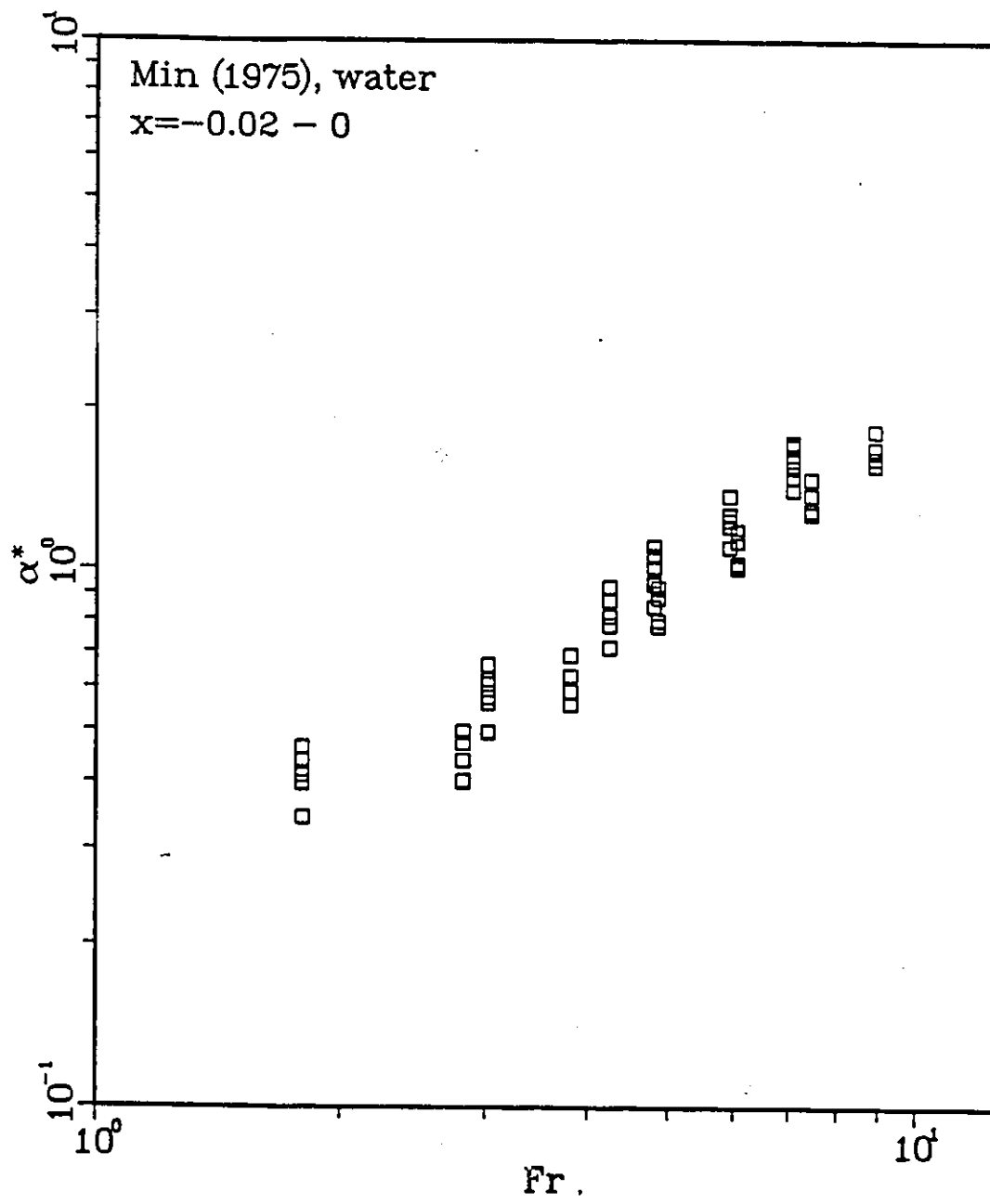


Figure 4.13: Effective vapor sheet thickness vs. Froude number

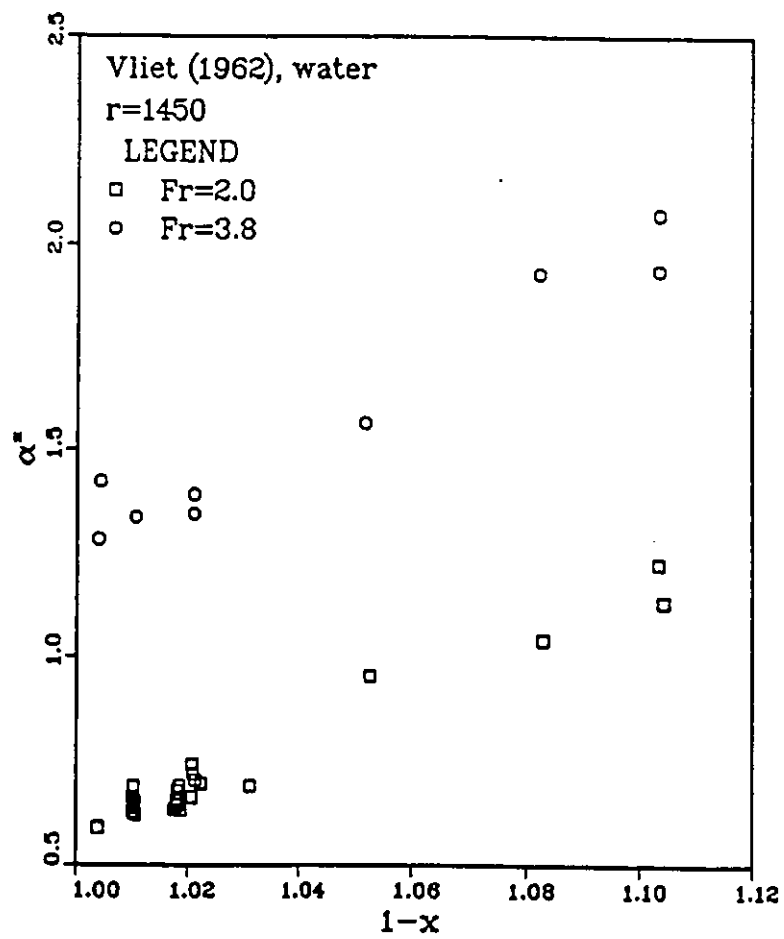


Figure 4.14: Effective vapor sheet thickness vs. $(1-x)$

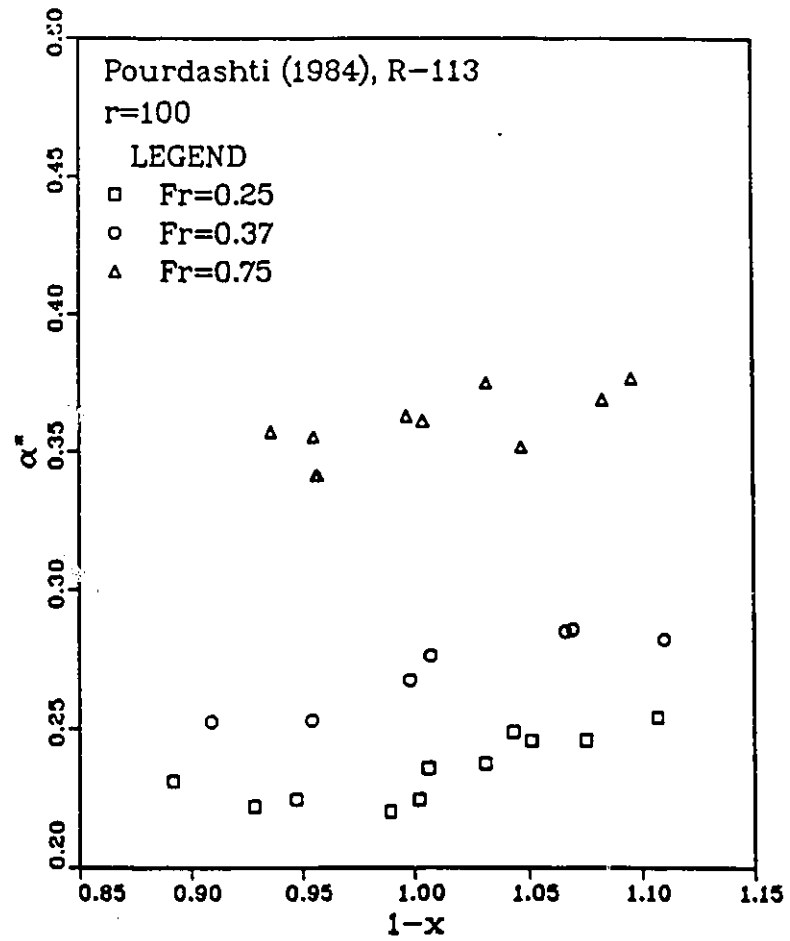


Figure 4.15: Effective vapor sheet thickness vs. $(1 - x)$

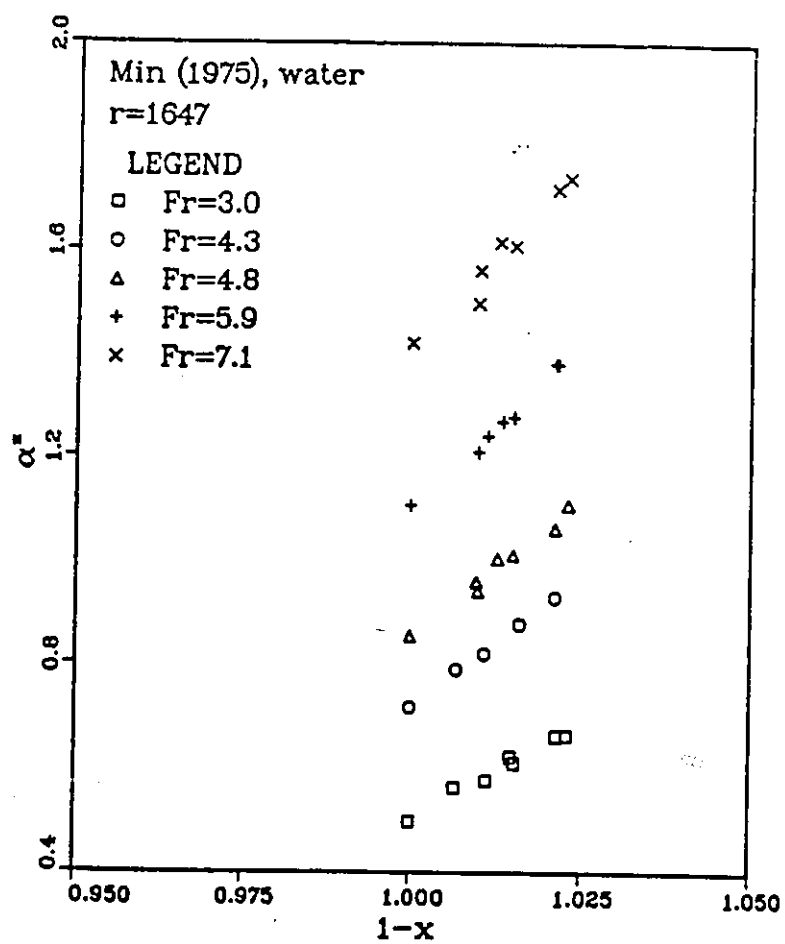


Figure 4.16: Effective vapor sheet thickness vs. $(1 - x)$

Table 4.3: Values of empirical variables obtained from different test sources

Investigator	ρ_f/ρ_g	a	b	c	fluid
Pourdashti (1984)	50	0.5501	0.3499	0.2574	Freon-113
Pourdashti (1984)	100	0.6812	0.4081	0.4105	Freon-113
Hasan et al. (1981)	334.7	0	0.2914	0.6590	isopropanol
Ungar and Eichhorn (1988)	341.8	0	0.3040	0.6020	isopropanol
Min (1975)	615	0	0.2340	0.9418	methanol
Vliet (1962)	1460	7.1713	0.3379	0.8443	water
Ungar and Eichhorn (1988)	1553	0	0.2640	0.9960	water
Diesselhorst et al. (1977)	1609	0	0.2692	0.7600	water
Min (1975)	1647	9.1792	0.1780	0.993	water

CHF is evaluated by $q_{max} = \phi \rho_g U_\infty h_{fg} / \pi$. The RMS error of the present prediction for each test fluid is listed in Table 4.4. In addition, Fig. 4.20 shows the comparison of measured CHF versus predicted CHF for all test data and represents 72% of the data with $\pm 25\%$ accuracy.

According to the results illustrated in Table 4.4, most test data give the RMS error lower than 25% or near 25% with only one exception of Min's (1975) water data. However, Min's methanol data gives around 20% of RMS error. It is acceptable even though the same data base has only 9.2% of RMS error with Ungar and Eichhorn's (1988) correlation. But one thing Ungar and Eichhorn did not point out in their study is that Min's water data gives more than 100% RMS error when using their correlation.

Hwang and Yao's (1984) CHF correlation has also been chosen to assess with the same test data used in the present analysis because only Hwang and Yao's correlation includes the quality effect for flow boiling across a single cylinder. However, Hwang and Yao's correlation has a narrower velocity range, which corresponds to $G_c \leq 2.8$ rather than $G_c \leq 8.0$ in present analysis. Therefore, both the RMS error for the flow condition of $G_c \leq 2.8$ and $G_c \leq 8.0$ have been calculated for the assessment of Hwang and Yao's correlation with all test fluids. These results are shown in Table 4.5. Though Hwang and Yao (1984) concluded that their correlation predicts the CHF well with their own experimental results, it seems not applicable to the other test results.

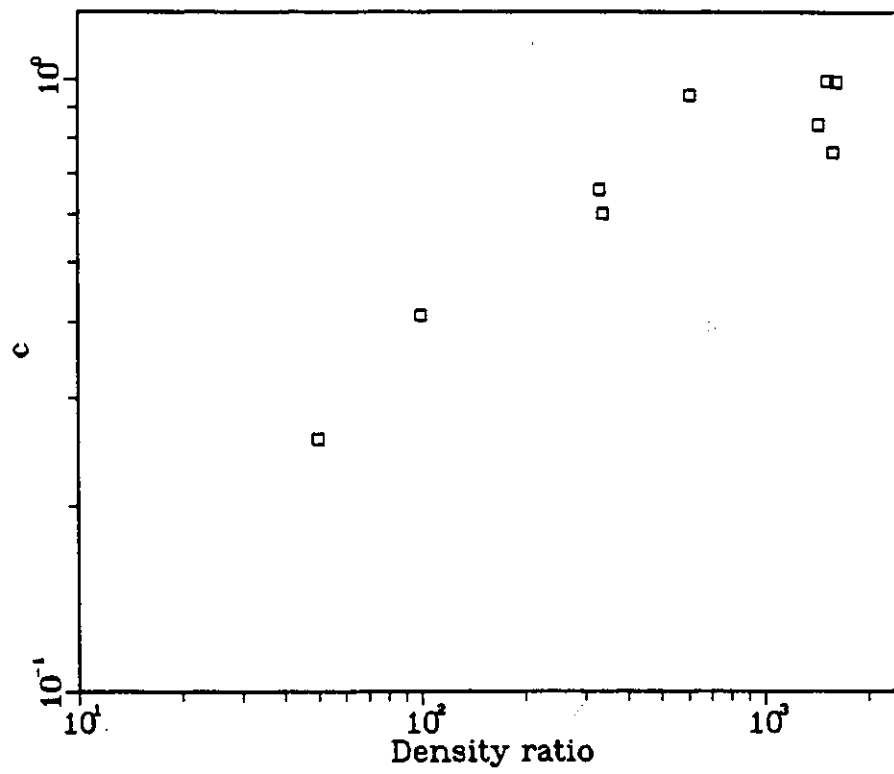


Figure 4.17: Exponential values of c vs. density ratio obtained from present study

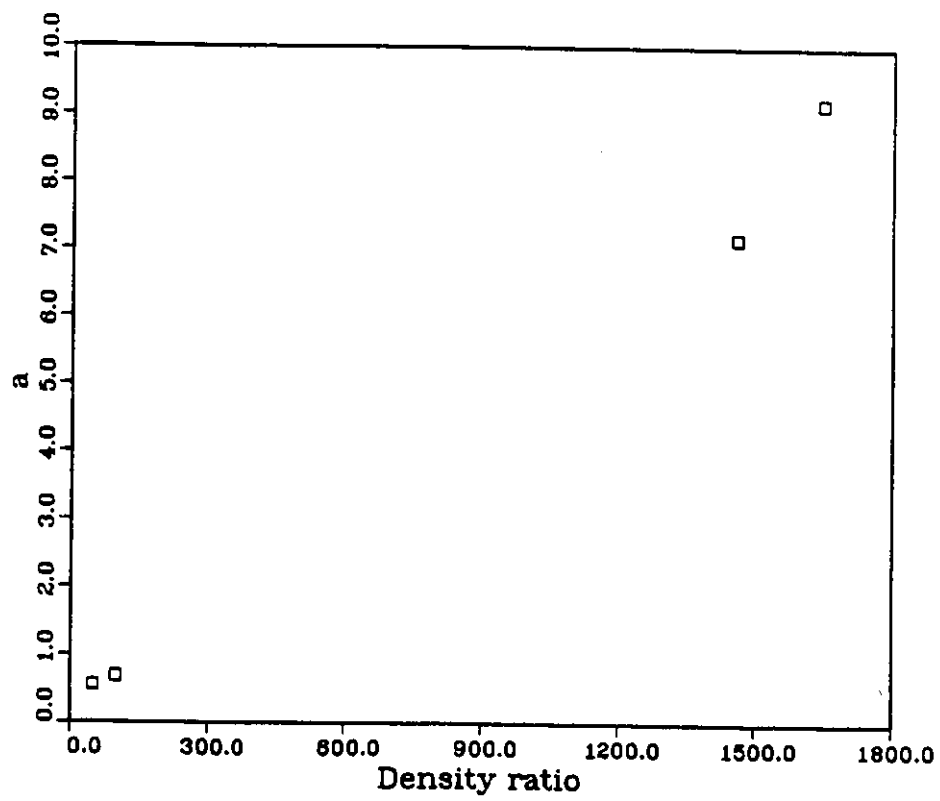


Figure 4.18: Exponential values of a vs. density ratio obtained from present study

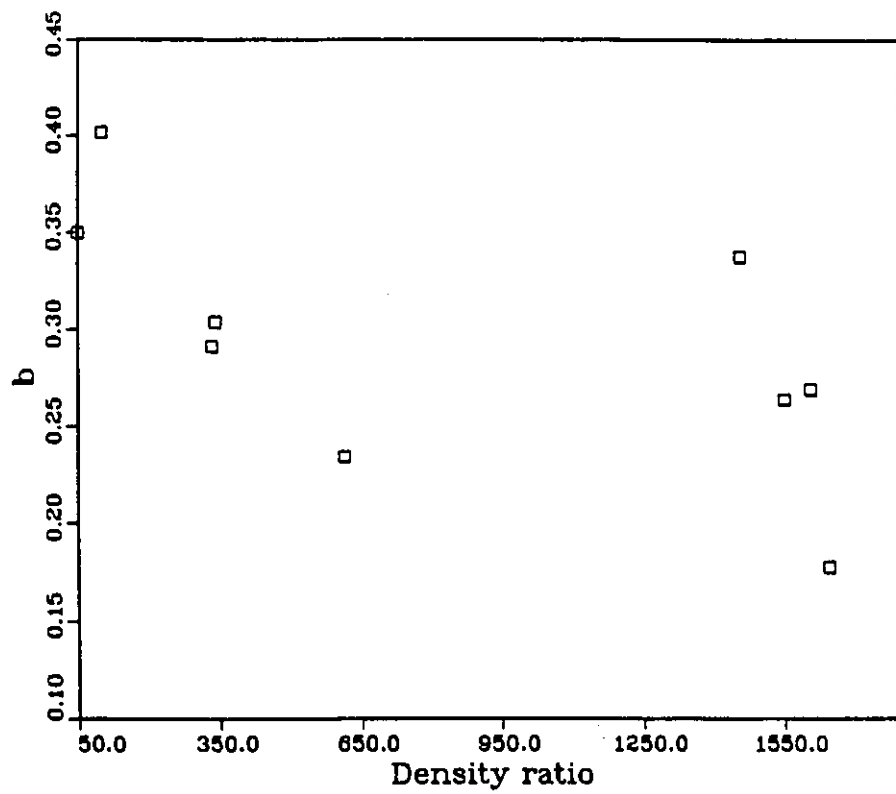


Figure 4.19: Constants b vs. density ratio obtained from present study

Table 4.4: Assessment of the present prediction of CHF to each test source

Data source	Number of data	Fluid	RMS error (%)
Pourdashti	40	Freon-113	21.75
Hasan et al.	19	isopropanol	7.36
Ungar and Eichhorn	15	isopropanol	8.64
Min	40	methanol	20.51
Vliet	81	water	27.74
Ungar and Eichhorn	45	water	18.64
Diesselhorst et al.	31	water	22.93
Min	62	water	65.63
	271*		22.08*

* Min's water data is not included.

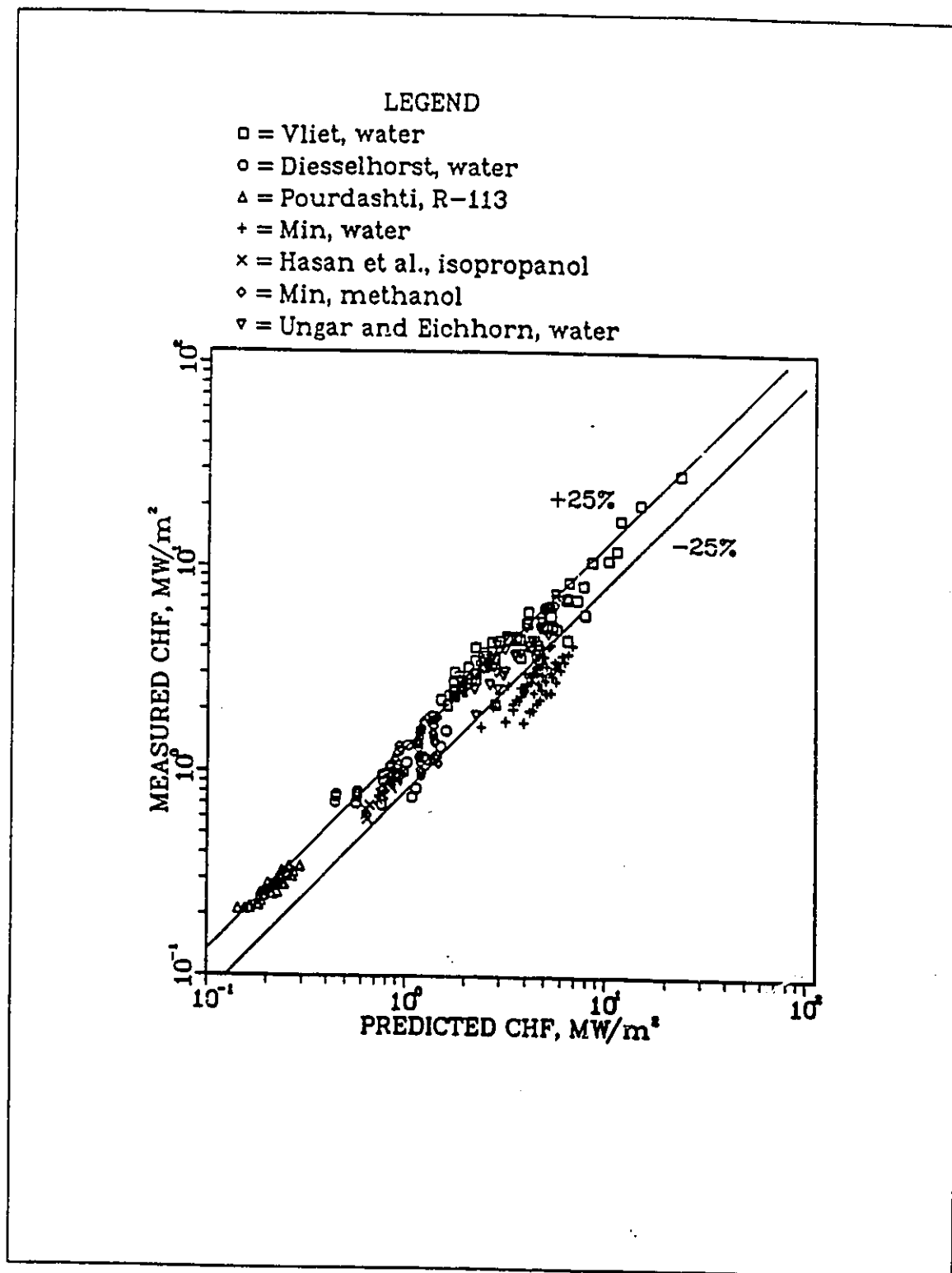


Figure 4.20: Comparison of CHF data predicted by Eq. (4.25) with measured CHF for different sources

Table 4.5: Assessment of the Hwang and Yao's prediction of CHF to each test source

Data source	Fluid	RMS error	No. of data	RMS error	No. of data
		(%) $G_c \leq 8$		(%) $G_c \leq 2.8$	
Pourdashti	Freon-113	46.75	40	45.78	36
Hasan et al.	isopropanol	32.94	19	31.37	4
Min	methanol	50.32	40	39.20	12
Vliet	water	66.46	81	58.16	42
Ungar and Eichhorn	water	61.84	45	51.59	10
Diesselhorst et al.	water	37.57	31	37.59	28
Min	water	48.47	62	39.44	24
		49.54*	197*	40.18*	78*

* The data for Pourdashti (1984) and Vliet (1962) are not included.

Chapter 5

Characterization of Velocity and Diameter for the Cross Flow

5.1 Blockage Effect for Flow Across Single Cylinder

In previous chapters, the use of velocity for cross-flow boiling is mainly the incoming flow velocity or the velocity in the bulk liquid because most analytical and experimental data on single-cylinder heat transfer refer to flow conditions at infinity, i.e., in the absence of channel walls. However, in engineering applications the flow experiences a significant blockage due to the presence of adjacent tubes and channel walls, which may exert an influence on heat transfer. Therefore, the following analysis of heat transfer for cross-flow boiling would include the factor of blockage effects for the fact of convenience.

5.1.1 Žukauskas and Žiugžda's correlation (1985)

The blockage factor used in Žukauskas and Žiugžda (1985) is defined as $K_q = D/w$, where D is the diameter of tube or cylinder and w the width of flow channel. Therefore the bulk velocity, U , varying across the cylinder due to the blockage effect

can be expressed in terms of a modified form of the Hiemenz (1911) relationship:

$$\frac{U}{U_\infty} = A_1^* \left(\frac{x'}{D} \right) + A_2^* \left(\frac{x'}{D} \right)^3 + A_3^* \left(\frac{x'}{D} \right)^5 \quad (5.1)$$

where

$$A_1^* = 3.6314 \left(1 + \frac{K_q}{2} \right)$$

$$A_2^* = -2.1709(1 - 12.1284K_q^{2.226} + 3.7376K_q)$$

and

$$A_3^* = -1.5144(1 + 18.542K_q^{2.277} - 6.878K_q)$$

and Hiemenz's original equation is written as:

$$\frac{U}{U_\infty} = 3.6314 \left(\frac{x'}{D} \right) - 2.1709 \left(\frac{x'}{D} \right)^3 - 1.5144 \left(\frac{x'}{D} \right)^5 \quad (5.2)$$

where x' is the arc length from the front stagnation point of the cylinder. At $K_q = 0$, equation (5.1) reduces to the Hiemenz relation, or equation (5.2).

5.1.2 Narrowest cross section model

In the study for single-phase and two-phase convective heat transfer by Hwang and Yao (1984), the blockage effect has also been considered. However, rather than the velocity defined in section 5.1.1, the effective velocity is defined as the average velocity in the narrowest cross-sectional area. The velocity can be related to the free stream velocity, U_∞ , as

$$\frac{U}{U_\infty} = \frac{w}{w - D} \quad (5.3)$$

5.1.3 Mean flow area model

Flow velocities used in Perkins and Leppert's (1962), Vliet's (1962) and Vliet and Leppert's (1964a) measurements are the mean test channel velocity corrected for test element blockage and for velocity profile. The profile correction is used since the element is small compared to channel width and thus is in the environment of the centerline velocity, which is a function of the Reynolds number based on the channel width. Blockage effect is based on the concept that the average velocity flows through a mean flow area, i.e., the mean flow area is defined as the ratio of

flow volume in the channel at the location of the test section to the test-section diameter. Figure 5.1 shows schematically the concept of derivations of mean flow area and average velocity. Thus, the mean flow area can be written as:

$$A_m = \frac{\text{Channel volume} - \text{tube volume}}{\text{tube diameter}}$$

or

$$A_m = \frac{\left(wD - \frac{\pi D^2}{4}\right)L}{D} = \left(w - \frac{\pi D}{4}\right)L \quad (5.4)$$

According to the conservation of volumetric rate of channel flow, $Q = U A_m = U_\infty A_\infty$, the average velocity can be obtained by:

$$U = U_\infty \left(\frac{A_\infty}{A_m}\right) \quad (5.5)$$

where $A_\infty/A_m = 4w/(4w - \pi D)$. Therefore, equation (5.5) can be rewritten as

$$U = \frac{U_\infty}{1 - \frac{\pi D}{4w}} \quad (5.6)$$

Vliet (1962) concluded that for forced convection non-boiling, the heat transfer coefficient is affected by blockage, and a similar effect may apply to the CHF — the CHF increases with blockage because the local velocity increases with the decreasing transverse pitch.

5.2 Blockage Effect for Flow Across Tube Bundles

A circular tube bundle is one of the most common heat transfer surfaces, particular in shell-and-tube exchangers. The most common tube arrays are staggered and in-line arrangements such as illustrated in Fig. 5.2.

5.2.1 Minimum intertube space model

A typical application of this model is used in Žukauskas's (1987) experiments. The blockage effect is described by the ratio of the transverse ($X_{tt} = X_t/D$), longitudinal ($X_{ll} = X_l/D$), or diagonal ($X_{dd} = X_d/D$) pitch, which are shown in Figure 5.2,

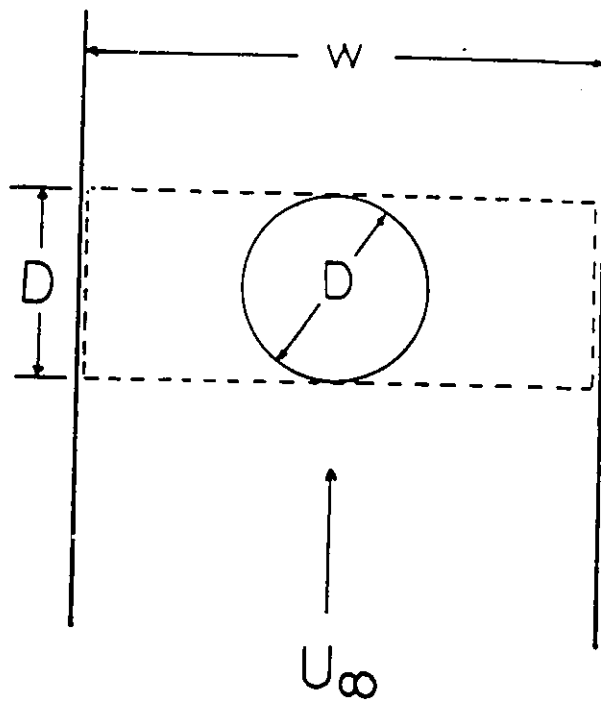


Figure 5.1: Control volume used to derive mean flow area across single cylinder

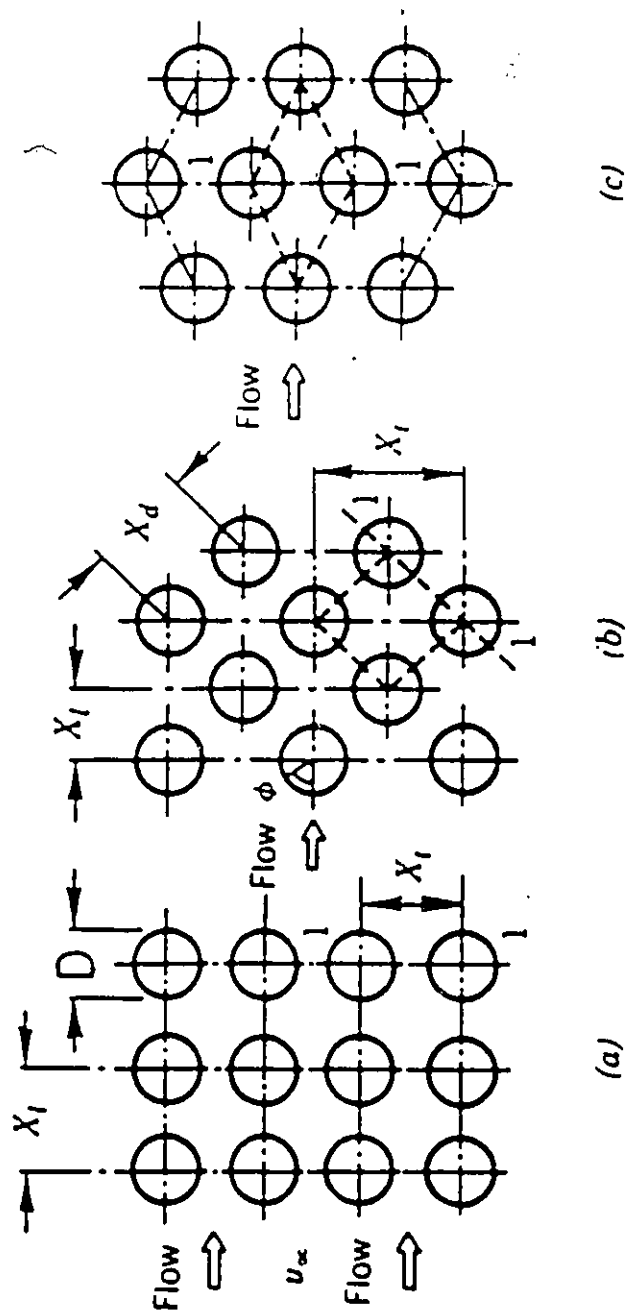


Figure 5.2: Most common types of tube bundles arrangements: (a) in-line. (b)–(c) staggered.

over the tube diameter. Žukauskas (1987) concluded that the transverse pitch has a significant effect on the velocity distribution in tube bundles. The average velocity in the minimum intertube space of a transverse tube row at a constant flow rate is given by:

$$U = U_{\infty} \frac{X_{tt}}{X_{tt} - 1} \quad (5.7)$$

and the blockage factor can be defined as the ratio of tube diameter of bundles over the transverse pitch and is equivalent to $1/X_{tt}$. The rate of velocity growth in the intertube space is mainly determined by the transverse pitch and it increases drastically at narrow pitches.

5.2.2 Mean flow area model

To extend the use of the mean flow area model for tube bundles, the control volume used to describe the derivation of a mean flow area is illustrated in Fig. 5.3. According to this control volume, the mean flow area is given by:

$$A_m = \frac{\left[wD' - n \left(\frac{\pi D^2}{4} \right) \right] L}{D'} \quad (5.8)$$

where n is the number of tubes in the control volume and D' the length of the control volume. Consequently, the average velocity can be obtained by substituting A_m into equation (5.5), yielding

$$U = U_{\infty} \frac{1}{1 - \frac{n}{wD'} \left(\frac{\pi D^2}{4} \right)} \quad (5.9)$$

In the following chapters, the use of the velocity for the analysis would refer to the mean flow area model because of the fact that most CHF's occur near the rear stagnation point and none of the models or correlations discussed above can characterize the flow velocity near the rear stagnation point successfully. However, the mean flow area model seems to be the most reasonable because the velocity is the average value across the cylinder or bundles.

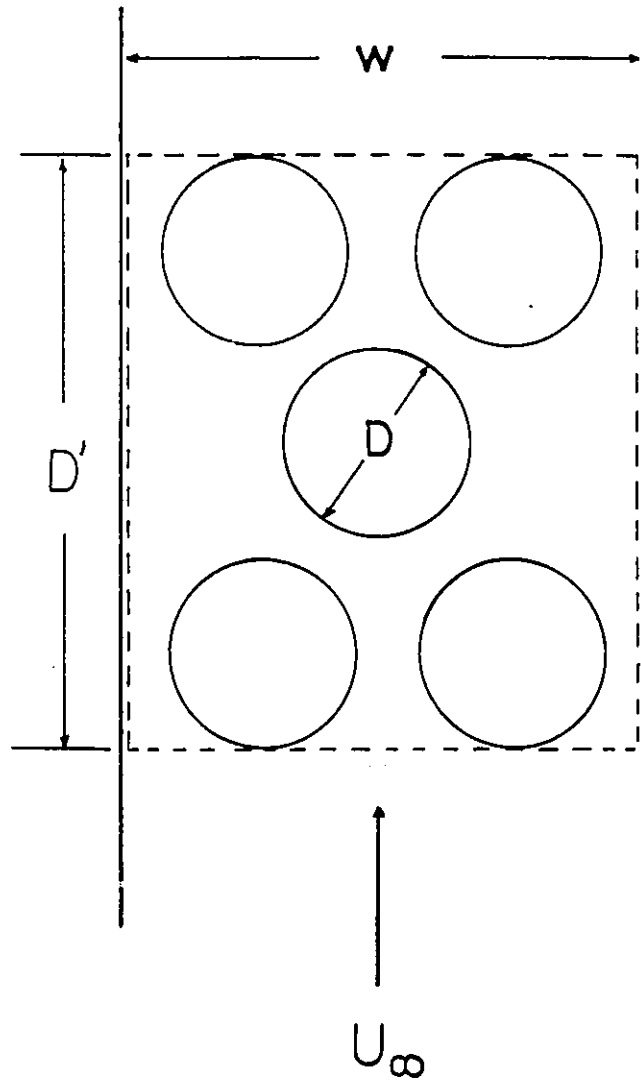


Figure 5.3: Control volume used to derive mean flow area across tube bundles

Chapter 6

Prediction Method for CHF in Axial Flow

The mechanism of boiling heat transfer including CHF has been introduced by Collier (1981). In present study the method for CHF prediction is only of interest for discussion.

6.1 CHF Prediction Methods for Water

6.1.1 Analytical model

CHF of annular flow in vertical tube is considered as the most practical system, where this type of CHF occurs as a result of liquid removal of the film (G_{LF}) leading to film dryout. The mechanisms of mass transfer from or to the liquid film are:

- Evaporation of liquid (q/h_{fg}) in the film;
- Entrainment of liquid droplets (G_E) from the film to the vapor core;
- Deposition of entrained liquid droplets (G_D) from the vapor core onto the liquid film.

The variation of liquid film flow rate with distance of the heated tube yields

$$\frac{dG_{LF}}{dL_h} = G_D - G_E - \frac{q}{h_{fg}} \quad (6.1)$$

As CHF is approached, the liquid film becomes very thin. Compared with other terms in equation (6.1), the dG_{LF}/dL_h term is assumed to be negligible. Then, if the evaporation rate (q/h_{fg}) exceeds the net deposition ($G_D - G_E$), the liquid film breaks down which leads to dryout. Equation (6.1) is for a steady state and it ignores the effect of evaporation by other mechanisms such as liquid flashing resulting from the channel pressure gradient.

Some predictive methods have been proposed to calculate the CHF in annular flow. These methods are different mainly in their treatments of entrainment, deposition and evaporation processes and in the choice of boundary conditions. Perhaps the most widely accepted model was developed by Whalley et al. (1974, 1975, 1978) and Hewitt and Hutchinson (1976) at Harwell. This model is rather general and superior to most empirical models. However, more work needs to be done in advancing understanding of entrainment and evaporation processes.

6.1.2 Empirical correlations

As pointed out by Collier (1981), two types of empirical correlating procedure have been adopted. They are:

1. Correlations of an empirical nature which make no assumption whatever about the mechanisms involved in the critical heat flux condition, but solely attempt to find a functional relationship between the critical heat flux and the independent variables.
2. Correlations where attempts have been made to look at and write down equations for the hydrodynamic and heat transfer processes occurring in the heated channel and to relate these to the critical heat flux condition.

Probably the most general form of the empirical correlations is that by Macbeth (1963), which belongs to the first category mentioned above and assumes that the CHF is a linear function of inlet subcooling. Thus

$$q_{max} = A + B\Delta h_{sub}$$

or

$$q_{max} = \frac{A' + B'\Delta h_{sub}}{C' + L_h}$$

where Δh_{sub} is the inlet subcooling, L_h the tube length and A' , B' and C' are related to the system parameters. Two of the most widely accepted correlations are the Bowring's correlation (1972) and Biasi's correlation (1967). The details of both correlations has been reviewed by Wong (1988).

6.2 CHF Prediction Methods for Non-aqueous Fluids

CHF prediction methods applicable to non-aqueous fluids can be categorized into three groups:

1. Fluid to fluid modelling of CHF: A technique that enables the CHF of a given system to be predicted using a modelling fluid rather than the working fluid (e.g. Barnett (1964), Stevens and Kirby (1964), and Ahmad (1973)).
2. Empirical dimensionless correlation: These are empirically derived dimensionless correlations that predict the CHF in a system regardless of the fluid type (e.g. Katto (1978), Katto and Ohno (1984), Shah (1979 and 1986) and Kutateladze(1963)).
3. Theoretical models: These theoretical models are based on the mechanism of CHF. They predict the CHF in a system within a dryout quality range regardless of fluid type used (e.g. Weisman and Pie (1983) and Whalley et al. (1973)).

All the above groups have been well reviewed by Kiameh (1986).

6.3 Table Look-up Method

The USSR Academy of Science (1977) has produced a series of standard tables of CHF. The CHF values are tabulated as a function of the local parameters G , x

and P for a tube of 8 mm inside diameter. The CHF for tube diameters other than 8 mm can be approximated by

$$q_{max} = q_{max,8mm} \left(\frac{D}{0.008} \right)^{-1/2} \quad (6.2)$$

for $4 \leq D \times 10^3 \leq 16$ mm.

Groeneveld et al. (1986a,1986b) developed an accurate and simple technique to predict CHF in water and non-aqueous fluids over a wide range of conditions. Groeneveld's look-up technique is also based on the local condition assumption. i.e.,

$$q_{max} = f(x, G, P, D) \quad (6.3)$$

The CHF table, or AECL-UO look-up table, is illustrated in Table 6.1. Although the standard CHF table is derived for very specific cases, i.e., upward flow in a uniformly heated 8 mm tube within table matrix conditions, it may also be used for other geometries and flow conditions: *CHF prediction for non-uniformly heat geometries, downward flow, flow in bundle geometries, etc. For these cases the table CHF value must be multiplied by correction factors K1 to K6.* The use of these correction factors is listed in Table 6.2. CHF values for a tube having a diameter other than 8 mm may be obtained by

$$q_{max} = q_{max,8mm} \left(\frac{D}{0.008} \right)^{-1/3} \quad (6.4)$$

for $4 \leq D \times 10^3 \leq 16$ mm.

The range of the table method is:

Pressure: 100 - 20,000 kPa
Mass flux: 0 - 7500 kg/m²/s
Quality: -50 to 100%

Groeneveld's look-up table is not dimensionless. However, the use of a CHF table for non-aqueous fluids can have two different approaches:

1. First method: The dimensional table is converted into dimensionless form by replacing P by ρ_f/ρ_g , q_{max} by $q_{max}/(Gh_{fg})$ and G by Ψ_K for Katto's scaling factor (Katto and Ohno (1984)), by Ψ_σ for Ahmad's first scaling factor (1973) and by Ψ_γ for Ahmad's second scaling factor (1973), where those expressions

Table 6.1: 1986 AECL-UO Critical Heat Flux look-up table (kW/m^2)

PRESS. (KPA)	G (KG/H.M.S)	QUALITY																				
		-0.50	-0.40	-0.30	-0.20	-0.15	-0.10	-0.05	0.00	0.05	0.10	0.15	0.20	0.25	0.30	0.40	0.50	0.60	0.70	0.80	0.90	1.00
100	0					3930	2430	2045	1200	1000	980	800	790	700	710	486	470	454	339	225	89	
100	50					4008	2500	2400	1700	1600	1500	1300	1250	1204	1126	899	850	800	501	355	150	
100	100					4800	4200	3100	2200	2100	2000	1450	1000	1000	955	950	950	900	660	430	135	
100	200					5600	4900	3800	2600	2550	2500	1500	1400	1300	1246	1200	1124	1100	854	474	460	
100	300					6400	5600	4500	3000	3000	3000	2500	2400	2300	2300	2168	1721	1448	818	481	450	
100	500					7200	6300	5200	3600	3550	3500	3200	3000	2500	2358	2350	2301	1220	700	651	440	
100	750					8000	7000	5900	5500	5000	4700	4249	4006	3104	3081	2933	2407	1317	911	700	400	
100	1000					8800	7600	6600	4800	4600	4406	3800	3600	3100	3300	3250	2503	1552	1100	600	300	
100	1500					9600	8200	7000	5500	4800	4600	4500	4451	4200	4000	3911	3105	2601	1100	600	300	
100	2000					10000	8800	7700	6000	5400	5300	5200	5100	5050	5000	4900	4549	2900	1100	600	300	
100	3000					10500	9400	8400	6500	6300	6000	5800	5700	5400	5300	5200	3900	2700	1100	600	300	
100	4000					11000	10000	9000	6900	6785	6550	6100	5300	5000	4600	4000	3000	2000	1000	500	300	
100	5000					11500	10600	9500	8000	7875	7600	7400	5200	4400	3800	3000	2500	1500	900	500	300	
100	7500					12000	11500	11000	10500	9233	9000	7500	5000	4000	3000	2200	2000	1200	800	400	300	
150	0					6273	4000	3500	2400	1350	1200	1100	1000	950	930	913	588	550	544	375	250	116
150	50					6344	4100	3600	2700	1900	1800	1700	1500	1450	1416	1329	1002	900	873	539	419	400
150	100					6397	4800	4200	3200	2400	2200	2000	1900	1800	1400	1093	1000	950	950	941	701	450
150	200					6488	5400	4800	3500	2700	2600	2500	2100	1600	1400	1200	1200	1166	1150	1000	1000	500
150	300					6571	6000	5400	4000	3000	2900	2800	2500	2400	1500	1400	1340	1300	1200	1200	800	500
150	500					6721	6600	6000	4500	3700	3600	3500	3100	2700	1600	1500	1500	1400	1200	1100	670	450
150	750					7400	7200	6600	5000	4300	4150	4000	3500	2800	1400	1399	1300	1200	1100	1000	650	400
150	1000					8000	7800	7200	5500	5200	4500	3800	3300	3100	1200	1105	1100	1100	1000	1000	600	350
150	1500					8600	8400	7600	6000	5500	4500	3800	3400	3200	1200	1105	1100	1100	1000	1000	600	350
150	2000					9200	9000	8200	6200	5700	4700	4000	3300	3100	2400	2000	1500	1300	1200	1100	550	300
150	3000					9800	9600	8740	5800	4800	4400	4300	4200	4100	3500	3000	2500	2000	1700	1500	550	300
150	4000					10200	10000	9636	6000	5000	4600	4500	4300	4200	3800	3300	3250	3200	2500	1000	500	300
150	5000					10700	10500	9900	6200	5145	4705	4500	4300	4200	3600	2800	2800	2600	2000	1000	450	300
150	7500					11200	11000	10400	9700	9500	8881	8000	6500	5000	3100	2600	2300	2000	1500	900	450	300
200	0					5971	4820	3700	2774	1450	277	262	248	233	219	204	175	146	117	87	58	29
200	50					6044	4955	4000	3094	2313	1794	1641	1562	1483	1405	1326	1244	1044	650	600	574	571
200	100					6099	5000	4100	3200	2600	2300	2100	2000	1900	1307	1200	1109	800	767	750	750	516
200	200					6193	5200	4300	3400	2800	2500	2300	2150	1400	1297	1200	1180	1100	1000	1000	1000	514
200	300					6278	5400	4450	3500	3000	2700	2400	2200	2150	2100	2000	1829	1497	904	417	400	350
200	500					6433	5600	4650	3700	3150	2900	2500	2400	2300	2250	2226	1759	1388	659	250	200	200
200	750					6610	5800	4850	3900	3300	3000	2650	2500	2314	2159	1818	1590	1463	1000	600	450	300
200	1000					6775	6000	5000	4000	3400	3000	2600	2500	2314	1939	1800	1700	1600	1000	600	450	300
200	1500					7083	6200	4600	3800	3300	3000	2600	2500	2350	2169	1800	1700	1500	1000	600	450	300
200	2000					7370	6400	4500	4000	3500	3000	2600	2500	2350	2150	2050	1800	1700	1400	1000	600	450
200	3000					7905	7000	5742	4287	3900	3000	2600	2500	2350	2150	2050	1800	1700	1400	1000	600	450
200	4000					8600	8500	6411	4800	4500	3000	2600	2500	2350	2150	2050	1800	1700	1400	1000	600	450
200	5000					8880	8607	6120	5267	4845	3000	2600	2500	2350	2150	2050	1800	1700	1400	1000	600	450
200	7500					9990	6990	6465	5581	5077	3000	2600	2500	2350	2150	2050	1800	1700	1400	1000	600	450
300	0					5631	4650	3667	2683	1700	325	308	291	274	257	240	205	171	137	103	68	34
300	50					5712	5063	4292	3296	2975	2481	2327	2248	2168	2014	1860	1551	1067	535	500	500	450
300	100					5772	5200	4500	3500	3400	3200	3000	2900	2800	2600	2400	2160	900	870	850	800	504
300	200					5877	5600	4700	3600	3500	3200	2900	2800	2600	1463	1400	1400	1376	1300	1200	1181	569
300	300					5971	4592	3324	3300	3250	3200	2800	2700	2600	2500	2400	2176	1785	1043	438	400	362
300	500					6142	4634	3367	3300	3250	3200	2700	2600	2600	2600	2500	1946	1341	811	200	200	200
300	750					6338	6000	5000	3900	3800	3200	2800	2700	2586	2475	2402	1526	868	490	418	250	200
300	1000					6521	6200	5200	4000	3900	3200	2800	2700	2500	2395	2208	2100	1800	1500	1200	700	400
300	1500					6862	6400	4154	4100	4000	3200	2800	2600	2538	2360	2100	1800	1500	1200	700	400	200
300	2000					7180	6600	4300	4200	4100	3200	2800	2600	2500	2294	2200	2100	1800	1500	1200	700	400
300	3000					8000	7778	5690	4800	4706	3200	2800	2600	2500	2294	2200	2100	1800	1500	1200	700	400
300	4000					8600	8443	6302	4518	4400	3200	2800	2600	2500	2294	2200	2100	1800	1500	1200	700	400
300	5000					8852	8419	4851	4967	4525	3200	2800	2600	2500	2294	2200	2100	1800	1500	1200	700	400
300	7500					10082	5816	5198	4236	4630	3200	2800	2600	2500	2300	2200	2100	1800	1500	1200	700	400
450	0					5363	4540	4222	2000	1968	380	360	340	320	300	280	240	200	160	150	120	110
450	50					5454	5035	4627	2330	2250	2195	2115	2035	1993	1950	1870	1660	1625	600	550	543	500
450	100					5522	5200	4100	3400	2900	2800	2700	2600	2550	2200	2000	1500	1123	1112	984	908	595
450	200					5639	5600	4150	3400	3000	2800	2600	2550	2500	2200	1700	1600	1500	1400	1306	890	595
450	300					5745	4145	2896	2850	2825	2800	2600	2500	2450	2400	2400	2300	2161	1583	755	447	408
450	500					5939	4187	2939	2900	2850	2800	2600	2400	2400	2400	2300	2245	1784	1123	451	250	236
450	750					6159	6000	4300	3400	3100	2800	2600	2300	2150	2000	1900	1355	879	707	467	350	316
450	1000					6365	6200	4400	3400	3200	2800	2600	2100	1966	1786	1200	700	606	593	500	3	

PRESS. G		QUALITY																				
(KPA) (KG/M.M.S)		-0.50	-0.40	-0.30	-0.20	-0.15	-0.10	-0.05	0.00	0.05	0.10	0.15	0.20	0.25	0.30	0.40	0.50	0.60	0.70	0.80	0.90	1.00
1500	0	8975	7713	6456	4525	4007	3488	2970	584	551	523	492	461	430	369	307	246	184	123	61		
1500	50	9088	7839	6576	4781	4200	3622	3044	4200	4000	3513	3256	3273	3115	2883	2567	2127	1149	1021	879	436	
1500	100	9171	7928	6666	4870	4250	3670	3090	4300	4100	3600	3350	3367	3200	2967	2651	2211	1231	1103	971	509	
1500	200	9315	8083	6820	5000	4400	3820	3240	4400	4200	3700	3450	3467	3300	3067	2751	2311	1331	1203	1071	609	
1500	300	9445	8222	6960	5140	4540	3960	3380	4500	4300	3800	3550	3567	3400	3167	2851	2411	1431	1303	1171	709	
1500	400	9683	8467	7200	5280	4680	4100	3520	4600	4400	3900	3650	3667	3500	3267	2951	2511	1551	1423	1291	809	
1500	500	9954	8767	7500	5420	4820	4240	3660	4700	4500	4000	3750	3767	3600	3367	3051	2611	1691	1563	1431	909	
1500	600	10207	9038	7770	5560	4960	4380	3800	4800	4600	4100	3850	3867	3700	3467	3151	2711	1831	1703	1571	1009	
1500	700	10478	9301	8040	5700	5100	4520	3940	4900	4700	4200	3950	3967	3800	3567	3251	2811	1971	1843	1711	1109	
1500	800	10759	9583	8320	5840	5240	4660	4080	5000	4800	4300	4050	4067	3900	3667	3351	2911	2071	1943	1811	1209	
1500	900	11040	9867	8600	5980	5380	4800	4220	5100	4900	4400	4150	4167	4000	3767	3451	3011	2171	2043	1911	1309	
1500	1000	11321	10145	8880	6120	5520	4940	4360	5200	5000	4500	4250	4267	4100	3867	3551	3111	2271	2143	2011	1409	
1500	1500	11602	10426	9160	6260	5660	5080	4500	5300	5100	4600	4350	4367	4200	3967	3651	3211	2411	2283	2151	1509	
1500	2000	11883	10707	9400	6400	5800	5220	4640	5400	5200	4700	4450	4467	4300	4067	3751	3311	2511	2383	2251	1609	
1500	2500	12164	10988	9640	6540	5940	5360	4780	5500	5300	4800	4550	4567	4400	4167	3851	3411	2651	2523	2391	1709	
1500	3000	12445	11269	9920	6680	6080	5500	4920	5600	5400	4900	4650	4667	4500	4267	3951	3511	2791	2663	2531	1809	
1500	3500	12726	11550	10200	6820	6220	5640	5060	5700	5500	5000	4750	4767	4600	4367	4051	3611	2931	2803	2671	1909	
1500	4000	13007	11831	10440	6960	6360	5720	5140	5800	5600	5100	4850	4867	4700	4467	4151	3711	3071	2943	2811	2009	
1500	4500	13288	12112	10680	7100	6500	5860	5280	5900	5700	5200	4950	4967	4800	4567	4251	3811	3171	3043	2911	2109	
1500	5000	13569	12393	10920	7240	6640	6000	5420	6000	5800	5300	5050	5067	4900	4667	4351	3911	3271	3143	3011	2209	
1500	5500	13850	12674	11160	7380	6780	6140	5560	6100	5900	5400	5150	5167	5000	4767	4451	4011	3371	3243	3111	2309	
1500	6000	14131	12955	11400	7520	6920	6280	5700	6200	6000	5500	5250	5267	5100	4867	4551	4111	3511	3383	3251	2409	
1500	6500	14412	13236	11640	7660	7060	6420	5840	6300	6100	5600	5350	5367	5200	4967	4651	4211	3651	3513	3383	2509	
1500	7000	14693	13517	11880	7800	7200	6560	5980	6400	6200	5700	5450	5467	5300	5067	4751	4311	3791	3663	3531	2609	
1500	7500	14974	13798	12120	7940	7340	6700	6120	6500	6300	5800	5550	5567	5400	5167	4851	4411	3931	3803	3671	2709	
1500	8000	15255	14079	12360	8080	7480	6840	6260	6600	6400	5900	5650	5667	5500	5267	4951	4511	4071	3943	3811	2809	
1500	8500	15536	14360	12600	8220	7620	6980	6400	6700	6500	6000	5750	5767	5600	5367	5051	4611	4171	4043	3911	2909	
1500	9000	15817	14641	12840	8360	7760	7120	6540	6800	6600	6100	5850	5867	5700	5467	5151	4711	4271	4143	4011	3009	
1500	9500	16098	14922	13080	8500	7900	7260	6680	6900	6700	6200	5950	5967	5800	5567	5251	4811	4371	4243	4111	3109	
1500	10000	16379	15203	13320	8640	8040	7400	6820	7000	6800	6300	6050	6067	5900	5667	5351	4911	4471	4343	4211	3209	
1500	10500	16660	15484	13560	8780	8180	7540	6960	7100	6900	6400	6150	6167	6000	5767	5451	5011	4571	4443	4311	3309	
1500	11000	16941	15765	13800	8920	8320	7680	7100	7200	7000	6500	6250	6267	6100	5867	5551	5111	4671	4543	4411	3409	
1500	11500	17222	16046	14040	9060	8460	7820	7240	7300	7100	6600	6350	6367	6200	5967	5651	5211	4751	4623	4491	3509	
1500	12000	17503	16327	14280	9200	8600	7960	7380	7400	7200	6700	6450	6467	6300	6067	5751	5311	4851	4723	4591	3609	
1500	12500	17784	16608	14520	9340	8740	8100	7520	7500	7300	6800	6550	6567	6400	6167	5851	5411	4951	4823	4691	3709	
1500	13000	18065	16889	14760	9480	8880	8240	7660	7500	7300	6900	6650	6667	6500	6267	5951	5511	5051	4943	4811	3809	
1500	13500	18346	17170	15000	9620	9020	8380	7800	7500	7300	7000	6750	6767	6600	6367	6051	5611	5151	5043	4911	3909	
1500	14000	18627	17451	15240	9760	9160	8520	7940	7500	7300	7100	6850	6867	6700	6467	6151	5711	5251	5143	5011	4009	
1500	14500	18908	17732	15480	9900	9300	8660	8080	7500	7300	7200	6950	6967	6800	6567	6251	5811	5351	5243	5111	4109	
1500	15000	19189	18013	15720	10040	9440	8800	8220	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4209	
1500	15500	19470	18294	15960	10180	9580	8940	8360	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4309	
1500	16000	19751	18575	16200	10320	9720	9080	8500	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4409	
1500	16500	20032	18856	16440	10460	9860	9220	8640	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4509	
1500	17000	20313	19137	16680	10600	10000	9360	8780	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4609	
1500	17500	20594	19418	16920	10740	10140	9500	8920	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4709	
1500	18000	20875	19699	17160	10880	10280	9640	9060	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4809	
1500	18500	21156	19980	17400	11020	10420	9780	9200	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	4909	
1500	19000	21437	20261	17640	11160	10560	9920	9340	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5009	
1500	19500	21718	20542	17880	11300	10700	10060	9480	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5109	
1500	20000	22000	20823	18120	11440	10840	10200	9620	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5209	
1500	20500	22281	21104	18360	11580	10980	10340	9760	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5309	
1500	21000	22562	21385	18600	11720	11120	10480	9900	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5409	
1500	21500	22843	21666	18840	11860	11260	10620	10040	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5509	
1500	22000	23124	21947	19080	12000	11400	10760	10180	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5609	
1500	22500	23405	22228	19320	12140	11540	10900	10320	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5709	
1500	23000	23686	22509	19560	12280	11680	11040	10460	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5809	
1500	23500	23967	22790	19800	12420	11820	11180	10600	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351	5243	5111	5909	
1500	24000	24248	23071	20040	12560	11960	11320	10740	7500	7300	7200	7000	7000	6800	6567	6251	5811	5351				

Table 6.2: CHF correction factors

Factor	Form	Comments
K_1 , subchannel or tube cross section factor	For $0.002 \text{ m} < D_{hy} < 0.016 \text{ m}$: $K_1 = \left(\frac{0.008}{D_{hy}} \right)^{1/3}$ For $D_{hy} > 0.016 \text{ m}$: $K_1 = \left(\frac{0.008}{0.016} \right)^{1/3} = 0.79$	Includes the observed diameter effect on CHF. This effect is quality-dependent and could be included by replacing the exponent $1/3$ by $(2 - X)/4$ (requires verification)
K_2 , bundle factor	$K_2 = \min[0.8, 0.8 \exp(-0.5X^{1/3})]$	Predicts the correct trend of quality on the bundle correction factor
K_3 , grid spacer factor	$K_3 = 1 + A \exp(-B L_p/D_{hy})$ $A = 1.5K^{0.5} \left(\frac{G}{1000} \right)^{0.2}, \quad B = 0.10$	Ideally A and B should be obtained for each grid spacer as the pressure loss coefficient K is insufficient to completely express the grid spacer effect
K_4 , heated length factor	$L/D_{hy} \geq 5: \quad K_4 = \exp\left(\frac{D_{hy}}{L} e^{2\alpha}\right)$ α evaluated from homogeneous model $\alpha = X/[X + \rho_v/\rho_l(1 - X)]$	Inclusion of α correctly predicts the diminishing length effect at subcooled conditions. The length effect is particularly noticeable if $L/D_{hy} < 10$ for subcooled burnout and $L/D_{hy} < 100$ for two-phase dryout
K_5 , axial flux distribution factor	$X > 0: \quad K_5 = q_{BLA}/q_{LOCAL}$ $X < 0: \quad K_5 = 1.0$	Tong's [17] F factor may also be used but only within narrow ranges of conditions
K_6 , flow factor, horizontal flow	Stratified flow ($G < G_1$) $K_6 = 0.0$ Nonstratified flow ($G > G_2$) $K_6 = 1.0$ Intermediate ($G_1 < G < G_2$) $K_6 = \frac{G - G_1}{G_2 - G_1}$	G_1 mass flux boundary between fully stratified and intermittent flow ($\text{kg/m}^2 \text{ s}$) G_2 mass flux boundary between intermittent and annular flow ($\text{kg/m}^2 \text{ s}$) To find G_1 and G_2 use Taitel and Dukler's [18] flow regime map
K_6 , flow factor, vertical flow	$G < -400 \text{ kg/m}^2 \text{ s} \text{ or } X = 0.0 \quad K_6 = 1.0$ $-50 < G < 10 \text{ kg/m}^2 \text{ s}$ $\text{CHF} = \text{CHF}_{G=0, X=0}(1 - \alpha)C_1$ $\left. \begin{array}{l} 10 < G < 100 \text{ kg/m}^2 \text{ s} \\ -400 < G < -50 \text{ kg/m}^2 \text{ s} \end{array} \right\} \text{linear interpolation}$	$\alpha < 0.8 \quad C_1 = 1.0$ $\alpha > 0.8 \quad C_1 = \frac{0.8 + 0.2 \rho_l/\rho_v}{\alpha + (1 - \alpha)\rho_l/\rho_v}$

of Ψ_K , Ψ_σ and Ψ_γ are illustrated in Table 6.3. Those scaling factors are commonly used in fluid modelling technique that enables the thermal and hydraulic behaviors of a given system to be predicted using a modelling fluid rather than the working fluid. The analysis shows that the prediction errors of the three dimensionless groups (Ψ_K , Ψ_σ and Ψ_γ) are approximately equal. The use of Ψ_K is recommended because of its simplest form. Thus, a dimensionless CHF table can be created and it is demonstrated in Table 6.4. (For detail, see Groeneveld et al. (1986b)).

2. Second method: The parameters of non-aqueous fluids are converted into the water-equivalent parameters so that a value of CHF can be obtained from the dimensional CHF table. Then, the real CHF for a non-aqueous fluid is calculated by:

$$CHF_{non-aqueous} = \left(\frac{CHF}{Gh_{fg}} \right) (Gh_{fg})_{non-aqueous}$$

The detail procedure can be seen in Leung (1986).

The deviations of the CHF values predicted by both methods are very little. In present work, the CHF values for non-aqueous fluids are predicted by the second method. In summary, Groeneveld's look-up CHF table predicts CHF adequately for a non-aqueous fluid as well as water. The accuracy for this table has been tested by Groeneveld et al. (1986a, 1986b).

Table 6.3: Dimensionless groups for flow parameter

Investigator	Form
Katto's scaling factor (Katto and Ohno (1984))	$\Psi_K = \frac{GD^{0.5}}{\sigma^{0.5}\rho_f^{0.5}}$
Ahmad's first scaling factor (1973)	$\Psi_\sigma = \left(\frac{G^2 D}{\rho_f \sigma}\right)^{2/3} \times \left(\frac{GD}{\mu_f}\right)^{-0.133} \times \left(\frac{GD}{\mu_g}\right)^{-0.2}$
Ahmad's second scaling factor (1973)	$\Psi_\gamma = \left(\frac{GD}{\mu_f}\right) \times \left(\frac{\gamma^{1/2} \mu_f^{2/3}}{D \rho_f^{1/2}}\right) \times \left(\frac{\mu_f}{\mu_g}\right)^{1/8}$

Table 6.4: Non-dimensional CHF table

Table 6.4 cont'd

[illegible]

Table 6.4 cont'd

λ	μ	1.00	.90	.80	.70	.60	.50	.40	.30	.25	.20	.15	.10	.05	0.00	-.05	-.10	-.15	-.20	-.25	-.30	-.40	-.50	-.60	-.70	-.80	-.90	1.00
34.7	1.02	984848	912771	842260	767901	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	1.01	984876	912800	842284	767921	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	1.00	984904	912824	842308	767941	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.99	984932	912848	842332	767965	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.98	984960	912872	842356	767989	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.97	984988	912896	842380	768013	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.96	985016	912920	842404	768037	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.95	985044	912944	842428	768061	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.94	985072	912968	842452	768085	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.93	985100	912992	842476	768109	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.92	985128	913016	842500	768133	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.91	985156	913040	842524	768157	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.90	985184	913064	842548	768181	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.89	985212	913088	842572	768205	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.88	985240	913112	842596	768229	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.87	985268	913136	842620	768253	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.86	985296	913160	842644	768277	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.85	985324	913184	842668	768301	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.84	985352	913208	842692	768325	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.83	985380	913232	842716	768349	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.82	985408	913256	842740	768373	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.81	985436	913280	842764	768397	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.80	985464	913304	842788	768421	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.79	985492	913328	842812	768445	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.78	985520	913352	842836	768469	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.77	985548	913376	842860	768493	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.76	985576	913400	842884	768517	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.75	985604	913424	842908	768541	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.74	985632	913448	842932	768565	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.73	985660	913472	842956	768589	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.72	985688	913496	842980	768613	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.71	985716	913520	843004	768637	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.70	985744	913544	843028	768661	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.69	985772	913568	843052	768685	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.68	985800	913592	843076	768709	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.67	985828	913616	843100	768733	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.66	985856	913640	843124	768757	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.65	985884	913664	843148	768781	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.64	985912	913688	843172	768805	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.63	985940	913712	843196	768829	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.62	985968	913736	843220	768853	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.61	985996	913760	843244	768877	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.60	986024	913784	843268	768901	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.59	986052	913808	843292	768925	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.58	986080	913832	843316	768949	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.57	986108	913856	843340	768973	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.56	986136	913880	843364	769000	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.55	986164	913904	843388	769024	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.54	986192	913928	843412	769048	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.53	986220	913952	843436	769072	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.52	986248	913976	843460	769096	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.51	986276	914000	843484	769120	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.50	986304	914024	843508	769144	676808	576654	467054	328660	194828	147085	91808	40353	15816	35816													
34.7	0.49	986332	914048	843532	769168	676808	576654	467054	328660	194828	147085	91808	40353	15816														

Chapter 7

Derivation of Correction Factor Method and Empirical Correlation for Cross-Flow CHF

As mentioned in Chapter 1, the objective of the present work is to predict the CHF in cross-flow boiling where a semi-empirical correlation has been presented in Chapter 4. In this chapter, a correction factor method is derived by relating the CHF's between cross flow and axial flow. To obtain the CHF for axial-flow boiling, Groeneveld's table look-up method is used as a standard in the present work. An empirical correlation for cross-flow CHF is also presented in this chapter.

7.1 CHF Relationship Between Cross and Axial Flows

The relationship of CHF's between cross flow and axial flow is defined as

$$KCHF = \frac{q_{max,cross}}{q_{max,axial}} \quad (7.1)$$

where

$q_{max,cross}$ CHF for cross flow

$q_{max,axial}$ CHF for axial flow

The $KCHF$ thus defined can be called the correction factor for correcting the CHF in axial flow to that of cross flow. It was mentioned in Chapter 1 that there are difficulties of the CHF in relating the cross and axial flows due to different flow patterns. In such circumstances, the observation of parametric trends on $KCHF$ may be an appropriate approach to attack this problem. Since the fluids involved in present study are not only water but also non-aqueous fluids such as Freon-113, dimensionless parameters are more appropriate to be used in the investigation. For axial flow, the dimensional parameters used in Groeneveld's look-up table as indicated in equation (6.3) are P , G and D , and the CHF is expressed by q_{max} . The dimensionless form of these parameters can be expressed by replacing q_{max} by Bo (Boiling number), P by r (density ratio), G by Fr (Froude number) and D by $D/0.008$, where Bo is $q_{max}/(Gh_{fg})$ and $D/0.008$ is the ratio of cylinder or tube diameter over 0.008 m. For cross flow, reviewing the existing correlations which are tabulated in Table 7.1, the dimensionless parameters used by these correlations are density ratio (r), vapor Weber number (We_g), liquid Weber number (We_f), or Katto's scaling factor (Ψ_k or $G^2D/\sigma/\rho_f$).

One of the important parameters, quality, is not used in these correlations because these correlations are developed on the basis of experimental data at saturated condition ($x = 0$). The empirical correlation presented by Hwang and Yao (1984) as illustrated in equation (3.34) has included the quality effect. However, their quality range can only apply to the saturated condition at $x \geq 0$. Consequently, the use of these correlations is limited to a narrower quality range. In present study, an attempt has been made to include the quality effect across the subcooled and saturated regions as well as the effects of pressure, mass flux and diameter. Table 7.2 lists the parameter ranges for various experimental sources used in this analysis.

7.2 Parametric Trends of $KCHF$

In present work the dimensionless parameters used to observe the trends of $KCHF$ are the density ratio (r), Froude number (Fr), quality (x) and $D/0.008$. The choice of $D/0.008$ is to be consistent with the use of Groeneveld's table look-up method. The question is why the Froude number is chosen rather than Katto's scaling factor (liquid Weber number) or Ahmad's first and second scaling factors. The analysis shows that Katto's and Ahmad's scaling factors have very similar

Table 7.1: CHF correlation for cross flow

Investigator	Form
Lienhard and Eichhorn (1976)	
For low U_∞	$\frac{\pi q_{max}}{\rho_g h_{fg} U_\infty} = 1 + \left(\frac{1}{We_g} \right)^{1/3}$
for high U_∞	$\frac{\pi q_{max}}{\rho_g h_{fg} U_\infty} = \frac{1}{169} \left(\frac{\rho_f}{\rho_g} \right)^{3/4} + \frac{1}{19.2} \left(\frac{\rho_f}{\rho_g} \right)^{1/2} \frac{1}{We_g^{1/3}}$
Katto and Haramura (1983b)	
For high U_∞	$\frac{q_{co}}{G h_{fg}} = 0.151 \left(\frac{\rho_a}{\rho_f} \right)^{0.467} \left(1 + \frac{\rho_a}{\rho_f} \right)^{1/3} \left(\frac{\sigma \rho_f}{G^2 D} \right)^{1/3}$
Yilmaz and Westwater (1980)	$\frac{q_{max}}{\rho_g h_{fg} U_\infty} = 0.369 We_g^{-0.26}$
Hasan et al. (1981)	$\frac{\pi q_{max}}{\rho_g h_{fg} U_\infty } = 0.000919 \frac{\rho_f}{\rho_g} \left(1 + \frac{16.3}{We_f^{1/3}} \right)$
Katto (1987)	$\frac{q_{co}}{G h_{fg}} = K \left(\frac{\sigma \rho_f}{G^2 D} \right)^m$ where $K = 0.00588 + 0.5 \left(\frac{\rho_a}{\rho_f} \right)^{1.11}$ $m = 0.42 \left(\frac{\rho_a}{\rho_f} \right)^{0.0428}$

Table 7.2: Sources of CHF data for cross-flow boiling used in present analysis

Parameter	Data sources				
	Broussard and Westwater (1985)	Yilmaz (1980)	Pourdashti (1984)	Kezios and Lo (1958)	Vliet (1962)
Density Ratio	205	205	50~100	800~1360	1460
Froude number	9~36	11~31	0.3~1.6	1.75~12	1.1~19
D/0.008	0.4~1.6	0.8	0.8	0.4	0.4~0.6
Quality	0	0	-0.11~0.11	-0.21~-0.04	-0.1~0
Fluid	Freon-113	Freon-113	Freon-113	water	water

effects on the $KCHF$ trend. This implies that the viscosity is not a major effect on the CHF occurrence. In addition, the Froude number shows a more smooth trend on $KCHF$ than Katto's scaling factor does. This means that the surface tension is not a major effect either. However, the Froude number doesn't involve any property of the fluid such as liquid density and vapor density. Thus, the Froude number leads to a result which is not as complicated as those using the other parameters.

The effects of these parameters on $KCHF$ trend are shown from Fig. 7.1 to Fig. 7.4. These figures show scattering distribution of $KCHF$ vs. all parameters and it seems not possible to observe clear parametric trends from these figures. Recall that in equation (7.1) the $q_{max,cross}$ and $q_{max,axial}$ should relate with these parameters directly, not the $KCHF$. Therefore, the observation of parametric trends on $q_{max,cross}$ and $q_{max,axial}$ is more useful than that on $KCHF$. Because $q_{max,cross}$ and $q_{max,axial}$ are dimensional while the parameters of Fr , r , x and $D/0.008$ are non-dimensional, equation (7.1) should be modified. Since the Boiling number (Bo) is used to represent the dimensionless form of CHF, equation (7.1) can be rewritten to

$$KCHF = \frac{\frac{q_{max,cross}}{Gh_{fg}}}{\frac{q_{max,axial}}{Gh_{fg}}} = \frac{Bo_{cross}}{Bo_{axial}} \quad (7.2)$$

The terms of Gh_{fg} shown in the numerator and denominator are equivalent to each other so that they can be eliminated. According to equation (7.2), the following analysis leads to the observation of parametric trends on Bo_{cross} and Bo_{axial} .

7.3 Parametric Trends of CHF

7.3.1 Mass flux effect

The Froude number Fr is usually used to represent the velocity or mass flux effect in the non-dimensional correlating technique. As discussed in Chapter 4, the Froude number has been used successfully in Ungar and Eichhorn's (1988) model. It is obvious that the flow patterns for axial and cross flows are completely different because the former is internal flow and the latter is external flow to the heated element. However, the external flow is still confined by the channel walls. Therefore, the Froude number hereby represents only the extent of velocity or mass flux effect in the flow channel. For cross-flow boiling, the Froude number is calculated based

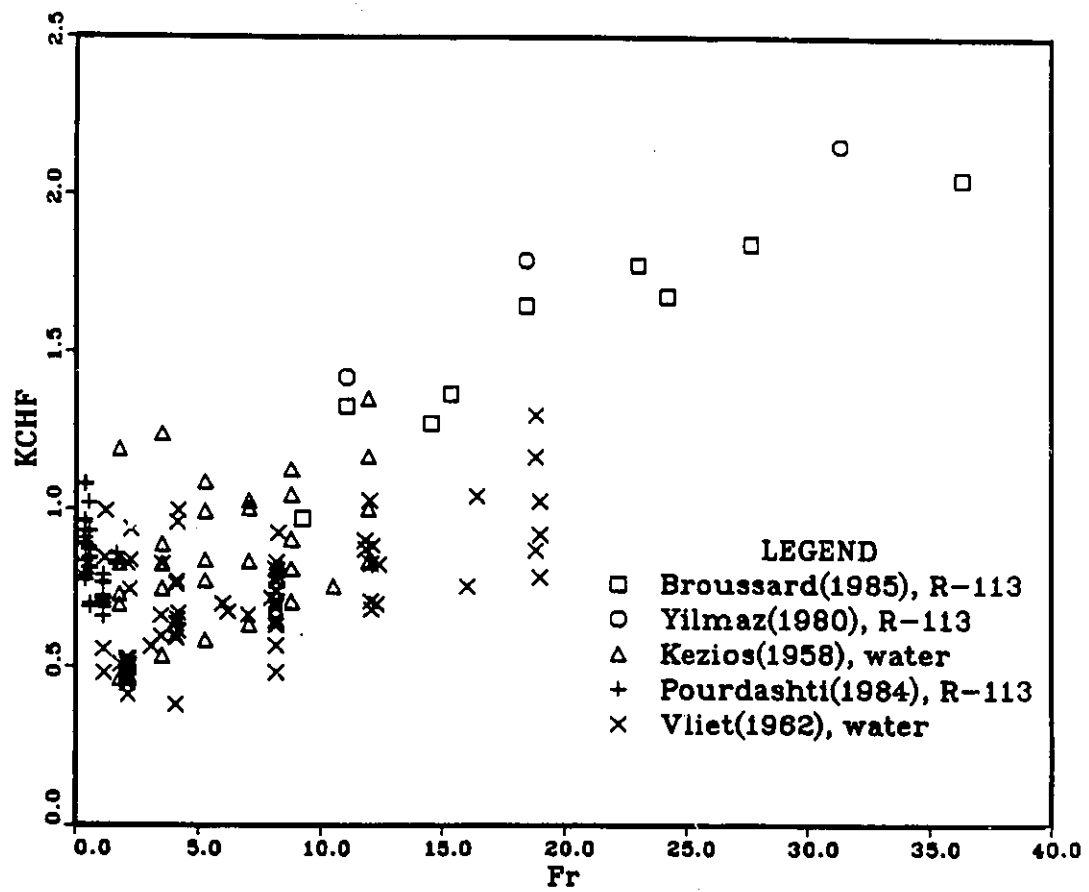


Figure 7.1: Distribution of $KCHF$ vs. Fr in present analysis

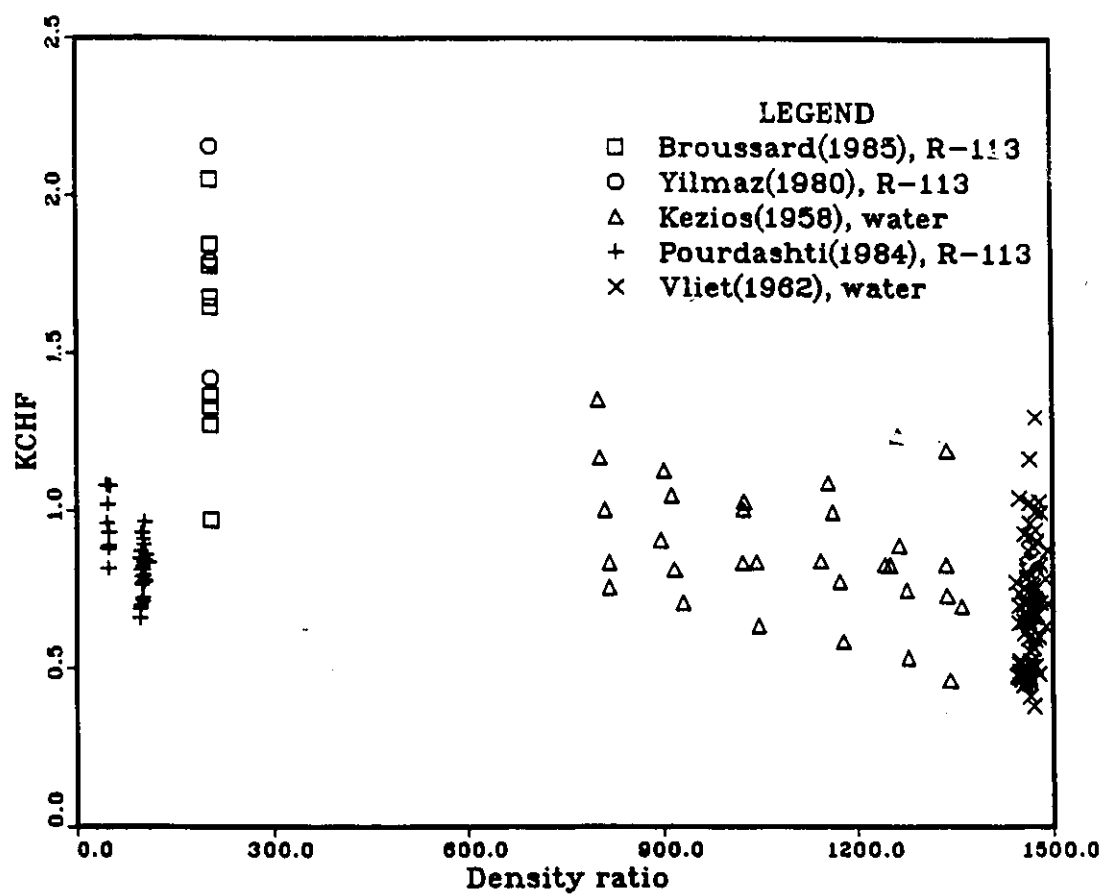


Figure 7.2: Distribution of $KCHF$ vs. r in present analysis

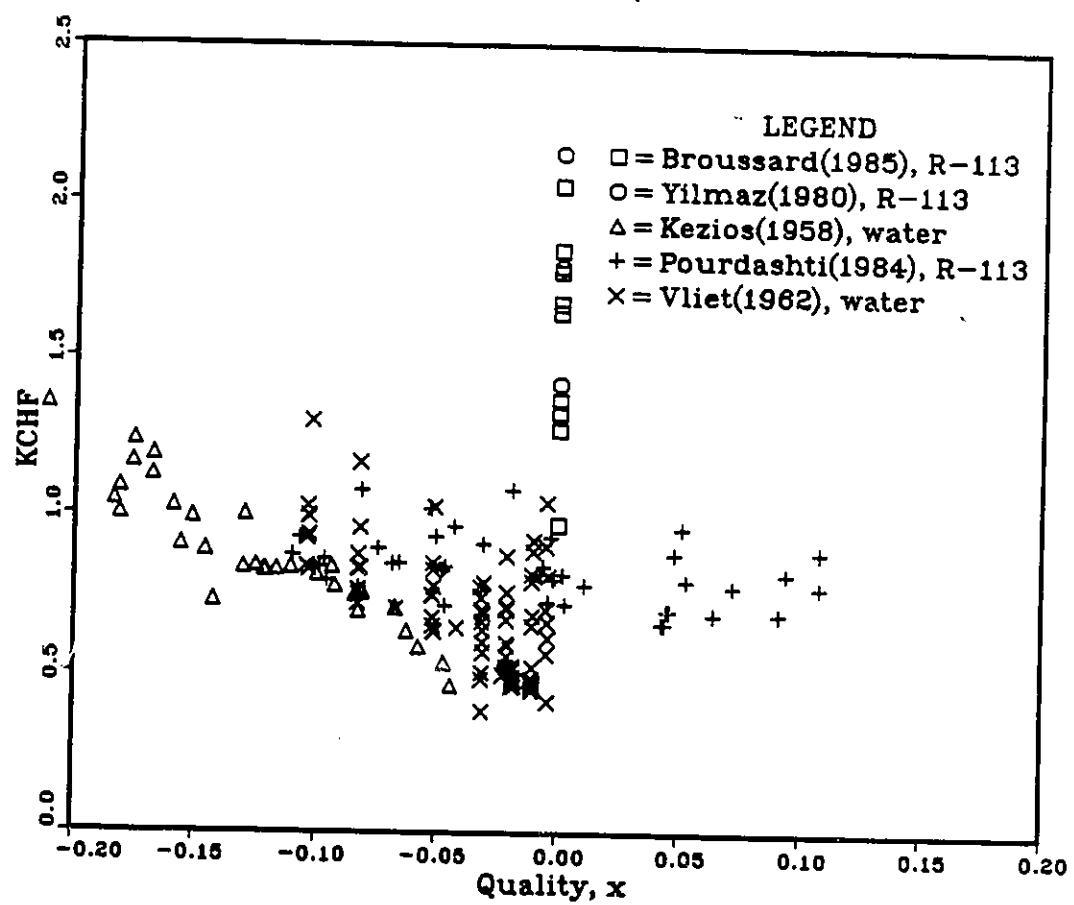


Figure 7.3: Distribution of $KCHF$ vs. x in present analysis

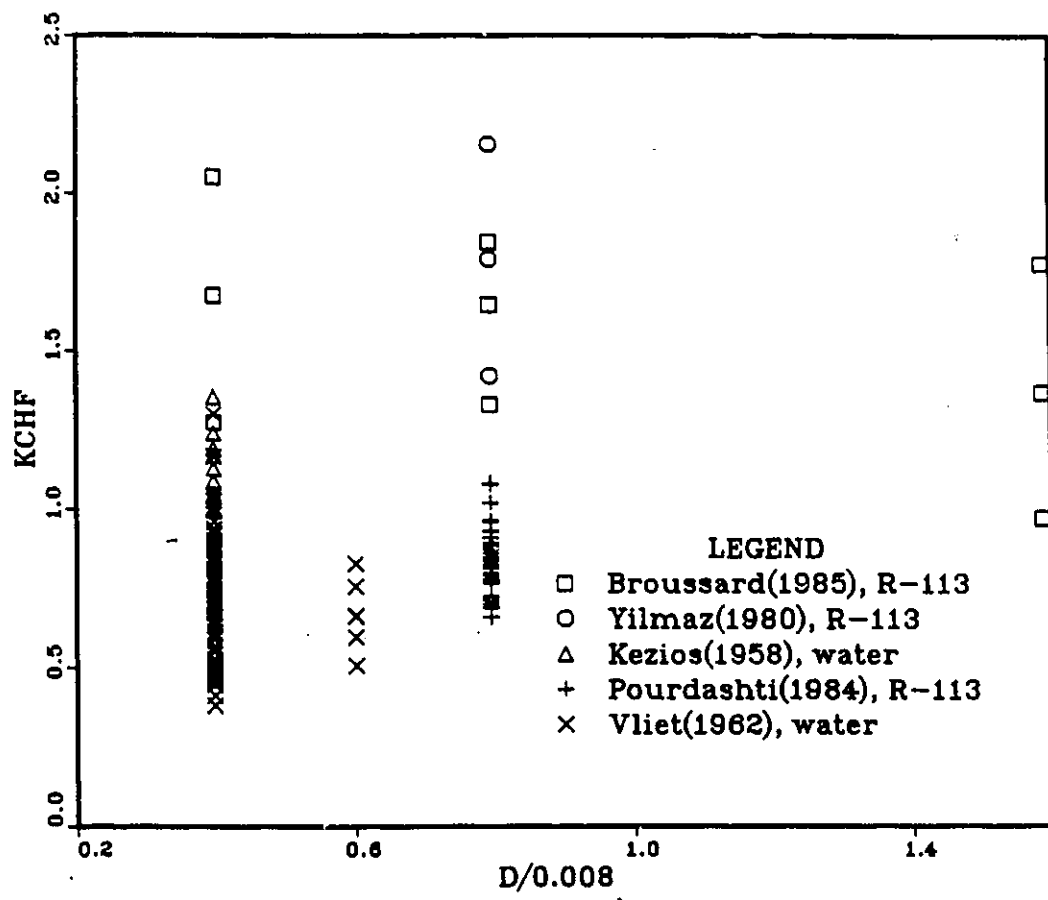


Figure 7.4: Distribution of $KCHF$ vs. $D/0.008$ in present analysis

on the velocity (U) which is obtained from the mean flow area model as discussed in Chapter 5. Fig. 7.5 shows the mass flux effect on the dimensionless CHF (Bo) for cross flow in different investigations, while Fig. 7.6 shows the same effect on boiling number for axial flow predicted by Groeneveld's table look-up method corresponding to the same test conditions of the previous figure. These two figures show that there are similar trends of Bo vs. Fr for both the cross and axial flows.

7.3.2 Quality effect

For quality less than zero, generally the Boiling number increases with decreasing quality for both axial and cross flows, and the absolute value of the slope for this trend becomes smaller when Fr becomes larger (see Figure 7.7 to 7.10), while in the region of quality greater than zero, the Boiling number decreases when quality increases. In Fig. 7.11 and Fig. 7.12, the trends for quality effect on Boiling number are continuous across the subcooled and saturated regions for both axial and cross flows, and the trends of quality effect on Boiling number for both cases are very similar to each other. Thus, the Boiling number can be correlated as a function of quality, or $Bo = f(.., x, ..)$, for axial and cross flows, respectively. However, in practical application it is more useful to express the quality effect by $Bo = f(.., 1 - x, ..)$. In such case, the parameter $1 - x$ has a positively proportional effect on Bo .

7.3.3 Diameter effect

In Chapter 6, the diameter effect on CHF in axial flow has been introduced by equations (6.2) and (6.4). They show that the CHF increases when the diameter decreases. For cross flow, the diameter effect has been introduced by Broussard and Westwater (1985), Beecher (1948), and Vliet and Leppert (1964a and 1964b) as discussed in Section 2.4. All of these investigations indicated that the CHF is inversely proportional to the diameter. Fig. 7.13 shows the effect of dimensionless diameter ($D/0.008$) on the trend of Bo for Broussard and Westwater's (1985) investigation.

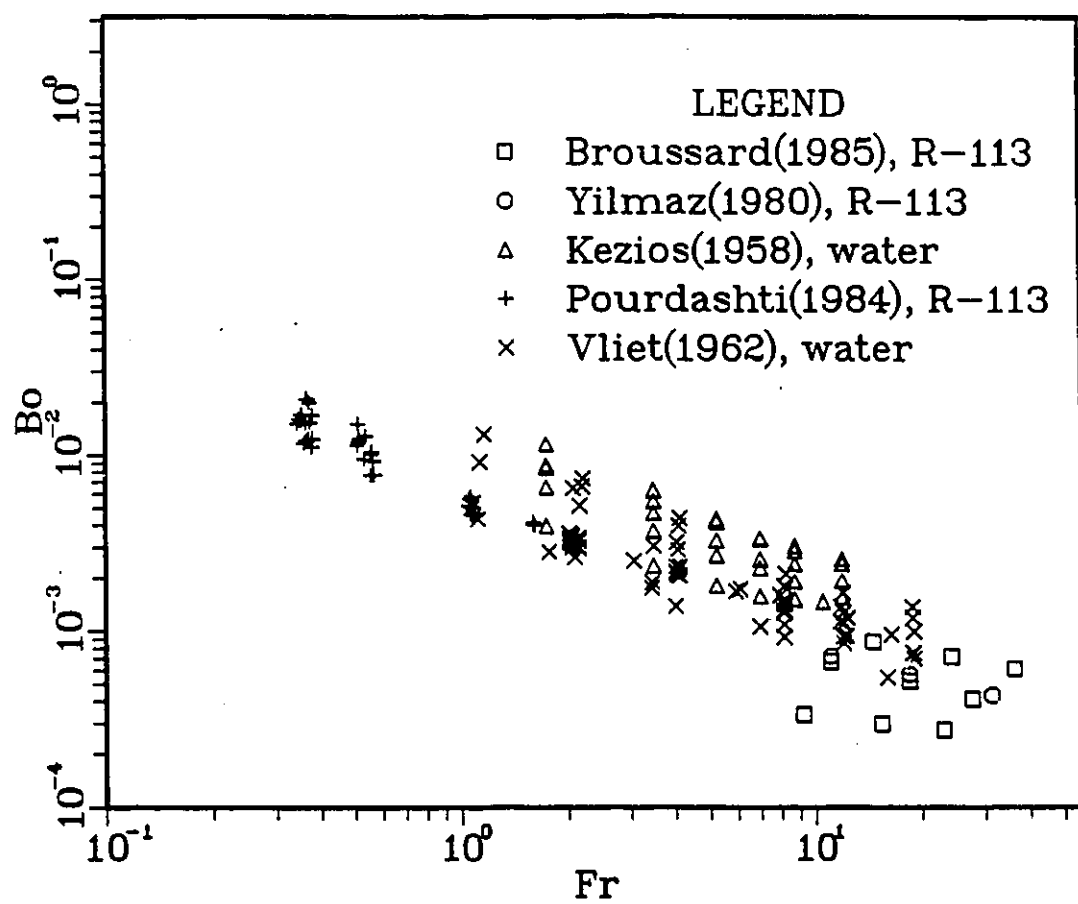


Figure 7.5: Dimensionless CHF vs. Fr for all cross flow experimental data used in present analysis

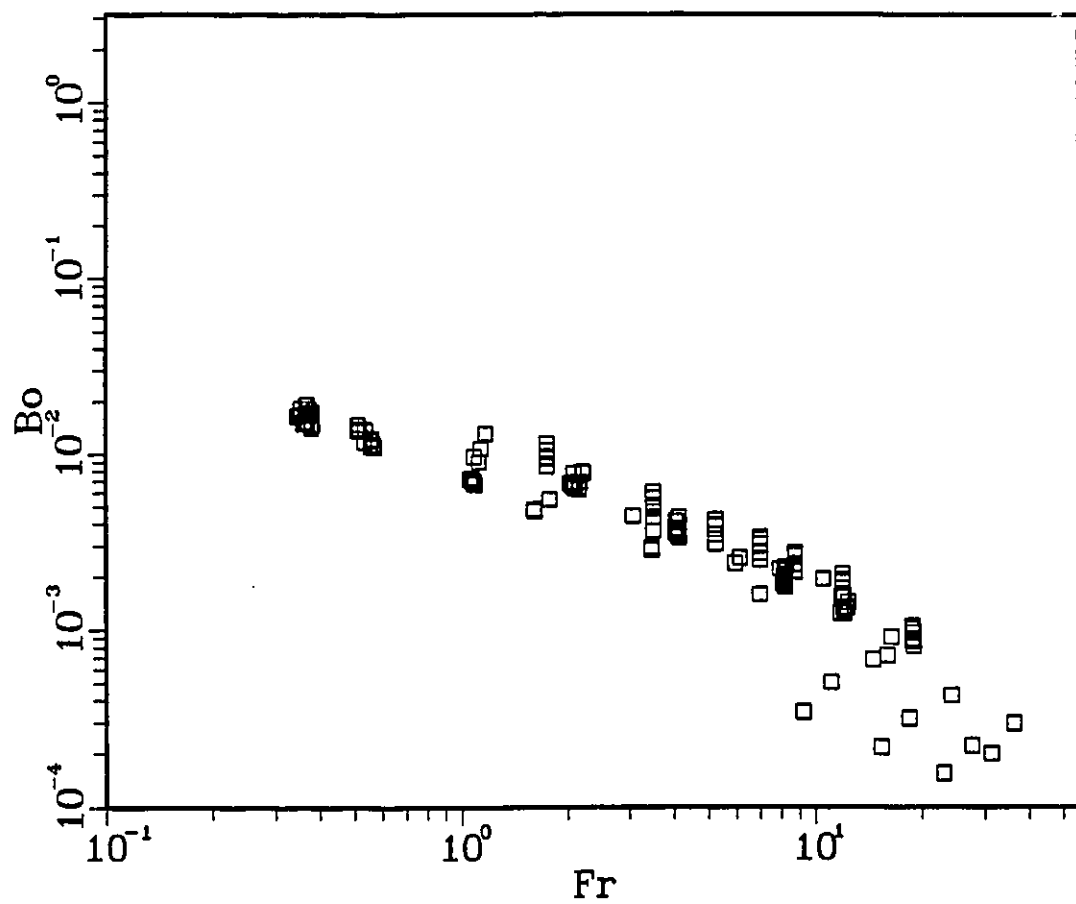


Figure 7.6: Dimensionless CHF for axial flow predicted by Groeneveld's look-up method vs. Fr with the same parameter ranges of previous figure

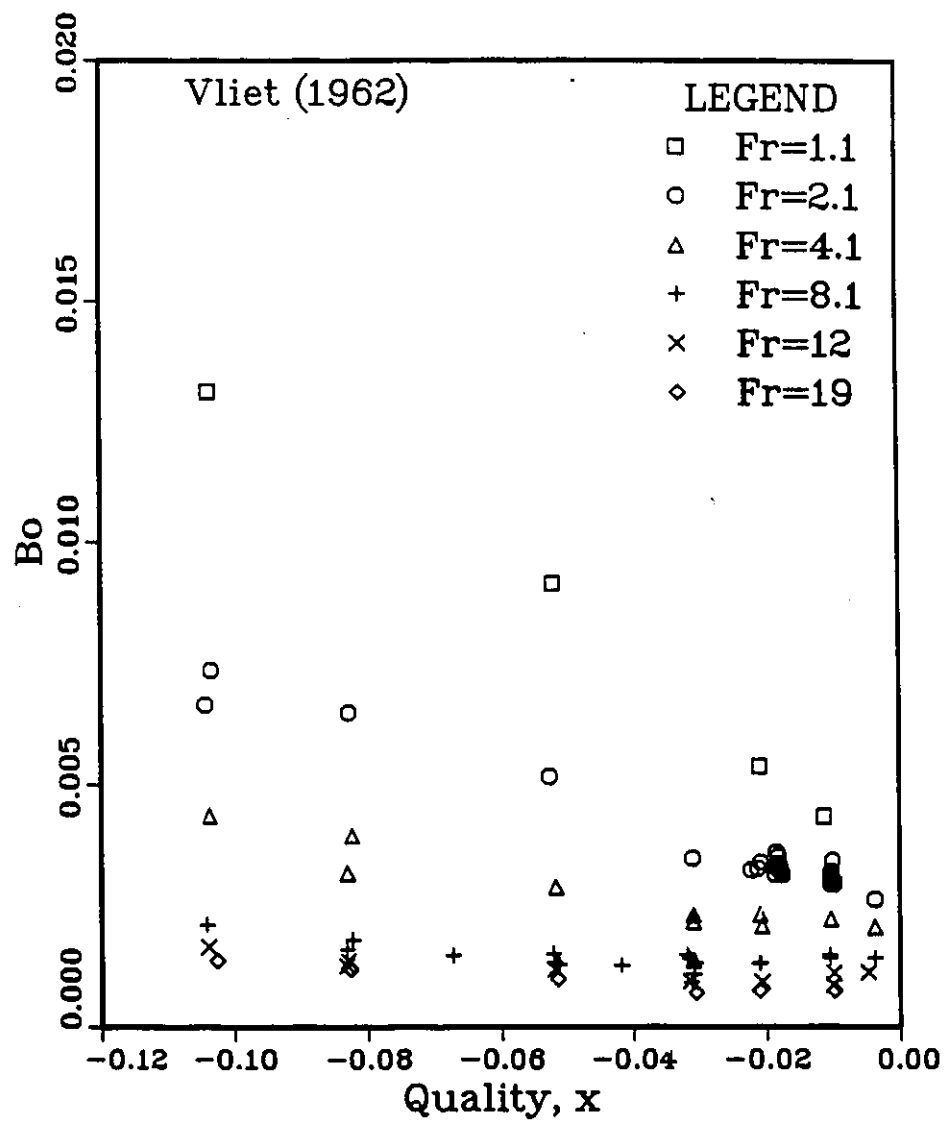


Figure 7.7: Quality effect on the dimensionless CHF for Vliet's (1962) cross flow data.

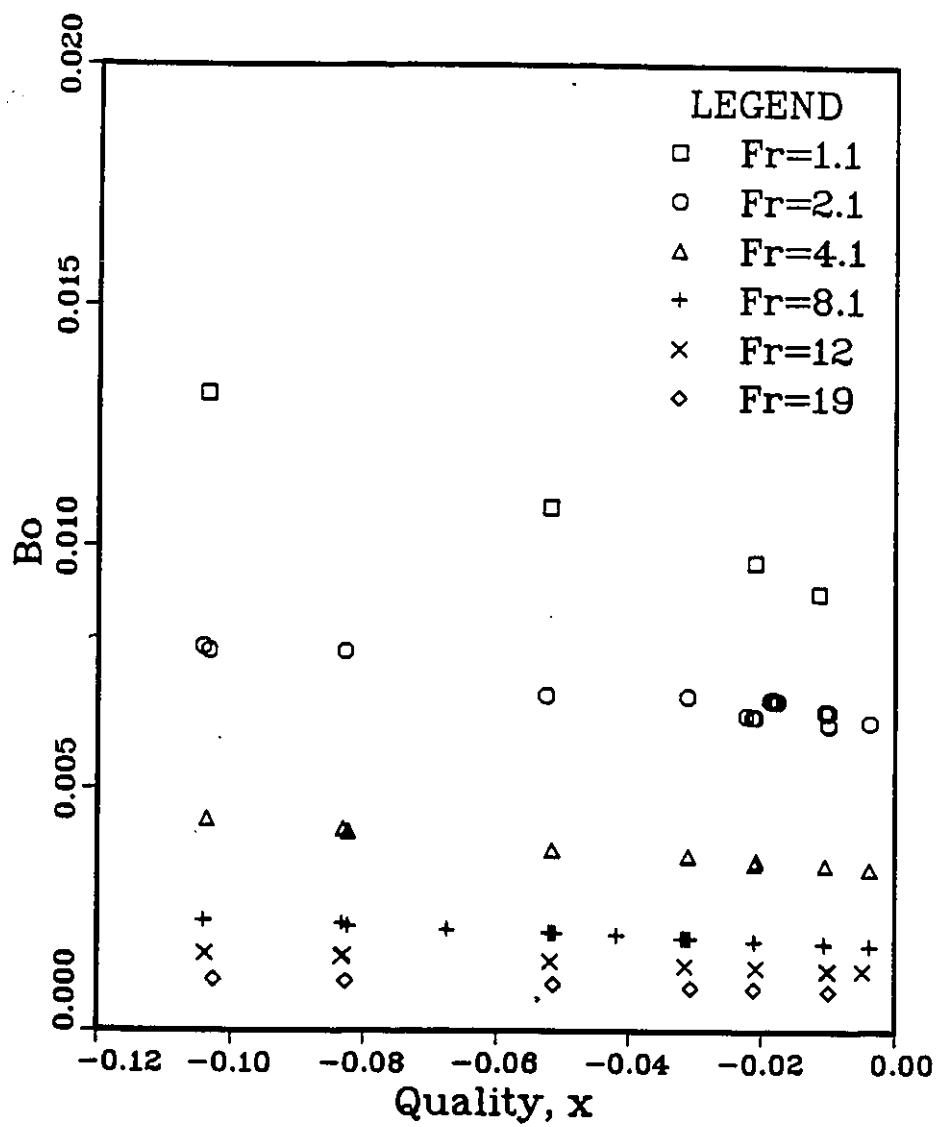


Figure 7.8: Quality effect on the dimensionless CHF for axial flow predicted by Groeneveld's table look-up method according to Vliet's test condition

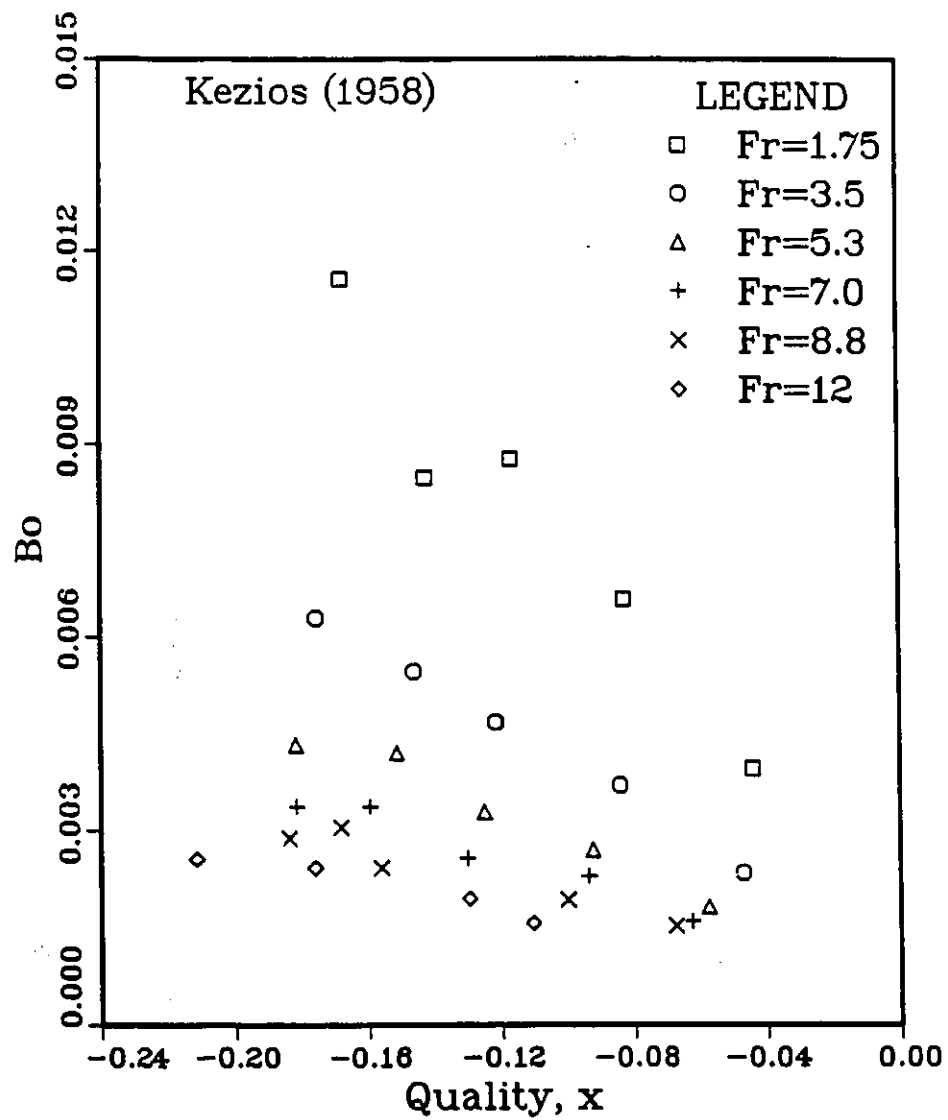


Figure 7.9: Quality effect on the dimensionless CHF for Kezios's (1958) cross flow data

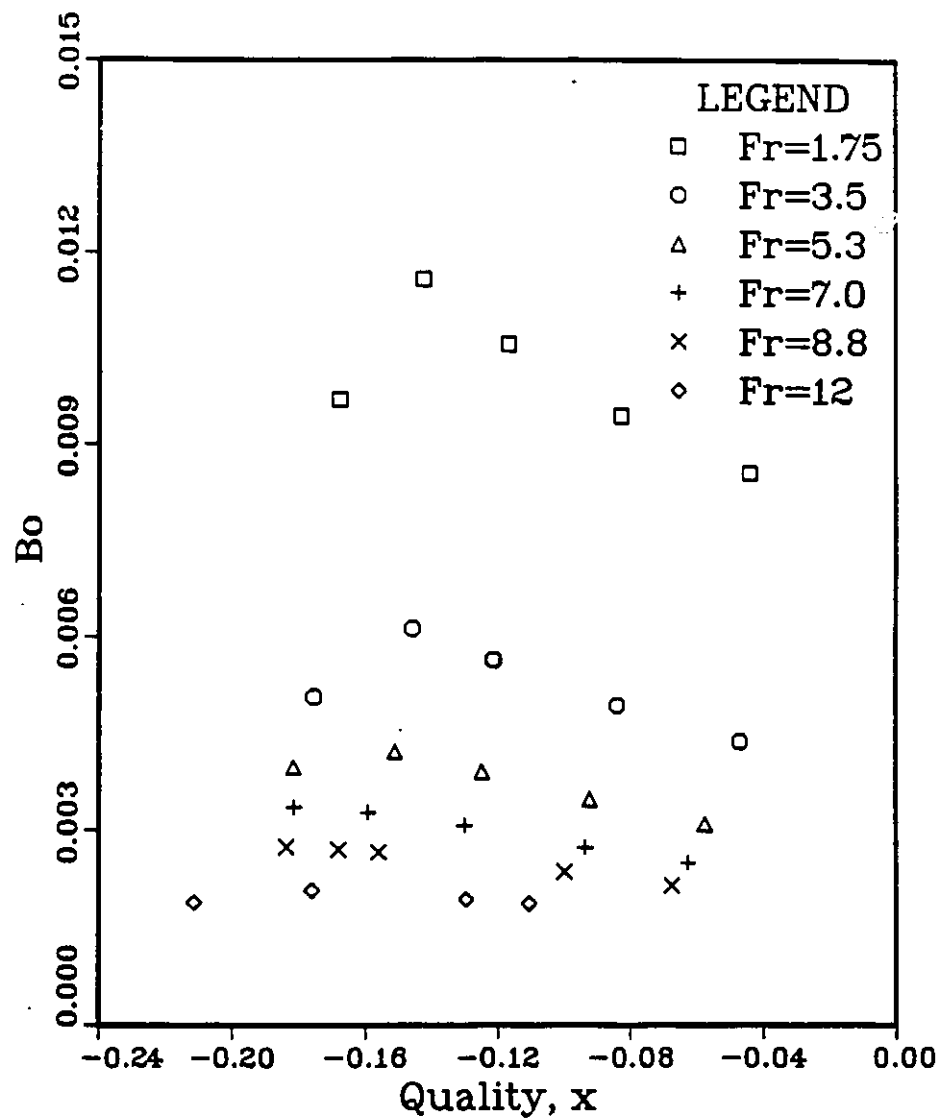


Figure 7.10: Quality effect on the dimensionless CHF for axial flow predicted by Groeneveld's table look-up method according to Kezios's test condition

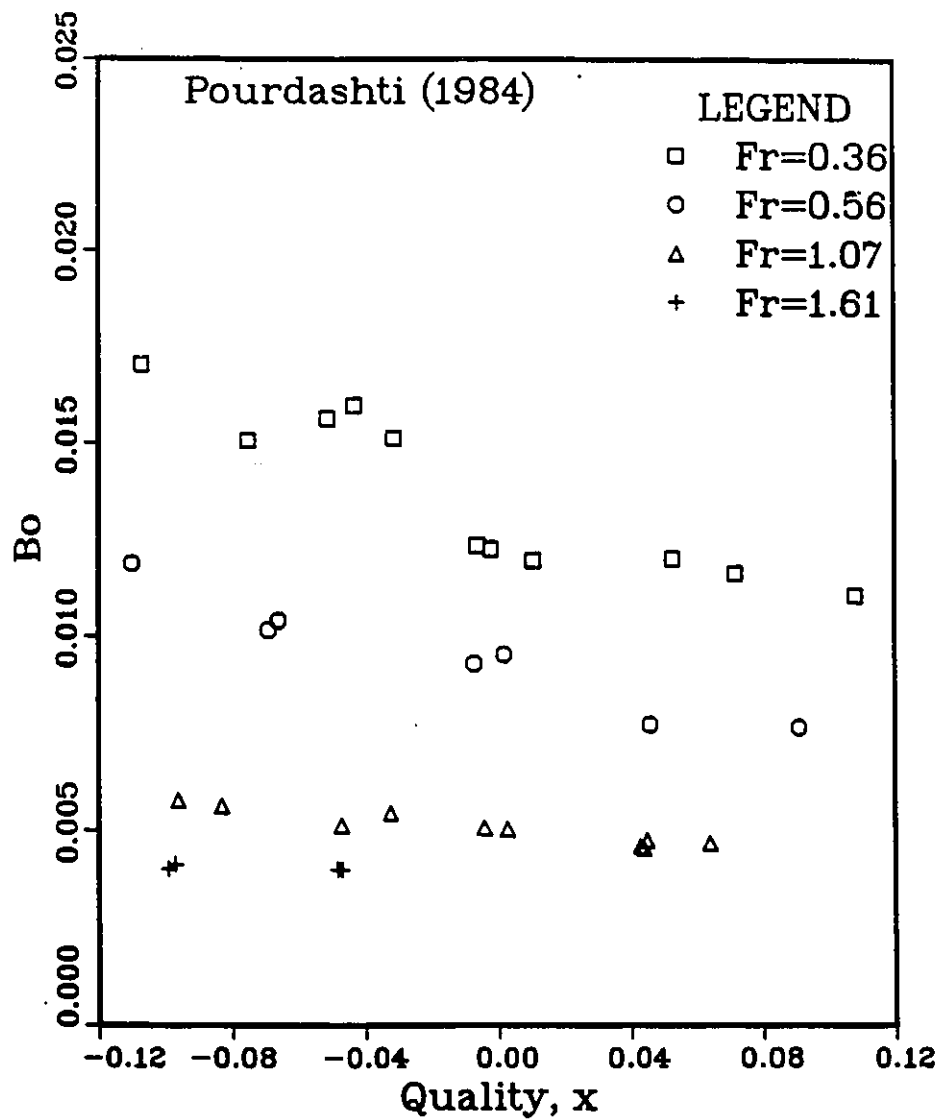


Figure 7.11: Quality effect on the dimensionless CHF for Pourdashti's (1984) cross flow data

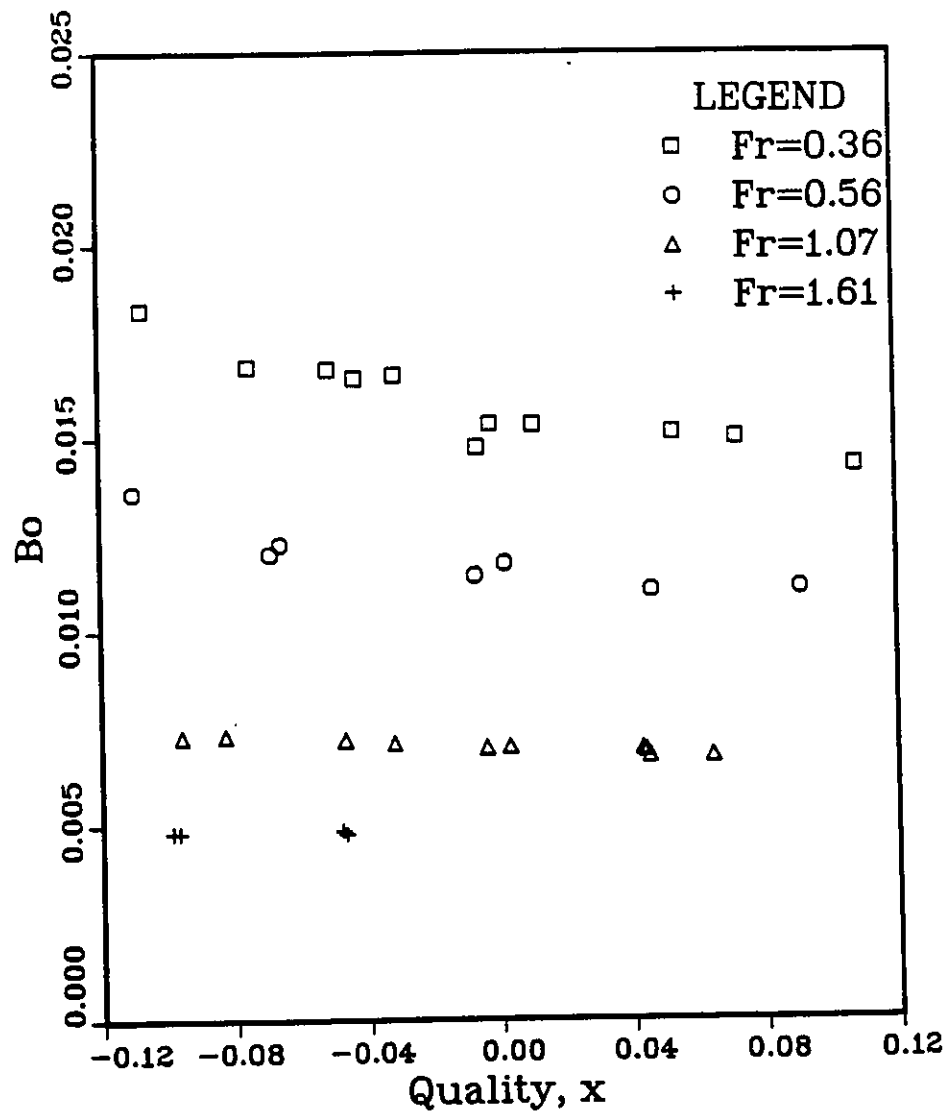


Figure 7.12: Quality effect on the dimensionless CHF for axial flow predicted by Groeneveld's table look-up method according to Pourdashti's test condition

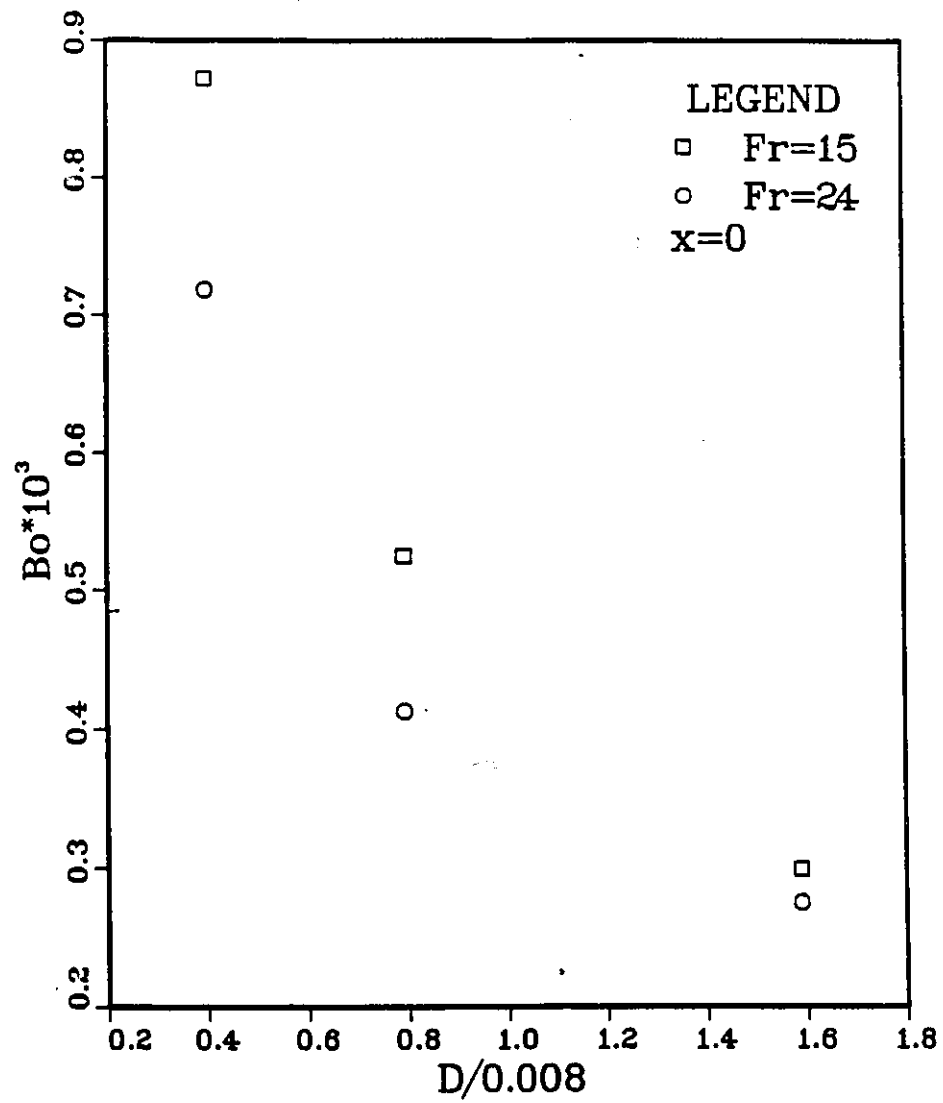


Figure 7.13: Dimensionless diameter effect on the trend of Bo for the investigation by Broussard and Westwater (1985)

7.3.4 Density ratio effect

Because of the scatter in the distribution of the cross-flow CHF data for different investigation sources, the density ratio effect on Bo is not clear. For axial-flow boiling, the CHF data predicted from Groeneveld's look-up table also shows a fluctuation at the same range of the test parameters of cross flow. However, the density ratio effect is still an important parameter to predict the CHF for both the cross and axial flows as shown in the latter analysis.

7.4 Derivation of $KCHF$

Since parametric trends of Bo_{cross} and Bo_{axial} have been observed in Section 7.3, which show that most parameters result in smooth trends on Bo_{cross} and Bo_{axial} , it implies that Bo_{cross} and Bo_{axial} relate directly with these parameters. According to equation (7.2), $KCHF$ should also relate with these parameters in spite of the scattering distributions of $KCHF$ vs. Fr , r , x and $D/0.008$ as shown from Fig. 7.1 to Fig. 7.4. Thus, an empirical correlation for $KCHF$ can be obtained by using the regression method:

$$KCHF = 3.25 Fr^{0.207} r^{-0.274} (1-x)^{0.04+0.003r} \left(\frac{D}{0.008} \right)^{0.094} \quad (7.3)$$

also equation (7.3) can be expressed by

$$\frac{q_{max,cross}}{q_{max,axial}} = 3.25 Fr^{0.207} r^{-0.274} (1-x)^{0.04+0.003r} \left(\frac{D}{0.008} \right)^{0.094} \quad (7.4)$$

From the result shown in equation (7.4), the CHF for cross-flow boiling can be predicted by multiplying the CHF obtained from Groeneveld's look-up table to the correction factor $KCHF$. The procedure is that:

1. Select the local parameters of test condition (e.g. P , G , D and x).
2. Calculate the CHF for axial flow from Groeneveld's look-up table with the inputs of P , G , D and x .
3. Calculate the dimensionless parameters Fr , r and $D/0.008$ from the inputs of P , G and D .

4. Calculate the CHF for cross-flow boiling by using equation (7.4).

The RMS error for the cross-flow CHF obtained from the above procedure and compared to the experimental data is 16.2%. For each test source, the RMS error is illustrated in Table 7.3.

7.5 Empirical CHF Correlation for Cross-flow Boiling

As emphasized before, the objective of this study is to predict the CHF in cross-flow boiling. After the observation of the parametric trends on Bo as discussed in Section 7.3, one question would come up, i.e. whether the CHF in cross flow can be predicted by correlating Bo_{cross} as a function of Fr , r , x and $D/0.008$ directly. If the result is more accurate than that predicted by using the correction factor, why bother to use the correction factor method. Therefore, an empirical correlation of Bo_{cross} vs. Fr , r , x and $D/0.008$ is also obtained by using the regression method:

$$Bo_{cross} = 0.018 Fr^{-0.758} r^{-0.29} (1-x)^{1.068+0.0048r} \left(\frac{D}{0.008} \right)^{-0.92} \quad (7.5)$$

For CHF in cross-flow boiling, it can be calculated by:

$$q_{max,cross} = Bo_{cross} \times Gh_{fg} \quad (7.6)$$

Recall that in Section 7.3.4 the density ratio effect on Bo can not be observed because of the scatter in the distribution of CHF data. But, from equation (7.5) this parameter shows a simply and inversely proportional effect on Bo_{cross} . This effect also can be seen schematically in Fig. 7.14. The RMS error for this correlation with the total 177 data is 13.4%. A summary of the assessment for this correlation to the individual data source is illustrated in Table 7.4. Note that Kezios and Lo's CHF data were taken for the fluid flowing horizontally and across the heated cylinder.

7.6 Discussion

Surprisingly we result in two prediction methods for CHF in cross-flow boiling in this chapter. The results show that the CHF for cross-flow boiling predicted by

Table 7.3: Assessment of the CHF predicted by correction factor method to individual test source

Data source	Number of data	RMS error (%)
Broussard and Westwater (1985)	9	18.6
Yilmaz (1980)	3	23.9
Pourdashti (1984)	40	14.6
Kezios and Lo (1958)	31	18.6
Vliet (1962)	94	15.1
All data	177	16.2

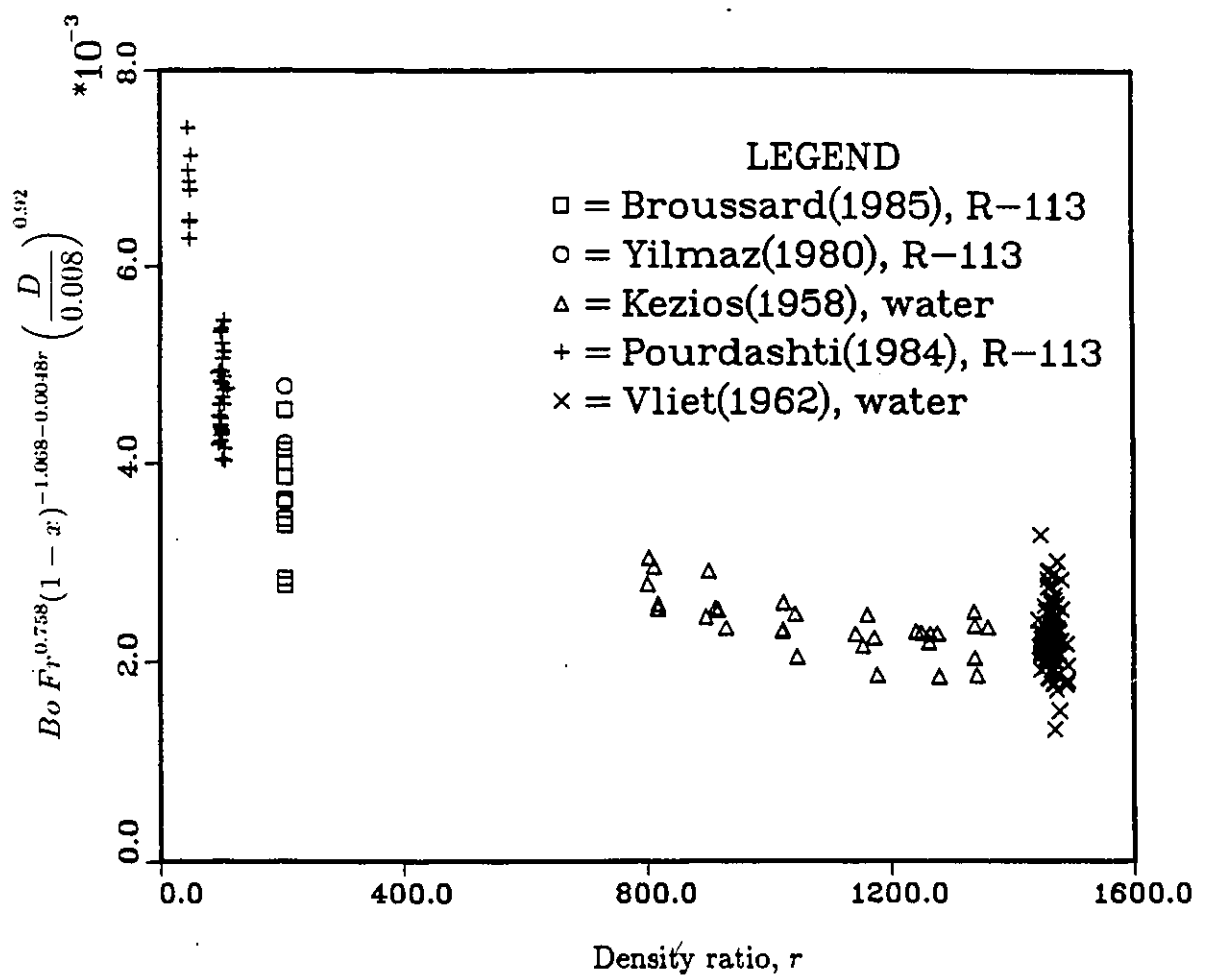


Figure 7.14: Density ratio effect on the trend of dimensionless CHF for all cross flow data

Table 7.4: Assessment of empirical CHF correlation for cross flow to each test source

Data source	Number of data	fluid	RMS error (%)
Broussard and Westwater (1985)	9	freon-113	19.4
Yilmaz (1980)	3	freon-113	13
Pourdashti (1984)	40	freon-113	10.4
Kezios and Lo (1958)	31	water	9.6
Vliet (1962)	94	water	14.9
All data	177		13.4

the empirical correlation illustrated in equation (7.5) is more accurate than that calculated by multiplying the correction factor to the axial-flow CHF predicted by Groeneveld's look-up table. However, the difference of the RMS error between both methods is less than 3%, which is due to the prediction error for Groeneveld's table look-up method with the actual value of axial-flow CHF.

The comparisons of cross-flow CHF predicted by both methods with the experimental data are illustrated from Fig. 7.15 to Fig. 7.21. From these figures, in most cases the equation (7.5) does predict the CHF more accurately than the correction factor method. The comparison of experimental CHF vs. the CHF predicted by the empirical correlation, equation (7.5), is shown in Fig. 7.22, which represents about 80% of the data with $\pm 15\%$ accuracy.

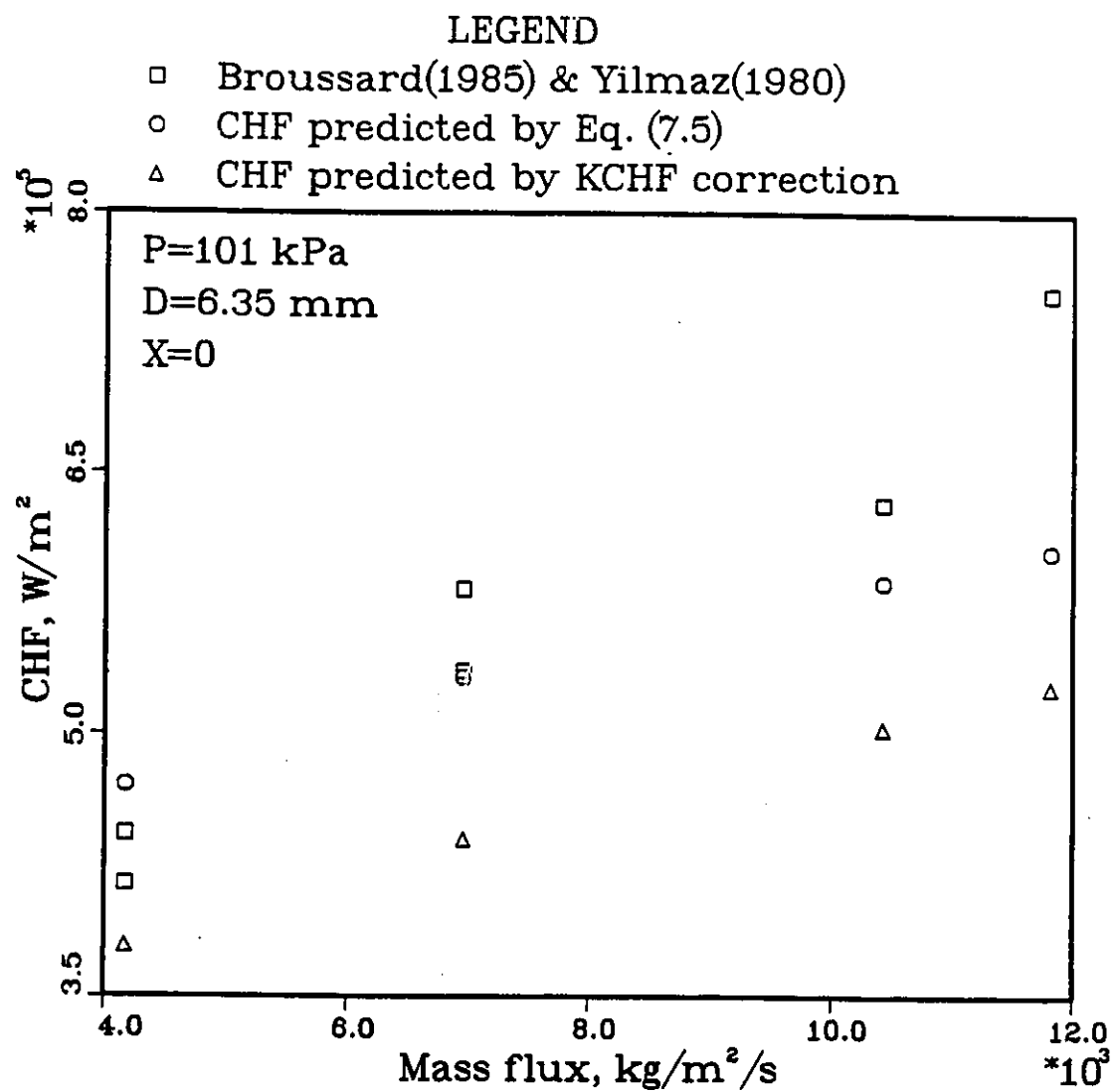


Figure 7.15: Comparison of measured CHF with the CHF's calculated by both prediction methods on the effect of mass flux

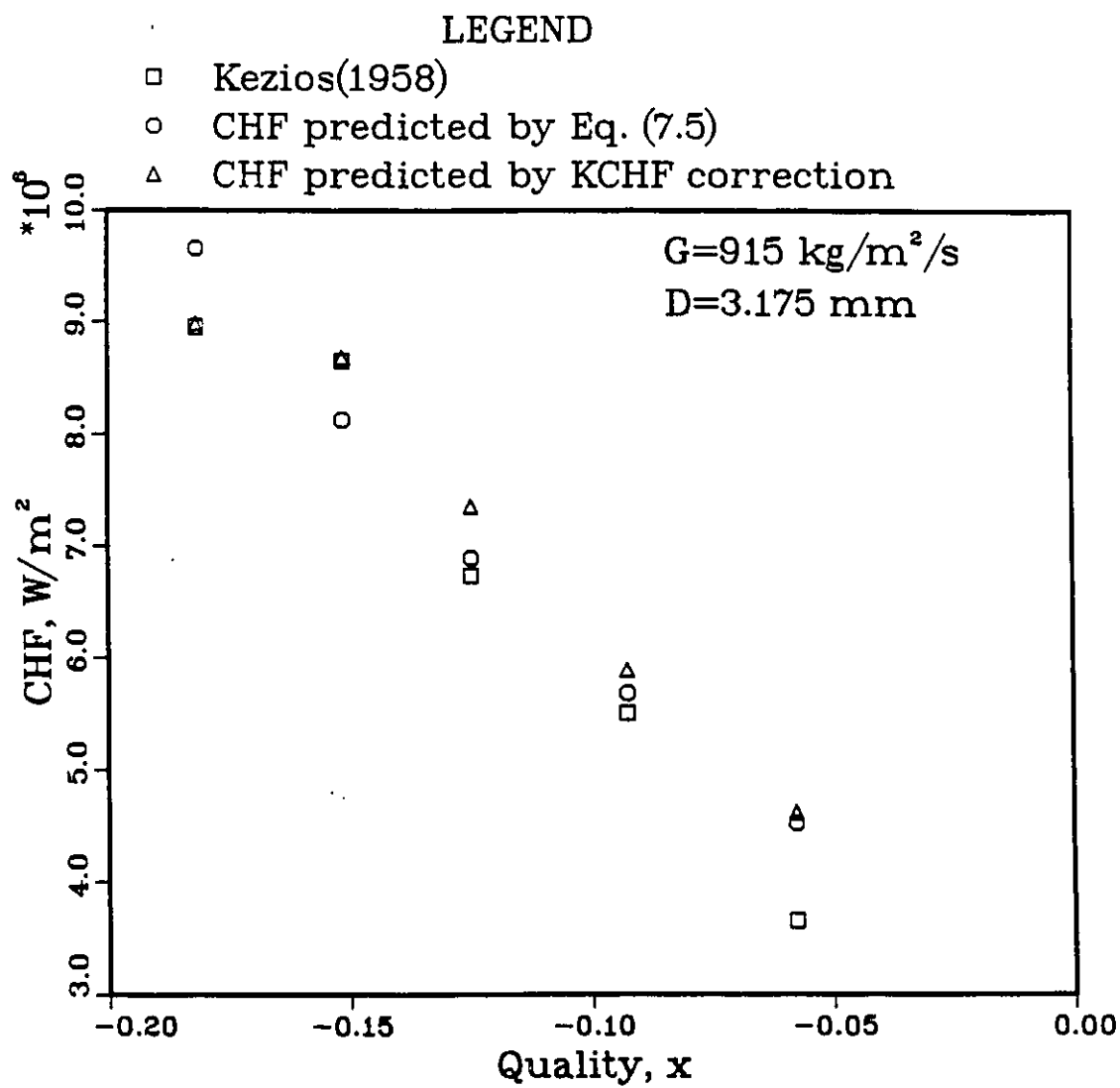


Figure 7.16: Comparison of measured CHF with the CHF's calculated by both prediction methods on the effect of quality

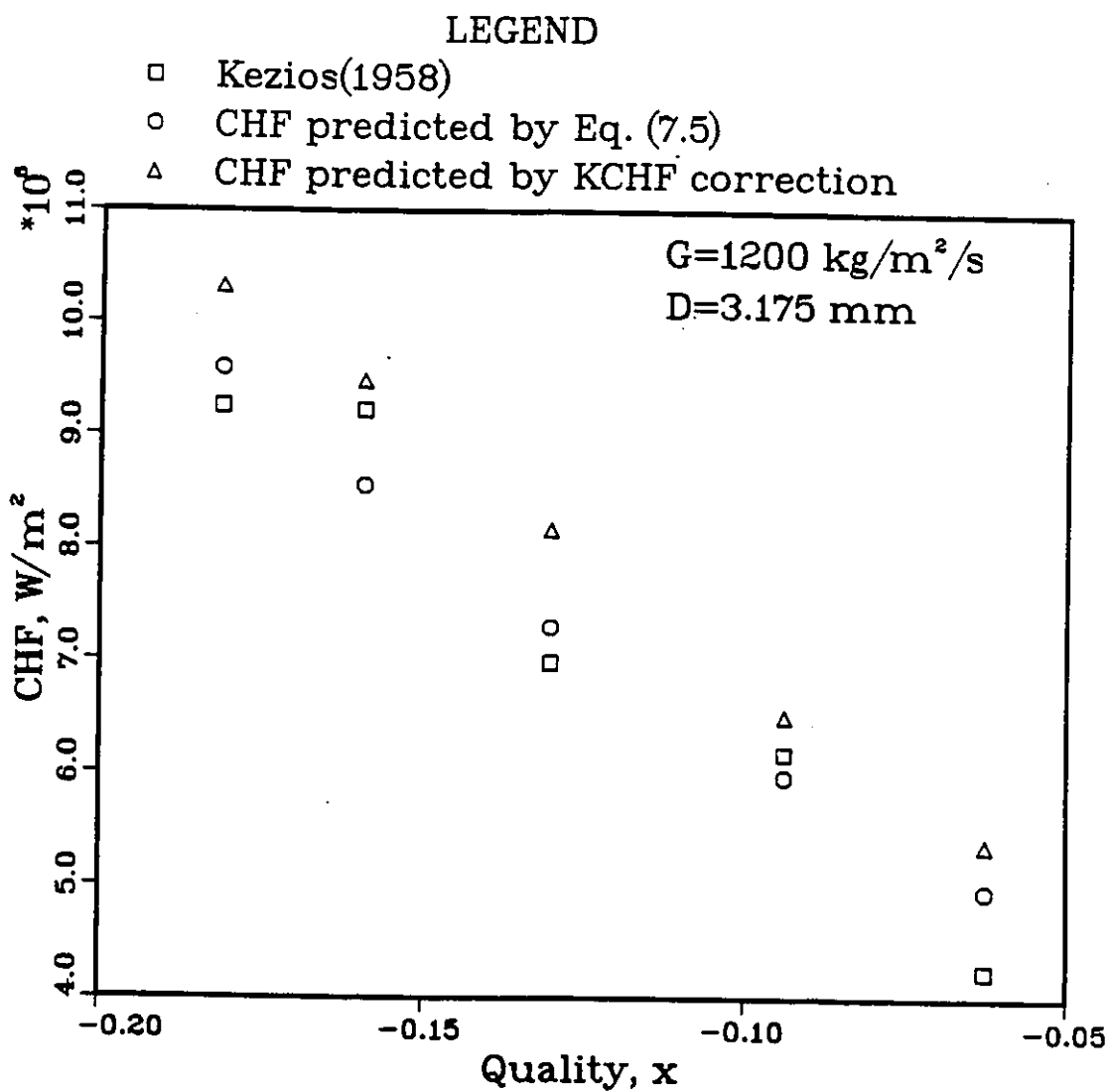


Figure 7.17: Comparison of measured CHF with the CHF's calculated by both prediction methods on the effect of quality

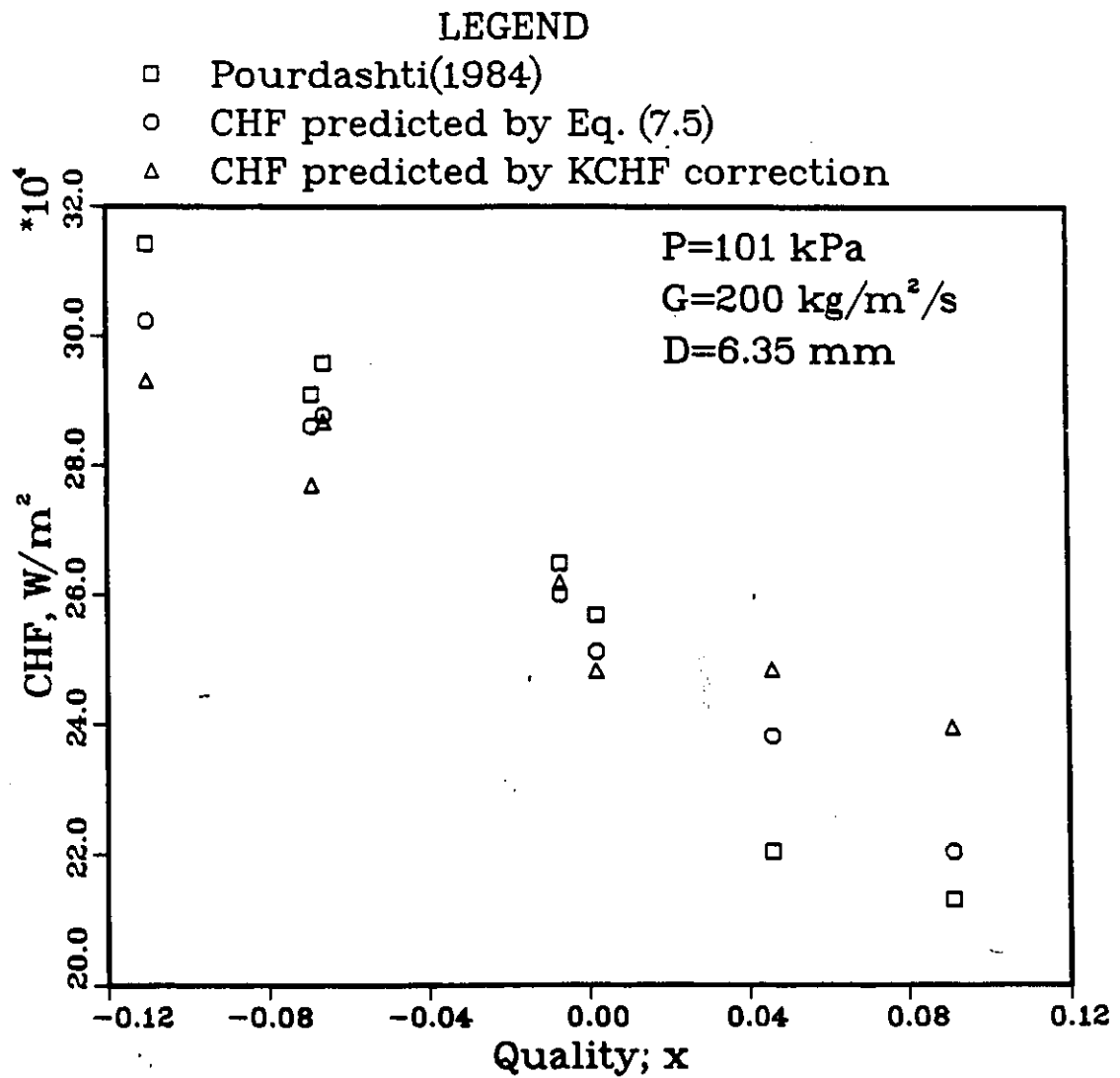


Figure 7.18: Comparison of measured CHF with the CHF's calculated by both prediction methods on the effect of quality

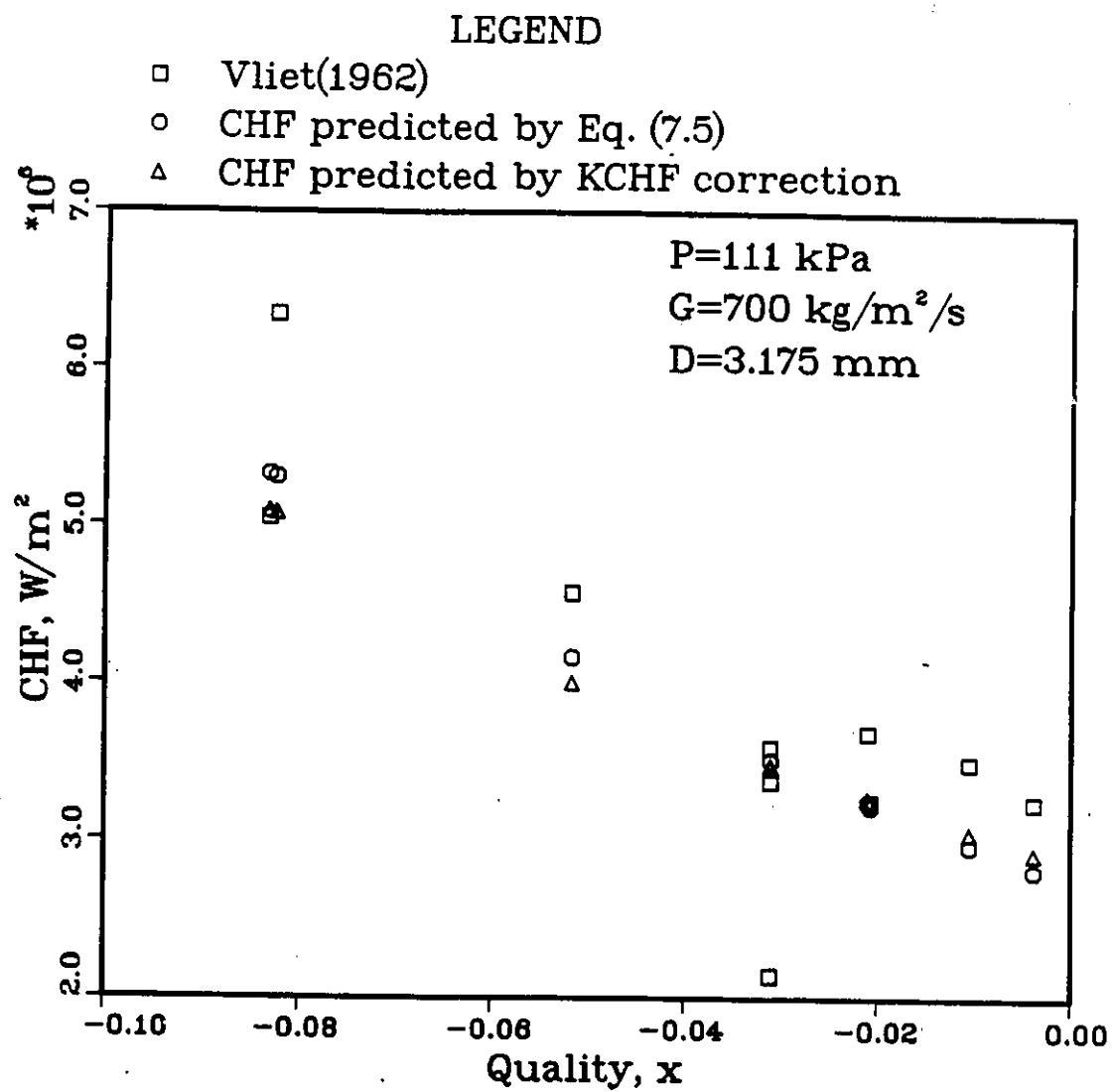


Figure 7.19: Comparison of measured CHF with the CHF's calculated by both prediction methods on the effect of quality

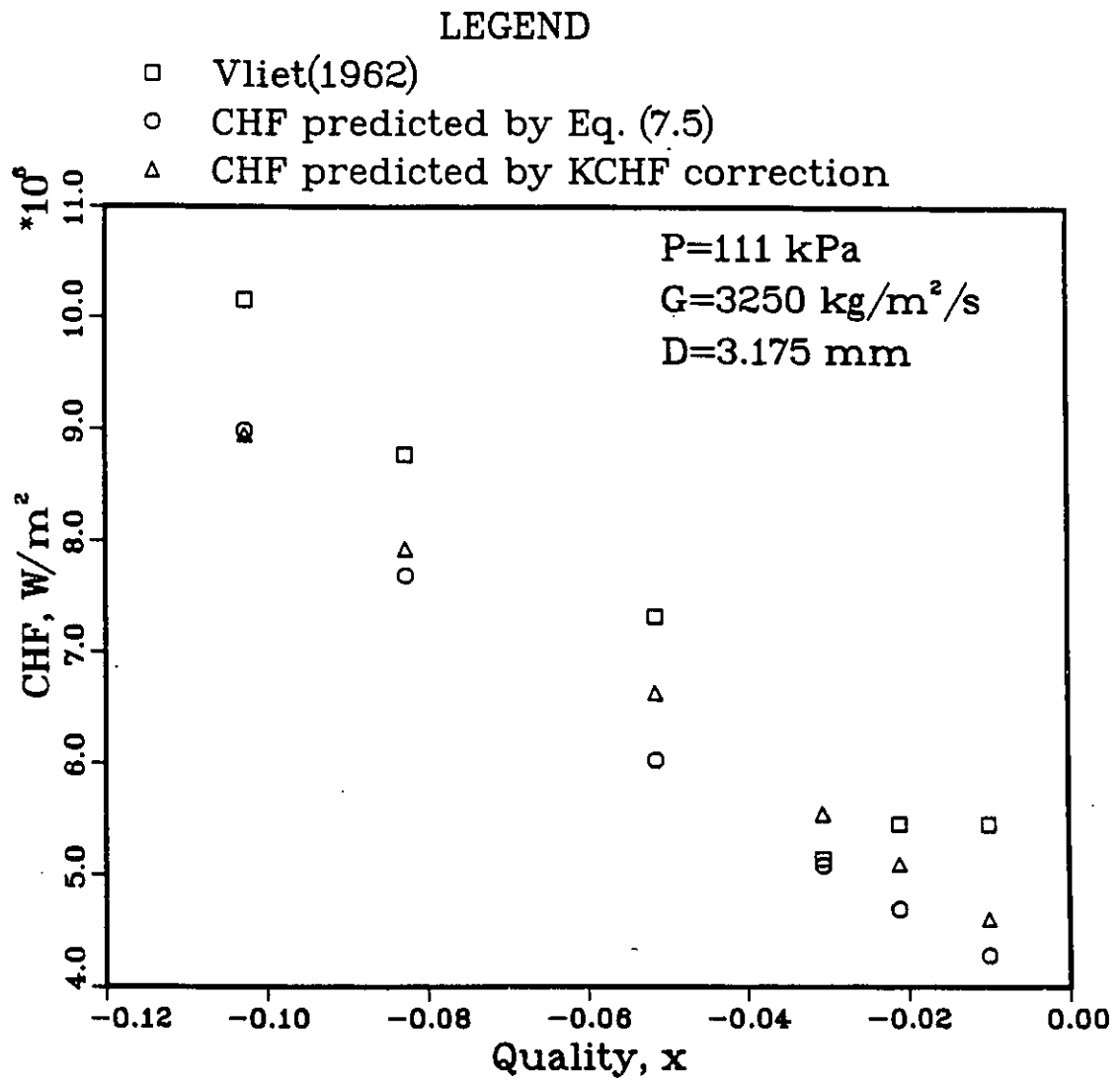


Figure 7.20: Comparison of measured CHF with the CHF's calculated by both prediction methods on the effect of quality

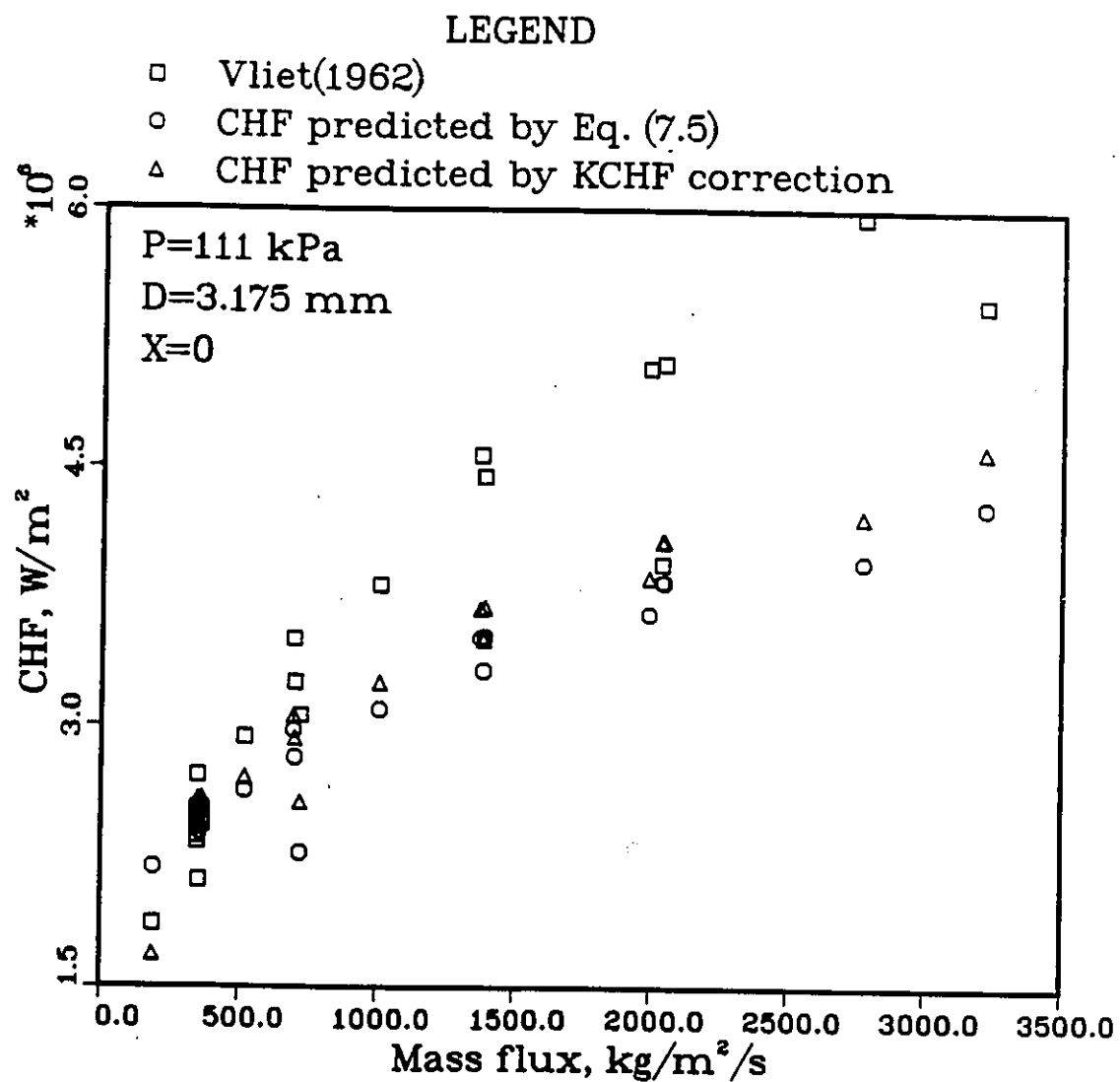


Figure 7.21: Comparison of measured CHF with the CHF's calculated by both prediction methods on the effect of mass flux

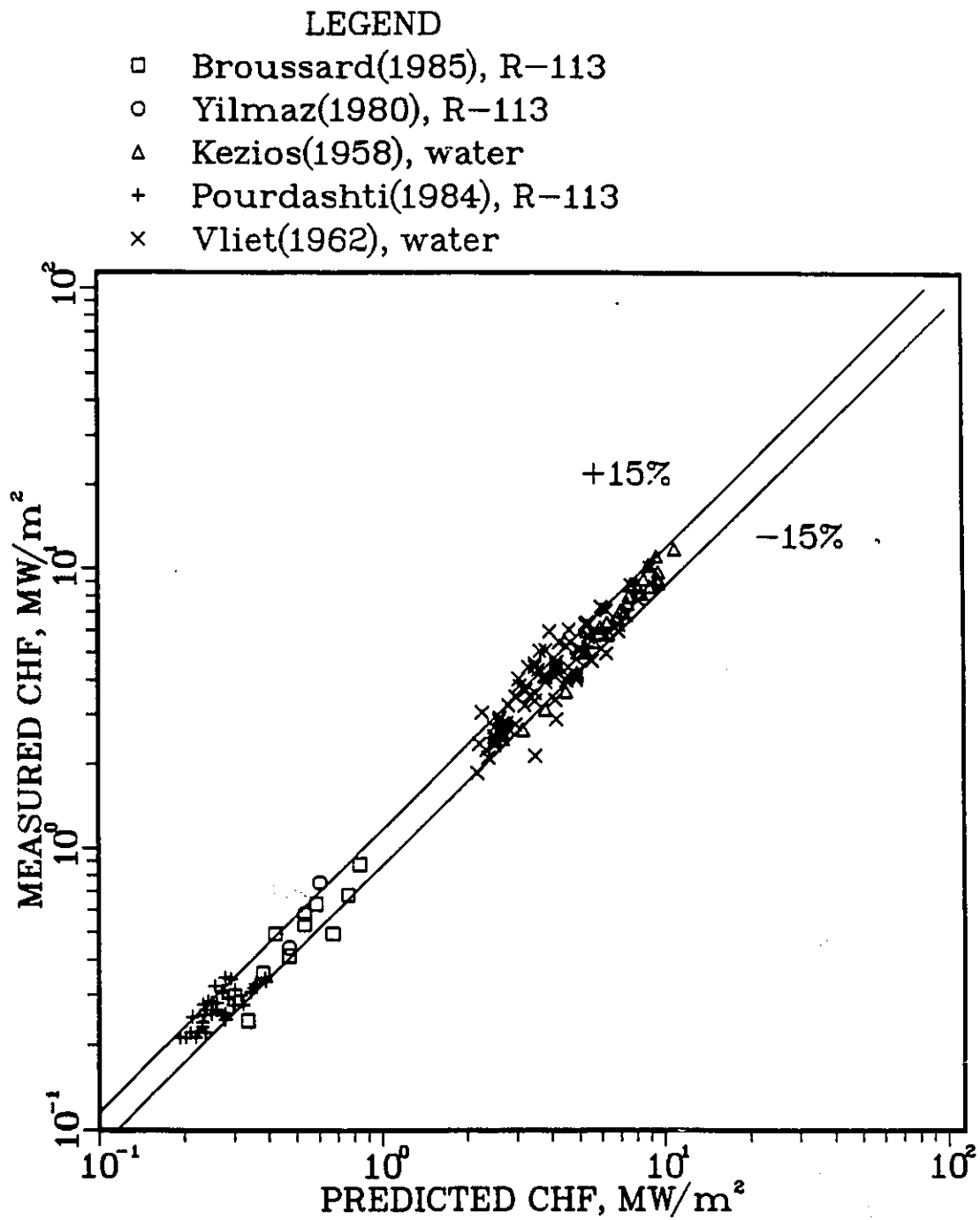


Figure 7.22: Comparison of measured CHF vs. the CHF predicted by equation (7.5)

Chapter 8

Discussion and Conclusion

A semi-empirical model for predicting the cross-flow CHF including the quality effect is presented in this work. The result (equation (4.15)) is analogous to that of Lienhard and Eichhorn (equation (3.14)) except the last bracket in equation (4.15). Because of the small value of E_c in the second term of the last bracket, equation (4.15) can be simplified to equation (4.25). Thus, equation (4.25) is applicable to a wider range of quality region and is identical to Lienhard and Eichhorn's (1976) result (see equation (3.14)) at the special case for $x = 0$. However, this model needs some empirical constants to obtain the key term of U_g/U_∞ . Ungar and Eichhorn's (1988) hydrodynamic prediction method is used to express U_g/U_∞ and to obtain these empirical constants on the basis of various experimental data. Compared with Ungar and Eichhorn's correlation for effective vapor sheet thickness, which is shown in equation (4.20), the correlation presented by this work has an additional empirical constant for the quality parameter (see equation (4.24)). Unlike Ungar and Eichhorn's correlation for α^* , which shows the coefficient b is constant, the coefficient b obtained from present work is expressed in equation (4.28) because the distribution of coefficient b varies with the density ratio and has no smooth trend. Therefore, the accuracy for this model would be affected in certain ranges of density ratio. The applicable range for this model is restricted for G_c up to the value of 8, where G_c is illustrated in equation (3.27). Thus, the applicable range is attributed to the low flow region according to Ungar and Eichhorn's (1988) observation. The other parameter ranges for this model are:

Density ratio: 50 - 1650

Quality: -11.0 to 11.0%

The diameter effect is not included in this analysis. However, according to the definition of Froude number, the diameter effect should have an implicit influence on the derivation of effective vapor sheet thickness. The experimental data used in this analysis are chosen according to the parameter range which meets the limitation that $G_c \leq 8$. Four types of fluid are used in this analysis: water, Freon-113, isopropanol and methanol. Because the model derived in present study is based on the assumption that the fluid flows across single cylinder, this model can not be applied to the bundle case. Efforts for applying this model to bundles should be made in the future.

An empirical correlation for cross-flow CHF is also developed in present work. In addition, the relationship of CHF's between cross flow and axial flow is investigated in the study, resulting in a correction factor to relate the CHF's in both cases. Unlike the semi-empirical CHF correlation for cross-flow boiling developed in Chapter 4, which can only apply to $G_c \leq 8$, the empirical CHF correlation for cross-flow boiling and the correction factor ($KCHF$) developed in Chapter 7 are applicable to a wider range of flow velocity (e.g. $G_c > 8$). The first consideration of choosing the experimental data for analysis is that the outside diameter in cross flow should be close to the range of $4 \leq D \times 10^3 \leq 16$ mm in order to correspond with the range used in Groeneveld's look-up table. However, because of the lack of experimental CHF data for cross flow, diameter which is in the range of $3 \leq D \times 10^3 \leq 4$ mm is also acceptable in the analysis.

The other parameter ranges are:

Froude number:	0.35 - 36
Density ratio:	50 - 1490
Quality:	-21 to 11%

Since the parameter range is different than that used in Chapter 4 for semi-empirical correlation, the experimental data sources used in both analyses may not be the same. Thus, only two types of fluid can be used in this analysis to satisfy the diameter range. They are water and Freon-113. However, the accuracy of the empirical correlation for cross flow is acceptable (13.4% of RMS error), which is indicated in Table 7.4. Compared with the result that the CHF calculated by multiplying a correction factor to the CHF predicted from Groeneveld's look-up table

(16.2% of RMS error), the CHF predicted by the empirical correlation is slightly more accurate. However, the analysis only involves limiting experimental data which have a narrower range than that used in Groeneveld's look-up table. If more data with wider parameter range are available, the deviation of both approaches with experimental data probably can be reduced to a satisfactory level.

In summary, the CHF of cross-flow boiling predicted by the empirical correlation, equation (7.5), is recommended. Though the CHF of cross flow predicted by the correction factor method is not as accurate as that predicted by equation (7.5), the idea to investigate the relationship of CHF's between these two different flow patterns is found with a reasonable result. Besides, the availability of axial-flow CHF data for wider parameter range (see Section 6.3) is very attractive to be used to predict the cross-flow CHF in which the experimental data is not available in certain range. The semi-empirical model presented in Chapter 4 is an improvement of Ungar and Eichhorn's (1988) hydrodynamic model and Lienhard and Eichhorn's (1976) *MESC* model to be applied for a wider quality range. However, more work has to be done, either the theoretical derivation or experimental investigation, to reduce the level of RMS error.

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Appendix A

Tabulation of Experimental CHF Data in Cross-flow Boiling

Because of the parameter ranges used in the semi-empirical model and the correction factor method as well as in the empirical CHF correlation are different, the experimental CHF data used in these analyses come from various sources. Table A.1 illustrates the cross-flow CHF data used in the semi-empirical model, while Table A.2 shows the cross-flow CHF data analyzed in the correction factor method and the empirical prediction method.

Table A.1: CHF data for cross-flow used in semi-empirical model

Fluid: Freon-113							
Source: Pourdashti (1984)							
Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
kPa	$kg/m^2/s$	m		W/m^2			
187.7	103	0.00635	-0.006	241000	1.1465	0.26632	0.2357
200.9	99.51	0.00635	-0.075	282200	1.1633	0.25787	0.2457
208.2	96.08	0.00635	-0.051	281900	1.1802	0.24949	0.2456
197.7	98.33	0.00635	0.011	221800	1.1762	0.25148	0.2201
201.1	100	0.00635	-0.002	230900	1.1676	0.25568	0.2244
197.7	96.6	0.00635	0.072	212000	1.1846	0.24737	0.2219
197.7	101.6	0.00635	0.108	212000	1.1574	0.2608	0.231
197.1	152	0.00635	-0.069	291000	0.9417	0.38975	0.2856
214.5	151.9	0.00635	-0.066	295800	0.9412	0.3901	0.2849
208.7	152	0.00635	-0.007	264900	0.94178	0.38969	0.2761
208	151.9	0.00635	0.046	220400	0.9414	0.38996	0.2529
211.4	148.3	0.00635	0.091	213100	0.95634	0.37948	0.2524
202.8	287.1	0.00635	-0.047	277400	0.60148	0.73938	0.3517
206.7	291	0.00635	0.003	275300	0.59296	0.75232	0.3629
213.4	288	0.00635	0.044	245800	0.59758	0.74526	0.3414
213.4	286.4	0.00635	0.043	246600	0.59962	0.74217	0.3414
396.1	95.79	0.00635	-0.082	347400	1.187	0.24621	0.2524
430.9	94.97	0.00635	-0.019	323300	1.1939	0.24286	0.2474
424.4	96.58	0.00635	0.051	278600	1.1844	0.24745	0.2361
404.5	95.21	0.00635	0.108	251000	1.196	0.2419	0.2293
418.2	131.8	0.00635	-0.053	342600	1.0225	0.33613	0.2745
405.1	137.3	0.00635	-0.003	305800	1.0001	0.35028	0.2666
411.8	131.7	0.00635	0.048	280600	1.0217	0.33663	0.2571
411.8	129.8	0.00635	0.094	255400	1.029	0.33213	0.2484
194.9	442.7	0.00635	-0.097	346000	0.40741	1.1407	0.4721
194.9	442.9	0.00635	-0.099	336600	0.40768	1.1399	0.462
202	438.7	0.00635	-0.047	329800	0.40901	1.1362	0.4662
205.8	437.3	0.00635	-0.048	329500	0.41061	1.1316	0.4624

(Table A.1 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	CHF <i>W/m²</i>	<i>St</i>	<i>Fr</i>	α^*
209	141.1	0.00635	-0.11	314200	0.99005	0.3568	0.282
202.1	97.75	0.00635	0.053	220800	1.1805	0.24938	0.2246
205.6	97.47	0.00635	-0.107	311900	1.1827	0.24829	0.2541
198.5	94.73	0.00635	-0.043	285000	1.1965	0.24162	0.2487
202.2	93.36	0.00635	-0.031	265600	1.2034	0.23838	0.2374
213	291.2	0.00635	0.064	255200	0.59333	0.75175	0.3571
206	293.9	0.00635	-0.096	319200	0.59422	0.75037	0.3766
206.1	290.8	0.00635	-0.083	307700	0.59887	0.74331	0.3688
206.2	291.7	0.00635	-0.032	298900	0.59485	0.74941	0.3749
202.4	288.8	0.00635	-0.004	275400	0.59979	0.74192	0.361
209.6	289.9	0.00635	0.045	258200	0.59632	0.74717	0.3554
202.3	143.2	0.00635	0.002	256900	0.98042	0.36315	0.2674

(Table A.1 cont'd)

Fluid: isopropanol							
Source: Hasan et al. (1981)							
Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
kPa	$kg/m^2/s$	m		W/m^2			
101	204.8	0.0015	0	585000	0.19026	2.3012	0.4909
101	256.9	0.0015	0	615000	0.14458	2.8868	0.588
101	378	0.0015	0	650000	0.088344	4.2476	0.7617
101	457.3	0.0015	0	685000	0.06864	5.1384	0.8601
101	593.1	0.0015	0	730000	0.048229	6.6643	0.9691
101	212.9	0.00081	0	770000	0.12444	3.2549	0.636
101	256.9	0.00081	0	775000	0.09782	3.9284	0.7266
101	320.8	0.00081	0	785000	0.073043	4.9049	0.8437
101	447.7	0.00081	0	865000	0.046472	6.8466	1.074
101	584.3	0.00081	0	900000	0.032097	8.9342	1.1992
101	191.6	0.0005	0	925000	0.10465	3.7286	0.697
101	200.4	0.0005	0	900000	0.098743	3.9	0.7071
101	275.3	0.0005	0	830000	0.064903	5.3571	0.8358
101	311.9	0.0005	0	865000	0.054784	6.0714	0.9326
101	339.1	0.0005	0	928000	0.048875	6.6	1.0316
101	341.3	0.0005	0	1000000	0.048443	6.6429	1.093
101	445.5	0.0005	0	955000	0.033469	8.6714	1.2093
101	447.7	0.0005	0	1070000	0.033239	8.7143	1.3205
101	596	0.0005	0	1000000	0.022185	11.6	1.3621

(Table A.1 cont'd)

Fluid: isopropanol							
Source: Ungar and Eichhorn (1988)							
Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α''
kPa	$kg/m^2/s$	m		W/m^2			
101	198.2	0.00159	0	539000	0.20497	2.16	0.4346
101	345	0.00159	0	644000	0.10317	3.77	0.695
101	484.4	0.00159	0	670000	0.066013	5.29	0.8358
101	697.3	0.00159	0	781000	0.040154	7.61	1.0137
101	227.5	0.00318	0	460000	0.2589	1.76	0.4122
101	249.6	0.00318	0	525000	0.23339	1.93	0.4771
101	389	0.00318	0	534000	0.13784	3	0.6111
101	565.2	0.00318	0	550000	0.085367	4.36	0.7088
101	587.2	0.00318	0	604000	0.081408	4.52	0.7684
101	719.3	0.00318	0	648000	0.29748	1.55	0.4137
101	220.2	0.00476	0	396000	0.33381	1.39	0.3607
101	315.6	0.00476	0	432000	0.22537	1.99	0.4636
101	469.8	0.00476	0	515000	0.14016	2.96	0.6256
101	616.6	0.00476	0	540000	0.099072	3.89	0.6932
101	638.6	0.00476	0	531000	0.094626	4.03	0.687

(Table A.1 cont'd)

Fluid: methanol							
Source: Min (1975)							
Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
Pa	$\text{kg/m}^2/\text{s}$	m		W/m^2			
101	142.5	0.00051	0	1100000	0.15786	2.6875	0.5099
101	172.5	0.00051	0	1210000	0.12452	3.2533	0.6304
101	217.5	0.00051	0	1420000	0.092466	4.102	0.8415
101	255	0.00051	0	1520000	0.074982	4.8093	0.9964
101	292.5	0.00051	0	1560000	0.062388	5.5165	1.1244
101	408.8	0.00051	0	1670000	0.039439	7.709	1.4876
101	435.1	0.00051	0	1810000	0.036165	8.2041	1.6386
101	495.1	0.00051	0	1840000	0.030174	9.3357	1.7869
101	142.5	0.00051	0	1100000	0.15786	2.6875	0.5099
101	172.5	0.00051	0	1170000	0.12452	3.2533	0.6162
101	217.5	0.00051	0	1380000	0.092466	4.102	0.8252
101	255	0.00051	0	1470000	0.074982	4.8093	0.9737
101	292.5	0.00051	0	1560000	0.062388	5.5165	1.1244
101	378.8	0.00051	0	1670000	0.043832	7.1432	1.4156
101	435.1	0.00051	0	1790000	0.036165	8.2041	1.6256
101	495.1	0.00051	0	1830000	0.030174	9.3357	1.7798
101	232.5	0.00081	0	1380000	0.11435	3.4794	0.8322
101	393.8	0.00081	0	1620000	0.057059	5.8926	1.3526
101	457.6	0.00081	0	1620000	0.046472	6.8466	1.4833
101	510.1	0.00081	0	1720000	0.039991	7.6323	1.6456
101	558.8	0.00081	0	1830000	0.035216	8.3618	1.8031
101	135	0.00081	0	980000	0.22149	2.0203	0.4405
101	176.3	0.00081	0	1100000	0.16151	2.6376	0.5796
101	232.5	0.00081	0	1200000	0.11435	3.4794	0.7561
101	273.8	0.00081	0	1300000	0.092623	4.0967	0.901
101	326.3	0.00081	0	1370000	0.073491	4.8824	1.0589
101	371.3	0.00081	0	1470000	0.061793	5.5558	1.2147
101	435.1	0.00081	0	1560000	0.049805	6.5099	1.401

(Table A.1 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	CHF <i>W/m²</i>	<i>St</i>	<i>Fr</i>	α^*
101	146.3	0.00081	0	950000	0.20183	2.1887	0.4572
101	172.5	0.00081	0	1020000	0.16578	2.5815	0.5419
101	217.5	0.00081	0	1160000	0.12444	3.2549	0.703
101	255	0.00081	0	1350000	0.10156	3.8161	0.8779
101	292.5	0.00081	0	1420000	0.084923	4.3773	1.0049
101	367.5	0.00081	0	1550000	0.062646	5.4997	1.2527
101	435.1	0.00081	0	1640000	0.049805	6.5099	1.4525
101	487.6	0.00081	0	1760000	0.042572	7.2955	1.6333
101	168.8	0.00165	0	960000	0.25737	1.7694	0.4931
101	322.5	0.00165	0	1150000	0.11857	3.3815	0.8714
101	386.3	0.00165	0	1220000	0.094019	4.05	1.0202
101	435.1	0.00165	0	1340000	0.080439	4.5611	1.1719

(Table A.1 cont'd)

Fluid: water							
Source: Vliet (1962)							
Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
kPa	$kg/m^2/s$	m		W/m^2			
111.9	174.7	0.003175	-0.01141	1861000	0.44874	1.0313	0.3546
111.4	170	0.003175	-0.02104	2240000	0.46204	0.99954	0.3918
111.2	180.1	0.003175	-0.05201	4038000	0.44232	1.0471	0.5865
110.1	188.3	0.003175	-0.1036	6057000	0.42995	1.0789	0.7009
111.4	327	0.003175	-0.00383	2114000	0.23261	1.9356	0.59
111.9	335.8	0.003175	-0.01006	2429000	0.22625	1.9832	0.656
111.4	335.9	0.003175	-0.00995	2461000	0.22625	1.9832	0.6625
111.4	318	0.003175	-0.01762	2461000	0.24158	1.8722	0.6328
111.6	318	0.003175	-0.01794	2524000	0.24158	1.8722	0.6432
111.6	318	0.003175	-0.01794	2524000	0.24158	1.8722	0.6432
111.6	318.1	0.003175	-0.01877	2461000	0.24158	1.8722	0.632
111.6	318.1	0.003175	-0.01845	2618000	0.24158	1.8722	0.6592
111.6	318	0.003175	-0.01804	2587000	0.24158	1.8722	0.654
111.6	318	0.003175	-0.01804	2587000	0.24158	1.8722	0.654
111.6	318.1	0.003175	-0.01845	2618000	0.24158	1.8722	0.6592
112.7	318	0.003175	-0.01825	2524000	0.24158	1.8722	0.6414
112.7	318	0.003175	-0.01825	2618000	0.24158	1.8722	0.6575
112.7	318	0.003175	-0.01825	2555000	0.24158	1.8722	0.6468
112.7	318	0.003175	-0.01825	2745000	0.24158	1.8722	0.6788
112.7	318	0.003175	-0.01857	2808000	0.24158	1.8722	0.6891
111.6	337.5	0.003175	-0.02239	2681000	0.22625	1.9832	0.696
111.2	337.4	0.003175	-0.02135	2713000	0.22625	1.9832	0.7028
111.2	337.3	0.003175	-0.02094	2808000	0.22625	1.9832	0.7194
111	325.1	0.003175	-0.03119	2776000	0.23702	1.9039	0.6893
111	343.9	0.003175	-0.05263	4353000	0.22419	1.9991	0.9538
110.1	333.3	0.003175	-0.08282	5300000	0.2348	1.9198	1.0401
110.3	354.5	0.003175	-0.1043	5773000	0.22017	2.0308	1.1318
110.8	357.1	0.003175	-0.1034	6435000	0.2182	2.0467	1.2241

(Table A.1. cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	CHF <i>W/m²</i>	<i>St</i>	<i>Fr</i>	α^*
111.4	322.5	0.003175	-0.01016	2713000	0.23702	1.9039	0.6874
111.4	322.5	0.003175	-0.01047	2366000	0.23702	1.9039	0.6266
111.6	322.5	0.003175	-0.01048	2524000	0.23702	1.9039	0.6541
111.6	322.5	0.003175	-0.01027	2524000	0.23702	1.9039	0.6542
112.9	322.4	0.003175	-0.01048	2492000	0.23702	1.9039	0.6465
112.5	322.4	0.003175	-0.01037	2492000	0.23702	1.9039	0.6473
112.3	322.4	0.003175	-0.01048	2334000	0.23702	1.9039	0.6196
112.3	322.4	0.003175	-0.01048	2492000	0.23702	1.9039	0.6476
112.5	322.4	0.003175	-0.01037	2429000	0.23702	1.9039	0.6361
112.3	322.4	0.003175	-0.01048	2461000	0.23702	1.9039	0.642
112.1	322.4	0.003175	-0.00995	2366000	0.23702	1.9039	0.6258
111	477.2	0.003175	-0.00435	2934000	0.14855	2.8241	0.9681
112.1	643.2	0.003175	-0.00384	3249000	0.10185	3.8078	1.2831
112.3	639.5	0.003175	-0.01048	3502000	0.10295	3.7761	1.3368
110.6	647.7	0.003175	-0.02104	3691000	0.10185	3.8078	1.3914
110.8	634.1	0.003175	-0.02073	3249000	0.10465	3.7285	1.2543
110.3	633.9	0.003175	-0.03108	3375000	0.10523	3.7126	1.2767
111	633.9	0.003175	-0.03108	2145000	0.10523	3.7126	0.9295
111.6	633.8	0.003175	-0.03109	3596000	0.10523	3.7126	1.3292
111.2	646.6	0.003175	-0.0517	4574000	0.10352	3.7602	1.5649
111.4	635.4	0.003175	-0.08233	6341000	0.10295	3.7761	1.9274
111	647.3	0.003175	-0.08305	5047000	0.10465	3.7285	1.6325
110.8	664.5	0.003175	-0.1037	7098000	0.10185	3.8078	2.0718
111	925.1	0.003175	-0.00477	3817000	0.063048	5.4737	1.8303
112.1	971.2	0.003175	-0.05203	4133000	0.060431	5.6482	1.9036
110.8	307.6	0.000812	-0.02073	3628000	0.25113	1.8087	0.5136
110.8	313	0.001626	-0.02073	3312000	0.24628	1.8404	0.614
110.8	321.1	0.002393	-0.02073	2997000	0.23928	1.888	0.6631
110.8	342.7	0.004806	-0.02083	2366000	0.22216	2.015	0.7415
111.9	584.3	0.000256	-0.00415	4416000	0.11522	3.4587	0.6657

(Table A.1 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	CHF <i>W/m²</i>	<i>St</i>	<i>Fr</i>	α^*
111.9	589.7	0.000502	-0.00415	4006000	0.11388	3.4905	0.7796
112.3	595	0.000812	-0.00404	4448000	0.11258	3.5222	0.9818
110.8	605.8	0.001623	-0.00373	4290000	0.11003	3.5857	1.2135
111.6	662.1	0.004803	-0.00415	3060000	0.098128	3.9188	1.4229
111.9	585.5	0.000256	-0.02105	5899000	0.11589	3.4429	0.7976
111.6	593.6	0.000502	-0.02084	4984000	0.11388	3.4905	0.8947
111.6	599	0.000810	-0.02073	4196000	0.11258	3.5222	0.9335
111.6	609.7	0.001626	-0.02063	4448000	0.11003	3.5857	1.2275
111.6	623.2	0.002393	-0.02053	3533000	0.10699	3.665	1.2011
111.6	666.5	0.004803	-0.02104	2871000	0.098128	3.9188	1.3447
111.2	589.3	0.000256	-0.05191	12150000	0.11658	3.427	1.2696
111.2	603	0.000812	-0.0518	8454000	0.11323	3.5063	1.4709
111.2	627.5	0.002393	-0.0517	5331000	0.10759	3.6491	1.559
110.8	614.7	0.000502	-0.1038	20500000	0.11258	3.5222	2.2296
111.2	620.2	0.000812	-0.1039	17100000	0.11129	3.5539	2.3186
112.7	631.1	0.001626	-0.1036	10760000	0.1088	3.6174	2.1364
110.6	631.2	0.001626	-0.1035	10730000	0.1088	3.6174	2.1443
112.7	645	0.002398	-0.1039	6940000	0.10581	3.6967	1.8189
110.8	692.2	0.004806	-0.1036	5142000	0.096603	3.9664	1.9364
111.9	1186	0.000256	-0.03109	10850000	0.045529	6.9492	2.0999
112.3	1194	0.000502	-0.03141	8139000	0.045103	6.9968	2.1128
110.6	1206	0.000812	-0.03108	6940000	0.044545	7.0603	2.198
110.3	1221	0.000256	-0.1039	28710000	0.045103	6.9968	3.9562

(Table A.1 cont'd)

Fluid: water							
Source: Diesselhorst et al. (1977)							
Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
kPa	$kg/m^2/s$	m		W/m^2			
101	132.2	0.0075	0	770000	0.79573	0.50902	0.1855
101	130.3	0.0075	0	710000	0.80364	0.50164	0.1746
101	132.2	0.0075	0	750000	0.79573	0.50902	0.1823
101	297.1	0.0075	0	900000	0.40647	1.1435	0.3149
101	292.3	0.0075	0	970000	0.41295	1.125	0.3281
101	331.5	0.0075	0	1000000	0.3644	1.2762	0.3615
101	347.8	0.0075	0	1060000	0.34699	1.3389	0.3874
101	140.9	0.006	0	800000	0.70258	0.60622	0.1918
101	139.9	0.006	0	750000	0.70615	0.60209	0.1831
101	138	0.006	0	700000	0.7134	0.59385	0.1737
101	139.9	0.006	0	750000	0.70615	0.60209	0.1831
101	140.9	0.006	0	780000	0.70258	0.60622	0.1886
101	140.9	0.006	0	770000	0.70258	0.60622	0.187
101	318.1	0.006	0	960000	0.33909	1.3691	0.3433
101	342.1	0.006	0	1000000	0.31427	1.4722	0.3697
101	371.8	0.006	0	1110000	0.28745	1.6001	0.4187
101	381.4	0.006	0	1350000	0.2796	1.6413	0.4864
101	132.2	0.004	0	800000	0.63088	0.69701	0.1803
101	130.3	0.004	0	690000	0.63824	0.6869	0.162
101	132.2	0.004	0	800000	0.63088	0.69701	0.1803
101	234.8	0.004	0	750000	0.37591	1.2374	0.2404
101	258.7	0.004	0	830000	0.34049	1.3637	0.2739
101	310.5	0.004	0	1180000	0.28051	1.6364	0.3918
101	343	0.004	0	1150000	0.25121	1.8082	0.412
101	119.8	0.003	0	1030000	0.60846	0.72901	0.1989
101	120.7	0.003	0	1070000	0.60451	0.73485	0.2049
101	124.6	0.003	0	1260000	0.58916	0.75818	0.2325
101	265.4	0.003	0	1150000	0.28447	1.6155	0.3491

(Table A.1 cont'd)

Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
kPa	$kg/m^2/s$	m		W/m^2			
101	287.5	0.003	0	1200000	0.2606	1.7496	0.3796
101	320	0.003	0	1330000	0.23094	1.9479	0.4386
101	375.6	0.003	0	1590000	0.19174	2.2862	0.5547

(Table A.1 cont'd)

Fluid: water

Source: Min (1975)

Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
kPa	$kg/m^2/s$	m		W/m^2			
98.54	192.4	0.00051	-0.00825	2016000	0.14823	2.829	0.4399
98.54	192.8	0.00051	-0.01445	2262000	0.14823	2.829	0.4732
98.54	193.5	0.00051	-0.02374	2453000	0.14823	2.829	0.4966
98.54	259.5	0.00051	-0.00618	2180000	0.10146	3.8191	0.5924
98.54	260.3	0.00051	-0.01445	2423000	0.10146	3.8191	0.6326
98.54	261.1	0.00051	-0.0227	2790000	0.10146	3.8191	0.6918
98.54	331.6	0.00051	-0.00618	2534000	0.073539	4.88	0.8031
98.54	332.8	0.00051	-0.01547	2957000	0.073539	4.88	0.886
98.54	333.7	0.00051	-0.0227	3172000	0.073539	4.88	0.9246
98.54	413.3	0.00051	-0.00618	2791000	0.054651	6.0823	1.0287
98.54	414.6	0.00051	-0.01445	3246000	0.054651	6.0823	1.1339
98.54	415.9	0.00051	-0.0227	3480000	0.054651	6.0823	1.1824
98.54	509.6	0.00051	-0.00722	3062000	0.040997	7.4968	1.2979
98.54	510.8	0.00051	-0.01341	3396000	0.040997	7.4968	1.3878
98.54	512.6	0.00051	-0.0227	3784000	0.040997	7.4968	1.4856
98.54	610.6	0.00051	-0.00722	3469000	0.031857	8.9821	1.6302
98.54	612	0.00051	-0.01341	3698000	0.031857	8.9821	1.6971
98.54	614.2	0.00051	-0.0227	4180000	0.031857	8.9821	1.8362
98.54	173.3	0.001024	-0.01063	2087000	0.25299	1.7968	0.3981
98.54	173.4	0.001024	-0.01269	2200000	0.25299	1.7968	0.4118
98.54	173.6	0.001024	-0.01579	2435000	0.25299	1.7968	0.4399
98.54	174.1	0.001024	-0.02301	2657000	0.25299	1.7968	0.4641
98.54	259.5	0.00081	-0.00650	2146000	0.13611	3.0305	0.5626
98.54	260.4	0.00081	-0.01475	2511000	0.13611	3.0305	0.6219
98.54	261.2	0.00081	-0.02301	2783000	0.13611	3.0305	0.6628
98.54	260	0.00081	-0.0112	2233000	0.13611	3.0305	0.576
98.54	260.4	0.00081	-0.01534	2441000	0.13611	3.0305	0.6099
98.54	261	0.00081	-0.02153	2776000	0.13611	3.0305	0.6623

(Table A.1 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	CHF <i>W/m²</i>	<i>St</i>	<i>Fr</i>	α^*
98.54	365.3	0.00081	-0.00670	2358000	0.08787	4.2651	0.7887
98.54	365.9	0.00081	-0.01083	2504000	0.08787	4.2651	0.8192
98.54	366.6	0.00081	-0.01599	2780000	0.08787	4.2651	0.8764
98.54	367.3	0.00081	-0.02116	3044000	0.08787	4.2651	0.9288
98.54	413.9	0.00081	-0.00981	2649000	0.074631	4.8263	0.941
98.54	414.7	0.00081	-0.01497	2955000	0.074631	4.8263	1.0102
98.54	415.7	0.00081	-0.02116	3201000	0.074631	4.8263	1.0623
98.54	413.8	0.00081	-0.00960	2721000	0.074631	4.8263	0.9584
98.54	414.3	0.00081	-0.01269	2919000	0.074631	4.8263	1.0033
98.54	416	0.00081	-0.02307	3408000	0.074631	4.8263	1.1071
98.54	510.1	0.00081	-0.00981	2994000	0.056329	5.9487	1.2087
98.54	511.1	0.00081	-0.01497	3256000	0.056329	5.9487	1.276
98.54	512.3	0.00081	-0.02116	3671000	0.056329	5.9487	1.3797
98.54	611.2	0.00081	-0.00981	3536000	0.043968	7.1272	1.5602
98.54	612.4	0.00081	-0.01497	3712000	0.043968	7.1272	1.6079
98.54	613.8	0.00081	-0.02116	4109000	0.043968	7.1272	1.718
98.54	510.4	0.00081	-0.01115	3110000	0.056329	5.9487	1.2396
98.54	510.8	0.00081	-0.01332	3220000	0.056329	5.9487	1.2677
98.54	512.4	0.00081	-0.02155	3670000	0.056329	5.9487	1.3791
98.54	611.1	0.00081	-0.00960	3327000	0.043968	7.1272	1.4958
98.54	611.9	0.00081	-0.01269	3727000	0.043968	7.1272	1.615
98.54	614.2	0.00081	-0.02299	4192000	0.043968	7.1272	1.7398
98.54	261.2	0.00081	0	1770000	0.13611	3.0305	0.4963
98.54	367.6	0.00081	0	2030000	0.08787	4.2651	0.7156
98.54	415.9	0.00081	0	2270000	0.074631	4.8263	0.8526
98.54	512.7	0.00081	0	2600000	0.056329	5.9487	1.1043
98.54	614.2	0.00081	0	3052000	0.043968	7.1272	1.4181
98.54	174.1	0.00102	0	1660000	0.25243	1.8004	0.344
98.54	193.5	0.00051	0	1750000	0.14823	2.829	0.4023
98.54	261.2	0.00051	0	1990000	0.10146	3.8191	0.5594

(Table A.1 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	CHF <i>W/m²</i>	<i>St</i>	<i>Fr</i>	α^*
98.54	333.7	0.00051	0	2440000	0.073539	4.88	0.786
98.54	415.9	0.00051	0	2710000	0.054651	6.0823	1.0125
98.54	512.7	0.00051	0	2980000	0.040997	7.4968	1.2802
98.54	614.2	0.00051	0	3300000	0.031857	8.9821	1.5824

(Table A.1 cont'd)

Fluid: water							
Source: Ungar and Eichhorn (1988)							
Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α^*
kPa	$kg/m^2/s$	m		W/m^2			
104.8	363.8	0.00081	0	3686000	0.08787	4.2651	1.0541
104.8	622.4	0.00081	0	5085000	0.042572	7.2955	2.0183
104.8	976.6	0.00081	0	5434000	0.022605	11.448	2.9019
104.8	306.4	0.00105	0	2544000	0.12944	3.1546	0.6979
104.8	392.6	0.00105	0	2968000	0.094266	4.0418	0.9429
104.8	421.3	0.00105	0	3097000	0.085948	4.3376	1.0266
104.8	641.5	0.00105	0	3768000	0.048825	6.6049	1.6275
104.8	660.7	0.00105	0	3780000	0.046891	6.8021	1.667
104.8	919.2	0.00105	0	4153000	0.029601	9.4638	2.234
104.8	1015	0.00105	0	4411000	0.025734	10.45	2.4758
104.8	344.7	0.0013	0	2674000	0.12767	3.1895	0.7777
104.8	976.6	0.0013	0	3815000	0.031587	9.0368	2.1272
104.8	258.5	0.00151	0	2531000	0.19854	2.2195	0.5947
104.8	363.8	0.00151	0	3168000	0.13105	3.1238	0.8987
104.8	430.9	0.00151	0	3219000	0.10572	3.6992	1.036
104.8	564.9	0.00151	0	3281000	0.074143	4.8501	1.2923
104.8	641.5	0.00151	0	3951000	0.062523	5.5077	1.6162
104.8	785.1	0.00151	0	3964000	0.047478	6.7408	1.8713
104.8	813.9	0.00151	0	4596000	0.045186	6.9874	2.1294
104.8	507.5	0.0016	0	3422000	0.088756	4.2326	1.219
104.8	325.5	0.00165	0	1886000	0.15885	2.6738	0.5769
104.8	526.6	0.00165	0	3328000	0.086271	4.3252	1.2265
104.8	689.4	0.00165	0	3070000	0.06023	5.6621	1.4144
104.8	316	0.00212	0	2589000	0.19142	2.2895	0.6876
104.8	344.7	0.00212	0	2454000	0.17253	2.4976	0.7075
104.8	497.9	0.00212	0	3019000	0.10918	3.6076	1.0741
104.8	593.6	0.00212	0	3328000	0.086898	4.3014	1.3091
104.8	651.1	0.00212	0	3459000	0.076923	4.7177	1.4382

(Table A.1 cont'd)

Pressure	Mass flux	Diameter	Quality	CHF	St	Fr	α''
kPa	$kg/m^2/s$	m		W/m^2			
104.8	727.7	0.00212	0	3574000	0.066304	5.2727	1.5922
104.8	833	0.00212	0	3686000	0.055225	6.0358	1.7831
104.8	938.3	0.00212	0	4173000	0.04692	6.799	2.1018
104.8	995.8	0.00212	0	3896000	0.043228	7.2153	2.0724
104.8	306.4	0.00038	0	4699000	0.066796	5.2438	1.1666
104.8	459.6	0.00038	0	6333000	0.038351	7.8657	2.0139
104.8	670.2	0.00038	0	6969000	0.022542	11.471	2.9212
104.8	277.7	0.00051	0	3940000	0.092466	4.102	0.9244
104.8	325.5	0.00051	0	3626000	0.074982	4.8093	0.998
104.8	325.5	0.00051	0	4388000	0.074982	4.8093	1.1352
104.8	440.4	0.00051	0	5010000	0.049839	6.5067	1.5986
104.8	478.7	0.00051	0	5141000	0.044438	7.0725	1.743
104.8	545.8	0.00051	0	6125000	0.037053	8.0626	2.187
104.8	603.2	0.00051	0	6125000	0.032213	8.9113	2.3695
104.8	699	0.00051	0	6464000	0.026172	10.326	2.7574
104.8	823.4	0.00051	0	7406000	0.020731	12.165	3.422
104.8	833	0.00051	0	6989000	0.020392	12.306	3.3111

Table A.2: CHF data for cross-flow used in correction factor and empirical prediction methods

Fluid: Freon-113							
Source: Broussard and Westwater (1985)							
Pressure	Mass flux	Diameter	Quality	Fr	CHF_{cross}	CHF_{axial}	$KCHF$
kPa	$kg/m^2/s$	m			W/m^2	W/m^2	
101.4	3879	0.00318	0	14.53941	497200	390300	1.274
101.4	6466	0.00318	0	24.23802	681600	406600	1.676
101.4	9699	0.00318	0	36.35420	876000	427000	2.052
101.4	4171	0.00635	0	11.06229	413900	311400	1.329
101.4	6952	0.00635	0	18.43848	535900	325300	1.647
101.4	10430	0.00635	0	27.66173	632800	342700	1.846
101.4	4911	0.0127	0	9.210444	243200	250100	0.9726
101.4	8185	0.0127	0	15.35263	360000	263100	1.368
101.4	12280	0.0127	0	23.02752	496800	279400	1.778

(Table A.2 cont'd)

Fluid: Freon-113

Source: Yilmaz (1980)

Pressure	Mass flux	Diameter	Quality	Fr	CHF_{cross}	CHF_{axial}	$KCHF$
kPa	$kg/m^2/s$	m			W/m^2	W/m^2	
101.4	4171	0.00635	0	11.06229	442500	311400	1.421
101.4	6952	0.00635	0	18.43848	583100	325300	1.792
101.4	11820	0.00635	0	31.34782	754100	349700	2.156

(Table A.2 cont'd)

Fluid: Freon-113

Source: Pourdashti (1984)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	<i>Fr</i>	<i>CHF_{cross}</i> <i>W/m²</i>	<i>CHF_{axial}</i> <i>W/m²</i>	<i>KCHF</i>
202	126.3	0.00635	-0.031	0.346012	265600	292200	0.9092
197.9	128.2	0.00635	-0.043	0.349658	285000	295400	0.9649
207.5	130	0.00635	-0.051	0.354185	281900	302600	0.9316
197.2	130.7	0.00635	0.072	0.360556	212000	273200	0.7761
204.8	131.9	0.00635	-0.107	0.354826	311900	335800	0.9288
202	132.3	0.00635	0.053	0.365804	220800	278100	0.794
197.2	133.1	0.00635	0.011	0.366285	221800	284200	0.7805
200.6	134.7	0.00635	-0.075	0.364202	282200	315800	0.8937
200.6	135.4	0.00635	-0.002	0.371894	230900	289100	0.7985
197.2	137.4	0.00635	0.108	0.379267	212000	273100	0.7763
187.5	139.4	0.00635	-0.006	0.381390	241000	287700	0.8377
208.2	191	0.00635	-0.11	0.513649	314200	359800	0.8735
202	193.8	0.00635	0.002	0.534083	256900	316200	0.8125
211	200.8	0.00635	0.091	0.556119	213100	308400	0.6911
214.4	205.6	0.00635	-0.066	0.559725	295800	348300	0.8491
207.5	205.6	0.00635	0.046	0.568940	220400	314700	0.7004
196.5	205.7	0.00635	-0.069	0.556520	291000	344100	0.8458
208.2	205.7	0.00635	-0.007	0.566536	264900	325800	0.8133
213	387.6	0.00635	0.043	1.074977	246600	373300	0.6606
202.7	388.5	0.00635	-0.047	1.058950	277400	388800	0.7134
213	389.8	0.00635	0.044	1.081788	245800	373000	0.6591
202	390.9	0.00635	-0.004	1.075778	275400	379700	0.7253
208.9	392.3	0.00635	0.045	1.085795	258200	368600	0.7004
205.5	393.6	0.00635	-0.083	1.064960	207700	400000	0.7694
206.2	393.9	0.00635	0.003	1.087397	275300	383000	0.7189
212.4	394	0.00635	0.064	1.092205	255200	369700	0.6904
205.5	394.8	0.00635	-0.032	1.081788	298900	390100	0.7664
205.5	397.7	0.00635	-0.096	1.072573	319200	402700	0.7926

(Table A.2 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	<i>Fr</i>	<i>CHF_{cross}</i> <i>W/m²</i>	<i>CHF_{axial}</i> <i>W/m²</i>	<i>KCHF</i>
204.8	591.8	0.00635	-0.048	1.613467	329500	399100	0.8255
202	593.8	0.00635	-0.047	1.617073	329800	394300	0.8365
194.4	599.2	0.00635	-0.097	1.609060	346000	402000	0.8607
194.4	599.4	0.00635	-0.099	1.608659	336600	402600	0.836
396.4	129.6	0.00635	-0.082	0.367808	347400	322100	1.078
403.3	128.9	0.00635	0.108	0.374218	251000	282900	0.8874
404	185.8	0.00635	-0.003	0.537688	305800	328600	0.9308
410.9	175.7	0.00635	0.094	0.510844	255400	312300	0.8176
410.9	178.2	0.00635	0.048	0.518056	280600	318600	0.8809
417.8	178.4	0.00635	-0.053	0.512447	342600	335400	1.021
424	130.7	0.00635	0.051	0.380949	278600	289900	0.961
430.9	128.5	0.00635	-0.019	0.373056	323300	298900	1.082

(Table A.2 cont'd)

Fluid: water							
Source: Kezios and Lo (1958)							
Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	<i>Fr</i>	<i>CHF_{cross}</i> <i>W/m²</i>	<i>CHF_{axial}</i> <i>W/m²</i>	<i>KCHF</i>
122	300.3	0.003175	-0.04418	1.753129	2665000	5754000	0.4632
120.3	304.1	0.003175	-0.08285	1.753129	4488000	6438000	0.6971
122.6	306.6	0.003175	-0.1167	1.753129	6020000	7258000	0.8295
122.4	308	0.003175	-0.1425	1.753129	5847000	7994000	0.7314
122.4	309	0.003175	-0.1678	1.753129	8009000	6717000	1.192
128.3	600.6	0.003175	-0.04718	3.506258	3147000	5885000	0.5347
128.5	607.7	0.003175	-0.08419	3.506258	5033000	6713000	0.7497
132.2	613.1	0.003175	-0.1213	3.506258	6407000	7728000	0.829
131.4	613.2	0.003175	-0.1216	3.506258	6406000	7735000	0.8282
129.8	616.1	0.003175	-0.146	3.506258	7524000	8446000	0.8909
130	618.2	0.003175	-0.1755	3.506258	8699000	7008000	1.241
139.7	902.7	0.003175	-0.05767	5.259387	3655000	6247000	0.585
140.3	912.4	0.003175	-0.09251	5.259387	5498000	7087000	0.7758
144.2	919.4	0.003175	-0.1249	5.259387	6732000	8005000	0.8409
141.7	924.1	0.003175	-0.1513	5.259387	8656000	8699000	0.9951
142.6	927.3	0.003175	-0.1816	5.259387	8955000	8225000	1.089
158	1202.7	0.003175	-0.06268	7.014783	4246000	6674000	0.6362
158.4	1214.5	0.003175	-0.09378	7.014783	6174000	7375000	0.8372
161.7	1225.4	0.003175	-0.1302	7.014783	6974000	8349000	0.8353
161.4	1232.1	0.003175	-0.1593	7.014783	9218000	8962000	1.029
161.7	1235.5	0.003175	-0.1814	7.014783	9249000	9203000	1.005
178.5	1502.1	0.003175	-0.06765	8.765646	5056000	7138000	0.7084
181.1	1517.2	0.003175	-0.09999	8.765646	6446000	7924000	0.8135
185.4	1537	0.003175	-0.1561	8.765646	8205000	9050000	0.9066
184.4	1540.1	0.003175	-0.1681	8.765646	10370000	9192000	1.128
182	1543.5	0.003175	-0.1838	8.765646	9830000	9372000	1.049
204.1	1805.8	0.003175	-0.08142	10.51650	5871000	7750000	0.7576
204.1	2065.1	0.003175	-0.1106	11.92173	7095000	8489000	0.8358

(Table A.2 cont'd)

Pressure	Mass flux	Diameter	Quality	Fr	CHF_{cross}	CHF_{axial}	$KCHF$
kPa	$kg/m^2/s$	m			W/m^2	W/m^2	
205.8	2075.3	0.003175	-0.1297	11.92173	8855000	8825000	1.003
207.9	2094.6	0.003175	-0.1761	11.92173	11170000	9548000	1.17
208.6	2102.5	0.003175	-0.2116	11.92173	11830000	8735000	1.354

(Table A.2 cont'd)

Fluid: water							
Source: Vliet (1962)							
Pressure	Mass flux	Diameter	Quality	Fr	CHF_{cross}	CHF_{axial}	$KCHF$
kPa	$kg/m^2/s$	m			W/m^2	W/m^2	
111.4	356.1	0.003175	-0.00383	2.107268	2114000	5112000	0.4134
111.9	1386.4	0.003175	-0.00383	8.204690	4448000	5500000	0.8087
112.1	700.5	0.003175	-0.00384	4.144841	3249000	5294000	0.6137
111.6	721.1	0.004803	-0.00415	3.468538	3060000	4621000	0.6622
111	519.8	0.003175	-0.00435	3.073925	2934000	5205000	0.5637
112.7	2773.2	0.003175	-0.00456	16.40938	5962000	5720000	1.042
111	1007.6	0.003175	-0.00477	5.960866	3817000	5444000	0.7012
110.6	1992	0.003175	-0.00487	11.78007	5079000	5621000	0.9035
111.4	3218.9	0.003175	-0.00995	18.99884	5457000	5930000	0.9203
111.4	365.8	0.003175	-0.00995	2.158830	2461000	5205000	0.4727
112.1	351.1	0.003175	-0.00995	2.072704	2366000	5204000	0.4546
111	2045.7	0.003175	-0.01006	12.07472	5110000	5768000	0.8861
111.9	365.8	0.003175	-0.01006	2.158830	2429000	5211000	0.4661
111.4	351.2	0.003175	-0.01016	2.072704	2713000	5201000	0.5216
111	2040	0.003175	-0.01016	12.03505	3943000	5769000	0.6835
111.6	351.2	0.003175	-0.01027	2.072704	2524000	5205000	0.4849
112.5	351.1	0.003175	-0.01038	2.072704	2492000	5215000	0.4779
112.5	351.1	0.003175	-0.01038	2.072704	2429000	5215000	0.4658
111.4	351.2	0.003175	-0.01047	2.072704	2366000	5206000	0.4545
112.9	351.1	0.003175	-0.01048	2.072704	2492000	5220000	0.4774
111.6	351.2	0.003175	-0.01048	2.072704	2524000	5208000	0.4846
112.3	696.5	0.003175	-0.01048	4.110277	3502000	5378000	0.6511
112.3	351.2	0.003175	-0.01048	2.072704	2334000	5214000	0.4477
112.3	351.2	0.003175	-0.01048	2.072704	2492000	5214000	0.478
112.3	351.2	0.003175	-0.01048	2.072704	2461000	5214000	0.4719
111.9	1390.2	0.003175	-0.01058	8.204690	4448000	5625000	0.7908
111.9	1375.6	0.003175	-0.01068	8.119696	4574000	5623000	0.8135
111.9	190.3	0.003175	-0.01141	1.122478	1861000	3869000	0.4811

(Table A.2 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	<i>Fr</i>	<i>CHF_{cross}</i> <i>W/m²</i>	<i>CHF_{axial}</i> <i>W/m²</i>	<i>KCHF</i>
111.4	346.3	0.003175	-0.01762	2.038140	2461000	5308000	0.4636
111.6	346.4	0.003175	-0.01794	2.038140	2524000	5314000	0.4749
111.6	346.4	0.003175	-0.01794	2.038140	2524000	5314000	0.4749
111.6	346.4	0.003175	-0.01804	2.038140	2587000	5315000	0.4866
111.6	346.4	0.003175	-0.01804	2.038140	2587000	5315000	0.4866
112.7	346.3	0.003175	-0.01825	2.038140	2524000	5328000	0.4737
112.7	346.3	0.003175	-0.01825	2.038140	2618000	5328000	0.4914
112.7	346.3	0.003175	-0.01825	2.038140	2555000	5328000	0.4796
112.7	346.3	0.003175	-0.01825	2.038140	2745000	5328000	0.5151
111.6	346.4	0.003175	-0.01845	2.038140	2618000	5322000	0.492
111.6	346.4	0.003175	-0.01845	2.038140	2618000	5322000	0.492
112.7	346.4	0.003175	-0.01857	2.038140	2808000	5333000	0.5265
111.6	346.5	0.003175	-0.01877	2.038140	2461000	5326000	0.462
110.8	690.7	0.003175	-0.02073	4.058715	3249000	5494000	0.5914
110.1	2045.8	0.003175	-0.02083	12.01805	4290000	6049000	0.7092
110.8	373.3	0.004806	-0.02083	1.782776	2366000	4667000	0.5069
112.3	3349.6	0.004806	-0.02084	16.00399	4133000	5460000	0.7568
111.2	367.4	0.003175	-0.02094	2.158830	2808000	5362000	0.5237
110.6	1393.3	0.003175	-0.02104	8.187691	4133000	5812000	0.7111
110.6	705.5	0.003175	-0.02104	4.144841	3691000	5504000	0.6706
111.6	725.9	0.004803	-0.02104	3.468538	2871000	4809000	0.5969
111.4	185.2	0.003175	-0.02104	1.087914	2240000	4030000	0.5558
112.1	1392.9	0.003175	-0.02115	8.187691	4101000	5821000	0.7045
111.6	3203.5	0.003175	-0.02115	18.82319	5457000	6259000	0.8719
111.2	367.4	0.003175	-0.02135	2.158830	2713000	5368000	0.5054
111.6	367.5	0.003175	-0.02239	2.158830	2681000	5386000	0.4978
109.5	3245.6	0.003175	-0.03065	18.99884	5142000	6542000	0.786
111.2	1392.3	0.003175	-0.03078	8.153694	4133000	5994000	0.6895

(Table A.2 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	<i>Fr</i>	<i>CHF_{cross}</i> <i>W/m²</i>	<i>CHF_{axial}</i> <i>W/m²</i>	<i>KCHF</i>
111	1392.4	0.003175	-0.03098	8.153694	3407000	5997000	0.5682
111	690.3	0.003175	-0.03108	4.041149	2145000	5628000	0.3812
110.3	690.4	0.003175	-0.03108	4.041149	3375000	5624000	0.6002
111.6	690.3	0.003175	-0.03109	4.041149	3596000	5632000	0.6386
111	354	0.003175	-0.03119	2.072704	2776000	5504000	0.5044
111	1466.4	0.004806	-0.03129	6.977281	3502000	5264000	0.6652
110.3	1392.8	0.003175	-0.03139	8.153694	2902000	6001000	0.4836
112.7	2094.3	0.003175	-0.03152	12.26170	4448000	6348000	0.7007
112.7	1395.3	0.003175	-0.03172	8.170692	4448000	6020000	0.7389
113.2	1395.4	0.003175	-0.03204	8.170692	4669000	6027000	0.7746
112.7	1397.4	0.003175	-0.04188	8.153694	4006000	6208000	0.6454
111.2	1402.3	0.003175	-0.05129	8.153694	4038000	6379000	0.633
111.4	3268.1	0.003175	-0.0515	18.99884	7319000	7137000	1.026
109.5	1399.9	0.003175	-0.05167	8.136695	4038000	6377000	0.6332
111.2	704.2	0.003175	-0.0517	4.093279	4574000	5914000	0.7734
111.9	2127.4	0.003175	-0.05192	12.36369	5678000	6867000	0.8269
111.2	196.1	0.003175	-0.05201	1.140043	4038000	4770000	0.8465
111.6	1402.5	0.003175	-0.05202	8.153694	4164000	6393000	0.6513
112.1	1057.8	0.003175	-0.05203	6.147851	4133000	6113000	0.676
111.6	1402.6	0.003175	-0.05212	8.153694	4763000	6395000	0.7449
111	374.5	0.003175	-0.05263	2.175829	4353000	5828000	0.747
109.3	1415.8	0.003175	-0.06732	8.187691	4700000	6661000	0.7056
111	1421.6	0.003175	-0.08232	8.187691	5773000	6931000	0.8329
111.4	713.8	0.003175	-0.08233	4.110277	6341000	6605000	0.96
111.4	3269.1	0.003175	-0.08264	18.82319	8770000	7512000	1.167
110.1	363	0.003175	-0.08282	2.089702	5300000	6394000	0.8289
109.3	2052.3	0.003175	-0.08301	11.81407	6309000	7204000	0.8758
111	704.9	0.003175	-0.08305	4.058715	5047000	6614000	0.7631

(Table A.2 cont'd)

Pressure <i>kPa</i>	Mass flux <i>kg/m²/s</i>	Diameter <i>m</i>	Quality	<i>Fr</i>	<i>CHF_{cross}</i> <i>W/m²</i>	<i>CHF_{axial}</i> <i>W/m²</i>	<i>KCHF</i>
110.8	1382.9	0.003175	-0.08315	7.961042	4984000	6927000	0.7195
110.8	2088	0.003175	-0.08325	12.01805	5994000	7220000	0.8301
110.8	3286	0.003175	-0.1026	18.82319	10160000	7811000	1.3
110.8	389	0.003175	-0.1034	2.227958	6435000	6861000	0.938
110.1	205.1	0.003175	-0.1036	1.174607	6057000	6083000	0.9956
110.8	753.8	0.004806	-0.1036	3.509365	5142000	6207000	0.8284
110.8	723.7	0.003175	-0.1037	4.144841	7098000	7116000	0.9975
110.3	2089.9	0.003175	-0.1039	11.96706	7760000	7539000	1.029
112.1	1441.3	0.003175	-0.1041	8.255686	6845000	7378000	0.9278
110.3	386	0.003175	-0.1043	2.210393	5773000	6882000	0.8388

Appendix B

Program for Predicting The Water CHF Data by Groeneveld's Table Look-up Method

This chapter includes two programs. The first one is a program which predicts the water CHF data by interpolation or extrapolation from the given conditions, while the second one is a data file which contains the CHF table as illustrated in Table 6.1. This data file is read by the first program.

```

SUBROUTINE CALCHF(P,G,D,X,CHF)
C.....
C
C   Subroutine CALCHF
C
C   This subroutine is used to obtain the CHF from table of discrete
C   values. The table was presented by Groeneveld et al..
C.....
C.....
C   Variable declaration:
C
C   Character:
C   GEOM      : a single character variable for channel geometry.
C   UNIT      : a single character variable for unit.
C   FLUID     : a single character variable for fluid type.
C   ORIENT    : a single character variable for channel orientation.
C   STATE     : a single character variable for state of the system.
C
C   Real:
C   X         : provided quality.
C   D         : provided diameter.
C   G         : provided mass flux.
C   P         : provided pressure.
C   INTRP3    : the interpolated value.
C   PX        : discrete value of quality.
C   PG        : discrete value of mass flux.
C   PP        : discrete value of pressure.
C   CHFTAB    : a set of 4410 discrete values of CHF arranged in three-
C               dimensional arrays.
C
C   Logical:
C   NFLG      : Program termination flag.
C   Flag      : Step termination flag.
C.....
C
C   REAL CHFTAB(21,14,15),PX(21),PG(14),PP(15),INTRP3
C   LOGICAL FLAG
C.....
C
C   Call up two common blocks to pass some variables in and out the
C   subroutine.
C.....
C
C   COMMON/TABLE/CHFTAB
C   COMMON/ROWS/PX,PG,PP,IX,IG,IP
C   PH2O = P
C   GH2O = G
C.....

```

Figure B.1: Program for the water CHF prediction by Groeneveld's table look-up method

```

C
C      Initialize the flag and range pointer.
C
C.....

      FLAG = .TRUE.
      IX = 20
      IG = 13
      IP = 14

C.....
C
C      Locate the discrete value of quality bounding the input value.
C
C.....

      DO 100 I=1,20
      IF((X .LE. PX(I+1)) .AND. FLAG)THEN
        IX = I
        FLAG = .FALSE.
      END IF
100  CONTINUE

C.....
C
C      Re-initialize the flag.
C      Locate the discrete value of mass flux bounding the input value.
C
C.....

      FLAG = .TRUE.
      DO 200 I=1,13
      IF((GH20 .LE. PG(I+1)) .AND. FLAG)THEN
        IG = I
        FLAG = .FALSE.
      END IF
200  CONTINUE

C.....
C
C      Re-initialize the flag.
C      Locate the discrete value of pressure bounding the input value.
C
C.....

      FLAG = .TRUE.
      DO 300 I=1,14
      IF((PH20 .LE. PP(I+1)) .AND. FLAG)THEN
        IP = I
        FLAG = .FALSE.
      END IF
300  CONTINUE

C.....
C

```

```

C      Calculate modification factor for tube diameter other than 0.008.*
C
C*****
      IF(D .GT. 0.016)THEN
        CK1 = 0.7937
      ELSE IF(D .LT. 0.002)THEN
        CK1 = 1.5874
      ELSE
        CK1 = (0.008/D)**(1./3)
      END IF
      CHF = INTRP3(X,GH20,PH20)*CK1
      RETURN
    END
    REAL FUNCTION INTRP3(X,G,P)

C*****
C      Function INTRP3
C
C      This function is used to set up the parameters for linear
C      interpolation in CHF evaluation. The parameters are the slope,
C      and two boundary points.
C*****

C*****
C      Variable declaration:
C
C      Real:
C      X      : provided quality.
C      G      : provided mass flux.
C      P      : provided pressure.
C      INTERP : the interpolated value.
C      PX     : discrete value of quality.
C      PG     : discrete value of mass flux.
C      PP     : discrete value of pressure.
C      CHFTAB : a set of 4410 discrete value of CHF arranged in three-
C               dimensional arrays.
C*****

      REAL INTERP,CHFTAB(21,14,15),PX(21),PG(14),PP(15)

C*****
C      Call up two common blocks to pass some variables in and out the
C      subroutine.
C*****

      COMMON/TABLE/CHFTAB
      COMMON/ROWS/PX,PG,PP,IX,IG,IP

```



```

C.....
C
C   Linear interpolate the pressure value.
C
C.....

PARAM = (P-PP(IP))/(PP(IP+1)-PP(IP))
C1    = INTERP(CHFTAB(IX,IG,IP),CHFTAB(IX,IG,IP+1),PARAM)
C2    = INTERP(CHFTAB(IX,IG+1,IP),CHFTAB(IX,IG+1,IP+1),PARAM)
C3    = INTERP(CHFTAB(IX+1,IG,IP),CHFTAB(IX+1,IG,IP+1),PARAM)
C4    = INTERP(CHFTAB(IX+1,IG+1,IP),CHFTAB(IX+1,IG+1,IP+1),PARAM)

C.....
C
C   Linear interpolate the mass flux value.
C
C.....

PARAM = (G-PG(IG))/(PG(IG+1)-PG(IG))
C5    = INTERP(C1,C2,PARAM)
C6    = INTERP(C3,C4,PARAM)

C.....
C
C   Linear interpolate the quality value.
C
C.....

PARAM = (X-PX(IX))/(PX(IX+1)-PX(IX))
INTRP3 = INTERP(C5,C6,PARAM)
RETURN
END
REAL FUNCTION INTERP(A1,A2,P)

C.....
C
C   Function INTERP
C
C   This function is used to interpolate value linearly.
C   It requires the slope of the function and two boundary points.
C
C.....

C.....
C
C   Variable declaration:
C
C   Real:
C   INTERP : the interpolated value.
C   P      : slope of the function.
C   A1,A2  : the two bounds.
C
C.....

INTERP = A1 + (A2-A1)*P

```

```

RETURN
END

```

SUBROUTINE READCHF

```

C*****
C
C   READ CHF DATA FROM CHFDA2.DAT
C*****
      IMPLICIT REAL (A-H,O-Z)
      DIMENSION CHF(15,14,21),CHFTAB(21,14,15)
      COMMON/TABLE/CHFTAB

      OPEN(UNIT=1,FILE='CHFDA2.DAT')
      DO 1 I=1,15
        READ(1,4) K
        4   FORMAT(/,11)
        DO 2 J=1,14
          READ(1,3) (CHF(I,J,K),K=1,21)
          3   FORMAT(3(T10,7F9.1,/))
        2   CONTINUE
      1   CONTINUE
      DO 100 I=1,15
        DO 200 J=1,14
          DO 300 K=1,21
            CHFTAB(K,J,I)=CHF(I,J,K)
          300 CONTINUE
        200 CONTINUE
      100 CONTINUE
      CLOSE(1)
      RETURN
      END
      BLOCK DATA TBCHF1
      REAL PX(21),PG(14),PP(15)
      COMMON/ROWS/PX,PG,PP,IX,IG,IP
      DATA PX/-.5,-.4,-.3,-.2,-.15,-.1,-.05,0.,.05,.1,.15,.2,.25,
+       .3,.4,.5,.6,.7,.8,.9,1./
      DATA PG/0.,50.,100.,200.,300.,500.,750.,1000.,1500.,2000.,
+       3000.,4000.,5000.,7500./
      DATA PP/100.,150.,200.,300.,450.,700.,1000.,1500.,2000.,
+       3000.,4500.,7000.,10000.,15000.,20000./
      END

```

PRESSURE ROW								
G	1	0.0	0.0	0.0	0.0	3930.0	3000.0	2100.0
		1300.0	950.0	875.0	800.0	750.0	700.0	675.0
		600.0	520.0	454.0	340.0	240.0	230.0	0.0
		0.0	0.0	0.0	0.0	4008.0	3500.0	2400.0
G	2	1700.0	1500.0	1300.0	1140.0	1000.0	950.0	875.0
		750.0	600.0	500.0	475.0	400.0	375.0	0.0
		0.0	0.0	0.0	0.0	4600.0	3700.0	2900.0
		2100.0	1700.0	1400.0	1200.0	1100.0	1000.0	900.0
G	3	800.0	700.0	600.0	520.0	450.0	430.0	0.0
		0.0	0.0	0.0	0.0	5200.0	4300.0	3400.0
		2600.0	2200.0	2100.0	1900.0	1600.0	1300.0	1100.0
		900.0	800.0	700.0	600.0	520.0	460.0	0.0
G	4	0.0	0.0	0.0	0.0	6000.0	4800.0	4200.0
		3600.0	3200.0	2900.0	2700.0	2500.0	2200.0	1900.0
		1100.0	900.0	820.0	750.0	650.0	500.0	0.0
		0.0	0.0	0.0	0.0	6200.0	5000.0	4200.0
G	5	3700.0	3300.0	3000.0	2900.0	2700.0	2400.0	2100.0
		1500.0	1100.0	1000.0	950.0	750.0	600.0	0.0
		0.0	0.0	0.0	0.0	6300.0	5100.0	4300.0
		3800.0	3400.0	3100.0	3000.0	2800.0	2500.0	2200.0
G	6	1800.0	1350.0	1100.0	980.0	775.0	620.0	0.0
		0.0	0.0	0.0	0.0	6400.0	5200.0	4400.0
		3900.0	3500.0	3200.0	3100.0	2850.0	2550.0	2350.0
		1900.0	1460.0	1200.0	875.0	620.0	450.0	0.0
G	7	0.0	0.0	0.0	0.0	6500.0	5300.0	4700.0
		3950.0	3550.0	3300.0	3150.0	2875.0	2600.0	2375.0
		2000.0	1600.0	1300.0	800.0	560.0	375.0	0.0
		0.0	0.0	0.0	0.0	6600.0	5400.0	5000.0
G	8	4000.0	3700.0	3400.0	3175.0	2900.0	2700.0	2400.0
		1830.0	1500.0	1100.0	725.0	450.0	300.0	0.0
		0.0	0.0	0.0	0.0	6800.0	5600.0	5200.0
		4100.0	4000.0	3500.0	3300.0	3000.0	2800.0	2500.0
G	9	1700.0	1400.0	900.0	650.0	380.4	225.0	0.0
		0.0	0.0	0.0	0.0	7000.0	5800.0	5300.0
		4200.0	4100.0	3800.0	3500.0	3100.0	2900.0	2600.0
		1800.0	1200.0	700.0	575.0	320.0	160.0	0.0
G	10	0.0	0.0	0.0	0.0	7130.0	6000.0	5400.0
		4300.3	4200.0	4000.0	3700.0	3200.0	3000.0	2700.0
		1400.0	1100.0	600.0	500.0	270.0	140.0	0.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
G	11	0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
G	12	0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
G	13	0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
G	14	0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0
		0.0	0.0	0.0	0.0	7250.0	6200.0	5700.0

Figure B.2: Data file of Groeneveld's CHF look-up table

		4400.6	4300.0	4200.0	3900.0	3400.0	3200.0	2800.0
		1200.0	900.0	550.0	425.0	230.0	130.0	0.0
PRESSURE ROW		2						
G	1	0.0	0.0	0.0	6273.0	4700.0	3400.0	2500.0
		1800.0	1200.0	1000.0	900.0	800.0	750.0	700.0
		650.0	600.0	525.0	450.0	375.0	300.0	0.0
G	2	0.0	0.0	0.0	5344.0	4800.0	3800.0	3000.0
		2200.0	1700.0	1500.0	1300.0	1100.0	1000.0	900.0
		800.0	700.0	600.0	539.0	450.0	420.0	0.0
G	3	0.0	0.0	0.0	6397.0	5000.0	4017.4	3200.0
		2400.0	1800.0	1600.0	1400.0	1150.0	1050.0	950.0
		850.0	750.0	675.0	580.0	510.0	450.0	0.0
G	4	0.0	0.0	0.0	6488.0	5444.0	4700.0	3800.0
		3100.0	2600.0	2400.0	2300.0	2100.0	1500.0	1400.0
		1000.0	850.0	775.0	700.0	600.0	500.0	0.0
G	5	0.0	0.0	0.0	6571.0	6050.0	5200.0	4400.0
		4000.0	3600.0	3000.0	2900.0	2800.0	2500.0	2200.0
		1200.0	1000.0	950.0	920.0	720.0	650.0	0.0
G	6	0.0	0.0	0.0	6700.0	6300.0	5300.0	4420.0
		4050.0	3650.0	3200.0	3100.0	2900.0	2600.0	2250.0
		1600.0	1300.0	1200.0	1000.0	850.0	700.0	0.0
G	7	0.0	0.0	0.0	6800.0	6350.0	5350.0	4450.0
		4100.0	3700.0	3300.0	3200.0	3000.0	2700.0	2500.0
		1900.0	1500.0	1250.0	1050.0	900.0	720.0	0.0
G	8	0.0	0.0	0.0	6850.0	6450.0	5400.0	4500.0
		4150.0	3725.0	3400.0	3250.0	3100.0	2800.0	2515.0
		2000.0	1630.0	1310.0	920.0	825.0	455.0	0.0
G	9	0.0	0.0	0.0	6900.0	6560.0	5445.0	4840.0
		4160.0	3745.0	3500.0	3300.0	3150.0	2850.0	2530.0
		2075.0	1650.0	1320.0	825.0	565.0	380.0	0.0
G	10	0.0	0.0	0.0	7000.0	6700.0	5550.0	5020.0
		4180.0	3800.0	3600.0	3400.0	3200.0	2900.0	2600.0
		1860.0	1520.0	1200.0	750.0	455.0	305.0	0.0
G	11	0.0	0.0	0.0	7500.0	7000.0	5700.4	5220.0
		4200.0	4100.0	3800.0	3600.0	3250.0	3000.0	2700.0
		1730.0	1420.0	910.0	675.0	385.0	230.0	0.0
G	12	0.0	0.0	0.0	8200.0	7200.0	5850.0	5320.0
		4250.0	4200.0	4000.0	3900.0	3350.0	3100.0	2800.0
		1630.0	1220.0	710.0	600.0	325.0	165.0	0.0
G	13	0.0	0.0	0.0	8900.0	7350.0	6050.0	5420.0
		4350.0	4300.0	4150.0	4100.0	3450.0	3200.0	2900.0

		1430.0	1110.0	610.0	525.0	275.0	146.0	0.0
G	14	0.0	0.0	0.0	9500.0	7450.0	6250.0	5720.0
		4450.0	4400.0	4300.0	4250.0	4650.0	3350.0	3100.0
		1230.0	910.0	560.0	450.0	240.0	136.0	0.0
PRESSURE ROW 3								
G	1	0.0	0.0	0.0	6300.0	4800.0	3800.0	2900.0
		2200.0	1500.0	1100.0	950.0	850.0	775.0	725.0
		675.0	625.0	550.0	480.0	380.0	300.0	0.0
G	2	0.0	0.0	0.0	6350.0	4955.0	4000.0	3200.0
		2500.0	1900.0	1600.0	1400.0	1300.0	1150.0	1126.0
		1000.0	750.0	650.0	550.0	460.0	430.0	0.0
G	3	0.0	0.0	0.0	6400.0	5050.0	4100.0	3250.0
		2600.0	2300.0	2100.0	1900.0	1700.0	1400.0	1300.0
		1100.0	800.0	750.0	600.0	540.0	500.0	0.0
G	4	0.0	0.0	0.0	6500.0	5800.4	4900.0	4000.0
		3550.0	3300.0	2800.0	2800.0	2400.0	1800.0	1500.0
		1200.0	900.0	850.0	800.0	700.0	600.0	0.0
G	5	0.0	0.0	0.0	6580.0	6040.0	5250.0	4450.0
		4050.0	3650.0	3200.0	3100.0	2850.0	2550.0	2250.0
		1500.0	1200.0	1100.0	925.0	810.0	651.0	0.0
G	6	0.0	0.0	0.0	6800.0	6400.0	5500.0	4700.0
		4100.0	3700.0	3300.0	3200.0	2950.0	2650.0	2300.0
		1900.0	1500.0	1300.0	1050.0	875.0	725.0	0.0
G	7	0.0	0.0	0.0	6900.0	6450.0	5900.0	4900.0
		4150.0	3725.0	3400.0	3250.0	3050.0	2750.0	2550.0
		2050.0	1600.0	1310.0	1060.0	910.0	728.0	0.0
G	8	0.0	0.0	0.0	7000.0	6500.0	6000.0	5000.0
		4200.0	3800.0	3500.0	3300.0	3150.0	2850.0	2560.0
		2070.0	1630.0	1320.0	920.0	630.0	460.0	0.0
G	9	0.0	0.0	0.0	7083.0	6600.0	6050.0	5030.0
		4235.0	3835.0	3550.0	3350.0	3200.0	2900.0	2580.0
		2100.0	1650.0	1330.0	845.0	570.0	385.0	0.0
G	10	0.0	0.0	0.0	7200.0	6700.0	6100.0	5050.0
		4300.0	3900.0	3600.0	3500.0	3300.0	2950.0	2700.0
		1885.0	1520.0	1210.0	770.0	460.0	310.0	0.0
G	11	0.0	0.0	0.0	7600.0	7150.0	6150.0	5240.0
		4400.0	4150.0	3900.0	3700.0	3400.0	3050.0	2800.0
		1755.0	1420.0	920.0	695.0	390.0	235.0	0.0
G	12	0.0	0.0	0.0	8400.0	7350.0	6200.0	5340.0
		4500.0	4350.0	4100.0	3900.0	3500.0	3150.0	2900.0
		1655.0	1220.0	720.0	620.0	330.0	170.0	0.0

G	13	0.0	0.0	0.0	9400.0	7500.0	6250.0	5440.0
		4650.0	4425.0	4200.0	4000.0	3600.0	3230.0	3000.0
		1455.0	1120.0	620.0	545.0	280.0	152.0	0.0
G	14	0.0	0.0	0.0	10100.0	7600.0	6350.0	5740.0
		4800.0	4500.0	4350.0	4200.0	3800.0	3370.0	3200.0
		1255.0	920.0	570.0	470.0	240.0	142.0	0.0
PRESSURE ROW 4								
G	1	0.0	0.0	0.0	6320.0	4900.0	3900.7	3000.0
		2250.0	1550.0	1200.0	960.0	860.0	785.0	740.0
		685.0	635.0	560.0	470.0	385.0	300.0	0.0
G	2	0.0	0.0	0.0	6375.0	5063.0	4100.0	3296.0
		2600.0	2300.0	2027.0	1700.0	1500.0	1614.0	1465.0
		1250.0	900.0	700.0	570.0	480.0	450.0	0.0
G	3	0.0	0.0	0.0	6470.0	5200.0	4200.0	3300.0
		2700.0	2500.0	2400.0	2000.0	1800.0	1700.0	1500.0
		1300.0	1000.0	850.0	650.0	575.0	550.0	0.0
G	4	0.0	0.0	0.0	6550.0	5850.0	5000.0	4150.0
		3650.0	3400.0	3200.0	2900.0	2500.0	2100.0	1800.0
		1600.0	1250.0	950.0	820.0	710.0	569.0	0.0
G	5	0.0	0.0	0.0	6600.0	6060.0	5275.0	4475.0
		4075.0	3675.0	3375.0	3175.0	2875.0	2675.0	2375.0
		1800.0	1400.0	1250.0	1000.0	830.0	575.0	0.0
G	6	0.0	0.0	0.0	6850.0	6450.0	5600.0	4800.0
		4150.0	3720.0	3420.0	3220.0	3000.0	2700.0	2400.0
		1950.0	1540.0	1310.0	1060.0	880.0	730.0	0.0
G	7	0.0	0.0	0.0	6950.0	6660.0	5925.0	4925.0
		4160.0	3740.0	3500.0	3250.0	3100.0	2800.0	2560.0
		2070.0	1670.0	1320.0	1070.0	920.0	731.0	0.0
G	8	0.0	0.0	0.0	7025.0	6700.0	6005.0	5005.0
		4205.0	3820.0	3650.0	3350.0	3150.0	2900.0	2600.0
		2090.0	1675.0	1325.0	960.0	835.0	470.0	0.0
G	9	0.0	0.0	0.0	7100.0	6800.0	6060.0	5050.0
		4245.0	3850.0	3720.0	3380.0	3250.0	2950.0	2650.0
		2115.0	1680.0	1350.0	870.0	575.0	395.0	0.0
G	10	0.0	0.0	0.0	7400.0	6900.0	6120.2	5080.0
		4330.0	4000.0	3800.0	3410.0	3350.0	3000.0	2750.0
		1900.0	1525.0	1230.0	795.0	470.0	320.0	0.0
G	11	0.0	0.0	0.0	7800.0	7300.0	6200.0	5280.0
		4500.0	4300.0	4000.0	3950.0	3500.0	3100.0	2900.0
		1770.0	1420.0	930.0	715.0	400.0	245.0	0.0

G	12	0.0	0.0	0.0	8700.0	7500.0	6400.0	5380.0
		4600.0	4475.0	4200.0	4100.0	3600.0	3200.0	3000.0
		1670.0	1230.0	730.0	640.0	340.0	180.0	0.0
G	13	0.0	0.0	0.0	9400.0	7850.0	6500.0	5480.0
		4800.0	4550.0	4300.0	4200.0	3700.0	3260.0	3100.0
		1470.0	1130.0	630.0	565.0	290.0	164.0	0.0
G	14	0.0	0.0	0.0	10100.0	7750.0	6600.0	5780.0
		4900.0	4625.0	4400.0	4300.0	3850.0	3390.0	3200.0
		1270.0	940.0	590.0	500.0	270.0	156.0	0.0
PRESSURE ROW 5								
G	1	0.0	0.0	0.0	6370.0	5000.0	3950.0	3050.0
		2300.0	1700.0	1300.0	1050.0	900.0	850.0	800.0
		730.0	645.0	570.0	500.0	410.0	390.0	0.0
G	2	0.0	0.0	0.0	6400.0	5100.0	4150.0	3350.0
		2750.0	2400.0	2100.0	1900.0	1700.0	1600.0	1520.0
		1450.0	1150.0	850.0	600.0	520.0	455.0	0.0
G	3	0.0	0.0	0.0	6520.0	5200.0	4320.0	3600.0
		3100.0	2700.0	2600.0	2300.0	2150.0	2000.0	1800.0
		1600.0	1300.0	950.0	725.0	620.0	560.0	0.0
G	4	0.0	0.0	0.0	6560.0	5900.0	5100.0	4200.0
		3750.0	3450.0	3300.0	3050.0	2650.0	2250.0	1950.0
		1660.0	1450.0	1050.0	900.0	730.0	580.0	0.0
G	5	0.0	0.0	0.0	6620.0	6080.0	5295.0	4495.0
		4090.0	3695.0	3395.0	3195.0	2895.0	2700.0	2500.0
		1870.0	1600.0	1270.0	1060.0	850.0	620.0	0.0
G	6	0.0	0.0	0.0	6700.0	6200.0	5400.0	4600.0
		4200.0	3740.0	3440.0	3240.0	3050.0	2720.0	2550.0
		1970.0	1670.0	1320.0	1160.0	890.0	740.0	0.0
G	7	0.0	0.0	0.0	6975.0	6885.0	5935.0	4935.0
		4170.0	3750.0	3470.0	3260.0	3150.0	2760.0	2580.0
		2090.0	1750.0	1325.0	1180.0	925.0	732.0	0.0
G	8	0.0	0.0	0.0	7030.0	6895.5	6010.0	5010.0
		4210.0	3850.0	3700.0	3375.0	3200.0	2850.0	2650.0
		2110.0	1760.0	1330.0	970.0	650.0	480.0	0.0
G	9	0.0	0.0	0.0	7105.0	6930.0	6070.0	5060.0
		4250.0	3900.0	3750.0	3400.0	3300.0	3000.0	2700.0
		2125.0	1785.0	1340.0	880.0	590.0	410.0	0.0
G	10	0.0	0.0	0.0	7600.0	7100.0	6140.0	5150.0
		4400.0	4100.0	3800.0	3450.0	3350.0	3050.0	2800.0
		1910.0	1600.0	1235.0	810.0	490.0	340.0	0.0
G	11	0.0	0.0	0.0	8100.0	7500.0	6300.0	5350.0

		4800.0	4450.0	4300.0	4000.0	3800.0	3150.0	3000.0
		1785.0	1440.0	940.0	725.0	405.0	255.0	0.0
G	12	0.0	0.0	0.0	9030.0	7700.0	6500.0	5450.0
		4700.0	4800.0	4400.0	4150.0	3700.0	3230.0	3100.0
		1685.0	1240.0	730.0	650.0	360.0	190.0	0.0
G	13	0.0	0.0	0.0	9700.0	7850.0	6600.0	5520.0
		4900.0	4675.0	4450.0	4250.0	3800.0	3280.0	3150.0
		1485.0	1140.0	630.0	575.0	310.0	176.0	0.0
G	14	0.0	0.0	0.0	10500.0	7900.0	6700.0	5820.0
		5000.0	4750.0	4475.0	4350.0	3900.0	3400.0	3250.0
		1285.0	950.0	590.0	510.0	290.0	168.0	0.0
PRESSURE ROW 6								
G	1	0.0	0.0	7600.0	6400.0	5100.0	4100.7	3300.0
		2700.0	2200.0	1800.0	1500.0	1200.0	1100.0	1000.0
		850.0	760.0	590.0	510.0	420.0	400.0	0.0
G	2	0.0	0.0	7800.0	6500.0	5270.0	4200.0	3400.0
		2900.0	2800.0	2350.0	2150.0	1900.0	1800.0	1700.0
		1500.0	1350.0	1000.0	700.0	550.0	470.0	0.0
G	3	0.0	0.0	7900.0	6600.0	5350.0	4350.0	3650.0
		3250.0	2900.0	2700.0	2500.0	2250.0	2050.0	1850.0
		1650.0	1400.0	1100.0	800.0	658.0	578.0	0.0
G	4	0.0	0.0	7950.0	6650.0	5950.0	4950.0	4400.0
		3800.0	3500.0	3400.0	3200.0	2800.0	2600.0	2200.0
		1700.0	1500.0	1300.0	1100.0	750.0	620.0	0.0
G	5	0.0	0.0	6986.0	6620.0	6100.0	5330.0	4530.0
		4120.0	3710.0	3450.0	3210.0	2930.0	2750.0	2600.0
		1950.0	1700.0	1410.0	1200.0	860.0	670.0	0.0
G	6	0.0	0.0	7208.0	6800.0	6300.0	5600.0	4700.0
		4250.0	3800.0	3600.0	3300.0	3100.0	2800.0	2650.0
		2100.0	1800.0	1500.0	1210.0	900.0	690.0	0.0
G	7	0.0	0.0	7480.0	7200.0	6900.0	6200.0	5300.0
		4400.0	3900.0	3700.0	3400.0	3200.0	2900.0	2700.0
		2130.0	1850.0	1550.0	1220.0	930.0	734.0	0.0
G	8	0.0	0.0	7696.0	7250.0	6910.0	6210.0	5320.0
		4405.9	3905.0	3750.0	3520.0	3250.0	3000.0	2750.0
		2128.0	1860.0	1560.0	980.0	660.0	490.0	0.0
G	9	0.0	0.0	8000.0	7500.0	6940.0	6080.0	5330.0
		4410.0	4200.0	3770.0	3600.0	3400.0	3050.0	2800.0
		2126.0	1863.0	1375.0	890.0	610.0	415.0	0.0
G	10	0.0	0.0	8200.0	7800.0	7200.0	6250.0	5335.0
		4800.0	4350.0	4100.0	3700.0	3500.0	3100.0	2900.0

		1915.0	1650.0	1240.0	820.0	510.0	345.0	0.0
G	11	0.0	0.0	9312.0	8500.0	7700.0	6400.0	5500.0
		4700.0	4525.0	4350.0	4200.0	3600.0	3200.0	3100.0
		1785.0	1445.0	950.0	735.0	410.0	265.0	0.0
G	12	0.0	0.0	10026.0	9500.0	7900.0	6600.0	5600.0
		4800.0	4700.0	4400.0	4250.0	3700.0	3260.0	3150.0
		1685.0	1245.0	740.0	660.0	370.0	210.0	0.0
G	13	0.0	0.0	10705.0	10200.0	8100.0	6700.0	5620.0
		5000.0	4800.0	4500.0	4300.0	3850.0	3300.0	3175.0
		1485.0	1145.0	640.0	585.0	330.0	186.0	0.0
G	14	0.0	0.0	12294.0	11000.0	8300.0	6800.0	5920.0
		5200.0	4970.0	4600.0	4400.0	3925.0	3410.0	3260.0
		1285.0	970.0	600.0	520.0	315.0	180.0	0.0
PRESSURE ROW		7						
G	1	0.0	0.0	8092.0	6551.0	5400.0	4600.0	3900.0
		3300.0	2800.0	2400.0	2100.0	1750.0	1500.0	1300.0
		1100.0	950.0	800.0	700.0	600.0	500.0	0.0
G	2	0.0	0.0	8195.0	6900.0	5800.0	4900.0	4100.0
		3400.0	2900.0	2600.0	2350.0	2150.0	2000.0	1900.0
		1750.0	1542.0	1204.0	900.0	650.0	530.0	0.0
G	3	0.0	0.0	8271.0	7000.0	6000.0	5200.0	4500.0
		4100.0	3900.0	3700.0	3500.0	3350.0	3200.0	3100.0
		2900.0	2500.0	2000.0	1400.0	1100.0	610.0	0.0
G	4	0.0	0.0	8403.0	7100.0	6170.0	5200.0	4600.0
		4200.0	4000.0	3900.0	3800.0	3750.0	3625.0	3500.0
		3300.0	2800.0	2300.0	1600.0	1200.0	630.0	0.0
G	5	0.0	0.0	8522.0	7200.0	6320.0	5500.0	4900.0
		4500.0	4300.0	4100.0	3900.0	3825.0	3775.0	3725.0
		3800.0	3100.0	2400.0	1800.0	1300.0	700.0	0.0
G	6	0.0	0.0	8738.0	7429.8	6600.0	5800.0	5100.0
		4800.0	4500.0	4300.0	4200.0	4100.0	4050.0	4000.0
		3900.0	3200.0	2500.0	1500.0	1000.0	720.0	0.0
G	7	0.0	0.0	8985.0	8100.0	7100.0	6300.0	5420.0
		5100.0	4700.0	4500.0	4300.0	4200.0	4125.0	4075.8
		3600.0	2272.0	1850.0	1300.0	932.0	732.0	0.0
G	8	0.0	0.0	9217.0	8150.4	7135.4	6350.0	5500.0
		5150.0	4800.0	4700.0	4600.0	4450.0	4250.0	4000.0
		3500.0	1900.0	1700.0	990.0	665.0	495.0	0.0
G	9	0.0	0.0	9647.0	8204.2	7270.0	6400.0	5575.0
		5200.0	5100.0	5000.0	4850.0	4500.0	4300.0	4050.0
		3200.0	1910.0	1380.0	910.0	620.0	420.0	0.0

G	10	0.0	0.0	10049.0	8307.9	7437.3	6410.2	5650.0
		5300.0	5246.0	5100.0	4700.0	4550.0	4350.0	4100.0
		2500.0	1660.0	1245.0	840.0	515.0	350.0	0.0
G	11	0.0	0.0	10798.0	9362.0	7900.0	6700.0	5789.8
		5482.9	5282.2	5120.0	4800.0	4200.0	3800.0	3600.0
		2000.0	1460.0	970.0	750.0	415.0	270.0	0.0
G	12	0.0	0.0	11497.0	10088.0	8135.0	6900.0	5830.0
		5562.1	5300.0	5000.0	4900.0	4100.0	3500.0	3350.0
		1800.0	1250.0	760.0	680.0	380.0	220.0	0.0
G	13	0.0	0.0	12161.0	10778.0	8492.6	7000.0	5870.9
		5700.0	5400.0	4900.0	4500.0	4000.0	3450.0	3300.0
		1700.0	1160.0	680.0	600.0	345.0	196.0	0.0
G	14	0.0	0.0	13716.0	12391.0	9007.4	7300.0	6060.4
		5815.5	5500.0	4800.0	4430.0	3950.0	3420.0	3270.0
		1500.0	1000.0	620.0	550.0	330.0	190.0	0.0

PRESSURE ROW 8

G	1	0.0	8975.0	8100.0	6900.0	5800.0	4900.0	4200.0
		3700.0	3300.0	3050.0	2850.0	2650.0	2450.0	2200.0
		1300.0	1100.0	1050.0	850.0	700.0	550.0	0.0
G	2	0.0	9088.0	8200.0	7050.0	6000.0	5100.0	4400.0
		3800.0	3500.0	3300.0	3200.0	3100.0	3000.0	2850.0
		2600.0	2100.0	1500.0	1151.0	850.0	560.0	0.0
G	3	0.0	9171.0	8300.0	7100.0	6100.0	5200.0	4600.0
		4300.0	4100.0	3900.0	3750.0	3600.0	3550.0	3500.0
		3300.0	3000.0	2400.0	2000.0	1200.0	700.0	0.0
G	4	0.0	9315.0	8320.0	7300.0	6500.0	5700.0	5200.0
		5000.0	4900.0	4800.0	4700.0	4600.0	4400.0	4000.0
		3600.0	3200.0	2800.0	2300.0	1500.0	800.0	0.0
G	5	0.0	9445.0	8600.0	7500.0	6800.0	6200.0	5900.0
		5600.0	5400.0	5300.0	5250.0	5200.0	5150.0	5050.0
		4500.0	4400.0	3600.0	2500.0	1700.0	850.0	0.0
G	6	0.0	9683.0	8700.0	7800.0	6900.0	6500.0	6300.0
		6100.0	5800.0	5900.0	5700.0	5500.0	5300.0	5100.0
		4600.0	4200.0	3400.0	1900.0	1058.0	750.0	0.0
G	7	0.0	9700.0	9100.0	8200.0	7400.0	6700.0	6400.0
		6150.0	6000.0	5700.0	5600.0	5100.0	4700.0	4400.0
		4000.0	3544.0	2100.0	1384.0	934.0	733.0	0.0
G	8	0.0	10207.0	9120.0	8220.0	7500.0	6856.0	6507.2
		6175.0	6050.0	5750.0	5400.0	5150.0	4750.0	4300.0
		3700.0	2081.0	1800.0	1000.0	667.0	500.0	0.0

G	9	0.0	10678.0	9700.0	8300.0	7631.0	6900.0	6570.6
		6225.0	5675.0	5300.0	5200.0	4900.0	4600.0	4200.0
		3400.0	2100.0	1390.0	930.0	622.0	425.0	0.0
G	10	0.0	11119.0	10100.0	8757.0	7858.3	6969.5	6719.0
		6300.0	5637.2	5200.0	5000.0	4560.0	4360.0	4050.0
		2600.0	1550.0	1250.0	860.0	517.0	355.0	0.0
G	11	0.0	11939.0	10893.0	9637.0	8147.9	7083.0	7060.7
		6400.0	5600.0	5120.0	4950.0	4300.0	4000.0	3700.0
		2100.0	1300.0	1000.0	770.0	420.0	275.0	0.0
G	12	0.0	12705.0	11714.0	10459.0	8457.6	7261.5	7069.0
		6450.0	5575.0	5050.0	4910.0	4200.0	3800.0	3400.0
		1900.0	1270.0	900.0	700.0	400.0	230.0	0.0
G	13	0.0	13433.0	12493.0	11239.0	9148.3	7500.0	7084.7
		6475.0	5500.0	4930.0	4600.0	4100.0	3550.0	3310.0
		1800.0	1200.0	730.0	630.0	360.0	208.0	0.0
G	14	0.0	15137.0	14317.0	13065.0	9858.6	8000.0	7100.0
		6500.0	5400.0	4810.0	4500.0	4000.0	3500.0	3260.0
		1600.0	1150.0	650.0	590.0	350.0	200.0	0.0

PRESSURE ROW 9

G	1	9832.0	8569.0	7595.0	6521.0	5581.0	4700.0	4000.0
		3695.0	3250.0	3000.0	2800.0	2600.0	2460.0	2210.0
		1780.0	1300.0	1100.0	900.0	690.0	595.0	0.0
G	2	9749.0	8692.0	7629.0	6550.0	5770.0	5000.0	4320.0
		3820.0	3520.8	3320.0	3230.0	3150.0	3100.0	3050.0
		2900.0	2600.0	2100.0	1500.0	1000.0	600.0	0.0
G	3	9836.0	8784.0	7729.0	6800.0	6000.0	5150.0	4575.0
		4320.0	4150.0	4000.0	3800.0	3800.0	3700.0	3600.0
		3400.0	3100.0	2600.0	2100.0	1400.0	762.0	0.0
G	4	9900.0	9300.0	8350.0	7370.0	6670.0	6000.0	5700.0
		5400.0	5200.0	5100.0	5000.0	4950.0	4900.0	4800.0
		4500.0	3900.0	3300.0	2500.0	1650.0	829.0	0.0
G	5	10124.0	9450.0	8620.0	7520.0	6700.0	6300.0	6200.0
		5800.0	5700.0	5600.0	5500.0	5450.0	5325.0	5200.0
		4900.0	4500.0	3900.0	3000.0	2000.0	900.0	0.0
G	6	10372.0	9700.0	8900.0	8200.0	7800.0	7600.0	7500.0
		7400.0	6900.0	6800.0	6400.0	6000.0	5600.0	5300.0
		4800.0	4400.0	3800.0	2846.0	1600.0	755.0	0.0
G	7	10656.0	9750.0	9150.0	8350.0	7950.0	7700.0	7600.0
		7350.0	7100.0	6790.0	6300.0	5800.0	5400.0	5000.0
		4500.0	3601.0	2549.0	1450.0	935.0	734.0	0.0
G	8	10921.0	9928.0	8971.0	8300.0	8200.0	8100.0	7800.0

		7323.7	7000.0	6745.0	5964.0	5790.0	5300.0	4900.0
		4000.0	2752.0	2000.0	1012.0	670.0	530.0	0.0
G	9	11414.0	10448.0	9536.0	8700.0	8500.0	8150.0	7850.0
		7200.0	6771.0	6400.0	5892.0	5600.0	5000.0	4600.0
		3800.0	2200.0	1500.0	975.0	625.0	460.0	0.0
G	10	11876.0	10934.0	10064.0	9000.0	8700.0	8170.0	7950.0
		7000.0	6156.7	5800.0	5400.0	5000.0	4500.0	4200.0
		2900.0	1800.0	1260.0	880.0	530.0	360.0	0.0
G	11	12735.0	11840.0	11047.0	9832.0	8944.0	8210.6	8100.0
		6770.8	5800.7	5600.0	5240.0	4600.0	4300.0	3850.0
		2450.0	1600.0	1200.0	800.0	425.0	280.0	0.0
G	12	13537.0	12685.0	11964.0	10713.0	9060.0	8251.0	8120.0
		6326.8	5691.6	5200.0	5000.0	4400.0	4100.0	3500.0
		2100.0	1425.0	1100.0	750.0	410.0	240.0	0.0
G	13	14299.0	13489.0	12836.0	11550.0	10000.0	8265.0	8130.0
		5400.0	5300.0	5000.0	4800.0	4200.0	3800.0	3315.0
		1900.0	1400.0	900.0	680.0	380.0	226.0	0.0
G	14	16083.0	15369.0	14876.0	13507.0	10300.0	8300.0	8140.0
		5100.0	5000.0	4850.0	4700.0	4020.0	3700.0	3260.0
		1850.0	1500.0	800.0	650.0	370.0	220.0	0.0
PRESSURE ROW		10						
G	1	8858.0	8031.0	7107.0	6000.0	5100.0	4346.0	3930.0
		3740.0	3540.0	3350.0	3170.0	3000.0	2840.0	2650.0
		2350.0	1750.0	1200.0	950.0	700.0	600.0	0.0
G	2	8993.0	8181.0	7258.0	6100.0	5400.0	4800.0	4300.0
		3840.0	3600.0	3400.0	3300.0	3225.0	3175.0	3150.0
		3050.0	2850.0	2450.0	1650.0	1100.0	700.0	0.0
G	3	9093.0	8291.0	7400.0	6420.0	5600.0	5000.0	4500.0
		4330.0	4180.0	4060.0	3860.0	3870.0	3780.0	3690.0
		3500.0	3200.0	2800.0	2200.0	1500.0	800.0	0.0
G	4	9287.0	8400.0	7665.0	7100.0	6500.0	5900.0	5500.0
		5350.0	5250.0	5175.0	5100.0	5025.0	4975.0	4900.0
		4700.0	4300.0	3700.0	2800.0	1700.0	819.0	0.0
G	5	9424.0	8657.0	7841.0	7200.0	6600.0	6300.0	6100.0
		5900.0	5800.0	5700.0	5600.0	5525.0	5475.0	5450.0
		5300.0	4900.0	4100.0	3200.0	1800.0	1000.0	0.0
G	6	9710.0	8973.0	8161.0	7700.0	7600.0	7520.0	7475.0
		7420.0	7000.0	6900.0	6700.0	6400.0	6100.0	5699.0
		5200.0	4600.0	3900.0	2900.0	1750.0	900.0	0.0
G	7	10036.0	9334.0	8527.0	8100.0	7800.0	7500.0	7425.0
		7350.0	7120.0	7000.0	6600.0	6200.0	5700.0	5350.0

		4800.0	3700.0	2850.0	2000.0	1500.0	800.0	0.0
G	8	10341.0	9671.0	8869.0	8500.0	8300.0	8200.0	8000.0
		7300.0	6900.0	6600.0	6300.0	5800.0	5400.0	5100.0
		4200.0	3312.0	2400.0	1875.0	1300.0	640.0	0.0
G	9	10909.0	10299.0	9505.0	8800.0	8700.0	8248.0	8100.0
		7250.0	6700.0	6500.0	6200.0	5400.0	5000.0	4850.0
		3850.0	2523.0	2000.0	1500.0	1000.0	465.0	0.0
G	10	11440.0	10885.0	10099.0	9200.0	9100.0	8300.0	8200.0
		7150.0	6100.0	6000.0	5900.0	5216.0	4792.0	4400.0
		2950.0	2122.0	1650.0	1100.0	800.0	365.0	0.0
G	11	12429.0	11977.0	11206.0	10010.0	9500.0	8333.7	8250.0
		7000.0	5750.0	5600.0	5500.0	4997.0	4400.0	4000.0
		2500.0	2015.0	1400.0	1050.0	600.0	290.0	0.0
G	12	13351.0	12997.0	12239.0	10947.0	10500.0	8419.4	8280.4
		6592.0	5500.0	5300.0	5047.0	4600.0	4100.0	3650.0
		2150.0	1900.0	1375.0	900.0	500.0	250.0	0.0
G	13	14228.0	13966.0	13221.0	11837.0	10800.0	8505.0	8300.0
		5870.0	5283.0	5100.0	4850.0	4300.0	3850.0	3320.0
		2000.0	1910.0	1200.0	850.0	450.0	240.0	0.0
G	14	16281.0	16234.0	15518.0	13919.0	11200.0	8623.0	8310.0
		5246.0	5000.0	4900.0	4730.0	4060.0	3800.0	3270.0
		1875.0	1920.0	1100.0	750.0	425.0	350.0	0.0

PRESSURE ROW 11

G	1	8083.0	7488.0	6800.0	5800.0	4800.0	4000.0	3500.0
		3250.0	3100.0	3000.0	2900.0	2600.0	2300.0	2000.0
		1400.0	800.0	600.0	400.0	320.0	260.0	0.0
G	2	8250.0	7658.0	6900.0	6000.0	5200.0	4400.0	3900.0
		3600.0	3400.0	3250.0	3125.0	3000.0	2900.0	2875.0
		2600.0	2000.0	1400.0	900.0	600.0	450.0	0.0
G	3	8374.0	7784.0	7000.0	6150.0	5300.0	4800.0	4400.0
		4200.0	4100.0	3950.0	3850.0	3775.0	3725.0	3675.0
		3475.0	2975.0	2275.0	1675.0	1100.0	760.0	0.0
G	4	8589.0	8004.0	7381.0	6700.0	6200.0	5800.0	5460.0
		5250.0	5100.0	4975.0	4875.0	4800.0	4600.0	4300.0
		3800.0	3200.0	2650.0	1800.0	1300.0	800.0	0.0
G	5	8783.0	8202.0	7567.0	6900.0	6500.0	6200.0	5800.0
		5700.0	5600.0	5425.0	5375.0	5200.0	5000.0	4700.0
		4400.0	4000.0	3400.0	2600.0	1700.0	950.0	0.0
G	6	9136.0	8562.0	7906.0	7400.0	7300.0	7200.0	7000.0
		6900.0	6500.0	6100.0	5600.0	5100.0	4600.0	4200.0
		3850.0	3300.0	2800.0	2392.0	1500.0	860.0	0.0

G	7	9539.0 6568.9 3400.0	8973.0 6200.0 3000.0	8294.0 5600.0 2611.0	7700.0 5100.0 2100.0	7200.0 4600.0 1200.0	6800.0 4200.0 775.0	6600.0 4000.0 0.0
G	8	9916.0 5976.0 3000.0	9357.0 5838.0 2400.0	8656.0 5173.0 2300.0	7816.0 4800.0 1950.0	7700.0 4295.0 1000.0	7311.7 3900.0 631.0	6586.0 3600.0 0.0
G	9	10617.0 5600.0 2500.0	10073.0 5400.0 1800.0	9330.0 5100.0 1644.0	8500.0 4700.0 1350.0	7700.0 4200.0 800.0	6800.0 3600.0 458.0	6000.0 3100.0 0.0
G	10	11273.0 5405.0 2200.0	10741.0 5319.0 1450.0	9959.0 5000.0 1346.0	8600.0 4722.0 710.0	7960.9 4000.0 510.0	7569.8 3400.0 290.0	5500.0 3050.0 0.0
G	11	12494.0 5000.0 1889.0	11986.0 4996.0 1200.0	11132.0 4798.0 1100.0	8700.0 4500.0 550.0	8227.3 3800.0 400.0	7716.9 3200.0 235.0	5327.0 2617.0 0.0
G	12	13633.0 5120.3 1527.0	13148.0 4944.0 1100.0	12226.0 4525.0 900.0	9215.0 4180.0 393.0	8700.0 3600.0 330.0	7747.9 3000.0 200.0	6216.0 2300.0 0.0
G	13	14716.0 5200.0 1366.0	14252.0 4586.0 1150.0	13266.0 4101.0 925.0	9890.0 3825.0 575.0	9100.0 3500.0 350.0	8000.0 2893.0 210.0	6624.0 2325.0 0.0
G	14	17251.0 5250.0 1800.0	16835.0 4400.0 1462.0	15698.0 4000.0 1000.0	10510.0 3900.0 600.0	9913.0 3553.0 450.0	8833.0 2969.0 325.0	7037.6 2355.0 0.0

PRESSURE ROW 12

G	1	7118.0 3200.0 700.0	6752.0 3050.0 500.0	6200.0 2800.0 420.0	5500.0 2750.0 325.0	4700.0 2250.0 300.0	3900.0 1550.0 230.0	3400.0 1000.0 0.0
G	2	7306.0 3550.0 1600.0	6927.0 3300.0 1169.0	6336.0 3100.0 800.0	5700.0 2950.0 535.0	4900.0 2800.0 425.0	4300.0 2578.0 375.0	3850.0 2100.0 0.0
G	3	7446.0 4100.0 2000.0	7057.0 4000.0 1450.0	6500.0 3900.0 1150.0	5800.0 3800.0 900.0	5000.0 3600.0 780.0	4500.0 3200.0 700.0	4250.0 2600.0 0.0
G	4	7888.0 5200.0 3000.0	7284.0 5100.0 2550.0	6852.0 4500.0 2000.0	6391.0 4600.0 1400.0	6000.0 4200.0 1157.0	5700.0 3800.0 750.0	5500.0 3400.0 0.0
G	5	7907.0 5400.0 3500.0	7487.0 5200.0 3172.0	7032.0 5000.0 2600.0	6500.0 4800.0 1937.0	6150.0 4600.0 1531.0	5900.0 4300.0 825.0	5600.0 3900.0 0.0

G	6	8305.0	7858.0	7361.0	6817.0	6500.0	6200.0	6000.0
		5800.0	5600.0	5400.0	5000.0	4400.0	4000.0	3700.0
		3100.0	2600.0	2100.0	1800.0	1300.0	800.0	0.0
G	7	8759.0	8282.0	7736.0	7131.0	6574.8	6181.1	5814.9
		5500.0	5203.0	4892.0	4350.0	3727.0	3597.0	3400.0
		2866.0	2350.0	2000.0	1650.0	978.0	667.0	0.0
G	8	9184.0	8678.0	8087.0	7425.0	6800.0	6281.0	5758.0
		5296.0	5100.0	4646.0	4057.0	3551.0	3356.0	3000.0
		2598.0	1850.0	1571.0	1400.0	675.0	400.0	0.0
G	9	9975.0	9416.0	8740.0	7556.4	7052.0	6296.4	5100.0
		4800.0	4600.0	4300.0	3896.0	3300.0	3100.0	2700.0
		1966.0	1325.0	992.0	900.0	450.0	325.0	0.0
G	10	10713.0	10104.0	9349.0	7808.0	7305.0	6343.0	4935.0
		4661.3	4323.0	3685.0	3265.0	3179.0	2917.0	2423.0
		1558.0	1023.0	776.0	500.0	330.0	180.0	0.0
G	11	12089.0	11387.0	10485.0	8226.5	7564.8	6558.4	4995.0
		4500.0	3832.0	3199.0	2932.0	2905.0	2521.0	1904.0
		1049.0	521.0	349.0	260.0	200.0	140.0	0.0
G	12	13373.0	12584.0	11544.0	8596.8	7982.4	7060.1	5694.0
		4600.0	3742.0	2929.0	2614.0	2435.0	2085.0	1670.0
		851.0	341.0	255.0	234.0	175.0	110.0	0.0
G	13	14593.0	13721.0	12550.0	9487.0	8696.0	7889.0	6408.0
		4650.0	3470.0	2800.0	2400.0	2350.0	2000.0	1466.0
		846.0	600.0	400.0	300.0	280.0	150.0	0.0
G	14	17447.0	16381.0	14903.0	10262.0	9817.0	8841.0	7042.0
		4700.0	3636.0	2941.0	2570.0	2500.0	2200.0	2143.0
		1700.0	900.0	500.0	450.0	375.0	200.0	0.0

PRESSURE ROW 13

G	1	4731.0	4566.0	4300.0	4000.0	3700.0	3400.0	3250.0
		3150.0	2900.0	2300.0	1781.0	1200.0	900.0	700.0
		600.0	498.0	399.0	300.0	230.0	200.0	0.0
G	2	4930.0	4745.0	4451.0	4100.0	3900.0	3700.0	3600.0
		3500.0	3300.0	3000.0	2800.0	2300.0	1800.0	1500.0
		1200.0	900.0	650.0	455.0	350.0	300.0	0.0
G	3	5078.0	4877.0	4600.0	4450.0	4250.0	4100.0	4050.0
		4025.0	3800.0	3500.0	3100.0	2700.0	2300.0	2000.0
		1600.0	1300.0	1000.0	700.0	500.0	450.0	0.0
G	4	5336.0	5108.0	4866.0	4610.0	4400.0	4300.0	4250.0
		4150.0	3900.0	3550.0	3300.0	3000.0	2700.0	2400.0
		2100.0	1700.0	1400.0	1100.0	900.0	550.0	0.0
G	5	5568.0	5315.0	5046.0	4760.0	4700.0	4651.8	4451.3

		4208.6	3725.0	3600.7	3353.0	3200.0	3164.0	2741.0
		2398.0	1826.0	1617.0	1477.0	1125.0	575.0	0.0
G	6	5600.0	5500.0	5300.0	5100.0	4900.0	4700.0	4500.0
		4300.0	3900.0	3700.0	3100.0	2900.0	2700.0	2400.0
		2200.0	1800.0	1400.0	1100.0	800.0	460.0	0.0
G	7	5800.0	5600.0	5500.0	5451.0	5053.3	4761.9	4463.0
		4125.1	3602.0	3428.0	3177.0	2800.0	2500.0	2135.0
		1683.0	1363.0	875.0	676.0	491.0	284.0	0.0
G	8	6924.0	6200.0	5908.0	5558.0	5099.5	4848.0	4448.2
		4052.0	3713.0	3376.0	3121.0	2600.0	2393.0	1844.0
		1061.0	810.0	595.0	425.0	350.0	310.0	0.0
G	9	7763.0	6900.0	6476.0	5772.0	5382.9	4947.8	4413.0
		3903.0	3581.0	3166.0	2699.0	2000.0	1873.0	1528.0
		871.0	550.0	379.0	320.0	300.0	283.0	0.0
G	10	8547.0	7982.0	6686.0	6400.0	5776.0	5066.0	4385.0
		3851.2	3458.0	2768.0	2178.0	1772.0	1526.0	1227.0
		666.0	386.0	300.0	250.0	230.0	166.0	0.0
G	11	11650.0	9289.0	8298.0	6700.0	6700.0	5473.0	4476.0
		3860.0	3232.0	2374.0	1883.0	1569.0	1385.0	1133.0
		542.0	267.0	223.0	220.0	209.0	168.0	0.0
G	12	12289.0	10508.0	9489.0	7170.0	7134.0	5995.0	4637.0
		3901.6	3110.0	2213.0	1705.0	1447.0	1309.0	1015.0
		454.0	227.0	200.0	200.0	177.0	160.0	0.0
G	13	13179.0	11414.0	10337.0	8300.0	8223.0	7226.0	5831.0
		4035.0	2929.0	2100.0	1658.0	1534.0	1370.0	1200.0
		703.0	450.0	300.0	225.0	177.0	165.0	0.0
G	14	14766.0	13009.0	11908.0	8703.0	8615.0	7704.0	6241.0
		4273.7	3561.0	2500.0	2391.0	2296.0	2100.0	2000.0
		1650.0	758.0	450.0	425.0	300.0	205.0	0.0
PRESSURE ROW		14						
G	1	3308.0	3249.0	3187.0	3117.0	3103.0	3046.0	2987.0
		2830.0	2206.0	1731.0	1395.0	1100.0	850.0	650.0
		500.0	420.0	360.0	300.0	240.0	180.0	0.0
G	2	3434.0	3363.0	3287.0	3204.0	3176.0	3087.0	2997.0
		2908.0	2700.0	2200.0	1999.0	1700.0	1500.0	1300.0
		1050.0	800.0	600.0	400.0	300.0	262.0	0.0
G	3	3528.0	3446.0	3361.0	3269.0	3200.0	3100.0	3000.0
		2950.0	2557.0	2400.0	2000.0	1800.0	1600.0	1400.0
		1200.0	1000.0	800.0	600.0	450.0	375.0	0.0
G	4	3690.0	3592.0	3490.0	3381.0	3250.0	3150.0	3010.0
		2960.0	2585.0	2500.0	2266.0	2053.0	1900.0	1700.0

G	6	8305.0	7858.0	7361.0	6817.0	6500.0	6200.0	6000.0
		5800.0	5600.0	5400.0	5000.0	4400.0	4000.0	3700.0
		3100.0	2600.0	2100.0	1800.0	1300.0	800.0	0.0
G	7	8759.0	8282.0	7736.0	7131.0	6574.8	6181.1	5814.9
		5500.0	5203.0	4892.0	4350.0	3727.0	3397.0	3400.0
		2866.0	2350.0	2000.0	1650.0	978.0	667.0	0.0
G	8	9184.0	8678.0	8087.0	7425.0	6600.0	6281.0	5758.0
		5296.0	5100.0	4646.0	4057.0	3551.0	3356.0	3000.0
		2598.0	1850.0	1571.0	1400.0	675.0	400.0	0.0
G	9	9975.0	9416.0	8740.0	7556.4	7052.0	6296.4	5100.0
		4800.0	4600.0	4300.0	3896.0	3300.0	3100.0	2700.0
		1966.0	1325.0	992.0	900.0	450.0	325.0	0.0
G	10	10713.0	10104.0	9349.0	7808.0	7305.0	6343.0	4935.0
		4661.3	4323.0	3685.0	3265.0	3179.0	2917.0	2423.0
		1558.0	1023.0	776.0	500.0	330.0	180.0	0.0
G	11	12089.0	11387.0	10485.0	8226.5	7564.8	6558.4	4995.0
		4500.0	3832.0	3199.0	2932.0	2905.0	2521.0	1904.0
		1049.0	521.0	349.0	260.0	200.0	140.0	0.0
G	12	13373.0	12584.0	11544.0	8596.8	7982.4	7060.1	5694.0
		4600.0	3742.0	2929.0	2614.0	2435.0	2085.0	1670.0
		851.0	341.0	255.0	234.0	175.0	110.0	0.0
G	13	14593.0	13721.0	12550.0	9487.0	8696.0	7889.0	6408.0
		4650.0	3470.0	2800.0	2400.0	2350.0	2000.0	1466.0
		846.0	600.0	400.0	300.0	280.0	150.0	0.0
G	14	17447.0	16381.0	14903.0	10262.0	9817.0	8841.0	7042.0
		4700.0	3636.0	2941.0	2570.0	2500.0	2200.0	2143.0
		1700.0	900.0	500.0	450.0	375.0	200.0	0.0

PRESSURE ROW 13

G	1	4731.0	4566.0	4300.0	4000.0	3700.0	3400.0	3250.0
		3150.0	2900.0	2300.0	1781.0	1200.0	900.0	700.0
		600.0	498.0	399.0	300.0	230.0	200.0	0.0
G	2	4930.0	4745.0	4451.0	4100.0	3900.0	3700.0	3600.0
		3500.0	3300.0	3000.0	2800.0	2300.0	1800.0	1500.0
		1200.0	900.0	650.0	455.0	350.0	300.0	0.0
G	3	5078.0	4877.0	4600.0	4450.0	4250.0	4100.0	4050.0
		4025.0	3800.0	3500.0	3100.0	2700.0	2300.0	2000.0
		1800.0	1300.0	1000.0	700.0	500.0	450.0	0.0
G	4	5336.0	5108.0	4866.0	4610.0	4400.0	4300.0	4250.0
		4150.0	3900.0	3550.0	3300.0	3000.0	2700.0	2400.0
		2100.0	1700.0	1400.0	1100.0	900.0	550.0	0.0
G	5	5568.0	5315.0	5046.0	4760.0	4700.0	4651.8	4451.3

		4208.6	3725.0	3600.7	3353.0	3200.0	3164.0	2741.0
		2398.0	1826.0	1617.0	1477.0	1125.0	575.0	0.0
G	6	5600.0	5500.0	5300.0	5100.0	4900.0	4700.0	4500.0
		4300.0	3900.0	3700.0	3100.0	2900.0	2700.0	2400.0
		2200.0	1800.0	1400.0	1100.0	800.0	460.0	0.0
G	7	5800.0	5600.0	5500.0	5451.0	5053.3	4761.9	4463.0
		4125.1	3602.0	3428.0	3177.0	2800.0	2500.0	2135.0
		1683.0	1363.0	875.0	676.0	491.0	284.0	0.0
G	8	6924.0	6200.0	5908.0	5558.0	5099.5	4848.0	4448.2
		4052.0	3713.0	3376.0	3121.0	2600.0	2393.0	1844.0
		1061.0	810.0	595.0	425.0	350.0	310.0	0.0
G	9	7763.0	6900.0	6476.0	5772.0	5382.9	4947.8	4413.0
		3903.0	3581.0	3166.0	2699.0	2000.0	1873.0	1528.0
		871.0	550.0	379.0	320.0	300.0	283.0	0.0
G	10	8547.0	7982.0	6686.0	6400.0	5776.0	5066.0	4385.0
		3851.2	3458.0	2768.0	2178.0	1772.0	1526.0	1227.0
		666.0	386.0	300.0	250.0	230.0	166.0	0.0
G	11	11650.0	9289.0	8298.0	6700.0	6700.0	5473.0	4476.0
		3860.0	3232.0	2374.0	1883.0	1569.0	1385.0	1133.0
		542.0	267.0	223.0	220.0	209.0	168.0	0.0
G	12	12289.0	10508.0	9489.0	7170.0	7134.0	5995.0	4637.0
		3901.8	3110.0	2213.0	1705.0	1447.0	1309.0	1015.0
		454.0	227.0	200.0	200.0	177.0	160.0	0.0
G	13	13179.0	11414.0	10337.0	8300.0	8223.0	7226.0	5831.0
		4035.0	2929.0	2100.0	1658.0	1534.0	1370.0	1200.0
		703.0	450.0	300.0	225.0	177.0	165.0	0.0
G	14	14786.0	13009.0	11908.0	8703.0	8615.0	7704.0	6241.0
		4273.7	3561.0	2500.0	2391.0	2286.0	2100.0	2000.0
		1650.0	758.0	450.0	425.0	300.0	205.0	0.0
PRESSURE ROW		14						
G	1	3308.0	3249.0	3187.0	3117.0	3103.0	3046.0	2987.0
		2830.0	2206.0	1731.0	1395.0	1100.0	850.0	650.0
		500.0	420.0	360.0	300.0	240.0	180.0	0.0
G	2	3434.0	3363.0	3287.0	3204.0	3176.0	3087.0	2997.0
		2908.0	2700.0	2200.0	1999.0	1700.0	1500.0	1300.0
		1050.0	800.0	600.0	400.0	300.0	262.0	0.0
G	3	3528.0	3446.0	3361.0	3269.0	3200.0	3100.0	3000.0
		2950.0	2557.0	2400.0	2000.0	1800.0	1600.0	1400.0
		1200.0	1000.0	800.0	600.0	450.0	375.0	0.0
G	4	3690.0	3592.0	3490.0	3381.0	3250.0	3150.0	3010.0
		2960.0	2585.0	2500.0	2266.0	2053.0	1900.0	1700.0

		1500.0	1300.0	1000.0	600.0	500.0	450.0	0.0
G	5	3837.0 2970.0 1488.0	3724.0 2800.0 1112.0	3608.0 2560.0 816.0	3482.0 2478.0 600.0	3300.0 2201.0 375.0	3203.0 2095.0 300.0	3025.0 1800.0 0.0
G	6	3838.0 3000.0 1000.0	3750.0 2900.0 800.0	3700.0 2500.0 600.0	3569.0 2100.0 450.0	3377.0 1700.0 300.0	3209.0 1600.0 178.0	3035.0 1400.0 0.0
G	7	3839.0 2800.0 769.0	3811.4 2700.0 517.0	3789.0 2300.0 400.0	3687.9 1800.0 400.0	3399.0 1441.0 210.0	3200.0 1300.0 175.0	3050.0 1168.0 0.0
G	8	3992.7 2620.7 425.0	3908.7 2183.0 263.0	3827.0 1806.0 242.0	3715.3 1466.0 225.0	3421.0 1130.0 200.0	3227.5 1000.0 170.0	3040.0 930.0 0.0
G	9	4300.0 2630.0 335.0	4100.0 2103.0 220.0	3947.0 1549.0 196.0	3770.0 1264.0 190.0	3465.0 1032.0 180.0	3246.0 850.0 167.0	3062.0 644.0 0.0
G	10	5530.0 2697.5 299.0	5091.0 2104.0 222.0	4433.0 1673.0 192.0	4180.0 1272.0 190.0	3670.0 950.0 175.0	3244.0 718.0 164.0	2715.0 529.0 0.0
G	11	6904.0 2726.0 431.0	5218.0 2288.0 288.0	5163.0 1722.0 220.0	4629.0 1299.0 210.0	4236.0 1017.0 185.0	3578.0 869.0 162.0	2987.0 634.0 0.0
G	12	7402.0 2953.0 514.0	6681.0 2525.0 324.0	6015.4 1906.0 260.0	4734.0 1476.0 225.0	4581.0 1134.0 200.0	3906.0 1000.0 180.0	3200.0 732.0 0.0
G	13	8573.0 3567.0 790.0	7329.0 2841.0 620.0	6303.0 2050.0 450.0	5703.0 1802.0 350.0	5380.8 1377.0 225.0	5158.0 1240.0 200.0	4356.0 1103.0 0.0
G	14	10140.0 4406.0 1562.0	9040.0 3800.0 1000.0	7881.0 2600.0 620.0	6401.0 2441.0 400.0	6089.0 2109.0 265.0	5567.0 1900.0 220.0	4749.0 1800.0 0.0
PRESSURE ROW		15						
G	1	1211.0 1000.0 357.0	1204.0 907.0 269.0	1202.0 791.0 200.0	1198.0 692.0 150.0	1107.0 607.0 100.0	1087.0 533.0 50.0	1066.0 457.0 0.0
G	2	1300.0 1030.0 544.0	1300.0 922.0 358.0	1294.0 813.0 295.0	1288.0 762.0 199.0	1133.0 687.0 170.0	1101.0 642.0 146.0	1069.0 598.0 0.0
G	3	1400.0 1040.0 606.0	1400.0 927.0 421.0	1362.0 820.0 365.0	1351.0 785.0 250.0	1191.0 713.0 185.0	1106.0 678.0 150.0	1070.0 642.0 0.0

G	4	1550.0 1050.0 679.0	1500.0 930.0 676.0	1500.0 850.0 675.0	1465.0 800.0 425.0	1344.0 760.0 323.0	1271.0 720.0 300.0	1070.0 675.0 0.0
G	5	1600.0 1079.5 789.0	1600.0 940.0 657.0	1587.0 900.0 456.0	1500.0 850.0 440.0	1400.0 800.0 183.0	1300.0 760.0 180.0	1150.0 700.0 0.0
G	6	1665.0 1155.3 430.0	1649.0 950.0 325.0	1640.0 910.0 250.0	1550.0 870.0 172.0	1458.3 810.0 150.0	1330.5 780.0 140.0	1194.4 725.0 0.0
G	7	1838.0 1204.7 400.0	1819.0 960.0 328.0	1740.0 920.0 170.0	1612.0 900.0 150.0	1531.3 830.0 120.0	1377.8 800.0 110.0	1250.0 750.0 0.0
G	8	2058.7 1254.0 387.0	1962.3 980.0 254.0	1826.5 925.0 140.0	1675.0 905.0 140.0	1604.0 850.0 105.0	1400.0 820.0 98.0	1317.7 770.0 0.0
G	9	2500.0 1261.0 375.0	2249.0 1054.0 250.0	1997.0 930.0 131.0	1800.0 810.0 130.0	1750.0 870.0 103.0	1712.0 840.0 97.0	1399.0 790.0 0.0
G	10	3261.0 1498.0 568.0	2878.0 1420.0 388.0	2458.0 1100.0 160.0	2098.0 950.0 110.0	1904.5 900.0 100.0	1753.5 860.0 99.0	1500.0 810.0 0.0
G	11	3939.0 1900.0 780.0	3176.0 1716.0 505.0	3017.0 1598.0 189.0	2671.0 1200.0 120.0	2233.0 1000.0 110.0	2000.0 900.0 107.0	1958.0 840.0 0.0
G	12	4365.0 2235.0 900.0	4000.0 1960.0 622.6	3891.0 1854.0 210.0	2939.0 1500.0 150.0	2576.0 1100.0 168.0	2300.0 1000.0 143.0	2250.0 900.0 0.0
G	13	4041.0 2357.0 965.0	4000.0 2150.0 740.2	3881.0 2015.0 500.0	3732.0 1745.0 230.0	3215.0 1300.0 210.0	2875.0 1200.0 204.0	2580.0 1000.0 0.0
G	14	5101.0 3200.0 1486.0	5017.0 3185.0 1034.0	4900.0 2594.0 750.0	3849.0 2400.0 325.0	3300.0 1500.0 250.0	3228.0 1400.0 240.0	3200.0 1200.0 0.0

Appendix C

Program for Predicting The Non-aqueous CHF Data by Groeneveld's Table Look-up Method

This chapter includes only one program which can predict the non-aqueous CHF data by interpolation or extrapolation from the given conditions. While the CHF data to be read by this program is the same as shown in Fig. B.2.

```

SUBROUTINE CALCHF(P,G,D,X,CHF,NFLG)
C*****
C
C      Subroutine CALCHF
C
C      This subroutine is used to obtain the CHF from table of discrete
C      values. The table was presented by Groeneveld et al..
C*****
C*****
C
C      Variable declaration:
C
C      Character:
C      GEOM      : a single character variable for channel geometry.
C      UNIT      : a single character variable for unit.
C      FLUID      : a single character variable for fluid type.
C      ORIENT     : a single character variable for channel orientation.
C      STATE      : a single character variable for state of the system.
C
C      Real:
C      X          : provided quality.
C      D          : provided diameter.
C      G          : provided mass flux.
C      P          : provided pressure.
C      INTRP3     : the interpolated value.
C      PX         : discrete value of quality.
C      PG         : discrete value of mass flux.
C      PP         : discrete value of pressure.
C      CHFTAB     : a set of 4410 discrete values of CHF arranged in three-
C                   dimensional arrays.
C
C      Logical:
C      NFLG       : Program termination flag.
C      Flag       : Step termination flag.
C*****
C
C      CHARACTER UNIT,GEOM,FLUID,ORIENT
C      REAL PX(21),PG(14),PP(15),INTRP3,LAT
C      LOGICAL FLAG,NFLG
C      COMMON/CHOICE/UNIT,GEOM,FLUID,ORIENT
C      COMMON/PROP/  VF,  VG,  VL,  RHOF, RHOG, RHOL,
C      +             HF,  HG,  HL,  CPF,  CPG,  CPL,
C      +             UF,  UG,  UL,  AKF,  AKG,  AKL,
C      +             PRF, PRG, PRL, HFG, PSAT, TSAT,
C      +             SIGMA
C      COMMON/CRIT/  TC,  PC,  VC,  W,  WEIGH, CO1,
C      +             CO2, CO3, CO4,  BO,  TO,  TREF
C      COMMON/ROWS/PX,PG,PP,IX,IG,IP
C
C      NFLG = .FALSE.
C      DEN = RHOF/RHOG

```

Figure C.1: Program for the non-aqueous CHF prediction by Groeneveld's table look-up method

```

C*****
C
C      Calculate the equivalent pressure of water & Psi_k for
C      non-aqueous fluid
C*****

      IF ((DEN .GE. 1.39) .AND. (DEN .LE. 10.75)) THEN
        TRY = -0.95*(ALOG10(DEN))**2.51
      IF (TRY .GE. 87.) THEN
        TRY = 86.99
      ELSE IF (TRY .LE. -87) THEN
        TRY = -86.99
      END IF
      PW = 17393.*EXP(TRY)+4792.8
      ELSE IF ((DEN .GT. 10.75) .AND. (DEN .LE. 84.6)) THEN
        TRY = -0.82*(ALOG10(DEN))**1.84
      IF (TRY .GE. 87.) THEN
        TRY = 86.99
      ELSE IF (TRY .LE. -87) THEN
        TRY = -86.99
      END IF
      PW = 25320.*EXP(TRY)+370.22
      ELSE IF ((DEN .GT. 84.6) .AND. (DEN .LE. 1624.5)) THEN
        TRY = -1.4*(ALOG10(DEN))**1.3
      IF (TRY .GE. 87.) THEN
        TRY = 86.99
      ELSE IF (TRY .LE. -87) THEN
        TRY = -86.99
      END IF
      PW = 53003.*EXP(TRY)+8.468
      ELSE
        NFLG = .TRUE.
        RETURN
      END IF
      PSI = G*0.008**0.5/SIGMA**0.5/RHOF**0.5
      PS = PW/22100
      IF ((PW .GE. 100) .AND. (PW .LE. 2000.)) THEN
        FUN = 7.7 - 59.6*PS + 693*PS**2 - 3449.3*PS**3
      ELSE IF ((PW .GT. 2000.) .AND. (PW .LE. 20500)) THEN
        FUN = 6.646 - 15.86*PS + 32.26*PS**2 - 51.45*PS**3
        + 44.23*PS**4 - 15.67*PS**5
      ELSE IF ((PW .GT. 20500.) .AND. (PW .LE. 21900)) THEN
        FUN = -32.4 + 75.63*PS - 43.167*PS**2
      ELSE
        NFLG = .TRUE.
        RETURN
      END IF
      GW = PSI*FUN/0.008**0.5
C*****
C
C      Initialize the flag and range pointer.
C*****

```

```

      FLAG = .TRUE.
      IX = 20
      IG = 13
      IP = 14
C*****
C
C   Locate the discrete value of quality bounding the input value.
C
C*****
      DO 100 I=1,20
      IF((X .LE. PX(I+1)) .AND. FLAG)THEN
        IX = I
        FLAG = .FALSE.
      END IF
100  CONTINUE
C*****
C
C   Re-initialize the flag.
C   Locate the discrete value of mass flux bounding the input value.
C
C*****
      FLAG = .TRUE.
      DO 200 I=1,13
      IF((GW .LE. PG(I+1)) .AND. FLAG)THEN
        IG = I
        FLAG = .FALSE.
      END IF
200  CONTINUE
C*****
C
C   Re-initialize the flag.
C   Locate the discrete value of pressure bounding the input value.
C
C*****
      FLAG = .TRUE.
      DO 300 I=1,14
      IF((PW .LE. PP(I+1)) .AND. FLAG)THEN
        IP = I
        FLAG = .FALSE.
      END IF
300  CONTINUE
C*****
C
C   Calculate modification factor for tube diameter other than 0.008.
C
C*****
      IF(D .GT. 0.018)THEN
        CK1 = 0.7937
      ELSE IF(D .LT. 0.002)THEN
        CK1 = 1.5874
      ELSE
        CK1 = (0.008/D)**(1./3)
      END IF
      QC = INTRP3(X,GW,PW)*CK1*1000
      IF ((PW .GE. 100.) .AND. (PW .LE. 8000.)) THEN

```



```

      LAT = 2214181. - 4627523.*PS + 13275135.*PS**2-17924341.*PS**3
      ELSE IF ((PW .GT. 8000.) .AND. (PW .LE. 21000.)) THEN
        LAT = 3102130. - 11492443.*PS+34371536.*PS**2-57226128.*PS**3
      + 47189435.*PS**4 - 15650016.*PS**5
      ELSE IF ((PW .GT. 21000.) .AND. (PW .LE. 21900)) THEN
        LAT = -84338385. + 179793761.*PS - 95310141.*PS**2
      ELSE
        NFLG = .TRUE.
        RETURN
      END IF
      CHF = QC/GW/LAT*G*HFG
      RETURN
    END
    REAL FUNCTION INTRP3(X,G,P)

```

```

C*****
C
C   Function INTRP3
C
C   This function is used to set up the parameters for linear
C   interpolation in CHF evaluation. The parameters are the slope,
C   and two boundary points.
C*****
C*****
C
C   Variable declaration:
C
C   Real:
C   X      : provided quality.
C   G      : provided mass flux.
C   P      : provided pressure.
C   INTERP : the interpolated value.
C   PX     : discrete value of quality.
C   PG     : discrete value of mass flux.
C   PP     : discrete value of pressure.
C   CHFTAB : a set of 4410 discrete value of CHF arranged in three-
C             dimensional arrays.
C*****
C
C   REAL INTERP,CHFTAB(21,14,15),PX(21),PG(14),PP(15)
C*****
C
C   Call up two common blocks to pass some variables in and out the
C   subroutine.
C*****
C
C   COMMON/TABLE/CHFTAB
C   COMMON/ROWS/PX,PG,PP,IX,IG,IP
C*****

```

```

C
C      Linear interpolate the pressure value.
C
C.....

PARAM = (P-PP(IP))/(PP(IP+1)-PP(IP))
C1    = INTERP(CHFTAB(IX,IG,IP),CHFTAB(IX,IG,IP+1),PARAM)
C2    = INTERP(CHFTAB(IX,IG+1,IP),CHFTAB(IX,IG+1,IP+1),PARAM)
C3    = INTERP(CHFTAB(IX+1,IG,IP),CHFTAB(IX+1,IG,IP+1),PARAM)
C4    = INTERP(CHFTAB(IX+1,IG+1,IP),CHFTAB(IX+1,IG+1,IP+1),PARAM)

C.....
C
C      Linear interpolate the mass flux value.
C
C.....

PARAM = (G-PG(IG))/(PG(IG+1)-PG(IG))
C5    = INTERP(C1,C2,PARAM)
C6    = INTERP(C3,C4,PARAM)

C.....
C
C      Linear interpolate the quality value.
C
C.....

PARAM = (X-PX(IX))/(PX(IX+1)-PX(IX))
INTRP3 = INTERP(C5,C6,PARAM)
RETURN
END
REAL FUNCTION INTERP(A1,A2,P)

C.....
C
C      Function INTERP
C
C      This function is used to interpolate value linearly.
C      It requires the slope of the function and two boundary points.
C
C.....

C.....
C
C      Variable declaration:
C
C      Real:
C      INTERP : the interpolated value.
C      P      : slope of the function.
C      A1,A2  : the two bounds.
C
C.....

INTERP = A1 + (A2-A1)*P
RETURN

```

END

SUBROUTINE READCHF

```
C.....
C
C   READ CHF DATA FROM CHFDA2.DATA
C
C.....
```

```
IMPLICIT REAL (A-H,O-Z)
DIMENSION CHF(15,14,21),CHFTAB(21,14,15)
COMMON/TABLE/CHFTAB
```

```
OPEN(UNIT=1,FILE='CHFDA2 DATA')
DO 1 I=1,15
  READ(1,4) K
  4   FORMAT(/,I1)
  DO 2 J=1,14
    READ(1,3) (CHF(I,J,K),K=1,21)
    3   FORMAT(3(T10,7F9.1,/))
  2   CONTINUE
1   CONTINUE
DO 100 I=1,15
  DO 200 J=1,14
    DO 300 K=1,21
      CHFTAB(K,J,I)=CHF(I,J,K)
    300 CONTINUE
  200 CONTINUE
100 CONTINUE
CLOSE(1)
RETURN
END
BLOCK DATA TBCHF1
REAL PX(21),PG(14),PP(15)
COMMON/ROWS/PX,PG,PP,IX,IG,IP
DATA PX/-.5,-.4,-.3,-.2,-.15,-.1,-.05,0,.05,.1,.15,.2,.25,
+ .3,.4,.5,.6,.7,.8,.9,1./
DATA PG/0.,50.,100.,200.,300.,500.,750.,1000.,1500.,2000.,
+ 3000.,4000.,5000.,7500./
DATA PP/100.,150.,200.,300.,450.,700.,1000.,1500.,2000.,
+ 3000.,4500.,7000.,10000.,15000.,20000./
END
```